

TECHNICAL REFERENCE MANUAL

State of Pennsylvania

Act 129 Energy Efficiency and Conservation Program &

Act 213 Alternative Energy Portfolio Standards

DRAFT June 2015

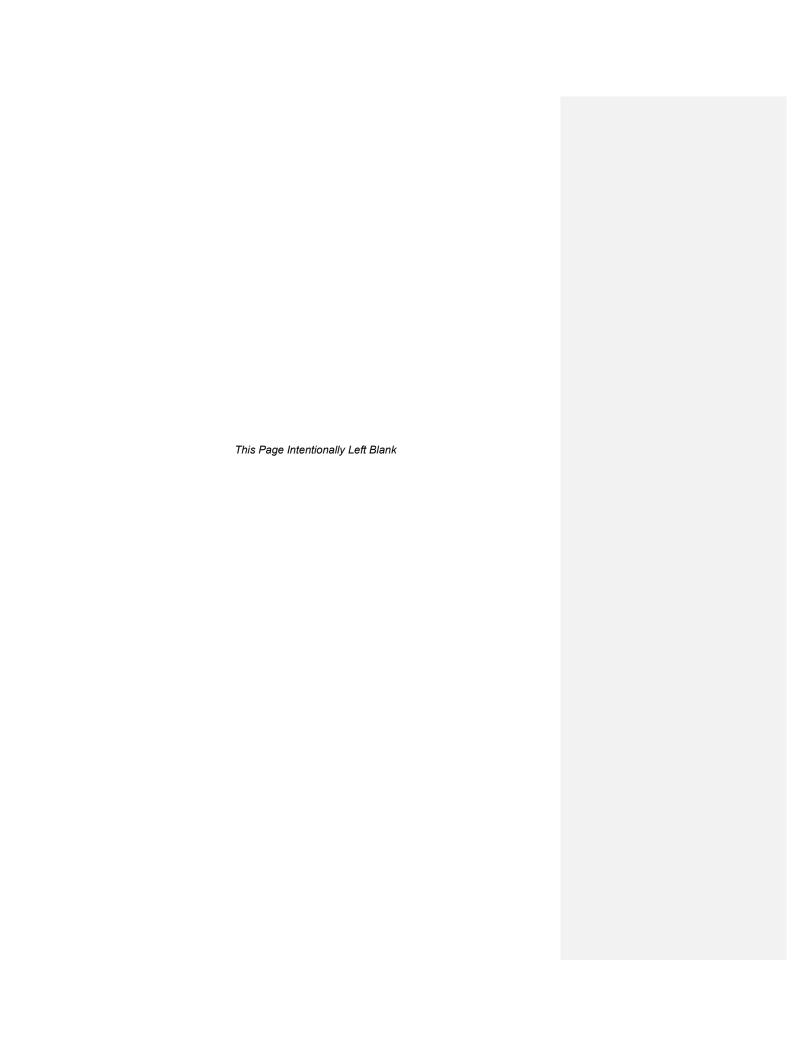


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1 INTRODUCTION

The Technical Reference Manual (TRM) was developed to measure the resource savings from standard energy efficiency measures. The savings' algorithms use measured and customer data as input values in industry-accepted algorithms. The data and input values for the algorithms come from Alternative Energy Portfolio Standards (AEPS) application forms¹, EDC program application forms, industry accepted standard values (e.g. ENERGY STAR standards), or data gathered by Electric Distribution Companies (EDCs). The standard input values are based on the best available measured or industry data.

Some electric input values were derived from a review of literature from various industry organizations, equipment manufacturers, and suppliers. These input values are updated to reflect changes in code, federal standards and recent program evaluations.

1.1 Purpose

The TRM was developed for the purpose of estimating annual electric energy savings and coincident peak demand savings for a selection of energy efficient technologies and measures. The TRM provides guidance to the Administrator responsible for awarding Alternative Energy Credits (AECs). The revised TRM serves a dual purpose of being used to determine compliance with the AEPS Act, 73 P.S. §§ 1648.1-1648.8, and the energy efficiency and conservation requirements of Act 129 of 2008, 66 Pa.C.S. § 2806.1. The TRM will continue to be updated on an annual basis to reflect the addition of technologies and measures as needed to remain relevant and useful.

Resource savings to be measured include electric energy (kWh) and electric capacity (kW) savings. The algorithms in this document focus on the determination of the per unit annualized energy savings and peak demand savings for the energy efficiency measures. The algorithms and methodologies set forth in this document must be used to determine EDC reported gross savings and evaluation measurement and verification (EM&V) verified savings.

For an Act 129 program, EDCs may, as an alternative to using the energy and demand savings values for standard measures contained in the TRM, use alternative methods to calculate *ex ante* savings and/or ask their evaluation contractor to use a custom method to verify *ex post* savings. The EDCs, however, must track savings estimated from the TRM protocols and alternative methods and report both sets of values in the quarterly and/or annual EDC reports. The EDCs must justify the deviation from the TRM *ex ante* and *ex post* protocols in the quarterly and/or annual reports in which they report the deviations. EDCs should be aware that use of a custom method as an alternative to the approved TRM protocol increases the risk that the PA PUC may challenge their reported savings. The alternative measurement methods are subject to review and approval by the Commission to ensure their accuracy after the reports are filed to the Commission.

1.2 Using the TRM

This section provides a consistent framework for EDC Implementation Conservation Service Providers (ICSPs) to estimate *ex ante* (claimed) savings and for EDC evaluation contractors to estimate *ex post* (verified) savings for Act 129 Energy Efficiency & Conservation (EE&C) programs.

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Purpose

¹ Note: Information in the TRM specifically relating to the AEPS Act is shaded in gray

1.2.1 MEASURE CATEGORIES

The TRM categorizes all non-custom measures into two categories: deemed measures and partially deemed measures. Methods used to estimate *ex ante* and/or *ex post* savings differ for deemed measures and partially deemed measures.

- Deemed measure protocols have specified "deemed energy and demand savings values" ²; no additional measurement or calculation is required to determine deemed savings. These protocols also may contain an algorithm with "stipulated variables" to provide transparency into deemed savings values and to facilitate the updating of deemed savings values in future TRMs. Stipulated variables should not be adjusted using customer-specific or program-specific information for calculating ex ante and/or ex post savings.
- Partially deemed measure protocols have algorithms with stipulated⁴ and "open variables"⁵, that require customer-specific input of certain parameters to calculate the energy and demand savings. Customer-specific or program-specific information is used for each open variable, resulting in multiple savings values for the same measure. Some open variables may have a default value to use when the open variable cannot be collected. Only variables specifically identified as open variables may be adjusted using customer-specific or program-specific information.

Note: Custom measures⁶ are considered too complex or unique to be included in the list of standard measures provided in the TRM and so are outside the scope of this TRM. Custom measures are determined through a custom-measure-specific process, which is described in Section 1.16 in this TRM.

1.2.2 CUSTOMER AND PROGRAM SPECIFIC DATA

The EDCs and their contractors (ICSPs and ECs) are encouraged to collect and apply customer-specific or program-specific data in the *ex ante* and/or *ex post* savings calculations for as many open variables as possible to reflect most accurate savings values. Site-specific data or information should be used for measures with important variations in one or more input values (e.g. delta watts, efficiency level, equipment capacity, operating hours). Customer-specific data comes directly from the measure application form or application process and/or EDC data gathering, such as, facility staff interviews, posted schedules, building monitoring systems (BMS), panel data, or metered data. In addition, standard input values for stipulated variables and default values for some open variables provided in this TRM are to be based on evaluations completed in Pennsylvania or best available measured or industry data, available from other jurisdictions or industry associations. The EDCs may use default values for open variables in the TRM if customer-specific or program-specific information is unreliable or the EDCs cannot obtain the information.

Values for exact variables that should be determined using customer-specific information are clearly described in the measure protocols in this TRM. This methodology will provide the EDCs

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² A stipulated value for a variable refers to a single input value to an algorithm, while a deemed savings estimate is the result of calculating the end result of all of the stipulated values in the savings algorithm.

³ A stipulated value for a variable refers to a single input value to an algorithm.

⁴ Ibid

⁵ Open variables are listed with a "default value" and an option for "EDC Data Gathering" in the TRM. When a measure indicates that an input to a prescriptive saving algorithm may take on a range of values, an average value is also provided in many cases. This value is considered the default input to the algorithm, and should be used when customer-specific information is not available.

⁶ This TRM does not provide calculations or algorithms for custom measures since the category covers a wide range of equipment, approaches, and measures. Where custom measures are discussed, the TRM requires site specific equipment, operating schedules, baseline and installed efficiencies, and calculation methodologies to estimate energy and demand savings.

with more flexibility to use customer-specific data, when available obtained from their application process and evaluations to improve the accuracy and reliability of savings.

1.2.3 END-USE CATEGORIES & THRESHOLDS FOR USING DEFAULT VALUES

The determination of when to use default values for open variables provided in the TRM in the *ex ante* and/or *ex post* savings calculations is a function of the savings impact and uncertainty associated with the measure⁷. The default values are appropriate for low-impact and low-uncertainty measures such as lighting retrofits in a small business facility. In contrast, customer-specific values are appropriate for high-impact and high-uncertainty measures, such as HVAC or lighting retrofits in universities or hospitals that have diverse facilities, and where those types of projects represent a significant share of program savings for a year.

The TRM organizes all measures⁸ into various end-use categories⁹ (e.g. lighting, HVAC, motors & VFDs). kWh savings thresholds are established at the end-use category level and should be used to determine whether customer-specific information is required for estimating *ex ante* and/or *ex post* savings. If a project involves multiple measures/technology¹⁰ types that fall under the same end-use category, the savings for all those measures/technology types should be grouped together to determine if the project falls below or above a particular threshold¹¹. Table 1-1Table 1-4 lists all the end-use categories and the sections for measures within a particular end-use category.

Table 1-1: End-Use Categories and Measures in the TRM12

End-Use Categories	List of Measures (Sections)
Reside	ntial Market Sector
Lighting <u>- 2.1</u>	2.4, 2.7, 2.29, 2.35, 2.36 <u>2.1.1 – 2.1.5</u>
HVAC <u>- 2.2</u>	2.1, 2.5, 2.10, 2.15, 2.16, 2.19, 2.28 <u>2.2.1 – 2.2.9</u>
Domestic Hot Water <u>- 2.3</u>	2.3 <u>.1 - 2.3.11, 2.6, 2.8, 2.9, 2.13, 2.14, 2.17, 2.18, 2.37, 2.41</u>
Appliances <u>– 2.4</u>	2.2, 2.11, 2.21, 2.23, 2.24, 2.25, 2.26, 2.27, 2.42 <u>2.4.1 – 2.4.104</u>
Building ShellConsumer Electronics – 2.5	2.20, 2.22, 2.30, 2.40 <u>2.5.1 – 2.5.3</u>
Building Shell – 2.6 Electronics	2.12, 2.33, 2.34 <u>2.6.1 – 2.6.67</u>
Miscellaneous <u>- 2.7</u>	2.31, 2.32, 2.38, 2.39 <u>2.7.1 – 2.7.2</u>
Commercial 8	Industrial Market Sector
Lighting <u>— 3.1</u>	3.2, 3.29 <u>3.1.1 – 3.1.74</u>

⁷ While the EDCs are required to collect and apply customer specific or program specific data for projects with savings at or above the established kWh thresholds in the TRM, they are allowed to use either default values or customer specific or program specific data for projects with savings below the thresholds.

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⁸ A measure is defined as a new installation, the replacement of an existing installation, or the retrofitting/modification of an existing installation of a building, of a system or process component, or of an energy using device in order to reduce energy consumption. e.g., the installation of a 14W CFL is one measure, and the installation of a 21W CFL is a separate measure; the installation of wall insulation, or the modification of an existing building to reduce air infiltration are two other measures..

⁹ An end-use is defined as the grouping of related technology types all associated with a similar application or primary function. E.g., CFLs, LEDs, fluorescent lamps, and lighting controls are all within the lighting end-use category; efficient water heaters, water heater blankets, water heater setback, and faucet aerators are all within the domestic hot water end-use category.

¹⁰ A technology is defined as the grouping of related measures in order to differentiate one type of measure from another. Each technology type may consist of multiple measures. e.g., CFLs, LEDs, and VFDs are all different technology types. A 14W CFL and a 21W CFL are different measures within the CFL technology type.

¹¹ For example, linear fluorescent lighting, CFL lighting and LED lighting are individual measures within the Lighting end-use category.

¹² Please note that this is not an exhaustive list of end-uses and that others may be included in future TRM updates.

End-Use Categories	List of Measures (Sections)	
HVAC <u>- 3.2</u>	3.5, 3.6, 3.17, 3.18, 3.32, 3.34, 3.39 <u>3.2.1 – 3.2.9</u>	
Motors & VFDs <u>– 3.3</u>	3.3 <u>.1 – 3.3.4</u> , 3.4	
Domestic Hot Water <u>- 3.4</u>	3.27, 3.28, 3.30, 3.31, 3.37, 3.38 <u>3</u> .4.1 – 3.4. <u>76</u>	
Refrigeration <u>— 3.5</u>	3.7, 3.8, 3.9, 3.10, 3.16, 3.20, 3.22, 3.23, 3.24, 3.25, 3.33, 3.35, 3.36 <u>3</u> .5.1 – 3.5.14	
Appliances <u>– 3.6</u>	3. <u>6.1</u> 26	
Food Service Equipment <u>– 3.7</u>	3. 13, 3.14, 3.19 <u>7.1 – 3.7.5</u>	
Building Shell – 3.8	<u>3.15</u> 3.8.1	
Consumer Electronics – 3.9	3.11, 3.12, 3.21 <u>3.9.1 – 3.9.3</u>	
Compressed Air – 3.10	<u>3.10.1 – 3.10.3</u>	
Miscellaneous – 3.11	3.11.1	
Agricultural Sector		
Agricultural Equipment	4.1 - 4.8, 4.2, 4.3, 4.4, 4.5, 4.6, 4.7, 4.8	

Table 1-2 shows the kWh thresholds¹³ for various end-use categories. For projects with savings of established kWh thresholds or higher, the EDCs are required to collect site-specific information for open variables used in the calculation of energy and demand savings. If savings for individual end-use categories within projects fall below the threshold, the EDCs may gather customerspecific data, or may use the default stipulated value for each open variable. The thresholds below are subject to review and adjustment by the EDC ECs in coordination with SWE to minimize the uncertainty of estimates. End-use metering is the preferred method of data collection for projects above the threshold, but trend data from BMS or panel data and billing analysis¹⁴ are acceptable substitutes. The EDCs are encouraged to meter projects with savings below the thresholds that have high uncertainty but are not required where data is unknown, variable, or difficult to verify. Exact conditions of "high uncertainty" are to be determined by the EDCs to appropriately manage variance. Metering completed by the ICSP may be leveraged by the evaluation contractor, subject to a reasonableness review¹⁵. This approach is intended to determine values for key variables and verify savings at a high level of rigor for projects that account for majority of the programs expected savings.

Table 1-2: kWh Savings Thresholds

End-Use Category	Expected kWh/yr Savings Threshold ¹⁶
C&I Lighting	>= 500,000
C&I HVAC	>= 250,000
C&I Motors & VFDs	>= 250,000
C&I Building Shell	>= 250,000
Agricultural Equipment	>= 250,000

¹³ These end-use specific thresholds were developed by the SWE based on review of methods used by other jurisdictions. In addition, the SWE also performed a sensitivity analyses using different thresholds based on all the energy efficiency projects (partially deemed/non-custom) implemented in Phase I (PY1 through PY4) of Act 129 Programs among all the EDCs.

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¹⁴ Billing analysis should be conducted using at least 12 months of billing data (pre- and post-retrofit).

¹⁵ EDC evaluation contractors must verify the project-specific M&V data (including pre and post metering results) obtained by the CSPs, as practicable, for projects in the evaluation sample. If the evaluation contractor determines that data collected by the CSPs are not reasonably valid, then the evaluator must perform measurements consistent with IPMVP options to collect post-retrofit information for projects that have estimated end-use savings above a threshold kWh/year level. The SWE reserves the right to audit and review claimed and verified impacts of any project selected in the evaluation sample.

¹⁶ In situations where an ICSP meters a project because the expected kWh savings are above the established threshold and then realizes that the actual savings are below the threshold, metered results should be used for reporting claimed and verified savings.

1.2.4 APPLICABILITY OF THE TRM FOR ESTIMATING Ex ANTE (CLAIMED) SAVINGS

For replacements and retrofits, the applicable date for determining which TRM version to use to estimate EDC claimed savings is the "in-service date" (ISD) or "commercial date of operation" (CDO) – the date at which the measure is "installed and commercially operable," and when savings actually start to occur. This is analogous to when a commercial customer's meter "sees" the savings under expected and designed-for operation. For most projects, this is obvious. For projects with commissioning, the CDO occurs after the commissioning is completed. For incented measures that have been installed, but are not being used because there is no occupant, or will not be used until another, unrelated installation/project is completed; the equipment is not "commercially operable." For these projects, the CDO is the date at which the customer begins using the incented equipment, not the date at which the equipment is energized. For new construction, the appropriate TRM must be based on the date when the building/construction permit was issued (or the date construction starts if no permit is required) because that aligns with codes and standards that define the baseline. Savings begin to accrue at the project's ISD.

1.3 **DEFINITIONS**

The TRM is designed for use with both the AEPS Act and Act 129; however, it contains words and terms that apply only to the AEPS or only to Act 129. The following definitions are provided to identify words and terms that are specific for implementation of the AEPS:

- Administrator/Program Administrator (PA) The Credit Administrator of the AEPS program that receives and processes, and approves AEPS Credit applications.
- <u>AEPS application forms</u> application forms submitted to qualify and register alternative energy facilities for alternative energy credits.
- Application worksheets part of the AEPS application forms.
- Alternative Energy Credits (AECs) A tradable instrument used to establish, verify, and measure compliance with the AEPS. One credit is earned for each 1000kWh of electricity generated (or saved from energy efficiency or conservation measures) at a qualified alternative energy facility.
- Coincidence Factor (CF) The ratio of the (1) sum of every unit's average kW load during the PJM peak load period (June through August, non-holiday weekdays, 2 pm to 6 pm) to the (2) sum of the non-coincident maximum kW connected load for every unit.
- <u>Direct Install (DI) Measure</u> A prescriptive measure implemented on site during an energy audit or other initial visit without the requirement of a diagnostic testing component. Examples of these DI measures that can be installed directly include the changing of an incandescent bulb to a CFL or LED or the installation of faucet aerators.
- <u>Early Retirement (ERET) Measure</u> The removal of equipment from service that is not scheduled to be replaced by either a more efficient option or a less efficient option and is deemed to be eligible for savings due to the nature of reduction in energy use by taking the equipment out of service.
- <u>EDC Reported Gross Savings</u> Also known as "EDC Claimed Savings" or "Ex Ante Savings". EDC estimated savings for projects and programs of projects which are completed and/or M&Ved. The estimates follow a TRM method or Site Specific M&V Protocols (SSMVP). The savings calculations/estimates follow algorithms prescribed by

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¹⁷ Pennsylvania Public Utility Commission Act 129 Phase II Order, Docket Number: M-2012-2289411 and M-2008-2069887, Adopted August 2, 2012, language in Section K.1.b.

the TRM or Site Specific M&V Protocols (SSMVP) and are based non-verified, estimated, stipulated, EDC gathered or measured values of key variables.

- <u>Efficiency Kits (KIT)</u> A collection of energy efficient upgrade measure materials that can be delivered to and installed by the end-user. Examples of these items are CFL light bulbs, LED nightlights, or faucet aerators.
- Replace on Burnout (ROB) Measure The replacement of equipment that has failed or is at the end of its service life with a model that is more efficient than required by the codes and standards in effect at the time of replacement, or is more efficient than standard practice if there are no applicable codes or standards. The baseline used for calculating energy savings for replace on burnout measures is the applicable code, standard or industry standard practice in the absence of applicable code or standards. The incremental cost for replacement on burnout measures is the difference between the cost of baseline and more efficient equipment. Examples of projects which fit in this category include replacement due to existing equipment failure, or imminent failure, as judged by a competent service specialist, as well as replacement of equipment which may still be in functional condition, but which is operationally obsolete due to industry advances and is no longer cost effective to keep.
- New Construction Measure (Substantial Renovation Measure) The substitution of efficient equipment for standard baseline equipment which the customer does not yet own or during the course of a major renovation project which removes existing, but operationally functional equipment. The baseline used for calculating energy savings is the construction of a new building or installation of new equipment that complies with applicable code, standard or industry standard practice in the absence of applicable code or standards in place at the time of construction/installation/substantial renovation. The incremental cost for a new construction or substantial renovation measure is the difference between the cost of the baseline and more efficient equipment. Examples of projects which fit in this category include installation of a new production line, construction of a new building, an addition to an existing facility, renovation of a plant which replaces an existing production line with a production line for a different product, substantial renovation of an existing building interior, replacement of an existing standard HVAC system with a ground source heat pump system.
- Realization Rate The ratio of "Verified Savings" to "EDC Reported Gross Savings".
- Retrofit Measure (RET) Measures which modify or add on to existing equipment with technology to make the system more energy efficient. Retrofit measures have a dual baseline: for the estimated remaining useful life of the existing equipment the baseline is the existing equipment; afterwards the baseline is the applicable code, standard, or industry standard practice expected to be in place at the time the unit would have been naturally replaced or retrofit. If there are no known or expected changes to the baseline standards, the standard in effect at the time of the retrofit is to be used. Incremental cost is the full cost of equipment retrofit. In practice, in order to avoid the uncertainty surrounding the determination of "remaining useful life" retrofit measure savings and costs sometimes follow replace on burnout baseline and incremental cost definitions. Examples of projects which fit this category include installation of a VFD on an existing HVAC system, or installation of wall or ceiling insulation.
- <u>Early Replacement Measure (EREP)</u> The replacement of existing equipment, which is functioning as intended and is not operationally obsolete, with a more efficient model primarily for purposes of increased efficiency. Early replacement measures have a dual baseline: for the estimated remaining useful life of the existing equipment the baseline is the existing equipment; afterwards the baseline is the applicable code, standard, or industry standard practice expected to be in place at the time the unit would have been naturally replaced. If there are no known or expected changes to the baseline standards, the standard in effect at the time of the early replacement is to be used. Incremental cost

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is the full cost of equipment replacement. In practice, in order to avoid the uncertainty surrounding the determination of "remaining useful life" early replacement measure savings and costs sometimes follow replace on burnout baseline and incremental cost definitions. Examples of projects which fit this category include upgrade of an existing production line to gain efficiency, upgrade an existing, but functional, lighting or HVAC system that is not part of a renovation/remodeling project, or replacement of an operational chiller with a more efficient unit.

- <u>Time of Sale (TOS) Measure</u> A measure implemented, usually incentivized at the retail level, that provides a financial incentive to the buyer or end user in order to promote the higher efficiency of the measure product over a standard efficiency product. Examples include the low-flow pre-rinse sprayers available to commercial kitchens and their applicable incentives to be purchased over standard flow sprayers.
- <u>Verified Gross Savings</u> Evaluator estimated savings for projects and programs of
 projects which are completed and for which the impact evaluation and EM&V activities
 are completed. The estimates follow a TRM method or Site Specific M&V Protocols
 (SSMVP). The savings calculations/estimates follow algorithms prescribed by the TRM
 or Site Specific M&V Protocols (SSMVP) and are based on verified values of stipulated
 variables, EDC or evaluator gathered data, or measured key variables.
- <u>Lifetime</u> The number of years (or hours) that the new high efficiency equipment is
 expected to function. These are generally based on engineering lives, but sometimes
 adjusted based on expectations about frequency of removal, remodeling or demolition.
 Two important distinctions fall under this definition; Effective Useful Life and Remaining
 Useful Life.
- Effective Useful Life (EUL) EUL is based on the manufacturers rating of the effective useful life; how long the equipment will last. For example, a CFL that operates x hours per year will typically have an EUL of y. A house boiler may have a lifetime of 20 years but the EUL is only 15 years since after that time it may be operating at a non-efficient point. It is an estimate of the median number of years that the measures installed under a program are still in place and operable.
- <u>Remaining Useful Life (RUL)</u> It applies to retrofit or early replacement measures. For
 example, if an existing working refrigerator is replaced with a high efficiency unit, the RUL
 is an assumption of how many more years the existing unit would have lasted.

1.4 GENERAL FRAMEWORK

In general, energy and demand savings will be estimated using TRM stipulated values, measured values, customer data and information from the AEPS application forms, worksheets and field tools

Three systems will work together to ensure accurate data on a given measure:

- The application form that the customer or customer's agent submits with basic information.
- Application worksheets and field tools with more detailed, site-specific data, input values and calculations.
- Algorithms that rely on standard or site-specific input values based on measured data. Parts or all of the algorithms may ultimately be implemented within the tracking system, application forms and worksheets and field tools.

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1.5 ALGORITHMS

The algorithms that have been developed to calculate the energy and or demand savings are typically driven by a change in efficiency level between the energy efficient measure and the baseline level of efficiency. The following are the basic algorithms.

```
= kW_{base} - kW_{ee}
\Delta kW_{peak}
                        = \Delta kW \times CF
∆kWh/yr
                        = ∆kW × EFLH
Where:
        ∆kW
                        = Demand Savings
        \Delta kW_{peak}
                        = Coincident Peak Demand Savings
        ∆kWh/vr
                        = Annual Energy Savings
        kW_{base}
                        = Connected load kW of baseline case.
        kW_{ee}
                        = Connected load kW of energy efficient case.
                        = Equivalent Full Load Hours of operation for the installed measure.
        EFLH
        CF
                        = Demand Coincidence Factors represent the fraction of connected load
                                 expected to be coincident with the PJM peak demand period as
                                 defined in Section 1.10.
```

Other resource savings will be calculated as appropriate.

Specific algorithms for each of the measures may incorporate additional factors to reflect specific conditions associated with a measure. This may include factors to account for coincidence of multiple installations or interaction between different measures.

1.6 DATA AND INPUT VALUES

The input values and algorithms are based on the best available and applicable data. The input values for the algorithms come from the AEPS application forms, EDC data gathering, or from standard values based on measured or industry data.

Many input values, including site-specific data, come directly from the AEPS application forms, EDC data gathering, worksheets and field tools. Site-specific data on the AEPS application forms and EDC data gathering are used for measures with important variations in one or more input values (e.g., delta watts, efficiency level, capacity, etc.).

Standard input values are based on the best available measured or industry data, including metered data, measured data from other state evaluations (applied prospectively), field data, and standards from industry associations. The standard values for most commercial and industrial measures are supported by end-use metering for key parameters for a sample of facilities and circuits.

For the standard input assumptions for which metered or measured data were not available, the input values (e.g., delta watts, delta efficiency, equipment capacity, operating hours, coincidence factors) were assumed based on best available industry data or standards. These input values were based on a review of literature from various industry organizations, equipment manufacturers and suppliers.

1.7 BASELINE ESTIMATES

The savings methods and assumptions can differ substantially based on the program delivery mechanism for each measure type. Within each of the measure protocols in the TRM, there is a

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Algorithms

definition for the measure's baseline efficiency, a critical input into the savings calculations. For most measures there will be at least two baselines that are most commonly used:

- One for market-driven choices -- often called "lost opportunity" and either replacing equipment that has failed (retrofit on burnout) or new installations (new construction)
- One for discretionary installations often called early replacement

For all new construction (NC) and replace on burnout (ROB) scenarios, the baseline may be a jurisdictional code, a national standard, or the prevailing level of efficiency in the marketplace. The Δ kW and Δ kWh savings calculations –are based on standard efficiency equipment versus new high-efficiency equipment. For all early replacement (EREP) scenarios, the baseline may be the existing equipment efficiency, but at some point the Δ kW and Δ kWh savings calculations must incorporate changes to the baseline for new installations, e.g. code or market changes. This approach encourages residential and business consumers to replace working inefficient equipment and appliances with new high-efficiency products rather than taking no action to upgrade or only replacing them with new standard-efficiency products.

All baselines are designed to reflect current market practices which are generally the higher of code or available equipment, that are updated periodically to reflect upgrades in code or information from evaluation results. Specifically for commercial and industrial measures, Pennsylvania has adopted the 2009 International Energy Conservation Code (IECC) per 34 Pa. Code Section 403.21, effective 12/31/09 by reference to the International Building code and the ICC electrical code. Per Section 501.1 of IECC 2009, "[t]he requirements contained in [chapter 5 of IECC 2009] are applicable to commercial buildings, or portions of commercial buildings. These commercial buildings shall meet either the requirements of ANSI/ASHRAE/IESNA Standard 90.1, Energy Standard for Buildings Except for Low-Rise Residential Buildings, or the requirements contain in [chapter 5 of IECC 2009]". As noted in Section 501.2, as an alternative to complying with Sections 502, 503, 504, and 505 of IECC 2009, commercial building projects "shall comply with the requirements of ANSI/ASHRAE/IESNA 90.1 in its entirety."

In accordance with IECC 2009, commercial protocols relying on code standards as the baseline condition may refer to either IECC 2009 or ASHRAE 90.1-2007 per the program design.

The baseline estimates used in the TRM are documented in baseline studies or other market information. Baselines will be updated to reflect changing codes, practices and market transformation effects, and will be handled in future versions of the TRM by describing the choice of and reasoning behind a shifting baseline assumption. In general, this TRM addresses the everchanging regulatory codes and recognized program standards of the energy efficiency market with the following guidance for applicable measures:

When an existing Federal standard expires in a given **calendar year**, then that change will be reflected in the **following program year**'s TRM ¹⁸. This applies only to measures where the Federal standard is considered the baseline as described in the TRM or otherwise required by law. In the case of a January 1st effective date for a new Federal standard, the previous standard will be said to have expired on December 31 of the previous calendar year, and thus the change will be reflected in the TRM to be released in June of that year. Likewise, it is proposed that when an existing ENERGY STAR Product Specification Version expires in a given **calendar year**, then that change will be reflected in the **following program year**'s TRM ¹⁹. This applies only to measures where the ENERGY STAR criterion is considered the eligibility requirement.

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Baseline Estimates Page

¹⁸ For new Federal standards that become effective on January 1st, the previous standards are considered to expire on December 31 of the prior calendar year.

¹⁹ For new ENERGY STAR product specifications that become effective on January 1st, the previous specifications are considered to expire on December 31 of the prior calendar year.

1.8 RESOURCE SAVINGS IN CURRENT AND FUTURE PROGRAM YEARS

AECs and energy efficiency and demand response reduction savings will apply in equal annual amounts corresponding to either PJM planning years or calendar years beginning with the year deemed appropriate by the Administrator, and lasting for the approved life of the measure for AEPS Credits. Energy efficiency and demand response savings associated with Act 129 can claim savings for up to fifteen years.

1.9 Prospective Application of the TRM

The TRM will be applied prospectively. The input values are from the AEPS application forms, EDC program application forms, EDC data gathering and standard input values (based on measured data including metered data and evaluation results). The TRM will be updated annually based on new information and available data and then applied prospectively for future program years. Updates will not alter the number of AEPS Credits, once awarded, by the Administrator, nor will it alter any energy savings or demand reductions already in service and within measure life. Any newly approved measure, whether in the TRM or approved as an interim protocol, may be applied retrospectively consistent with the EDC's approved plan. If any errors are discovered in the TRM or clarifications are required, those corrections or clarifications should be applied to the associated measure calculations for the current program year, if applicable.

1.10 ELECTRIC RESOURCE SAVINGS

Algorithms have been developed to determine the annual electric energy and electric coincident peak demand savings. Annual electric energy savings are calculated and then allocated separately by season (summer and winter) and time of day (on-peak and off-peak). Summer coincident peak demand savings are calculated using a demand savings algorithm for each measure that includes a coincidence factor.

Table 1-3: Periods for Energy Savings and Coincident Peak Demand Savings

Period	Energy Savings	Coincident Peak Demand Savings
Summer	May through September	June through August (excluding weekends and holidays)
Winter	October through April	N/A
Peak	8:00 a.m. to 8:00 p.m. MonFri.	2:00 p.m. to 6:00 p.m.
8:00 p.m. to 8:00 a.m. MonFri., 12 a.m. to 12 a.m. Sat/Sun & holidays		N/A

The time periods for energy savings and coincident peak demand savings were chosen to best fit the Act 129 requirement, which reflects the seasonal avoided cost patterns for electric energy and capacity that were used for the energy efficiency program cost effectiveness purposes. For energy, the summer period May through September was selected based on the pattern of avoided costs for energy at the PJM level. In order to keep the complexity of the process for calculating energy savings' benefits to a reasonable level by using two time periods, the knee periods for spring and fall were split approximately evenly between the summer and winter periods.

For capacity, the definition of summer peak is adopted from PJM which is applied statewide in this TRM. Only the summer peak period is defined for the purpose of this TRM. The coincident summer peak period is defined as the period between the hour ending 15:00 Eastern Prevailing

Time²⁰ (EPT) and the hour ending 18:00 EPT during all days from June 1 through August 31, inclusive, that is not a weekend or federal holiday.²¹

1.11 POST-IMPLEMENTATION REVIEW

The Administrator will review AEPS application forms and tracking systems for all measures and conduct field inspections on a sample of installations. For some programs and projects (e.g., custom, large process, large and complex comprehensive design), post-installation review and on-site verification of a sample of AEPS application forms and installations will be used to ensure the reliability of site-specific savings' estimates.

1.12 ADJUSTMENTS TO ENERGY AND RESOURCE SAVINGS

1.12.1 COINCIDENCE WITH ELECTRIC SYSTEM PEAK

Coincidence factors are used to reflect the portion of the connected load savings or generation that is coincident with the system peak period.

1.12.2 Measure Retention and Persistence of Savings

The combined effect of measure retention and persistence is the ability of installed measures to maintain the initial level of energy savings or generation over the measure life. If the measure is subject to a reduction in savings or generation over time, the reduction in retention or persistence is accounted for using factors in the calculation of resource savings (e.g., in-service rates for residential lighting measures).

It is important to note that the Commission's Phase II Implementation Order, dated August 2, 2012, provides clarification on the accumulation and reporting of savings from Act 129 programs in Phase II. This order states on page 26 that "Savings reduction targets can be considered cumulative in two different ways - at the end of a phase and among phases. The Act 129 programs are cumulative at the end of a phase such that the savings at the end of a phase must show that the total savings from measures installed during the phase are equal to or greater than the established reduction target. Therefore, if any measures are installed whose useful life expires before the end of the phase, another measure must be installed or implemented during that phase which replenishes the savings from the expired measure." This means that reported savings for Phase II must take into account the useful life of measures. For example, savings for a measure with a useful life of two years installed in the first program year of Phase II cannot be counted towards the established reduction target unless another measure is installed or implemented to replenish the savings form the expired measures.

It is also important to note that the 2008 Pennsylvania Act 129 legislation states that the Total Resource Cost test shall be used to determine program cost effectiveness, and defines the TRC test as:

"-A STANDARD TEST THAT IS MET IF, OVER THE EFFECTIVE LIFE OF EACH PLAN NOT TO EXCEED 15 YEARS, THE NET PRESENT VALUE OF THE AVOIDED MONETARY COST OF SUPPLYING ELECTRICITY IS GREATER THAN THE NET PRESENT VALUE OF THE MONETARY COST OF ENERGY EFFICIENCY CONSERVATION MEASURES."

Post-Implementation Review

 $^{^{\}rm 20}$ This is same as the Daylight Savings Time (DST)

²¹ PJM Manual 18B for Energy Efficiency Measurement & Verification

Thus when TRC ratios are calculated for Act 129 programs, the life for any measure cannot be longer than 15 years.

1.12.3 Interactive Measure Energy Savings

Throughout the TRM, the iInteractive effect of thermostatically sensitive building components ion of energy savings is accounted for in specific measure protocols as appropriate. For all other measures, interaction of energy savings is zero. In instances where there is a measurable amount of interaction between two energy consuming sources, the energy or peak demand savings are accounted for in either the algorithms or in the modeling software used to determine energy savings.

For example, in a residential protocol where the lighting load has a direct effect on the energy used to condition the space, the TRM provides an interactive effect value to be used in the savings algorithm for certain measures. Other measures rely on the characteristics of the modeling software that account for the effect within a building, such as a new construction protocol software that will apply the effects for a measureable difference in the baseline and efficient buildings.

Likewise in Commercial and Industrial applications, the TRM accounts for the internal gains affected by implementing certain measures, also by using deemed values within the measure algorithms or by site-specific analysis where warranted, such as in the case of custom C&I measures. For example, the use of electronically commutated motors and the reduced heat output that affects the space cooling energy shall be specified by the measure protocol and where no interaction is present then the energy savings is zero.

For Residential lighting, the energy and peak demand savings is increased by an amount specified in the algorithm to account for HVAC interaction depending on the presence of air conditioning.

For Residential New Construction, the interaction of energy and peak demand savings is accounted for in the home energy rating tool that compares the efficient building to the baseline or reference building and calculates savings.

For Commercial and Industrial (C&I) lighting, the energy and peak demand savings is increased by an amount specified in the algorithm to account for HVAC interaction depending on the presence of air conditioning.

For C&I custom measures, interaction is accounted for in the site-specific analysis where relevant.

1.12.4 VERIFIED GROSS ADJUSTMENTS

Evaluation activities at a basic level consist of verification of the installation and operation of measures. In many cases, the number of widgets found on-site may differ from the number stated on the application, which represents the number of widgets paid for by the program. When the number of widgets found on-site is less than what is stated on the application, the savings will be adjusted by a realization rate. For example, if an application states 100 widgets but an on-site inspection only finds 85, the realization rate applied is 85% (assuming no other discrepancies). On-site widget counts within 5% of the application numbers can be considered to be within reasonable error without requiring realization rate adjustment.

On the other hand, if the number of widgets found on-site is more than what is stated on the application, the savings will be capped at the application findings. For example, if an application states 100 widgets but an on-site inspection finds 120, the realization rate applied is 100% (assuming no other discrepancies).

1.13 CALCULATION OF THE VALUE OF RESOURCE SAVINGS

The calculation of the value of the resources saved is not part of the TRM. The TRM is limited to the determination of the per unit resource savings in physical terms at the customer meter.

In order to calculate the value of the energy savings for reporting cost-benefit analyses and other purposes, the energy savings are determined at the customer level and then increased by the amount of the transmission and distribution losses to reflect the energy savings at the system level. The energy savings at the system level are then multiplied by the appropriate avoided costs to calculate the value of the benefits.

System Savings

= (Savings at Customer) X (T&D Loss Factor)

Value of Resource Savings = (System Savings) X (System Avoided Costs) + (Value of

Other Resource Savings)

Please refer to the 2013 TRC Order²² for a more detailed discussion of other resource savings.

1.14 Transmission and Distribution System Losses

The electric energy consumption reduction compliance targets for Phase II of Act 129 are established at the retail level i.e. based on forecasts of sales. The energy savings must be reported to the Commission at the customer meter level, which is used to determine if EDCs have met their statutory targets for Phase II. For the purpose of calculating cost-effectiveness of Act 129 programs, the value of both energy and demand savings shall be calculated at the system level. The TRM calculates the energy savings at the customer meter level. These savings need to be increased by the amount of transmission and distribution system losses in order to determine the energy savings at the system level. The electric line loss factors multiplied by the savings calculated from the algorithms will result in savings at the system level.

The EDC specific electric line loss factors filed in its Commission approved EE&C Plans, or other official reports filed with the Commission should be applied to gross up energy savings from the customer meter level to the system level. The EDCs are allowed to use alternate loss factors calculated to reflect system losses at peaking conditions when available to gross up demand savings to the system level. The Commissions encourages the use of the most recent and accurate values for line loss factors for energy and demand known to the EDCs, regardless of what was filed in the original Phase II EE&C Plans.

1.15 Measure Lives

Measure lives are provided at the beginning of each measure, as well as in in Appendix A: Measure Lives, for informational purposes and for use in other applications such as reporting lifetime savings or in benefit cost studies that span more than one year. For the purpose of calculating the Total Resource Cost (TRC) Test for Act 129, measures cannot claim savings for more than 15 years.

In general, avoided cost savings for programs where measures replace units before the end of their useful life are measured from the efficient unit versus the replaced unit for the remaining life of the existing unit, then from the efficient unit versus a new standard unit for the remaining efficient measure's life. Specific guidance is provided through the 2013 TRC Order.

²² See 2012 PA Total Resource Cost (TRC) Test; 2009 PA Total Resource Cost Test Final Order, at Docket Nos. M-2012-2300653 and M-2009-2108601, (2013 TRC Test Final Order), entered August 30, 2012.

1.16 CUSTOM MEASURES

Custom measures are considered too complex or unique to be included in the list of standard measures provided in the TRM. Also included are measures that may involve metered data, but require additional assumptions to arrive at a 'typical' level of savings as opposed to an exact measurement

While TRM measures are reviewed and approved by the PA PUC through the TRM update process, custom measures do not undergo the same approval process. The EDCs are not required to submit savings protocols for C&I custom measures to the Commission or the SWE for each measure/technology type prior to implementing the custom measure, however, the Commission recommends that site-specific custom measure protocols be established in general conformity to the International Performance Measurement and Verification Protocol (IPMVP)²³ or Federal Energy Management Program²⁴ M&V Guidelines. All evaluation sampled custom projects require a Site-Specific Measurement and Verification Plan (SSMVP) developed or approved for use by the EDC evaluator which must be available for SWE review. During Phase I of Act 129, the TWG developed custom measure protocols (CMPs) for calculating the energy and demand savings for several custom measures. CMPs approved during Phase I are considered available for use in Phase II by EDCs²⁵. The qualification for and availability of AEPS Credits and energy efficiency and demand response savings are determined on a case-by-case basis.

In addition, certain mass market programs in the residential sector are a subset of custom measures. These programs offer measures, or groups of measures, which are not included in the TRM. As with the C&I CMPs, during Phase I of Act 129, the TWG developed mass market protocols ("MMPs") for calculating the energy and demand savings associated with residential behavioral modification and low-income weatherization programs. MMPs approved during Phase I are considered available for use in Phase II by the EDCs.

An AEPS application must be submitted, containing adequate documentation fully describing the energy efficiency measures installed or proposed and an explanation of how the installed facilities qualify for AECs. The AEPS application must include a proposed evaluation plan by which the Administrator may evaluate the effectiveness of the energy efficiency measures provided by the installed facilities. All assumptions should be identified, explained and supported by documentation, where possible. The applicant may propose incorporating tracking and evaluation measures using existing data streams currently in use provided that they permit the Administrator to evaluate the program using the reported data.

To the extent possible, the energy efficiency measures identified in the AEPS application should be verified by the meter readings submitted to the Administrator.

1.17 IMPACT OF WEATHER

To account for weather differences within Pennsylvania, the Equivalent Full Load Hours (ELFH) for C&I HVAC measures are calculated based on the degree day scaling methodology. The EFLH values reported in the 2012 Connecticut Program Savings Documentation were adjusted using full load hours (FLH) from the US Department of Energy's ENERGY STAR Calculator²⁶. Degree day scaling ratios were calculated using heating degree day and cooling degree day values for seven Pennsylvania cities: Allentown, Erie, Harrisburg, Philadelphia, Pittsburgh, Scranton, and

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²³ http://www.evo-world.org/index.php?option=com_content&task=view&id=272&Itemid=279

²⁴ www1.eere.energy.gov/femp/pdfs/mv_guidelines.pdf

²⁵ If the CMPs use a top 100 hours approach for calculating peak demand savings, the protocol must be revised to address the new peak demand window definition prior to use.

²⁶ http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/ASHP_Sav_Calc.xls

Williamsport. These reference cities provide a representative sample of the various climate and utility regions in Pennsylvania.

In addition, several protocols in this TRM rely on the work and analysis completed in California, where savings values are adjusted for climate. These measures include Refrigeration – Auto Closers (Section 3.5.11 Section 3.5.11) (Section 3.5.11) and Refrigeration – Suction Pipes Insulation (Section 3.5.143.5.14). There—While there are sixteen California climate zones and seven Pennsylvania cities, all protocols relying on California work paper data will use a single climate zone. Very low risk is associated with this assumption due to the small contribution of savings from these measures to the overall portfolios (<0.1%) and the inherent differences in climate when comparing California weather to Pennsylvania weather, Based on comparable average dry bulb, wet bulb, and relative humidity as well as comparable cooling degree hours, the TRM uses California climate zone 114 to best estimate the savings of refrigeration measures. Each of the seven reference cities are mapped to a California climate zone as shown in Table 1-1 based on comparable number of cooling degree hours and average wet bulb temperatures. Section 3.22 and 3.24 will follow this mapping table.

Table 1-1: California CZ Mapping Table

Reference City	California Climate Zone
Allentown	15
Erie	9
Harrisburg	15
Philadelphia	15
Pittsburgh	4
Scranton	4
Williamsport	15

Furthermore, all the Pennsylvania zip codes are mapped to a reference city as shown in Appendix G: Zip Code Mapping Appendix G. In general, zip codes were mapped to the closest reference city because the majority of the state resides in ASHRAE climate zone 5. However, Philadelphia and a small area southwest of Harrisburg are assigned to ASHRAE climate zone 4. Therefore, any zip code in ASHRAE climate zone 4 were manually assigned to Philadelphia, regardless of distance.

1.18 Measure Applicability Based on Sector

Protocols for the residential sector quantify savings for measures typically found in residential areas under residential meters. Likewise, protocols for the C&I or Agriculture sectors quantify savings for measures typically found in C&I areas under C&I meters. However, there is some overlap where measure type, usage and the sector do not match.

Protocols in the residential and C&I sections describe measure savings based on the *application* or *usage characteristics* of the measure rather than how the measure is *metered*. For example, if a measure is found in a residential environment but is metered under a commercial meter, the residential sector protocol is used. On the other hand, if a measure is found in a commercial or agricultural environment but is metered under a residential meter, the commercial or agricultural sector protocol is used. This is particularly relevant for residential appliances that frequently appear in small commercial spaces (commercial protocol) and residential appliances that are used in residential settings but are under commercial meters (multi-family residences). In

addition, air sealing, duct sealing and ceiling/attic and wall insulation protocols and standards for residential measures should be used to estimate savings in two to four units multifamily complexes whereas air sealing and insulation protocols and standards for C&I measures should be applied in multifamily complexes with more than four units. Depending on the scale, an agricultural facility could be metered under a range of meters, but the agricultural measure protocol will supersede the meter type in the same fashion as listed for the other sectors.

1.19 ALGORITHMS FOR ENERGY EFFICIENT MEASURES

The following sections present measure-specific algorithms. Section 2 addresses residential sector measures and Section 3 addresses commercial and industrial sector measures. Section 4 addresses agricultural measures for residential, commercial, and industrial market sectors.

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2 RESIDENTIAL MEASURES

The following section of the TRM contains savings protocols for residential measures. This TRM does include an updated energy-to-demand factor for residential energy efficiency measures affecting the electric water heating end use. Due to time constraints, energy-to-demand factors for all other residential energy efficiency measures will be reviewed and updated in future TRMs.

2.1 LIGHTING

2.1.1 ENERGY STAR LIGHTING

Measure Name	ENERGY STAR Lighting
Target Sector	Residential Establishments
Measure Unit	Light Bulb or Fixture
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	CFL: 5.2 years ^{27,28} LED: 14.7 years ²⁹
Vintage	Replace on Burnout

Savings for residential energy efficient lighting products are based on a straightforward algorithm that calculates the difference between baseline and new wattage and the average daily hours of usage for the lighting unit being replaced. An "in-service" rate is used to reflect the fact that not all lighting products purchased are actually installed.

The parameter estimates in this section are for residential use only. If the split between residential and non-residential installations is unknown (e.g., an upstream program), EDCs can conduct data gathering to determine the percentage of bulbs sold and installed in various types of non-residential applications. EDCs should use the CF and hours of use by business type present in 3.1 Lighting for non-residential bulb savings estimates.

ELIGIBILITY

Definition of Efficient Equipment

In order for this measure protocol to apply, the high-efficiency equipment must be a screw-in ENERGY STAR CFL (general service or specialty bulb), screw-in ENERGY STAR LED bulb (general service or specialty bulb), LED fixture, ENERGY STAR fluorescent torchiere, ENERGY STAR indoor fluorescent fixture, ENERGY STAR outdoor fluorescent fixture, or an ENERGY STAR ceiling fan with a fluorescent light fixture. 30

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²⁷ Jump et al "Welcome to the Dark Side: The Effect of Switching on CFL Measure Life" indicates that the "observed life" of CFLs with an average rated life of 8,000 hours is 5.2 years due to increased on/off switching.

²⁸ Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets, this shift is assumed not to occur unitl mid-2020. Measure life is reduced to 5 years for CFLs installed June 2015 – May 2016. For every subsequent year, CFL measure life should be reduced by one year.

²⁹ All LED bulbs listed on the qualified ENERGY STAR product list have a lifetime of at least 15,000 hours. Assuming 2.8 hours per day usage, this equales to 14.7 years.

³⁰ The protocol also applies to products that are pending ENERGY STAR qualification.

Definition of Baseline Equipment

The baseline equipment is assumed to be a socket, fixture, torchiere, or ceiling fan with a standard or specialty incandescent light bulb(s).

An adjustment to the baseline wattage for general service and specialty screw-in CFLs and LEDs is made to account for the Energy Independence and Security Act of 2007 (EISA 2007), which requires that all general service lamps and some specialty lamps between 40W and 100W meet minimum efficiency standards in terms of amount of light delivered per unit of energy consumed. The standard was phased in between January 1, 2012 and January 1, 2014. This adjustment affects any efficient lighting where the baseline condition is assumed to be a general service, standard screw-in incandescent light bulb, or specialty, screw-in incandescent lamp.

For upstream buy-down, retail (time of sale), or efficiency kit programs, baseline wattages can be determined using the tables included in this protocol below. For direct install programs where wattage of the existing bulb is known, and the existing bulb was in working condition, wattage of the existing lamp removed by the program may be used in lieu of the tables below.

ALGORITHMS

The general form of the equation for the ENERGY STAR or other high-efficiency lighting energy savings algorithm is:

Total Savings = Number of Units \times Savings per Unit

ENERGY STAR CFL Bulbs (screw-in):

$$\Delta kWh/yr = \frac{Watts_{base} - Watts_{CFL}}{1000 \frac{kW}{W}} \times CFL_{hours} \times (1 + IE_{kWh}) \times 365 \frac{days}{yr} \times ISR_{CFL}$$

$$\Delta kW_{peak} = \frac{Watts_{base} - Watts_{CFL}}{1000 \frac{W}{kW}} \times CF \times (1 + IE_{kW}) \times ISR_{CFL}$$

ENERGY STAR LED Bulbs (screw-in):

$$\begin{split} &\Delta kWh/yr \\ &= \frac{Watts_{base} - Watts_{LED}}{1000 \frac{W}{kW}} \times CFL_{hours} \times (1 + IE_{kWh-LED}) \times 365 \frac{days}{yr} \times ISR_{CFL} \\ &\Delta kW_{peak} \end{split}$$

$$\begin{split} &= \frac{Watts_{base} - Watts_{LED}}{1000 \frac{W}{kW}} \times CF \times (1 + IE_{kW-LED}) \times 365 \frac{days}{yr} \times ISR_{CFL} \end{split}$$

ENERGY STAR Torchieres:

$$\begin{split} &\Delta \, kWh/yr \\ &= \frac{Watts_{\,base} - Watts_{\,Torch}}{1000\,\frac{W}{kW}} \times Torch_{\,hours} \times (1 + IE_{kWh}) \times 365 \frac{days}{yr} \times ISR_{\,Torch} \\ &\Delta kW_{peak} \end{split}$$

$$&= \frac{Watts_{\,base} - Watts_{\,Torch}}{1000\,\frac{W}{kW}} \times CF \times (1 + IE_{kW}) \times ISR_{\,Torch}$$

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ENERGY STAR Indoor CFL Fixture (hard-wired, pin-based):

$$\Delta kWh/yr = \frac{Watts_{base} - Watts_{IF}}{1000 \frac{W}{kW}} \times IF_{hours} \times (1 + IE_{kWh}) \times 365 \frac{days}{yr} \times ISR_{IF}$$

$$\Delta kW_{peak} = \frac{Watts_{base} - Watts_{IF}}{1000 \frac{W}{kW}} \times CF \times (1 + IE_{kW}) \times ISR_{IF}$$

ENERGY STAR Indoor LED Fixture (hard-wired, pin-based):

$$\Delta kWh/yr = \frac{Watts_{base} - Watts_{IF}}{1000 \frac{W}{kW}} \times IF_{hours} \times (1 + IE_{kWh}) \times 365 \frac{days}{yr} \times ISR_{IF}$$

$$\Delta kW_{peak} = \frac{Watts_{base} - Watts_{IF}}{1000 \frac{W}{kW}} \times CF \times (1 + IE_{kW-LED}) \times ISR_{IF}$$

ENERGY STAR Outdoor Fixture (hard wired, pin-based):

$$\Delta \, kWh/yr \\ = \frac{Watts_{\,base} - Watts_{OF}}{1000 \, \frac{W}{kW}} \times OF_{\,hours} \times 365 \, \frac{days}{yr} \times ISR_{OF} \\ \Delta kW_{peak} \\ = \frac{Watts_{\,base} - Watts_{OF}}{1000 \, \frac{W}{kW}} \times CF \times ISR_{OF}$$

Ceiling Fan with ENERGY STAR Light Fixture:

$$\Delta \, kWh/yr \\ = \frac{Watts_{\,base} - Watts_{fan}}{1000 \, \frac{W}{kW}} \times Fan_{\,hours} \times (1 + IE_{kWh}) \times 365 \frac{days}{yr} \times ISR_{fan} \\ \Delta \, kW_{peak} \\ = \frac{Watts_{\,base} - Watts_{fan}}{1000 \, \frac{W}{kW}} \times CF \times (1 + IE_{kW}) \times ISR_{fan}$$

DEFINITION OF TERMS

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Table 2-1: ENERGY STAR Lighting - References

Component	TypeUnit Value		Sources
Watts _{base} , Wattage of baseline case lamp/fixture	Variable <u>Watts</u>	EDC Data Gathering ³¹ or Table 2-2, Table 2-3 Table 2-3 & Table 2-4Table 2-4	
Watts _{CFL} , Wattage of CFL	Variable Watts	EDC Data Gathering	Data Gathering
CFL _{hours} , Average hours of use per day per CFL	$\frac{hours}{day}$ Fixed	2.8	5
IE _{kWh} , HVAC Interactive Effect for CFL energy	Variable <u>None</u>	EDC Data Gathering Default=Table 2-5	6
<i>IE_{kW}</i> , HVAC Interactive Effect for CFL demand	Variable None	EDC Data Gathering Default= Table 2-5 Table 2-5	6
ISR _{CFL} , In-service rate per CFL	Fixed <u>%</u>	97% ³²	2
Watts _{LED} , Wattage of LED	Variable Watts	EDC Data Gathering	Data Gathering
<i>IE</i> _{kWh-LED} , HVAC Interactive Effect for LED energy	Variable <u>None</u>	EDC Data Gathering Default=Table 2-6Table 2-6	6
IE _{kW-LED} , HVAC Interactive Effect for LED demand	Variable <u>None</u>	EDC Data Gathering Default=Table 2-6Table 2-6	6
Watts _{Torch} , Wattage of ENERGY STAR torchiere	Variable Watts	EDC Data Gathering	Data Gathering
Torch _{hours} , Average hours of use per day per torchiere	hours Fixed	3.0	1
ISR _{Torch} , In-service rate per Torchiere	Fixed%	83%	2
Watts _{IF} , Wattage of ENERGY STAR Indoor Fixture	Variable <u>Watts</u>	EDC Data Gathering	Data Gathering
<i>IF</i> _{hours} , Average hours of use per day per Indoor Fixture	FixedNone	2.6	1
ISR _{IF} , In-service rate per Indoor Fixture	Fixed%	95%	2
Watts _{OF} , Wattage of ENERGY STAR Outdoor Fixture	Variable Watts	EDC Data Gathering	Data Gathering
<i>OF</i> _{hours} , Average hours of use per day per Outdoor Fixture	hours Fixed	4.5	1
ISR _{OF} , In-service rate per Outdoor Fixture	Fixed%	87%	2
CF , Demand Coincidence	Fixed%	9.1%	3

SECTION 2: Residential Measures

Lighting

³¹ EDCs may use the wattage of the replaced bulb for directly installed program bulbs ³² For direct install program bulbs, EDCs have the option to use an evaluated ISR when verified through PA program primary research.

Component	Typ e <u>Unit</u>	Value	Sources
Factor			
Watts _{Fan} , Wattage of ENERGY STAR Ceiling Fan light fixture	Variable <u>Watts</u>	EDC Data Gathering	Data Gathering
Fanhours, Average hours of use per day per Ceiling Fan light fixture	hours Fixed day	3.5	4
ISR _{Fan} , In-service rate per Ceiling Fan fixture	Fixed%	95%	4

VARIABLE INPUT VALUES

Baseline Wattage Values - General Service Lamps

Baseline wattage is dependent on lumens, shape of bulb, and EISA qualifications. Commonly used EISA exempt bulbs include 3-way bulbs, globes with ≥5" diameter or ≤749 lumens, and candelabra base bulbs with ≤1049 lumens. See EISA legislation for the full list of exemptions.

For direct installation programs where the removed bulb is known, and the bulb is in working condition, EDCs may use the wattage of the replaced bulb in lieu of the tables below. 33 For bulbs with lumens outside of the lumen bins provided, EDCs should use the manufacturer rated comparable wattage as the Watts_{Base}. For EISA exempt bulbs, EDCs also have the option of using manufacturer rated comparable wattage as the Watts_{Base}, rather than the tables below.

To determine the Watts_{Base} for General Service Lamps³⁴, follow these steps:

- 1. Identify the rated lumen output of the energy efficient lighting product
- 2. Identify if the bulb is EISA exempt³⁵
- 3. In Table 2-2, find the lumen range into which the lamp falls (see columns (a) and (b).
- Find the baseline wattage (Watts_{Base}) in column (c) or column (d). If the bulb is exempt from EISA legislation, use column (c), else, use column (d).

Lighting

³³ Bulbs that are not installed during the home visit do not qualify for this exemption. This includes bulbs that are left for homeowners to install. In these instances, baseline wattages should be estimated using Table 2-2,

Table 2-3 Table 2-3, & Table 2-4.

³⁴ General Service Lamps are medium screw based bulbs that are not globe, bullet, candle,flood, reflector, or decorative shaped. These bulbs do encompass both twist/spiral and a-lamp shaped bulbs.

³⁵ The EISA 2007 standards apply to general service incandescent lamps. A complete list of the 22 incandescent lamps exempt from EISA 2007 is listed in the United States Department of *Energy Impact of EISA 2007 on General Service Incandescent Lamps: FACT SHEET.*

http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/general_service_incandescent_factsheet.pdf

Table 2-2: Baseline Wattage by Lumen Output for General Service Lamps (GSL)36

Minimum Lumens (a)	Maximum Lumens (b)	Incandescent Equivalent Watts _{Base} (Pre-EISA 2007) (c)	Watts _{Base} (Post-EISA 2007) (d)	Watts _{base} post 2020 ³⁷ (e)
2000	2600	150	72	23
1600	1999	100	72	23
1100	1599	75	53	18
800	1099	60	43	15
450	799	40	29	9
310	449	25	25	25

Baseline values in Table 2-2 column (e), Watts_{base} post 2020, should only be used in cost-effectiveness calculations for bulbs expected to be in use past 2020, such as LEDs. For these bulbs, Watts_{base} column (d) should be used for the savings calculations until 2020, followed by the values in column (e) for the remainder of the measure life.

For bulbs that do not fall within EISA regulations, such as exempt bulbs and bulbs with lumens greater than 2,600, the manufacturer rated equivalent wattage should be used as the baseline. The manufacturer rated wattage can vary by bulb type, but is usually clearly labeled on the bulb package. Note the EISA 2007 standards apply to general service incandescent lamps. A complete list of the 22 incandescent lamps exempt from EISA 2007 is listed in the United States Energy Independence and Securities Act.

Baseline Wattage Values - Specialty Bulbs

ENERGY STAR provides separate equivelantequivalent incandescent wattages for specialty and decorative bulb shapes. These shapes include candelabra base, globe, bullet, and shapes other than A-lamp bulbs.³⁸ For these bulbs, use the Watts_{Base} from

Table 2-3

Table 2-3

For EISA exempt specialty bulbs, use the Wattsbase value in column (c) in

Table 2-3

Table 2-3. Commonly used EISA exempt bulbs include 3-way bulbs, globes with ≥5" diameter or ≤749 lumens, and candelabra base bulbs with ≤1049 lumens. See the EISA legislation for the full list of exemptions.

To determine the Watts_{Base} for specialty/decorative lamps, follow these steps:

- 1. Identify the rated lumen output of the energy efficient lighting product
- 2. Identify if the bulb is EISA exempt

SECTION 2: Residential Measures

³⁶ Lumen bins and incadescent equivalent wattages from ENERGY STAR labeling requirements, Version 1.0 http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Lamps%20V1.0%20Final%20Draft%20Specification.pdf. EISA Standards from: United States Department of Energy. Impact of EISA 2007 on General Service Incandescent Lamps: FACT SHEET.

http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/general_service_incandescent_factsheet.pdf

³⁷ Example of cost-effectiveness calculation using column (e): If the LED life is 14.7 years, cost-effectiveness models for 2014 would model the first six years using column (d) as the Wattsbase, and the remaining 8.7 years using the Wattsbase, in column (e).
³⁸ ANSI Shapes for decorative bulbs: B, BA, C, CA, DC, F, and G

In

Table 2-3

3. Table 2-3, find the lamp shape of the bulb (see columns (a) or (b)).

n

Tahle 2-3

- 4. Table 2-3, find the lumen range into which the lamp falls (see columns (a) or (b)).
- 5. Find the baseline wattage (Watts_{Base}) in column (c) or column (d). If the bulb is exempt from EISA legislation, use column (c), else, use column (d).

Table 2-3: Baseline Wattage by Lumen Output for Specialty Lamps³⁹

Lumen Bins (decorative)	Lumen Bins (globe) (b)	Incandescent Equivalent Watts _{Base} (Pre-EISA 2007) (c)	Watts _{Base} (Post-EISA 2007) (d)
	1100-1300	150	72
	650-1099	100	72
	575-649	75	53
500-699	500-574	60	43
300-499	350-499	40	29
150-299	250-349	25	25
90-149		15	15
70-89		10	10

Baseline Wattage Values - Reflector or Flood Lamps

Reflector (directional) bulbs fall under legislation different from GSL and other specialty bulbs. For these bulbs, EDCs can use the manufacturer rated equivalent wattage as printed on the retail packaging, or use the default WattsBase (column (c)) in <u>Table 2-4Table 2-4</u> below.

Table 2-4. Default Baseline Wattage for Reflector Bulbs⁴⁰

Bulb Type (a)	Incandescent Equivalent (Pre-EISA) (b)	Watts _{Base} (Post-EISA) (c)
PAR20	50	35
PAR30	50	35
R20	50	45
PAR38	60	55
BR30	65	EXEMPT

³⁹ Lumen bins and incadescent equivalent wattages from ENERGY STAR labeling requirements, Version 1.0 http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Lamps%20V1.0%20Final%20Draft%20Specification.pdf. EISA Standards from: United States Department of Energy. Impact of EISA 2007 on General Service Incandescent Lamps: FACT SHEET.

⁴⁰ Based on manufacturer recommended replacements for EISA affected lamps. Manufacturer ratings may differ from the list above, in which case EDCs should default to the manufacturer equivalent rating.

Bulb Type (a)	Incandescent Equivalent (Pre-EISA) (b)	Watts _{Base} (Post-EISA) (c)
BR40	65	EXEMPT
ER40	65	EXEMPT
BR40	75	65
BR30	75	65
PAR30	75	55
PAR38	75	55
R30	75	65
R40	75	65
PAR38	90	70
PAR38	120	70
R20	≤ 45	EXEMPT
BR30	≤ 50	EXEMPT
BR40	≤ 50	EXEMPT
ER30	≤ 50	EXEMPT
ER40	≤ 50	EXEMPT

Interactive Effects Values

In the absence of EDC data gathering and analysis, the default values for Energy and Demand HVAC Interactive Effects are below. Due to the differences between LED and CFL technologies, these bulb types have separate interactive effects values.

Table 2-5: CFL Energy and Demand HVAC Interactive Effects by EDC41

EDC	IE _{kWh}	IE _{kW}
Duquesne	8%	13%
FE (Met-Ed)	-8%	13%
FE (Penelec)	1%	10%
FE (Penn Power)	0%	20%
FE (WPP)	-2%	30%
PPL	-6%	12%
PECO	9%	14%

SECTION 2: Residential Measures

⁴¹ HVAC Interactive Effects modeled through REM/Rate models, using EDC specific inputs. Values were weighted to the saturation of HVAC equipment and housing types present in each EDC service territory as reported in the Pennsylvania Statewide Residential End-Use and Saturation Study, 2012.

Table 2-6. LED Energy and Demand HVAC Interactive Effects by EDC42

EDC	IE _{kWh-LED}	IE _{kW-LED}
Duquesne	8%	10%
FE (MetEd)	-7%	11%
FE (Penn Elec)	1%	10%
FE (Penn Power)	0%	16%
FE (WPP)	-3%	27%
PPL	-6%	17%
PECO	9%	24%

MEASURE LIFE

Residential CFL Measure Life is 5.2 years 43,44

Residential LED Measure Life is 14.7 years45.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs", Final Report, October 1, 2004,. p. 104 (Table 9-7). This table adjusts for differences between logged sample and the much larger telephone survey sample and should, therefore, have less bias.
- 2. The ISR is based on an installation rate "trajectory" and includes savings for all program bulbs that are believed to ultimately be installed. Evaluations of the PECO Smart Lighting Discounts program determined a first year ISR of 78% for customers that purchased a bulb through a retailer or were provided a CFL through a give-a-way program⁴⁶. For future installations, the recommendations of the Uniform Methods Project ("UMP") can be

⁴² HVAC Interactive Effects modeled through REM/Rate models, using EDC specific inputs. Values were weighted to the saturation of HVAC equipment and housing types present in each EDC service territory as reported in the Pennsylvania Statewide Residential End-Use and Saturation Study, 2012.

Jump et al "Welcome to the Dark Side: The Effect of Switching on CFL Measure Life" indicates that the "observed life" of CFLs with an average rated life of 8,000 hours is 5.2 years due to increased on/off switching.

⁴⁴ Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets, this shift is assumed not to occur unitl mid-2020. Measure life is reduced to 5 years for CFLs installed June 2015 – May 2016. For every subsequent year, CFL measure life should be reduced by one year.

⁴⁵All LED bulbs listed on the qualified ENERGY STAR product list have a lifetime of at least 15,000 hours. Assuming 2.8 hours per day usage, this equates to 14.7 years.

⁴⁶ Evaluation Research Report: PECO Smart Lighting Discounts Program, Navigant Consulting, September 20, 2012

incorporated. The UMP recommends using the findings from the evaluation of the 2006-2008 California Residential Upstream Lighting Programs, which estimated that 99% of program bulbs get installed within three years, including the program year. Discounting the future savings back to the current program year reduces the ISR to 97%.

- EmPOWER Maryland 2012 Final Evaluation Report: Residential Lighting Program, Prepared by Navigant Consulting and the Cadmus Group, Inc., March 2013, Table 50.
- 4. ENERGY STAR Ceiling Fan Savings Calculator (Calculator updated April 2009). Hours based on ENERGY STAR calculator for the Mid-Atlantic region defer to this value since it is recognized that ceiling fans are generally installed in high-use areas such as kitchens, living rooms and dining rooms. Ceiling fans are also installed in bedrooms, but the overall average HOU for this measure is higher than the average of all CFLs (2.8) and indoor fixtures (2.6) since these values incorporate usage in low-use areas such as bathrooms and hallways where ceiling fans are generally not installed.
- Nexus Market Research, "Residential Lighting Markdown Impact Evaluation", Final Report, January 20, 2009. Table 6-1.

Additionally, the following studies were reviewed and analyzed to support the "Residential Lighting Markdown Inpact Evaluation":

- a. Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs", Final Report, October 1, 2004. Table 9-7.
- b. CFL Metering Study, Final Report. Prepared for PG&E, SDG&E, and SCE by KEMA, Inc. February 25, 2005. Table 4-1.
- c. Nexus Market Research, ""Process and Impact Evaluation of the Efficiency Maine Lighting Program"", April 2007. Table 1-7."
- d. Nexus Market Research, "Residential Lighting Markdown Impact Evaluation", Final Report, January 20, 2009. Table 6-1.
- KEMA, Inc., "Final Evaluation Report: Upstream Lighting Program." Prepared from the California Public Utilities Commission, Febuary 8, 2010. Table 18.
- f. Itron, Inc. "Verification of Reported Energy and Peak Savings from the EmPOWER Maryland Energy Efficiency Programs." Prepared for the Maryland Public Service Commission, April 21, 2011. Table 3-6.
- g. TecMarket Works, "Duke Energy Residential Smart Saver CFL Program in North Carolina and South Carolina", February 2011. Table 29.
- h. Glacier Consulting Group, LLC. "Adjustments to CFL Operating Hours-Residential." Memo to Oscar Bloch, Wisconsin DOA. June 27, 2005.
- New Jersey's Clean Energy Program Residential CFL Impact Evaluation and Protocol Review. KEMA, Inc. September 28, 2008. pg. 21.
- 6. GDS Simulation Modeling, September-November 2013.
- Lumen bins and Pre-EISA baselines are consistent with ENERGY STAR lamp labeling requirements, Version 1.0. Post-EISA baselines are the maximum EISA complaint equivalent incandescent wattages based on EISA lumen bins.

2.1.2 RESIDENTIAL OCCUPANCY SENSORS

Measure Name	ENERGY STAR Occupancy Sensors
Target Sector	Residential Establishments
Measure Unit	Occupancy Sensor
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	10 years ⁴⁷
Vintage	Retrofit

ELIGIBILITY

This protocol is for the installation of occupancy sensors inside residential homes or common areas.

ALGORITHMS

$$\Delta \, kWh/yr \qquad \qquad = \frac{Watts_{controlled}}{1000 \frac{W}{kW}} \times (RH_{old} - RH_{new}) \times 365 \frac{days}{yr}$$

$$\Delta kW_{peak} \qquad \qquad = 0$$

DEFINITION OF TERMS

Table 2-7: Residential Occupancy Sensors Calculations Assumptions

Component	Type <u>Unit</u>	Value	Source
Watts _{controlled} , Wattage of the fixture being controlled by the occupancy sensor	Variab le <u>kW</u>	EDC's Data Gathering	AEPS Application; EDC's Data Gathering
RH _{old} , Daily run ⁴⁸ hours before installation	Fixed <u>Hours</u>	2.8	1
RH_{new} , Daily run hours after installation	Fixed Hours	2.0 (70% of RHold)	2

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SECTION 2: Residential Measures

⁴⁷ GDS Associates, Inc. (2007). Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures. Prepared for The New England State Program Working Group.

MEASURE LIFE

The expected measure life is 10 years 49.

SOURCES

- Nexus Market Research, "Residential Lighting Markdown Impact Evaluation", Final Report, January 20, 2009. Table 6-1. Reference Table 2-1: ENERGY STAR Lighting for full citation.
- Lighting control savings fractions consistent with current programs offered by National Grid, Northeast Utilities, Long Island Power Authority, NYSERDA, and Energy Efficient Vermont

GDS Associates, Inc. (2007). Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures. Prepared for The New England State Program Working Group.

SECTION 2: Residential Measures

Lighting

2.1.3 ELECTROLUMINESCENT NIGHTLIGHT

Measure Name	Electroluminescent Nightlight
Target Sector	Residential Establishments
Measure Unit	Nightlight
Unit Energy Savings	29.49 kWh
Unit Peak Demand Reduction	0 kW
Measure Life	8 years ⁵⁰
Vintage	Replace on Burnout

Savings from installation of plug-in electroluminescent nightlights are based on a straightforward algorithm that calculates the difference between existing and new wattage and the average daily hours of usage for the lighting unit being replaced. An "installation" rate is used to modify the savings based upon the outcome of participant surveys, which will inform the calculation. Demand savings is assumed to be zero for this measure.

ELIGIBILITY

This measure documents the energy savings resulting from the installation of an electroluminescent night light instead of a standard night light. The target sector is primarily residential.

ALGORITHMS

The general form of the equation for the electroluminescent nightlight energy savings algorithm is:

$$\Delta \, kWh/yr \qquad \qquad = \frac{((W_{base} \, \times \, HOU_{base} \,) - \, (W_{ee} \, \times \, HOU_{ee} \,)) \, \times 365 \frac{days}{yr} \times \, ISR_{NL}}{1000 \frac{W}{kW}}$$

 ΔkW_{peak} = 0 (assumed)

DEFINITION OF TERMS

⁵⁰ Southern California Edison Company, "LED, Electroluminescent & Fluorescent Night Lights", Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2 & p. 3.

Table 2-8: Electroluminescent Nightlight - References

Component	Type Unit	Value	Sources
$W_{\rm NL_{\perp}}W_{\rm ee}$, Watts per electroluminescent nightlight	Variable Watts	0.03 or EDC Data Gathering Default = 0.03	1
$W_{ ext{base}}W_{ ext{base}}$. Watts per baseline nightlight	Variable Watts	7 (for incandescent- bulbs) or EDC Data Gathering_ Default := 7	2
h _{NL_} HOU _{ee} , Hours-of-Use per day of electroluminescent nightlight	Fixed_hours day	24	3
$h_{inc}HOU_{base}$, Hours per baseline nightlight	Fixed hours day	12	2
ISR _{NL_} , In-Service Rate per electroluminescent nightlight	Variable None	0.97 or EDC Data Gathering Default ÷= 0.97	PA CFL ISR value
Measure Life (EUL)	Fixed	8	4

DEEMED ENERGY SAVINGS

=
$$\left((7 \times 12) - (.03 \times 24) \right) \times \frac{_{365} \frac{days}{yr}}{_{1000} \frac{W}{kW}} \times 0.97 = 29.49 \, kWh$$

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Limelite Equipment Specification. Personal Communication, Ralph Ruffin, El Products, 512-357-2776/ ralph@limelite.com.
- Southern California Edison Company, "LED, Electroluminescent & Fluorescent Night Lights", Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2 & p. 3.
- 3. As these nightlights are plugged in without a switch, the assumption is they will operate 24 hours per day.
- 4. Southern California Edison Company, "LED, Electroluminescent & Fluorescent Night Lights", Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2 & p. 3.

SECTION 2: Residential Measures

2.1.4 LED NIGHTLIGHT

Measure Name	LED Nightlight
Target Sector	Residential Establishments
Measure Unit	LED Nightlight
Unit Energy Savings	22 - <u>25.49</u> kWh
Unit Peak Demand Reduction	0 kW
Measure Life	8 years ⁵¹
Vintage	Replace on Burnout

Savings from installation of LED nightlights are based on a straightforward algorithm that calculates the difference between existing and new wattage and the average daily hours of usage for the lighting unit being replaced. An "installation" rate is used to modify the savings based upon the outcome of participant surveys, which will inform the calculation. Demand savings is assumed to be zero for this measure.

ELIGIBILITY

This measure documents the energy savings resulting from the installation of an LED night light instead of a standard night light. The target sector is primarily residential.

ALGORITHMS

Assumes a 1 Watt LED nightlight replaces a 7 Watt incandescent nightlight. The nightlight is assumed to operate 12 hours per day 52 , 365 days per year; estimated useful life is 8 years (manufacturer cites 11 years 100,000 hours). Savings are calculated using the following algorithm:

$$\Delta kWh/yr = ((W_{base} - W_{NL}) \times \left(\frac{HOU \times 365 \frac{days}{yr}}{1000 \frac{W}{kW}}\right)) \times ISR$$

$$\Delta kW_{peak} = 0 \text{ (assumed)}$$

SECTION 2: Residential Measures

⁵¹ Southern California Edison Company, "LED, Electroluminescent & Fluorescent Night Lights", Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2. and p.3.

DEFINITION OF TERMS

Table 2-9: LED Nightlight - References

Component	Type <u>Unit</u>	Value	Sources
Wattsbase Wbase, Watts per baseline	Variable <u>None</u>	Default = 7 Watts; EDC Data Gathering Default = 7	EDC Data Gathering
Watts _{NL} W _{NL} ,Watts per LED Nightlight	Variable <u>None</u>	Default= 1 Watt; EDC Data Gathering Default = 1	EDC Data Gathering
NL _{hours} HOU , Hours-of-Use	Fixed_hours_day	12	1
ISR _{NL} , In-Service Rate per electroluminescent nightlight	Fixed %	0.97 or Default = 97%	PA CFL ISR value
EUL	Fixed	8 years	4

DEEMED SAVINGS

The default energy savings is based on a delta watts assumption ($W_{\text{base}} - W_{\text{NL}}$) of 6 watts.

∆kWh

$$= \left((6) \times \left(\frac{12 \times 365 \frac{days}{yr}}{1000 \frac{W}{kW}} \right) \right) \times .97 = 25.49 \ kWh^{53}$$

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

1. Southern California Edison Company, "LED, Electroluminescent & Fluorescent Night Lights", Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2 & p. 3.

SECTION 2: Residential Measures

2.1.5 HOLIDAY LIGHTS

Measure Name	Holiday Lights
Target Sector	Residential Applications
Measure Unit	One 25-bulb Strand of Holiday lights
Unit Energy Savings	10.6 kWh per strand
Unit Peak Demand Reduction	0 kW
Measure Life	10 years ^{54,55}
<u>Vintage</u>	Replace on Burnout

<u>Light Emitting Diode (LED) holiday lights are a relatively new application for this existing technology.</u> LED holiday lights reduce <u>light strand</u> energy consumption up <u>by</u> to 90%. Up to 25 strands can be connected end-to-end in terms of residential grade lights. Commercial grade lights require different power adapters and as a result, more strands can be connected end-to-end.

ELIGIBILITY

This protocol documents the energy savings attributed to the installation of LED holiday lights indoors and outdoors. LED lights must replace traditional incandescent holiday lights.

ALGORITHMS

$$\Delta kWh/yr_{C9} = \frac{[(INC_{C9} - LED_{C9}) \times \#BULBS \times \#STRANDS \times HR]}{1000\frac{W}{kW}}$$

$$\Delta kWh/yr_{C7} = \frac{[(INC_{C7} - LED_{C7}) \times \#BULBS \times \#STRANDS \times HR]}{1000\frac{W}{kW}}$$

$$\Delta kWh/yr_{mini} = \frac{[(INC_{mini} - LED_{mini}) \times \#BULBS \times \#STRANDS \times HR]}{1000\frac{W}{kW}}$$

Key assumptions

- All estimated values reflect the use of residential (25ct.). per strand).) bulb LED holiday lighting.
- Secondary impacts for heating and cooling were not evaluated.
- It is assumed that 50% of rebated lamps are of the "mini" variety, 25% are of the "C7" variety, and 25% are of the "C9" variety. If the lamp type is known or fixed by program design, then the savings can be calculated as described by the algorithms above. Otherwise, the savings for the "mini", "C7", and "C9" varieties should be weighted by 0.5, 0.25 and 0.25 respectively.

SECTION 2: Residential Measures

⁵⁴ http://www.energyideas.org/documents/factsheets/HolidayLighting.pdf

⁵⁵ The DSMore Michigan Database of Energy Efficiency Measures: Based on spreadsheet calculations using collected data: Franklin Energy Services; "FES-L19 – LED Holiday Lighting Calc Sheet"

DEFINITION OF TERMS

Table 2-10: Holiday Lights Assumptions

Parameter	TypeUnit	Value	Source
<i>LED_{mini}</i> , Wattage of LED mini bulbs	Fixed Watts	0.08 ₩	1
<i>INC</i> _{mini} , Wattage of incandescent mini bulbs	<u>Watts</u> Fixed	0.48 W	1
LED _{C7} , Wattage of LED C7 bulbs	<u>Watts</u> Fixed	0.48 ₩	1
INC _{C7} , Wattage of incandescent C7bulbs	<u>Watts</u> Fixed	6.0 ₩	1
LED _{C9} , Wattage of LED C9 bulbs	<u>Watts</u> Fixed	2.0 ₩	1
INC _{C9} , Wattage of incandescent C9 bulbs	<u>Watts</u> Fixed	7.0 ₩	1
W _{Mini} _Percentage of holiday lights that are "mini"	Fixed%	50%	1
W _{C7} , Percentage of holiday lights that are "C7"	Fixed%	25%	1
W _{C9} , Percentage of holiday lights that are "C9"	Fixed%	25%	1
# _{Bulbs} , Number of bulbs per strand	Variable <u>bulbs/strand</u>	Variable EDC Data Gathering Default: 25 per strand	EDC- Data- Gathering Section- 2.38.1-3
#Strands , Number of strands of lights per package	Variablestrands/package	Variable EDC Data Gathering Default: 1 strand	EDC- Data- Gathering Section- 2.38.1-3
Hr, Annual hours of operation	FixedHours/yr	150	1

DEEMED SAVINGS

The deemed savings for installation of LED C9, C7, and mini lights is 18.7 kWh, 20.7 kWh, and 1.5 kWh, respectively. The weighted average savings are 10.6 kWh per strand. There are no demand savings as holiday lights only operate at night. Since the lights do not operate in the summer, the coincidence factor for this measure is 0.0.

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MEASURE LIFE

Measure life is 10 years. 56,51

EVALUATION PROTOCOL

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings. As these lights are used on a seasonal basis, verification must occur in the winter holiday season. Given the relatively small amount of impact evaluation risk that this measure represents, and given that the savings hinge as heavily on the actual wattage of the supplanted lights than the usage of the efficient LED lights, customer interviews should be considered as an appropriate channel for verification.

SOURCES

- The DSMore Michigan Database of Energy Efficiency Measures: Based on spreadsheet calculations using collected data.
- 2. http://www.energyideas.org/documents/factsheets/HolidayLighting.pdf
- 3. Key Assumptions:
 - All estimated values reflect the use of residential (25ct.). per strand).) bulb LED holiday lighting.
 - b. Secondary impacts for heating and cooling were not evaluated.
 - c. It is assumed that 50% of rebated lamps are of the "mini" variety, 25% are of the "C7" variety, and 25% are of the "C9" variety. If the lamp type is known or fixed by program design, then the savings can be calculated as described by the algorithms.follows. Otherwise, the savings for the "mini", "C7", and "C9" varieties should be weighted by 0.5, 0.25 and 0.25 respectively.

Lighting

⁵⁶The DSMore Michigan Database of Energy Efficiency Measures: Based on spreadsheet calculations using collected data:

Franklin Energy Services; "FES-L19 – LED Holiday Lighting Calc Sheet"

ESF. http://www.energyideas.org/documents/factsheets/HolidayLighting.pdf

2.2 HVAC

2.2.1 ELECTRIC HVAC

Measure Name	Electric HVAC		
Target Sector	Residential Establishments		
Measure Unit	AC Unit, ASHP Unit, or GSHP Unit		
Unit Energy Savings	Varies		
Unit Peak Demand Reduction	Varies		
Measure Life	Varies (See Appendix A?)		
Vintage	Replace on Burnout, Retrofit (Maintenance and Proper Sizing), Early Replacement		

The method for determining residential high-efficiency cooling and heating equipment energy impact savings is based on algorithms that determine a central air conditioner or heat pump's cooling/heating energy use and peak demand contribution. Input data is based both on fixed assumptions and data supplied from the high-efficiency equipment AEPS application form or EDC data gathering.

The algorithms applicable for this program measure the energy savings directly related to the more efficient hardware installation.

Larger commercial air conditioning and heat pump applications are dealt with in Section 3.2.

ELIGIBILITY

This measure requires the purchase of an ENERGY STAR Air Conditioner, Air Source Heat Pump, Ground Source Heat Pump, proper sizing of a central air conditioner, central air conditioner or air source heat pump maintenance, installation of a desuperheater on an existing Ground Source Heat Pump, or installation of a new high efficiency fan on an existing furnace. The baseline condition is an existing standard efficiency electric heating system, a gas or electric furnace with a standard efficiency furnace fan, or a ground source heat pump without a desuperheater.

The following sections detail how this measure's energy and demand savings were determined.

ALGORITHMS

Central A/C and Air Source Heat Pump (ASHP) (High Efficiency Equipment Only)

This algorithm is used for the installation of new high efficiency A/C and ASHP equipment.

$$\begin{array}{ll} \Delta kWh/yr & = \Delta kWh_{cool} + \Delta kWh_{heat} \\ = \frac{CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e}\right) \times EFLH_{cool} \\ \\ \Delta kWh_{heat} (\text{ASHP Only}) & = \frac{CAPY_{heat}}{1000\frac{W}{kW}} \times \left(\frac{1}{HSPF_b} - \frac{1}{HSPF_e}\right) \times EFLH_{heat} \\ \\ \Delta kWh_{peak} & = \frac{CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{1}{EER_b} - \frac{1}{EER_e}\right) \times CF \end{array}$$

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Central A/C (Proper Sizing)58

This algorithm is specifically intended for new units (Quality installation).

$$\Delta kWh/yr = \frac{CAPY_{cool}}{(SEER_q \times 1000 \frac{W}{kW})} \times EFLH_{cool} \times PSF$$

$$\Delta kWh_{peak} = \frac{CAPY_{cool}}{(EER_q \times 1000 \frac{W}{kW})} \times CF \times PSF$$

Central A/C and ASHP (Maintenance)

This algorithm is used for measures providing services to maintain, service or tune-up central A/C and ASHP units. The tune-up must include the following at a minimum:

- · Check refrigerant charge level and correct as necessary
- Clean filters as needed
- · Inspect and lubricate bearings
- · Inspect and clean condenser and, if accessible, evaporator coil

$$\begin{array}{ll} \Delta kWh/yr & = \Delta kWh_{cool} + \Delta kWh_{heat} \\ \Delta kWh_{cool} & = \frac{CAPY_{cool}}{(1000\frac{W}{kW} \times SEER_m)} \times EFLH_{cool} \times MF_{cool} \\ \Delta kWh_{heat}(ASHP\ Only) & = \frac{CAPY_{heat}}{(1000\frac{W}{kW} \times HSPF_m)} \times EFLH_{heat} \times MF_{heat} \\ \Delta kW_{peak} & = \frac{CAPY_{cool}}{(1000\frac{W}{kW} \times EER_m)} \times CF \times MF_{cool} \\ \end{array}$$

Ground Source Heat Pumps (GSHP)

This algorithm is used for the installation of new GSHP units. For GSHP systems over 65,000 BTUh, see commercial algorithm stated in Section 3.2.3.

$$\begin{array}{lll} \Delta kWh & = \Delta kWh_{cool} + \Delta kWh_{heat} \\ \mathrm{COP}_{\mathrm{sys}} & = COP_{g} \times GSHPDF \\ \mathrm{EER}_{\mathrm{sys}} & = EER_{g} \times GSHPDF \\ \\ \Delta kWh_{cool} & = \frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{SEER_{b}} - \frac{1}{EER_{\mathrm{sys}} \times GSER}\right) \times EFLH_{cool} \\ \Delta kWh_{heat} & = \frac{CAPY_{heat}}{1000 \frac{W}{kW}} \times \left(\frac{1}{HSPF_{b}} - \frac{1}{COP_{\mathrm{sys}} \times GSOP}\right) \times EFLH_{heat} \\ \Delta kW & = \frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{EER_{b}} - \frac{1}{EER_{\mathrm{sys}} \times GSPK}\right) \times CF \end{array}$$

GSHP Desuperheater

This algorithm is used for the installation of a desuperheater for a GSHP unit.

$$\Delta kWh = \frac{EFDSH \times \frac{1}{EF_{Base}} \times HW \times 365 \frac{days}{yr} \times 8.3 \frac{lb}{gal} \times 1 \frac{Btu}{lb \cdot {}^{\circ}F} \times (T_{hot} - T_{cold})}{3413 \frac{Btu}{kWh}}$$

$$= 567 \ kWh$$

$$= EDSH \times ETDF$$

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⁵⁸ Proper sizing requires Manual J calculations, following of ENERGY STAR QI procedures, or similar calculations.

Furnace High Efficiency FanThis algorithm is used for the installation of new high efficiency furnace fans.

∆kWh_{heat} = HFS = 0, 0 = PDFS = CFS ΔkWh_{cool} ΔkW_{peak}

DEFINITION OF TERMS

Table 2-11: Residential Electric HVAC - References

	Component	Type Unit	Value	Sources
	CAPY _{cool} , The cooling capacity (output in Btuh) of the central air conditioner or heat pump being installed ⁵⁹ ⁶⁰	Variable_ None	EDC Data Gathering_	AEPS Application; EDC Data Gathering
	CAPY _{heat} , The heating capacity (output in Btuh) of the central air conditioner or heat pump being installed ⁶¹ .2-	Variable_ None	EDC Data Gathering_	AEPS Application; EDC Data Gathering
		Btu W⋅h Fixed Non €	Replace on Burnout: 13 SEER_(Central A/C) or 14 SEER (ASHP)	1
	SEER _b , Seasonal Energy Efficiency Ratio of the Baseline Unit		Early Retirement	
		BtuVariableN wh	EDC Data Gathering	
	Natio of the baseline offic		Default = 11 (Central A/C) or	13; EDC Data Gathering
			12 (ASHP)- Early Retirement: Default 10- SEER or EDC Data- Gathering	Gathering
	SEER _e , Seasonal Energy Efficiency Ratio of the qualifying unit being installed ⁶²	Btu Variable None	EDC Data Gathering_	AEPS Application; EDC Data Gathering
	SEER _m , Seasonal Energy Efficiency Ratio of the Unit receiving maintenance	BtuVariableN one	EDC Data Gathering_ Default= 11 (Central A/C) or 12 (ASHP)	13; EDC Data Gathering
	EER _b , Energy Efficiency Ratio of the Baseline Unit	Btu W⋅hFixedNon e	Replace on Burnout: 11.3	2
		Wh Variable None	EDC Data Gathering_ Default= 8.69	14; EDC Data Gathering

HVAC

⁵⁹ This data is obtained from the AEPS Application Form or EDC's data gathering based on the model number.

⁶⁰ This data is obtained from the AEPS Application Form based on the model number or from EDC data gathering.

^{61 &}lt;u>Ibid.</u>
62 This data is obtained from the AEPS Application Form or EDC's data gathering based on the model number bid.

Component	Type <u>Unit</u>	Value	Sources
\textit{EER}_{e} , Energy Efficiency Ratio of the unit being installed $^{\underline{639}}$	$\frac{\frac{Btu}{W \cdot h} Fixed \underline{Non}}{\underline{\mathbf{e}}}$	$\frac{11.3}{13}$ $(11.3/13) \times XSEER$	2
EER _g , EER of the ground source heat pump being installed. Note that EERs of GSHPs are measured differently than EERs of air source heat pumps (focusing on entering water temperatures rather than ambient air temperatures). The equivalent SEER of a GSHP can be estimated by multiplying EERg by 1.02	Btu _{W-h} Variable <u>N</u> one	EDC Data Gathering_	AEPS Application; EDC's Data Gathering
EER _{sys} , Ground Source Heat Pump effective system EER	Btu W⋅h One	Calculated	Calculated
<i>EER_m</i> , Energy Efficiency Ratio of the Unit receiving maintenance	Btu W⋅h One	EDC Data Gathering_ Default= 8.69	14; EDC Data Gathering
GSER , Factor used to determine the SEER of a GSHP based on its EERg	Btu W⋅h EixedNon <u>e</u>	1.02	3
<i>EFLH</i> _{cool} , Equivalent Full Load Hours of operation during the cooling season for the average unit	hours yr Default	Allentown Cooling = 487 Hours Erie Cooling = 389 Hours Harrisburg Cooling = 551 Hours Philadelphia Cooling = 591 Hours Pittsburgh Cooling = 432 Hours Scranton Cooling = 417 Hours Williamsport Cooling = 422 Hours	4
	Optional	An EDC can either use the Alternate EFLH Table or estimate its own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Table 2-12); EDC Data Gathering

63 Ibid.

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Component	Type Unit	Value	Sources
<i>EFLH</i> _{heat} , Equivalent Full Load Hours of operation during the heating season for the average unit	hours yr Hours	Allentown Heating = 1,193 Hours Erie Heating = 1,349 Hours Harrisburg Heating = 1,103 Hours Philadelphia Heating = 1,060 Hours Pittsburgh Heating = 1,209 Hours Scranton Heating = 1,296 Hours Williamsport Heating = 1,251 Hours	4
	Optional	An EDC can either use the Alternate EFLH Table or estimate its own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Table 2-13); EDC Data Gathering
PSF , Proper Sizing Factor or the assumed savings due to proper sizing and proper installation	FixedNone	<u>0.05</u> 5%	5
MF _{cool} , Maintenance Factor or assumed savings due to completing recommended maintenance on installed cooling equipment	FixedNone	<u>0.05</u> 5%	15
MF _{heat} , Maintenance Factor or assumed savings due to completing recommended maintenance on installed heating equipment	FixedNone	<u>0.05</u> 5%	15
CF , Demand Coincidence Factor (See Section 1.5)	Fixed%	70% 64.7%	6
HSPF _b , Heating Seasonal	$\frac{\frac{Btu}{W \cdot h} Fixed \underline{Non}}{\underline{\mathbf{e}}}$	Replace on Burnout: 8.27.7	7
Performance Factor of the Baseline Unit	Btu W⋅h one	Early Retirement: Early Replacement: -EDC Data Gathering_ Default = 6.9Default=	EDC Data Gathering20
HSPF_{e} , Heating Seasonal Performance Factor of the unit being installed 64	Btu W∙h VariableN one	EDC Data Gathering_	AEPS Application; EDC's Data Gathering
HSPF _m , Heating Seasonal Performance Factor of the unit receiving maintenance	Btu W⋅h E	6. <u>98</u>	13 20
${\it COP_g}$, Coefficient of Performance. This is a measure of the efficiency of a heat pump	Variable <u>Non</u> <u>e</u>	EDC Data Gathering	AEPS Application; EDC's Data Gathering

⁶⁴ This data is obtained from the AEPS Application Form or EDC's data gathering.

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Co	mponent	Type Unit	Value	Sources
	SHPDF , Ground Source Heat mp De-rate Factor	FixedNone	0.885	19 (Engineering Estimate - See 2.1.5System Performance of Ground Source Heat Pumps)
	PP _{sys} , Ground Source Heat Pump ective system COP	Variable	Calculated	Calculated
HS	SOP , Factor to determine the SPF of a GSHP based on its OPg	FixedNone	3.413	8
the cor	SPK , Factor to convert EERg to equivalent EER of an air nditioner to enable comparisons the baseline unit	<u>None</u> Fixed	0.8416	9
	DSH , Energy Factor per superheater	<u>None</u> Fixed	<u>0.17</u> 17%	10, 11
	<i>DSH</i> ⁶⁵ , Fixed savings per superheater	<u>kWh/yr</u> Fixed	576 - <u>567</u> kWh	Calculated
	base, Energy Factor of Electric ater Heater	NoneFixed	0.904	Table 2-42
HV	V, Daily Hot Water Use	FixedGallon s/day	50	Table 2-42
Th	, Hot Water Temperature	<u>°F</u> Fixed	123	Table 2-42
T _c ,	Cold Water Temperature	<u>°F</u> Fixed	55	Table 2-42
	DF , Fixed "Energy to Demand ctor per desuperheater	<u>None</u> Fixed	0.00008294	Table 2-42
	OSH , Assumed peak-demand vings per desuperheater	Fixed <u>kW</u>	0.05- kW	Calculated
	S , Assumed heating season vings per furnace high efficiency	FixedkWh	311- kWh	16
	S , Assumed cooling season vings per furnace high efficiency	Fixed <u>kWh</u>	135- kWh	17
	OFS , Assumed peak-demand vings per furnace high efficiency	Fixed <u>kW</u>	0.114 kW	18

ALTERNATE EQUIVALENT FULL LOAD HOUR (EFLH) TABLES

Table 2-12 and Table 2-13 below show cooling EFLH and heating EFLH, respectively, by city and for each EDC's housing demographics. EFLH values are only shown for cities that are close to customers in each EDC's service territory. In order to determine the most appropriate EFLH

⁶⁵ GSHP desuperheaters are generally small, auxiliary heat exchangers that uses superheated gases from the GSHP's compressor to heat water. This hot water then circulates through a pipe to the home's storage water heater tank.

value to use for a project, first select the appropriate EDC, then, from that column, pick the closest city to the project location. The value shown in that cell will be the EFLH value to use for the project.

Table 2-12: Alternate Cooling EFLH

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	431	528	453	N/A	N/A	N/A	523
Erie	N/A	418	N/A	413	N/A	397	N/A
Harrisburg	487	N/A	506	580	N/A	N/A	N/A
Philadelphia	N/A	N/A	536	N/A	N/A	N/A	651
Pittsburgh	N/A	468	N/A	458	417	448	N/A
Scranton	376	454	N/A	N/A	N/A	N/A	N/A
Williamsport	N/A	N/A	N/A	447	N/A	N/A	N/A

Table 2-13: Alternate Heating EFLH

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	1112	1057	1122	N/A	N/A	N/A	1320
Erie	N/A	1204	N/A	1317	N/A	1376	N/A
Harrisburg	1028	N/A	1035	1077	N/A	N/A	N/A
Philadelphia	N/A	N/A	1001	N/A	N/A	N/A	1165
Pittsburgh	N/A	1068	N/A	1175	1274	1234	N/A
Scranton	1203	1151	N/A	N/A	N/A	N/A	N/A
Williamsport	N/A	N/A	N/A	1218	N/A	N/A	N/A

SYSTEM PERFORMANCE OF GROUND SOURCE HEAT PUMPS

Ground Source heat pump nameplate AHRI ratings do not include auxiliary pumping energy for ground loop water distribution. Based on McQuay heat pump design guidelines (Ref. #19), it is estimated that approximately a 1/3 HP pump would be required to be paired with a 2.5 ton Ground Source Heat Pump (assuming 3 GPM//ton design flow and 200 ft./ton of 1-inch tubing). At 7.5 GPM, a 1/3 HP pump would consume approximately 0.23 kW (7.5 GPM @ 30 ft. head). Assuming a 2 kW load for the heat pump itself, this would amount to a roughly 11.5% increase in system energy. The system COP de-rate factor would then be 0.885.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200. Code of Regulations 10 CFR 430. http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/75
- 2. Average EER for SEER 13 units as calculated by EER = -0.02 × SEER² + 1.12 × SEER based on U.S. DOE Building America House Simulation Protocol, Revised 2010.
- 3. VEIC estimate. Extrapolation of manufacturer data.

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- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners⁶⁶ and 40% oversizing of heat pumps.⁶⁷
- Northeast Energy Efficiency Partnerships, Inc., "Strategies to Increase Residential HVAC Efficiency in the Northeast", (February 2006): Appendix C Benefits of HVAC Contractor Training: Field Research Results 03-STAC-01, page 46.
- 6. Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal.

 Aug/Sept. 2011. Found at http://www.sciencedirect.com/science/article/pii/S1040619011001941 Based on an analysis of six different utilities by Proctor Engineering.
- 7. Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200. Code of Regulations 10 CFR 430. http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/75
- 8. Engineering calculation, HSPF/COP=3.413.
- 9. VEIC Estimate. Extrapolation of manufacturer data.
- "Residential Ground Source Heat Pumps with Integrated Domestic Hot Water Generation: Performance Results from Long-Term Monitoring", U.S. Department of Energy, November 2012.
- 11. Desuperheater Study, New England Electric System, 1998 42 U.S.C.A 6295(i) (West Supp. 2011) and 10 C.F.R. 430.32 (x) (2011).
- Northeast Energy Efficiency Partnerships, Inc., "Benefits of HVAC Contractor Training", (February 2006): Appendix C Benefits of HVAC Contractor Training: Field Research Results 03-STAC-01.
- 13. Minimum Federal Standards for new Central Air Conditioners and Air Source Heat Pumps between 1990 and 2006 based on VEIC estimates-2014 Pennsylvania Residential Baseline Study. The Act 129 2014 Residential Baseline Study may be found at http://www.puc.pa.gov/Electric/pdf/Act129/SWE-2014 PA Statewide Act129 Residential Baseline Study.pdf
- 14. The same EER to SEER ratio used for SEER 13 units applied to SEER 10 units. EERm = (11.3/13) * 10.
- 2013 Illinois Statewide TRM (Central Air Conditioning in Wisconsin, Energy Center of Wisconsin, May 2008)
- 16. Scott Pigg (Energy Center of Wisconsin), "Electricity Use by New Furnaces: A Wisconsin Field Study", Technical Report 230-1, October 2003, page 20. The average heating-mode savings of 400 kWh multiplied by the ratio of average heating degree days in PA compared to Madison, WI (5568/7172).
- 17. Ibid, page 34. The average cooling-mode savings of 88 kWh multiplied by the ratio of average EFLH in PA compared to Madison, WI (749/487).

lbid, page 34. The average kW savings of 0.1625 multiplied by the coincidence factor from

- 18. <u>Table 2-11Table 2-1.</u>
- 49. McQuay Application Guide 31-008, Geothermal Heat Pump Design Manual, 2002.

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⁶⁶ Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE, February 1, 1999. Confirmed also by *Central Air Conditioning in Wisconsin, a compilation of recent field research*. Energy Center of Wisconsin. May 2008, emended December 15, 2010, http://ecw.org/sites/default/files/241-1_0.pdf

⁶⁷ ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

<u>19.</u>

20. Based on building energy model simulations and residential baseline characteristics determined from the 2014 Residential End-use Study and applied to an HSPF listing for 12 SEER Air Source Heat Pumps at https://www.ahridirectory.org on July 28th, 2014.

2.2.2 FUEL SWITCHING: ELECTRIC HEAT TO GAS/PROPANE/OIL HEAT

Measure Name	Fuel Switching: Electric Heat to Gas/Propane/Oil Heat
Target Sector	Residential Establishments
Measure Unit	Gas, Propane, or Oil Heater
Unit Energy Savings	Variable based on system and location
Unit Peak Demand Reduction	Variable based on system and location
Measure Life	20 years ⁶⁸
Vintage	Replace on Burnout

This protocol documents the energy savings attributed to converting from an existing electric heating system to a new natural gas, propane, or oil furnace or boiler in a residential home.

The baseline for this measure is an existing residential home with an electric primary heating source. The heating source can be electric baseboards, electric furnace, or electric air source heat pump.

ELIGIBILITY

The target sector primarily consists of single-family residences.

The retrofit condition for this measure is the installation of a new standard efficiency natural gas, propane, or oil furnace or boiler. To encourage adoption of the highest efficiency units, older units which meet outdated ENERGY STAR standards may be incented up through the given sunset dates (see table below).

ENERGY STAR Product Criteria Version	ENERGY STAR Effective Manufacture Date	Act 129 Sunset Date ^a
ENERGY STAR Furnaces Version 4.0	February 1, 2013	N/A
ENERGY STAR Furnaces Version 3.0	February 1, 2012	May 31, 2014
ENERGY STAR Furnaces Version 2.0, Tier II units	October 1, 2008	May 31, 2013

^a Date after which Act 129 programs may no longer offer incentives for products meeting the criteria for the listed ENERGY STAR version."

EDCs may provide incentives for equipment with efficiencies greater than or equal to the applicable ENERGY STAR requirement per the following table.

Equipment	Energy Star Requirements ⁶⁹
Gas Furnace	 AFUE rating of 95% or greater Less than or equal to 2.0% furnace fan efficiency Less than or equal to 2.0% air leakage

PA 2010 TRM Appendix A: Measure Lives. Note that PA Act 129 savings can be claimed for no more than 15 years.
 Residential Furnace and Boiler Energy Star product criteria. http://www.energystar.gov/index.cfm?c=furnaces.pr_crit_furnaces
 and http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=BO

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Equipment	Energy Star Requirements ⁶⁹
Oil Furnace	 AFUE rating of 85% or greater Less than or equal to 2.0% furnace fan efficiency Less than or equal to 2.0% air leakage
Boiler	AFUE rating of 85% or greater

ALGORITHMS

The energy savings are the full energy consumption of the electric heating source minus the energy consumption of the fossil fuel furnace blower motor. EDC's may use billing analysis using program participant data to claim measure savings, in lieu of the defaults provided in this measure protocol. The energy savings are obtained through the following formulas:

Heating savings with electric baseboards or electric furnace (assumes 100% efficiency): Energy Impact:

Heating savings with electric furnace (assumes 100% efficiency):

Energy Impact:

$$\frac{\Delta kWh/yr_{elec\ furnace}}{\Delta kWh/yr_{elec\ furnace}} = \frac{CAPY_{elec\ heat} \times EFLH_{elec\ furnace}}{3412\frac{Btu}{kWh}}$$

Heating savings with electric baseboards (assumes 100% efficiency):

Energy Impact:

$$\Delta \, kWh/yr_{elec\,bb\,heat} \qquad = \frac{CAPY_{elec\,heat} \times EFLH_{elec\,bb}}{3412\frac{Btu}{kWh}} - \frac{HP_{motor} \times \left(746\frac{W}{hp}\right) \times EFLH_{fuel\,furnace}}{\eta_{motor} \times 1000\frac{W}{kW}}$$

Heating savings with electric air source heat pump:

Energy Impact:

$$\Delta kWh_{ASHP \, heat} = \frac{CAPY_{ASHP \, heat} \times EFLH_{ASHP}}{HSPF_{ASHP} \times 1000 \frac{W}{kW}} - \frac{HP_{motor} \times \left(746 \frac{W}{HP}\right) \times EFLH_{fuel \, furnace}}{\eta_{motor} \times 1000 \frac{W}{kW}}$$

For boilers, the annual pump energy consumption is negligible (<50 kWh per year) and not included in this calculation. 70

There are no peak demand savings as it is a heating only measure.

Although there is a significant electric savings, there is also an associated increase in natural gas energy consumption. While this gas consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel energy is obtained through the following formulas:

Gas consumption with fossil fuel furnace:

Gas Consumption (MMBtu) =
$$\frac{CAPY_{fuel\ heat} \times EFLH_{fuel\ furnace}}{AFUE_{fuel\ heat} \times 1,000,000 \frac{Btu}{MMBtu}}$$

⁷⁰ Pump motors are typically 1/25 HP. With 1,000 hour runtime and 80% assumed efficiency, this translates to 37 kWh.

DEFINITION OF TERMS

The default values for each term are shown in Table 2-14Table 2-38.

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 ${\it Table 2-14: Default\ values\ for\ algorithm\ terms, Fuel\ Switching, Electric\ Heat\ to\ Gas\ Heat}$

	Term	Type Units	Value	Source
	CAPY _{elec heat} , Total heating capacity of existing electric baseboards or electric furnace	$\frac{\text{Variable}}{hr}$	Nameplate	EDC Data Gathering
	CAPY _{ASHP heat} , Total heating capacity of existing electric ASHP	$\frac{\text{Variable}}{hr}$	Nameplate	EDC Data Gathering
	CAPY _{fuel heat} , Total heating capacity of new natural gas furnace	$\frac{\text{Variable}}{hr}^{Btu}$	Nameplate	EDC Data Gathering
	<i>EFLH_{ASHP}</i> , Equivalent Full Load Heating hours for Air Source Heat Pumps	hours yr Default	Allentown = 1,193 Erie = 1,349 Harrisburg = 1,103 Philadelphia = 1,060 Pittsburgh = 1,209 Scranton = 1,296 Williamsport = 1,251	2014 PA TRM Table 2-11 Table 2-11, in Electric HVAC section
	rumps	Optional	An EDC can either use the Alternate EFLH Table or estimate it's own EFLH based on customer billing data analysis.	Alternate EFLH Table (See <u>Table</u> 2-15Table 2-15) or EDC Data Gathering
	EFLH _{elec fumace_} . Equivalent Full Load Heating hours for Electric Forced Air	hours Default	Allentown = 1,000 Erie = 1,075 Harrisburg = 947 Philadelphia = 934 Pittsburgh = 964 Scranton = 1,034 Williamsport = 1,011	1
	Furnaces	Optional	An EDC can either use the Alternate EFLH Table or estimate it's own EFLH based on customer billing data analysis.	Alternate EFLH Table (See <u>Table</u> 2-16Table 2-16) or EDC Data Gathering
	EFLH _{elec bb.} , Equivalent Full Load Heating hours for Electric	hours Default	Allentown = 1,321 Erie = 1,396 Harrisburg = 1,265 Philadelphia = 1,236 Pittsburgh = 1,273 Scranton = 1,357 Williamsport = 1,354	1
	Baseboard systems	Optional	An EDC can either use the Alternate EFLH Table or estimate it's own EFLH based on customer billing data analysis.	Alternate EFLH Table (See <u>Table</u> 2-17Table 2-17) or EDC Data Gathering
 	EFLH _{fuel furnace_} , Equivalent Full Load	hours Default	Allentown = 1,022 Erie = 1,098 Harrisburg = 969 Philadelphia = 955 Pittsburgh = 985 Scranton = 1,056	1

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Term	Type Units	Value	Source
Heating hours for Fossil Fuel		Williamsport = 1,033	
Furnace systems	Optional	An EDC can either use the Alternate EFLH Table or estimate it's own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Table 2-18) or EDC Data Gathering
<i>EFLH_{fuel boiler}</i> , Equivalent Full Load Heating hours for Fuel Boilers	hours Default	Allentown = 1,334 Erie = 1,411 Harrisburg = 1,279 Philadelphia = 1,249 Pittsburgh = 1,283 Scranton = 1,371 Williamsport = 1,354	1
	Optional	An EDC can either use the Alternate EFLH Table or estimate it's own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Table 2-19) or EDC Data Gathering
HSPF _{ASHP} ., Heating Seasonal Performance Factor for existing heat pump	$\frac{Btu}{W \cdot hr}$ Variable	EDC Data Gathering Default = 7.7	2010 PA TRM <u>Table 2-11</u> Table 2-11
		Nameplate	EDC Data Gathering
 AFUE _{fuel heat.} , Annual Fuel Utilization Efficiency for the new gas furnace	Variable <u>%</u>	EDC Data Gathering Default = 95% (natural gas/propane furnace) 95% (natural gas/propane steam boiler) 95% (natural gas/propane hot water boiler) 85% (oil furnace) 85% (oil steam boiler)	ENERGY STAR requirement
		Nameplate	EDC Data Gathering
<i>HP_{motor.}</i> , Gas furnace blower motor horsepower	Variable <u>hp</u>	EDC Data Gathering Default = ½ hp	Average blower motor capacity for gas furnace (typical range = 1/4 hp to 3/4 hp)
		Nameplate	EDC Data Gathering
$\eta_{motor.}$, Efficiency of furnace blower motor	Variable <u>%</u>	EDC Data Gathering Default = 50%	Typical efficiency of ½ hp blower motor

Alternate Equivalent Full Load Hour (EFLH) Tables

Table 2-15 Table 2-15 Table 2-39 through Table 2-19 Table 2-43 below, show heating EFLH by city and for each EDC's housing demographics. In order to determine the most appropriate EFLH value to use for a project, first select the type of electric heating equipment being replaced, then

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the appropriate EDC. Next, from the column, pick the closest city to the project location. The value shown in that cell will be the EFLH value to use for the project.

Table 2-15: Alternate Heating EFLH for Air Source Heat Pumps

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	1112	1057	1122	1165	1265	1226	1320
Erie	1255	1204	1273	1317	1420	1376	1494
Harrisburg	1028	974	1035	1077	1174	1138	1219
Philadelphia	986	940	1001	1039	1134	1098	1165
Pittsburgh	1124	1068	1133	1175	1274	1234	1347
Scranton	1203	1151	1218	1261	1365	1321	1445
Williamsport	1161	1110	1175	1218	1320	1278	1392

Table 2-16: Alternate Heating EFLH for Electric Furnaces

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	914	890	952	991	1079	1037	1100
Erie	986	964	1027	1064	1150	1108	1183
Harrisburg	866	837	900	940	1027	986	1041
Philadelphia	854	827	893	931	1018	976	1021
Pittsburgh	882	854	914	950	1033	994	1068
Scranton	945	922	983	1020	1107	1064	1144
Williamsport	924	902	961	998	1085	1043	1118

Table 2-17: Alternate Heating EFLH for Electric Baseboard Heating

	PPL	Penelec	Met Ed	West	Duquesne	Penn	PECO
				Penn		Power	
Allentown	1355	1204	1280	1334	1351	1355	1326
Erie	1432	1287	1360	1408	1426	1430	1395
Harrisburg	1300	1144	1224	1280	1298	1299	1271
Philadelphia	1272	1115	1194	1247	1268	1269	1242
Pittsburgh	1301	1158	1230	1281	1297	1431	1277
Scranton	1389	1245	1317	1369	1385	1385	1366
Williamsport	1373	1230	1303	1351	1371	1371	1394

Table 2-18: Alternate Heating EFLH for Fossil Fuel Furnaces

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	934	919	985	1023	1116	1071	1106
Erie	1007	995	1060	1098	1188	1144	1190
Harrisburg	887	865	931	973	1064	1018	1048
Philadelphia	873	855	922	962	1055	1007	1027
Pittsburgh	900	882	945	982	1067	1024	1075
Scranton	965	951	1016	1053	1144	1099	1149
Williamsport	944	931	993	1031	1121	1078	1124

Table 2-19: Alternate Heating EFLH for Fossil Fuel Boilers

	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Allentown	1366	1214	1289	1346	1363	1364	1347
Erie	1445	1299	1370	1422	1440	1440	1417
Harrisburg	1312	1155	1234	1290	1308	1309	1291
Philadelphia	1281	1125	1205	1261	1278	1280	1260
Pittsburgh	1315	1169	1240	1294	1311	1311	1292
Scranton	1400	1256	1330	1378	1399	1397	1386

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	PPL	Penelec	Met Ed	West Penn	Duquesne	Penn Power	PECO
Williamsport	1384	1238	1313	1365	1382	1383	1364

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

State of Pennsylvania

Measure life = 20 years 71

$\textcolor{red}{\textbf{S}} \, \texttt{OURCES}$

Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
calculated from kWh consumption for cooling and heating. Models 40% oversizing of heat
systems.

⁷¹ PA 2010 TRM Appendix A: Measure Lives. Note that PA Act 129 savings can be claimed for no more than 15 years.

2.2.3 DUCTLESS MINI-SPLIT HEAT PUMPS

Measure Name	Ductless Heat Pumps
Target Sector	Residential Establishments
Measure Unit	Ductless Heat Pumps
Unit Energy Savings	Variable based on efficiency of systems
Unit Peak Demand Reduction	Variable based on efficiency of systems
Measure Life	15 years ⁷²
Vintage	Replace on Burnout

Rev Date: June 2015

ENERGY STAR ductless "mini-split" heat pumps utilize high efficiency SEER/EER and HSPF energy performance factors of 14.5/12 and 8.2, respectively, or greater. This technology typically converts an electric resistance heated home into an efficient single or multi-zonal ductless heat pump system. Homeowners have choice to install an ENERGY STAR qualified model or a standard efficiency model.

ELIGIBILITY

This protocol documents the energy savings attributed to ductless mini-split heat pumps with energy efficiency performance of 14.5/12 SEER/EER and 8.2 HSPF or greater with inverter technology. The baseline heating system could be an existing electric resistance heating, a lower-efficiency ductless heat pump system, a ducted heat pump, electric furnace, or a non-electric fuel-based system. The baseline cooling system can be a standard efficiency heat pump system, central air conditioning system, or room air conditioner. In addition, this could be installed in new construction or an addition. For new construction or addition applications, the baseline assumption is a standard-efficiency ductless unit. The DHP systems could be installed as the primary heating or cooling system for the house or as a secondary heating or cooling system for a single room.

ALGORITHMS

The savings depend on three main factors: baseline condition, usage (primary or secondary heating system), and the capacity of the indoor unit. The algorithm is separated into two calculations: single zone and multi-zone ductless heat pumps. The savings algorithm is as follows:

SINGLE ZONE

$$\begin{array}{ll} \Delta \, kWh/yr & = \Delta \, kWh/yr_{cool} + \Delta \, kWh/yr_{heat} \\ \Delta \, kWh/yr_{heat} & = \frac{CAPY_{heat}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{HSPF_{base}} - \frac{1}{HSPF_{ee}}\right) \times EFLH_{heat} \\ \Delta \, kWh/yr_{cool} & = \frac{CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{SEER_{base}} - \frac{1}{SEER_{ee}}\right) \times EFLH_{cool} \end{array}$$

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⁷² DEER values, updated October 10, 2008. Various sources range from 12 to 20 years, DEER represented a reasonable

mid-range. http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

The measure energy efficiency performance is based on ENERGY STAR minimum specification requirements as specified in ARHI and CEE directory for ductless mini-split heat pumps. Ductless heat pumps fit these criteria and can easily exceed SEER levels of 16 or greater.

$$\Delta kW_{peak}$$

$$= \frac{CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base}} - \frac{1}{EER_{ee}}\right) \times CF$$

MULTI-ZONE+

$$\begin{array}{l} \Delta kWh/yr \\ \Delta kWh/yr_{heat} \\ \end{array} = \frac{\left[CAPY_{heat}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{HSPF_{base~1}} - \frac{1}{HSPF_{ee}} \right) \times EFLH_{heat} \right]_{zone~1} \\ + \frac{\left[CAPY_{heat}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{HSPF_{base~2}} - \frac{1}{HSPF_{ee}} \right) \times EFLH_{heat} \right]_{zone~2} \\ + \frac{\left[CAPY_{heat}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{HSPF_{base~n}} - \frac{1}{HSPF_{ee}} \right) \times EFLH_{heat} \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{SEER_{base~1}} - \frac{1}{SEER_{ee}} \right) \times EFLH_{cool} \right]_{zone~1} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{SEER_{base~2}} - \frac{1}{SEER_{ee}} \right) \times EFLH_{cool} \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{SEER_{base~1}} - \frac{1}{SEER_{ee}} \right) \times EFLH_{cool} \right]_{zone~1} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~1} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{1}{EER_{ee}} \right) \times CF \right]_{zone~2} \\ + \frac{\left[CAPY_{cool}}{1000\frac{W}{kW}} \times \left(\frac{OF \times DLF}{EER_{base~1}} - \frac{$$

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Table 2-20: DHP - Values and References

<u>Term</u> Component	<u>Unit</u> Type	Values	Sources
CAPY _{coof} , The cooling (at 47° F) capacity of the Ductless Heat Pump unit	Hour Varia	EDC Data Gathering	AEPS Application; EDC Data Gathering
CAPY _{heat} , The heating (at 47° F) capacity of the Ductless Heat Pump unit	Btu hour	EDC Data Gathering	AEPS Application; EDC Data Gathering
EFLH primary , Equivalent Full Load Hours of the primary system – If the unit is installed as the primary heating or cooling system, as defined in Table 2-21 Table 2-30	hours year Fixe d	Allentown Cooling = 487 Hours Allentown Heating = 1,193 Hours Erie Cooling = 389 Hours Erie Heating = 1,349 Hours Harrisburg Cooling = 551 Hours Harrisburg Heating = 1,103 Hours Philadelphia Cooling = 591 Hours Philadelphia Heating = 1,060 Hours Pittsburgh Cooling = 432 Hours Pittsburgh Heating = 1,209 Hours Scranton Cooling = 417 Hours Scranton Heating = 1,296 Hours Williamsport Cooling = 422 Hours Williamsport Heating = 1,251 Hours	1
	hours year Opti onal	An EDC can either use the Alternate EFLH Table 2-13 or estimate its own EFLH based on customer billing data analysis	EDC Data Gathering
EFLH secondary _ Equivalent Full Load Hours of the secondary system — If the unit is installed as the secondary heating or cooling system, as defined in Table 2-21	hours Fixe year d	Allentown Cooling = 243 Hours Allentown Heating = 800 Hours Erie Cooling = 149 Hours Erie Heating = 994 Hours Harrisburg Cooling = 288 Hours Harrisburg Heating = 782 Hours Philadelphia Cooling = 320 Hours Philadelphia Heating = 712 Hours Pittsburgh Cooling = 228 Hours Pittsburgh Heating = 848 Hours Scranton Cooling = 193 Hours Scranton Heating = 925 Hours Williamsport Cooling = 204 Hours Williamsport Heating = 875 hours	2, 3

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TermComponent	<u>Unit</u> Type	Values	Sources
<i>HSPF</i> _{base} , "Heating Seasonal Performance Factor"- heating efficiency of baseline unit	$\frac{Btu}{W \cdot h}$ Fixed	Standard DHP: 8.27.7 Electric resistance: 3.413 ASHP: 8.27.7 Electric furnace: 3.242 No existing or non-electric heating: use standard DHP: 8.27.7	4, 6
SEER _{base} "Seasonal Energy Efficiency Ratio" - Cooling efficiency of baseline unit	$\frac{Btu}{W \cdot h}$ Fixed	DHP; or ASHP, or: 14- eCentral AC: 13 Room AC: 11.3 No existing cooling for primary space: use DHP, ASHP, or cCentral AC: 13 No existing cooling for secondary space: use Room AC: 11.3	5, 6, 7
HSPF _{ee_} "Heating Seasonal Performance Factor"- heating efficiency of installed DHP	Btu W·h le	Based on nameplate information. Should be at least ENERGY STAR.	AEPS Application; EDC Data Gathering
SEER _{ee_} "Seasonal Energy Efficiency Ratio" - Cooling efficiency of installed DHP	Btu W⋅h le	Based on nameplate information. Should be at least ENERGY STAR.	AEPS Application; EDC Data Gathering
OF, "Oversize factor" factor to account for the fact that the baseline unit is typically 40%-50% oversized	None	Depends on baseline condition: Central AC=1.5 Central ASHP=1.4 Electric Furnace=1.4 Electric Baseboard=1.4 Room AC: 1.0 Ductless Heat Pump: 1.0	1
DLF, "Duct Leakage Factor" accounts for the fact that a % of the energy is lost to duct leakage and conduction for ducted systems, but not ductless ones	None	Depends on baseline condition: Central AC=1.15 Central ASHP=1.15 Electric Furnace=1.15 Electric Baseboard=1.00 Room AC: 1.00 Ductless Heat Pump: 1.0	
CF_Coincidence Factor	%Fixed	<u>64.7%</u> 70%	8
EER _{base.} The Energy Efficiency Ratio of the baseline unit	$\frac{Btu}{W \cdot h}$ Fixed	= (11.3/13) X SEER _b for DHP or central AC = 9.8 room AC	5,9
EER _{ee_} The Energy Efficiency Ratio of the installed DHP	Btu Wh le	= (11.3/13) X SEER _e Based on nameplate information. Should be at least ENERGY STAR.	AEPS Application; EDC Data Gathering
L F	Fixed	25 %	10

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DEFINITION OF HEATING ZONE

Definition of primary and secondary heating systems depends primarily on the location where the source heat is provided in the household, and shown in Table 2-21.

Table 2-21: DHP - Heating Zones

Component	Definition
Primary Heating Zone	Living room Dining room House hallway Kitchen areas Family Room Recreation Room
Secondary Heating Zone	Bedroom Bathroom Basement Storage Room Office/Study Laundry/Mudroom Sunroom/Seasonal Room

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, a heat pump's lifespan is 15 years.74

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings. A sample of pre- and post-metering is recommended to verify heating and cooling savings but billing analysis will be accepted as a proper form of savings verification and evaluation.

SOURCES

- 1. Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners⁷⁵ and 40% oversizing of heat pumps.⁷⁶
- 2. Secondary cooling load hours based on room air conditioner "corrected" EFLH work paper that adjusted the central cooling hours to room AC cooling hours; see Section 2.2.5 Room AC Retirement measure.
- 3. Secondary heating hours based on a ratio of HDD base 68 and base 60 deg F. The ratio is used to reflect the heating requirement for secondary spaces is less than primary

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Teer Values, updated October 10, 2008. Various sources range from 12 to 20 years, DEER represented a reasonable-mid range. http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10.1.08.xls
 Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE,

February 1, 1999. Confirmed also by Central Air Conditioning in Wisconsin, a compilation of recent field research. Energy Center of Wisconsin. May 2008, emended December 15, 2010

76 ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

- space as the thermostat set point in these spaces is generally lowered during unoccupied time periods.
- 4. COP = 3.413 HSPF for electric resistance heating. Electric furnace efficiency typically varies from 0.95 to 1.00 and thereby assumed a COP 0.95 = 3.242.
- U.S. Federal Standards for Residential Air Conditioners and Heat Pumps. Effective 1/1/2015.
 http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/75Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170, 7200.
- Air-Conditioning, Heating, and Refrigeration Institute (AHRI); the directory of the available ductless mini-split heat pumps and corresponding efficiencies (lowest efficiency currently available). Accessed 8/16/2010.
- SEER based on average EER of 9.8 for room AC unit. From Pennsylvania's Technical Reference Manual.
- 8. Based on an analysis of six different utilities by Proctor Engineering. From Pennsylvania's Technical Reference Manual.
- 8. Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. Found at http://www.sciencedirect.com/science/article/pii/S1040619011001941
- 9. Average EER for SEER 13 unit. From Pennsylvania's Technical Reference Manual.
- 10. The load factor is used to account for inverter-based DHP units operating at partial loads. The value was chosen to align savings with what is seen in other jurisdictions, based on personal communication with Bruce Manclark, Delta-T, Inc., who is working with Northwest Energy Efficiency Alliance (NEEA) on the Northwest DHP Project http://www.nwductless.com/, and the results found in the "Ductless Mini Pilot Study" by KEMA, Inc., June 2009. This adjustment is required to account for partial load conditions and because the EFLH used are based on central ducted systems which may overestimate actual usage for baseboard systems.

2.2.4 ENERGY STAR ROOM AIR CONDITIONERS

Measure Name	Room Air Conditioners
Target Sector	Residential Establishments
Measure Unit	Room Air Conditioner
Unit Energy Savings	Varies
Unit Peak Demand Reduction	0.059 kWVaries
Measure Life	9 years ⁷⁷
Vintage	Replace on Burnout

ELIGIBILITY

This measure relates to the purchase and installation of a room air conditioner meeting ENERGY STAR criteria.

ALGORITHMS

The general form of the equation for the ENERGY STAR Room Air Conditioners (RAC) measure savings algorithm is:

Total Savings = Number of Room Air Conditioners \times Savings per Room Air Conditioner

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of room air conditioners. The number of room air conditioners will be determined using market assessments and market tracking.

As of June 1, 2014 RAC units will have a "CEER" rating as well as an "EER". CEER is the "Combined Energy Efficiency Ratio", which incorporates standby power into the calculation. This will be the value used in the $\Delta \, kWh/yr$ calculation.

$$\begin{array}{ll} \Delta kWh/yr & = CAPY_{RAC}/1000 \times (1/EER_b - 1/EER_{ee}) * EFLH_{RAC} \\ \Delta kWh/yr & = \frac{CAPY}{1000\frac{W}{kW}} \times \left(\frac{1}{CEER_{base}} - \frac{1}{CEER_{ee}}\right) \times EFLH_{RAC} \\ \\ \Delta kW & = DSav_{RAC} \times CF \\ \Delta kW_{peak} & = \frac{CAPY}{1000\frac{W}{kW}} \times \left(\frac{1}{CEER_{base}} - \frac{1}{CEER_{ee}}\right) \times CF \end{array}$$

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⁷⁷ Appliance Magazine, 2008.

DEFINITION OF TERMS

Table 2-22: ENERGY STAR Room AC - References

Component	Unit	Value	Sources
CAPY, The cooling capacity of the room air conditioner (RAC) being installed	$\frac{Btu}{hr}$	EDC Data Gathering	
$CEER_{base}$, Combined Energy Efficiency ratio of the baseline unit	$\frac{Btu}{W \cdot h}$	Federal Standard Values in: Table 2-23Table 2-23 -Table 2-24Table 2-24 -Table 2-25Table 2-25	1
CEER _{ee} , Combined Energy efficiency ratio of the RAC being installed	$\frac{Btu}{W \cdot h}$	EDC Data Gathering Default = ENERGY STAR values in: Table 2-23Table 2-23 -Table 2-24Table 2-24 -Table 2-25Table 2-25	<u>2</u>
EFLH _{RAC} , Equivalent full load hours of the RAC being installed	hours year	Table 2-26 Table 2-26 or alternate EFLH _{cool} values × an Adjustment Factor in Section 2.2.5	3
CF , Demand coincidence factor	%	Default: 30% Or EDC data gathering	<u>4</u>

Table 2-23 lists the minimum federal efficiency standards as of June 2014 and minimum ENERGY STAR efficiency standards for RAC units of various capacity ranges and with and without louvered sides. Units without louvered sides are also referred to as "through the wall" units or "built-in" units. Note that the new federal standards are based on the Combined Energy Efficiency Ratio Mmetric (CEER), which is a metric that incorporates energy use in all modes, including standby and off modes.⁷⁸

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⁷⁸ Federal standards: U.S. Department of Energy. *Federal Register*. 164th ed. Vol. 76, August 24, 2011.

Table 2-23: RAC (without reverse cycle) Federal Minimum Efficiency and ENERGY STAR Version 3.19 Standards 79

	Capacity (Btu/h)	Federal Standard CEER, with louvered sides	ENERGY STAR <u>CEER</u> , with louvered sides	Federal Standard <u>CEER</u> , without louvered sides	ENERGY STAR CEER, without louvered sides	
ı	< 6,000	≥11.0	11.0 2	10.0	10.24	
ı	6,000 to 7,999	211.0	11. <u>0</u> 2	10.0	10. <u>2</u> 4	
1	8,000 to 10,999			9.6	9. <u>7</u> 8	
l	11,000 to 13,999	≥10.9	11. <u>2</u> 3	9.5		
l	14,000 to 19,999	≥10.7	11. <u>1</u> 2	9.3	9.8 <u>9.7</u>	
	20,000 to 24,999	≥9.4	9.8	9.4		
	≥25,000	≥9.0			<u>9.8</u>	

Table 2-24 lists the minimum federal efficiency standards and minimum ENERGY STAR efficiency standards for casement-only and casement-slider RAC units. Casement-only refers to a RAC designed for mounting in a casement window with an encased assembly with a width of 14.8 inches or less and a height of 11.2 inches or less. Casement-slider refers to a RAC with an encased assembly designed for mounting in a sliding or casement window with a width of 15.5 inches or less.

Table 2-24: Casement-Only and Casement-Slider RAC Federal Minimum Efficiency and ENERGY STAR Version 3.10 Standards (effective 2014 TRM)

Casement	Federal Standard CEER	ENERGY STAR CEER
Casement-only	≥ 9.5	<u>9.9≥ 10.0</u>
Casement-slider	≥ 10.4	<u>10.8≥ 10.9</u>

Table 2-25 lists the minimum federal efficiency standards and minimum ENERGY STAR efficiency standards for reverse-cycle RAC units.

Table 2-25: Reverse-Cycle RAC Federal Minimum Efficiency Standards and ENERGY STAR Version 3.19 Standards (effective 2014 TRM)

	Capacity (Btu/h)	Federal Standard CEER, with louvered sides	ENERGY STAR <u>CEER</u> , with louvered sides	Federal Standard CEER, without louvered sides	ENERGY STAR <u>CEER</u> , without louvered sides
l	< 14,000	n/a	n/a	≥ 9.3	<u>9.7≥ 9.8</u>
	≥ 14,000	II/a	II/a	≥ 8.7	<u>9.1≥ 9.2</u>
	< 20,000	≥ 9.8	<u>10.3</u> ≥ 10.4	n/a	n/a
l	≥ 20,000	≥ 9.3	≥9.8	II/a	II/a

Table 2-26

Table 2-26 provides deemed EFLH by city and default energy savings values (assuming CAPY=8,000 Btuh⁸⁰, louvered sides, no reverse cycle) if efficiency and capacity information is

http://www.energystar.gov/ia/partners/product_specs/program_regs/Room_Air_Conditioner_Program_Requirements_Version_3.pdf

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⁷⁹ Federal standards: U.S. Department of Energy. Federal Register. 164th ed. Vol. 76, August 24, 2011.
ENERGY STAR standards: ENERGY STAR Program Requirements Product Specification for Room Air Conditioners, Eligibility Criteria Version 3.10. October 1, 2013, June 22, 2012.

unknown. Alternate $EFLH_{cool}$ values from Table 2-12 in Section 2.2.1 may be used in conjunction with the Adjustment Factor (AF) in Section 2.2.5 to find $EFLH_{RAC}$ if desired.

Table 2-26: Deemed EFLH and Default Energy Savings81,82

City	EFLH _{RAC}	ΔkWh <u>/yr</u>	<u>ΔkW_{peak}</u>
Allentown	151	-14 3.0	.0059
Erie	121	-11 2.4	.0059
Harrisburg	171	-16 3.4	.0059
Philadelphia	183	-17 3.6	.0059
Pittsburgh	134	-13 2.6	.0059
Scranton	129	-12 2.5	.0059
Williamsport	131	-12 2.6	.0059

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Federal standards: U.S. Department of Energy. Code of Federal Regulations. 10 CFR, part 430.32(b). Effective June 1, 2014. http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41
- ENERGY STAR Program Requirements Product Specification for Room Air Conditioners, Eligibility Criteria Version 3.10. October 1, 2013. http://www.energystar.gov/products/specs/system/files/ENERGY%20STAR%20Version%203.1%20Room%20Air%20Conditioner%20Program%20Requirements.pdf
- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners and 40% oversizing of heat pumps.
- 2.4. Consistent with CFs found in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008.⁸⁴

⁸⁰ Pennsylvania Residential Baseline Study, April 2014.

⁸⁴⁻Values taken from ENERGY STAR calculator except for EFLH estimates, which can be found in Section 2.11 (Room AC Retirement), Table 2-23.

⁸² Assumes 10,000 Btu/h capacity and 9.8 EER for baseline unit and 10.8 EER for ENERGY S TAR unit.

⁸³ ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

⁸⁴ In the absence of better, Pennsylvania-specific data, this is the same source and value as the Mid-Atlantic and Illinois TRMs.

2.2.5 ROOM AC (RAC) RETIREMENT

Measure Name	Room A/C Retirement
Target Sector	Residential Establishments
Measure Unit	Room A/C
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	4 <u>years</u> ⁸⁵
Vintage	Early Retirement, Early Replacement

This measure is defined as retirement and recycling <u>without replacement</u> of an *operable* but older and inefficient room AC (RAC) unit that would not have otherwise been recycled. The assumption is that these units will be permanently removed from the grid rather than handed down or sold for use in another location by another EDC customer, and furthermore that they would not have been recycled without this program. This measure is quite different from other energy-efficiency measures in that the energy/demand savings is not the difference between a pre- and post- configuration, but is instead the result of complete elimination of the existing RAC.

ELIGIBILITY

Furthermore, tThe savings are *not* attributable to the customer that owned the RAC, but instead are attributed to a *hypothetical user of the equipment had it not been recycled*. Energy and demand savings is the estimated energy consumption of the retired unit over its remaining useful life (RUL).

ALGORITHMS

Although this is a fully deemed approach, any of these values can and should be evaluated and used to improve the savings estimates for this measure in subsequent TRM revisions.

Retirement-Only

All EDC programs are currently operated under this scenario. For this approach, impacts are based only on the existing unit, and savings apply only for the remaining useful life (RUL) of the unit.

$$\begin{split} \Delta \, kWh/yr & = \left(\frac{CAPY}{1000\frac{W}{kW} \times EER_{RetRAC}}\right) \times EFLH_{RAC} \\ \Delta kW_{peak} & = \left(\frac{CAPY}{1000\frac{W}{kW} \times EER_{RetRAC}}\right) \times CF_{RAC} \end{split}$$

Replacement and Recycling

It is not apparent that any EDCs are currently implementing the program in this manner, but the algorithms are included here for completeness. For this approach, the ENERGY STAR upgrade measure would have to be combined with recycling via a turn-in event at a retail appliance store, where the old RAC is turned in at the same time that a new one is purchased. Unlike the

SECTION 2: Residential Measures

⁸⁵ See Measure Life section of this measure.

retirement-only measure, the savings here are attributed to the customer that owns the retired RAC, and are based on the old unit and original unit being of the same size and configuration. In this case, two savings calculations would be needed. One would be applied over the remaining life of the recycled unit, and another would be used for the rest of the effective useful life, as explained below.

For the remaining useful life (RUL) of the existing RAC: The baseline value is the EER of the retired unit.

$$\begin{split} \Delta \, kWh/yr & = \frac{CAPY}{1000\frac{W}{kW}} \times \left(\frac{1}{EER_{RetRAC}} - \frac{1}{EER_{ee}}\right) \times EFLH_{RAC} \\ \Delta kW_{peak} & = \frac{CAPY}{1000\frac{W}{kW}} \times \left(\frac{1}{EER_{retRAC}} - \frac{1}{EER_{ee}}\right) \times CF_{RAC} \end{split}$$

After the RUL for (EUL-RUL) years: The baseline EER would revert to the minimum Federal appliance standard \underline{C} EER. As of June 1, 2014 RAC units will have a "CEER" rating in addition to an "EER". CEER is the "Combined Energy Efficiency Ratio", which incorporates standby power into the calculation. This will be the value used in the $\Delta kWh/yr$ calculation. (CEER was not used in the previous equations however since older units were not qualified with this metric).

$$\begin{array}{ll} \Delta kWh & = \textit{EFLH}_{RAC} * (\textit{CAPY/1000}) * (\textit{1/EER}_b - \textit{1/EER}_{ES}) \\ \Delta kWh/yr & = \frac{\textit{CAPY}}{1000\frac{\textit{W}}{\textit{kW}}} \times \left(\frac{1}{\textit{CEER}_{base}} - \frac{1}{\textit{CEER}_{ee}}\right) \times \textit{EFLH}_{RAC} \\ \Delta kW_{peak} & = \frac{\textit{CAPY/1000}}{1000\frac{\textit{W}}{\textit{kW}}} \times \left(\frac{1}{\textit{CEER}_{base}} - \frac{1}{\textit{CEER}_{ee}}\right) \times \textit{CF}_{RAC} \\ \Delta kW_{peak} & = \frac{\textit{CAPY}}{1000\frac{\textit{W}}{\textit{kW}}} \times \left(\frac{1}{\textit{CEER}_{base}} - \frac{1}{\textit{CEER}_{ee}}\right) \times \textit{CF}_{RAC} \end{array}$$

DEFINITION OF TERMS

Correction of ES RAC EFLH Values:

An additional step is required to determine EFLH $_{RAC}$ values. Normally, the EFLH values from the ENERGY STAR Room AC Calculator would be used directly,—however, the current (July 2010) ES Room AC calculator EFLHs appear unreasonably high and are in the range of those typically used for the Central AC calculator . In reality, RAC full load hours should be much lower than for a CAC system- and, as such, the EFLH $_{RAC}$ values were- calculated from CAC EFLH values as follows:

 $EFLH_{RAC}$ = $EFLH_{cool} \times AF$

Where:

Note that when the ENERGY STAR RAC calculator values are eventually corrected in the ES calculator, the corrected EFLH_{ES-RAC} values can be used directly and this adjustment step can be ignored and/or deleted.

SECTION 2: Residential Measures

Table 2-27: Room AC Retirement Calculation Assumptions

	Component	Type <u>Unit</u>	Value	Sourc es
	$\it EFLH_{RAC}$, Equivalent Full Load Hours of operation for the installed measure. In actuality, the number of hours and time of operation can vary drastically depending on the RAC location (living room, bedroom, home office, etc.).	Varies hours yr	Table 2-28	1
	$\mathit{EFLH}_\mathit{cool}$, Full load hours from REM/Rate modeling	hours yr VariesHou	Table 2-28	1
		hours yr	The Alternate <i>EFLH_{COOL}</i> values in Table 2-12 may be used	
	AF , Adjustment factor for correcting current ES Room AC calculator EFLHs.	FixedNone	0.31	2
	$\ensuremath{\textit{CAPY}}$, Rated cooling capacity (size) of the RAC unit.	Variable <u>Btuh</u>	EDC Data Gathering Default : 7,87040,000	3
	$\textit{EER}_{\textit{RefRAC}}$, The Energy Efficiency Ratio of the unit being retired-recycled.	$\frac{Btu}{W \cdot h} Variable$	Default: 9.07; or EDC Data Gathering	4
	<u>CEER_{base}</u> , (for a <u>8,000</u> 10,000-Btuh unit), The Energy Efficiency Ratio of a RAC that just meets the minimum federal appliance standard efficiency.	Fixed $\frac{Btu}{W \cdot h}$	<u>10.9</u> 9.8	5
	<u>CEERee</u> , (for a <u>8,00040,000</u> Btuh unit), The Energy Efficiency Ratio for an ENERGY STAR RAC.	$\frac{Btu}{W \cdot h}$ Variable	EDC Data Gathering Default=-11.210.8	5
•	CF _{rac} , Demand Coincidence Factor (See Section 1.5), which is 0.58 from the 2010 PA TRM for the "ENERGY STAR Room Air Conditioner" measure.	Default%	EDC Data Gathering Default= 30%0.58	6 <u>7</u>
	RAC Time Period Allocation Factors	Fixed%	65.1%, 34.9%, 0.0%, 0.0%	6
•	Measure Life (EUL)	<u>Years</u> Fixed	4	See- source notes

Table 2-28: RAC Retirement-Only EFLH and Energy Savings by City86

City	Original Hours (EFLH cool)	Corrected Hours (EFLH _{RAC})	Energy Impact (kWh)	Demand Impact (kW)
Allentown	487	151	166	0.6395
Erie	389	121	133	
Harrisburg	551	171	188	
Philadelphia	591	183	202	
Pittsburgh	432	134	148	
Scranton	417	129	143	
Williamsport	422	131	144	

MEASURE LIFE

Room Air Conditioner Retirement = 4 years

From the PA TRM, the EUL for an ENERGY STAR Room Air Conditioner is 10 years, but the TRM does not provide an RUL for RACs. However, as shown in Table 2-29, the results from a recent evaluation of ComEd's appliance recycling program⁸⁷ found a median age of 21 to 25 years for recycled ACs. For a unit this old, the expected life of the savings is likely to be short, so 4 years was chosen as a reasonable assumption based on these references:

- DEER database, presents several values for EUL/RUL for room AC recycling: http://www.deeresources.com/deer2008exante/downloads/EUL Summary 10-1-08.xls
 - a. DEER 0607 recommendation: EUL=9, RUL=1/3 of EUL = 3 years. The 1/3 was defined as a "reasonable estimate", but no basis given.
 - b. 2005 DEER: EUL=15, did not have recycling RUL
 - c. Appliance Magazine and ENERGY STAR calculator: EUL=9 years
 - d. CA IOUs: EUL=15, RUL=5 to 7
- "Out With the Old, in With the New: Why Refrigerator and Room Air Conditioner Programs Should Target Replacement to Maximize Energy Savings," National Resources Defense Council, November 2001, page 21, 5 years stated as a credible estimate.
- From the PA TRM June 2010, if the ratio of refrigerator recycling measure life to ENERGY STAR measure life is applied: (8/13) * 10 years (for RAC) = 6 years for RAC recycling.

⁸⁶ Table 2-28 should be used with a master "mapping table" that maps the zip codes for all PA cities to one of the representative cities above. This mapping table would also be used for the TRM ENERGY STAR Room Air Conditioning measure. This table will be developed in the context of the TWG.

⁸⁷ Residential Appliance Recycling Program Year 1 Evaluation Report – Final Report, prepared for Commonwealth Edison by Itron (under contract to Navigant Consulting), November 2009.

Table 2-29: Preliminary Results from ComEd RAC Recycling Evaluation⁸⁸

	Age in Years							N		
Appliance Type	0 to 5	6 to 10	11 to 15	16 to 20	21 to 25	26 to 30	31 to 35	36 to 40	Over 40	
Room Air Conditioners	0%	5%	7%	18%	37%	18%	5%	6%	5%	_

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners⁸⁹ and 40% oversizing of heat pumps.⁹⁰
- 2. Mid Atlantic Version 1.0. <u>April 28, 2010</u> Draft. Prepared by Vermont Energy Investment Corporation. An adjustment to the ES RAC EFLHs of 31% was used for the "Window A/C" measure. The average ratio of EFLH for Room AC provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008⁹¹ to FLH for Central Cooling for the same location (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls is 31%. This factor was applied to the EFLH for Central Cooling provided for PA cities and averaged to come up with the assumption for EFLH for Room AC."
- Statewide average capacity of RAC units, 2014 Pennsylvania Residential Baseline <u>Study</u>.10,000 Btuh is the typical size assumption for the ENERGY STAR Room AC <u>Savings calculator</u>. It is also used as the basis for PA TRM ENERGY STAR Room AC measure savings calculations, even though not explicitly stated in the TRM. For example:
 - a. Energy savings for Allentown = 74 kWh and EFLH = 784 hrs:

784 * (10,000/1000) * (1/9.8 - 1/10.8) = 74 kWh.

b. CPUC 2006-2008 EM&V, "Residential Retrofit High Impact Measure Evaluation Report", prepared for the CPUC Energy Division, February 8, 2010, page 165, Table 147 show average sizes of 9,729 and 10,091 Btuh.

⁸⁸ Navigant Consulting evaluation of ComEd appliance recycling program.

⁸⁹ Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE, February 1, 1999. Confirmed also by *Central Air Conditioning in Wisconsin, a compilation of recent field research.* Energy Center of Wisconsin. May 2008, emended December 15, 2010.

⁹⁰ ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

⁹¹ Accesed: http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf>

- 4. Massachusetts TRM, Version 1.0, October 23, 2009, "Room AC Retirement" measure, Page 52-54.__Assumes an existing/recycled unit EER=9.07, reference is to weighted 1999 AHAM shipment data.__This value should be evaluated and based on the actual distribution of recycled units in PA and revised in later TRMs if necessary. Other references include:
 - a. ENERGY STAR website materials on Turn-In programs, if reverse-engineered indicate an EER of 9.16 is used for savings calculations for a 10 year old RAC. Another statement indicates that units that are at least 10 years old use 20% more energy than a new ES unit which equates to: 10.8 EER/1.2 = 9 EER http://www.energystar.gov/ia/products/recycle/documents/RoomAirConditionerTurn-InAndRecyclingPrograms.pdf
 - b. "Out With the Old, in With the New: Why Refrigerator and Room Air Conditioner Programs Should Target Replacement to Maximize Energy Savings." National Resources Defense Council, November 2001.— Page 3, Cites a 7.5 EER as typical for a room air conditioner in use in 1990s.- However, page 21 indicates an 8.0 EER was typical for a NYSERDA program.
- ENERGY STAR <u>Version 3.1</u> and Federal Appliance Standard minimum <u>C</u>EERs for a 6000-799910,000 Btuh unit with louvered sides. <a href="http://www.energystar.gov/index.cfm?c=roomac.pr_crit_room_achttp://www.energystar.gov/products/specs/system/files/ENERGY%20STAR%20Version%203.1%20Room%20Air%20Conditioner%20Program%20Requirements.pdf
- PA TRM June 2010, coincident demand factor and Time Period Allocation Factors for ENERGY STAR Room AC.
- Consistent with CFs found in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008.

SECTION 2: Residential Measures

⁹² In the absence of better, Pennsylvania-specific data, this is the same source and value as the Mid-Atlantic and Illinois TRMs.

2.2.6 DUCT INSULATION AND SEALING

Measure Name	Duct Sealing
Target Sector	Residential Establishments
Measure Unit	Office Equipment Device
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	(15max, 20 actual for TRC) years ⁹³
Vintage	Retrofit

This measure describes evaluating the savings associated with performing duct sealing using mastic sealant or metal tape to the distribution system of homes with either central air conditioning or a ducted heating system.

Three methodologies for estimating the savings associated with sealing ducts are provided. The first two require the use of a blower door and the third requires careful inspection of the duct work.

- Modified Blower Door Subtraction this technique is described in detail on p.44 of the Energy Conservatory Blower Door Manual this method involves performing a whole house depressurization test, an envelope depressurization test that excludes duct leakage, and finally a duct leakage pressurization test under envelope depressurization. The subtraction of the envelope leakage in the second test from the whole house leakage in the first test, multiplied by a correction factor determined by the third test will provide an accurate measurement of the duct leakage to the outside. This technique is described in detail on p.44 of the Energy Conservatory Blower Door Manual; http://www.energyconservatory.com/sites/default/files/documents/mod 3-4 dg700 new flow rings cr tpt no fr switch manual ce 0.pdf
- 2. RESNET Test 803.7 this method involves the pressurization of the house to 25 Pascals with reference to outside and a simultaneous pressurization of the duct system to reach equilibrium with the envelope or inside pressure of zero Pascals. A blower door is used to pressurize the building to 25 Pascals with reference to outside, when that is achieved the duct blaster is used to equalize the pressure difference between the duct system and the house. The amount of air required to bring the duct system to zero Pascals with reference to the building is the amount of air leaking through the ductwork to the outside. This technique is described in detail in section 803.7 of the RESNET Standards: http://www.resnet.us/professional/standards

4.-

- 2.3. Evaluation of Distribution Efficiency this methodology requires the evaluation of three duct characteristics below, and use of the Building Performance Institutes 'Distribution Efficiency Look-Up Table; http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf
 - a. Percentage of duct work found within the conditioned space
 - b. Duct leakage evaluation
 - c. Duct insulation evaluation

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⁹³ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

ELIGIBILITY

The efficient condition is sealed duct work throughout the unconditioned space in the home. The existing baseline condition is leaky duct work within the unconditioned space in the home.

ALGORITHMS

Methodology 1: Modified Blower Door Subtraction

a) Determine Duct Leakage rate before and after performing duct sealing

$$CFM50_{DL}$$
 = $(CFM50_{whole\ house} - CFM50_{envelope\ only}) \times SCF$

 b) Calculate duct leakage reduction, convert to CFM25DL and factor in Supply and Return Loss Factors

$$= (CFM50_{DL} (pre) - CFM50_{DL} (post)) \times CONV \times (SLF + RLF)$$

c) Calculate Energy Savings

$$\Delta kWh/yr_{cooling} = \frac{\left(\frac{\Delta CFM25_{DL}}{\left(\frac{Cap_{cool}}{12,000\frac{Btuh}{ton}}\right) \times TCFM} \times EFLH_{cool} \times Cap_{cool}}{\left(\frac{SEER \times 1000\frac{W}{bW}}{1000}\right)}\right)}$$

$$\Delta kWh/yr_{heating} = \frac{\left(\frac{\Delta CFM25_{DL}}{\left(\frac{Cap_{heat}}{12,000\frac{Btuh}{ton}}\right) \times TCFM} \times EFLH_{heat} \times Cap_{heat}}{\left(COP \times 3412\frac{Btu}{kWh}\right)}\right)}$$

Methodology 2: RESNET Test 803.7

a) Determine Duct Leakage rate before and after performing duct sealing

$$\Delta CFM_{25DB}$$
 = CFM_{25BASE} - CFM_{25EE}

b) Calculate Energy Savings

$$\Delta \, kWh/yr_{cooling} = \frac{\left(\frac{\Delta CFM_{25DB}}{\left(\frac{Cap_{cool}}{12,000\frac{Btuh}{ton}}\right) \times TCFM} \times EFLH_{cool} \times Cap_{cool}\right)}{\left(SEER \times 1000\frac{W}{kW}\right)}$$

SECTION 2: Residential Measures

$$\Delta \, kWh/yr_{heating} = \frac{\left(\frac{\Delta CFM_{25DB}}{\left(\frac{Cap_{heat}}{12,000\frac{Btuh}{ton}}\right) \times TCFM} \times EFLH_{heat} \times Cap_{heat}}{\left(COP \times 3412\frac{Btu}{kWh}\right)}\right)}$$

Methodology 32: Evaluation of Distribution Efficiency

Determine Distribution Efficiency by evaluating duct system before and after duct sealing using Building Performance Institute "Distribution Efficiency Look-Up Table"

$$\Delta \, kWh/yr_{cooling} \qquad = \begin{cases} \frac{\left(\frac{DE_{after} - DE_{before}}{DE_{after}}\right) \times EFLH_{cool} \times Cap_{cool}}{SEER \times 1000 \frac{W}{kW}} \\ \\ \Delta \, kWh/yr_{heating} \qquad = \begin{cases} \frac{\left(\frac{DE_{after} - DE_{before}}{DE_{after}}\right) \times EFLH_{heat} \times Cap_{heat}}{DE_{after}} \\ \\ COP \times 3412 \frac{Btu}{kWh} \end{cases}$$

Summer Coincident Peak Demand Savings

$$\Delta kW_{peak} \hspace{1.5cm} = \frac{\Delta kWhcooling}{EFLH_{cool}} \times CF$$

DEFINITION OF TERMS

Table 2-30: Duct Sealing - Values and References

Term	Type <u>Unit</u>	Value	Source
CF , Demand Coincidence Factor (See Section 1.5) for central AC systems	Fixed <u>%</u>	<u>Default = 64.7%0.70</u>	<u>11</u>
CFM50 _{whole house} , Duct leakage at 50 Pascal pressure differential	$\frac{ft^3}{min} Variable$	EDC Data Gathering	EDC Data Gathering
<u>CFM_{25DB}</u> , <u>Cubic feet per minute of air</u> <u>leaving the duct system at 25 Pascals</u>	$\frac{ft^3}{min}$	EDC Data Gathering	<u>12</u>
<u>CFM_{25BASE}</u> , Standard Duct Leakage test result at 25 Pascal pressure differential of the duct system prior to sealing, calculated from the duct blaster fan flow chart	$\frac{ft^3}{min}$	EDC Data Gathering	<u>12</u>
CFM _{25EE} , Standard Duct Leakage test result at 25 Pascal pressure differential of the duct system after sealing, calculated from the duct blaster fan flow chart	$\frac{ft^3}{min}$	EDC Data Gathering	<u>12</u>
CFM50 _{envelope only} , Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential	$\frac{ft^3}{min}$ Variable	EDC Data Gathering	EDC Data Gathering

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Rev Date: June 2015

Term	Type <u>Unit</u>	Value	Source	
SCF, Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure in duct system with registers sealed, and using look up table provided by Energy Conservatory	Variable <u>No</u> ne	Table 4, on pg 45 of Minneapolis Blower Door™ Operation Manual for Model 3 and Model 4 Systems (Source 10)	7, 10	
Conv , Conversion factor from CFM50 to CFM25	FixedNone	0.64	2	
SLF , Supply Loss Factor (% leaks sealed located in Supply ducts x 1)	Variable <u>No</u> <u>ne</u>	EDC Data Gathering Default =0.5	4, EDC Data Gathering	
RLF , Return Loss Factor (Portion of % leaks sealed located in Return ducts x 0.5)	Variable <u>No</u> ne	EDC Data Gathering Default = 0.25	6, EDC Data Gathering	
Cap _{cool} , Capacity of Air Cooling System	Variable <u>Btu</u> <u>h</u>	EDC Data Gathering	EDC Data Gathering	
Cap _{heat} , Capacity of Air Heating System	Btuh	EDC Data Gathering	EDC Data Gathering	
$\ensuremath{\textit{TCFM}}$, Conversion from tons of cooling to CFM	Fixed CFM ton	400	7	
SEER , Efficiency of cooling equipment	$\frac{Btu}{W \cdot h}$ Variable	EDC Data Gathering Default = 10	8, EDC Data Gathering	
COP , Efficiency of Heating Equipment	Variable <u>No</u> <u>ne</u>	EDC Data Gathering Default = 2.0	8, EDC Data Gathering	
EFLH _{cool} , Cooling equivalent full load hours	hours Default	Allentown Cooling = 487 Hours Erie Cooling = 389 Hours Harrisburg Cooling = 551 Hours Philadelphia Cooling = 591 Hours Pittsburgh Cooling = 432 Hours Scranton Cooling = 417 Hours Williamsport Cooling = 422_Hours	<u>Table 2-11</u> <u>Table 2-11</u>	
	Optional	An EDC can either use the Alternate EFLH Table or estimate its own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Section 2.2.12.1.4); EDC Data Gathering	

	Term	Type <u>Unit</u>	Value	Source
			Allentown Heating = 1,193 Hours	
			Erie Heating = 1,349 Hours	
1			Harrisburg Heating = 1,103 Hours	
		hours year Default	Philadelphia Heating = 1,060 Hours	Table 2-11 Table 2-11
Ì	EFLH _{heat} , Heating equivalent full load hours		Pittsburgh Heating = 1,209 Hours	Tubic 2 TT
			Scranton Heating = 1,296 Hours	
			Williamsport Heating = 1,251 Hours	
Ì		Optional	An EDC can either use the Alternate EFLH Table or estimate its own EFLH based on customer billing data analysis.	Alternate EFLH Table (See Section 2.2.1); EDC Data Gathering
	DE _{after} , Distribution energy after duct sealing	Variable<u>No</u> <u>ne</u>	Variable	7, 9
	DE _{before} , Distribution energy before duct sealing	Variable<u>No</u> ne	Variable	7, 9

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The assumed lifetime of this measure is 20-years ⁹⁴. The actual duct sealing measure cost should be used.

SOURCES

- Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.neep.org/Assets/uploads/files/emv/emv-library/measure-life-GDS%5B1%5D.pdf
- 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions. To convert CFM50 to CFM25 you multiply by 0.64 (inverse of the "Can't Reach Fifty" factor for CFM25; see Energy Conservatory Blower Door Manual).

⁹⁴ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

- 3. Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply side leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from http://www.energyconservatory.com/download/dbmanual.pdf
- 4. _Assumes 50% of leaks are in supply ducts (Illinois Statewide TRM 2013).
- 5. Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than "average" (e.g. pulling return air from a super-heated attic), or can be adjusted downward to represent significantly less energy loss (e.g. pulling return air from a moderate temperature crawl space). More information provided in "Appendix E Estimating HVAC System Loss from Duct Airtightness Measurements" from http://www.energyconservatory.com/sites/default/files/documents/duct_blaster_manual_s_eries_b dg700.pdf http://www.energyconservatory.com/download/dbmanual.pdf
- 6. _Assumes 50% of leaks are in return ducts (Illinois Statewide TRM 2013).
- 7. Illinois Statewide TRM, 2013, Section 5.3.4.
- 8. Minimum Federal Standards for new Central Air Conditioners and Air Source Heat Pumps between 1990 and 2006 based on VEIC estimates.
- Building Performance Institute, Distribution Efficiency Table, http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf
- 10. Minneapolis Blower Door™ Operation Manual for Model 3 and Model 4 Systems. http://www.energyconservatory.com/sites/default/files/documents/mod 3-4 dg700 new flow rings - cr - tpt - no fr switch manual ce 0.pdf
- 11. Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941
- 12. Resnet Energy Services Network, Standards for Performance Testing. http://www.resnet.us/standards/DRAFT_Chapter_8_July_22.pdf

2.2.7 FURNACE WHISTLE

Measure Name	Furnace Whistle
Target Sector	Residential Establishments
Measure Unit	Furnace whistle (to promote regular filter change-out)
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	14 years ⁹⁵
Vintage	Retrofit

ELIGIBILITY

Savings estimates are based on reduced furnace blower fan motor power requirements for winter and summer use of the blower fan motor. This furnace whistle measure applies to central forcedair furnaces, central AC and heat pump systems. Each table in this protocol (33 through 39) presents the annual kWh savings for each major urban center in Pennsylvania based on their respective estimated full load hours (EFLH). Where homes do not have A/C or heat pump systems for cooling, only the annual heating savings will apply.

ALGORITHMS

$$\Delta kWh/yr \\ \Delta kWh/yr_{heat} \\ \Delta kWh/yr_{cool} \\ = kW_{motor} \times EFLH_{heat} \times EI \times ISR \\ = kW_{motor} \times EFLH_{cool} \times EI \times ISR$$

$$= \frac{\Delta kWh/yr_{cool}}{EFLH_{cool}} \times CF$$

DEFINITION OF TERMS

 $^{^{\}rm 95}$ See Appendix A, assumed to be the life of the HVAC unit.

Table 2-31: Furnace Whistle - References

Component	TypeU nit	Value	Sources
MkWkW _{motor.} , Average motor full load electric demand (kW)	<u>kW</u> Fix ed	0.5 kW	1, 2
EFLH _{Heat} , Estimated Full Load Hours (Heating) for the EDC region	hours yr	Variable. See Table 2-32	TRM Table 2-11 Table 2-11
EFLH _{Cool} , Estimated Full Load Hours (Cooling) for the EDC region.	hours yr Fixed	Variable. See Table 2-32	TRM Table 2-11 Table 2-11
EI , Efficiency Improvement	Fixed None <u>%</u>	15%	3 <u>. 6</u>
ISR , In-service Rate	Fixed None <u>%</u>	0.47 <u>.</u> 4 <u>%</u>	4
CF , Coincidence Factor	Fixed None <u>%</u>	70% 64.7%	<u>5</u>
-Measure EUL	Fixed	14	TRM Appendix- A (Life- assumed to be- the life of CAC- unit)

Table 2-32: EFLH for various cities in Pennsylvania (TRM Data)

City	Cooling load hours	Heating load hours	Total load hours
Allentown	487	1,193	1,681
Erie	389	1,349	1,739
Harrisburg	551	1,103	1,654
Philadelphia	591	1,060	1,651
Pittsburgh	432	1,209	1,641
Scranton	417	1,296	1,713
Williamsport	422	1,251	1,673

DEFAULT SAVINGS

The following table presents the assumptions and the results of the deemed savings calculations for each EDC. $\frac{1}{2} \int_{-\infty}^{\infty} \frac{1}{2} \left(\frac{1}{2} \int_{-\infty}$

SECTION 2: Residential Measures

Table 2-33: Assumptions and Results of Deemed Savings Calculations (Pittsburgh, PA)

	Blower Motor kW	Pittsburgh EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,209	604	695	91	0.474	43
Cooling	0.5	432	216	248	32	0.474	15
Total		1,641	820	944	123		58

 $\Delta kW_{peak} = 0.024 \text{ kW (Pittsburgh)}$

Table 2-34: Assumptions and Results of Deemed Savings Calculations (Philadelphia, PA)

	Blower Motor kW	Philadelphia EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,060	530	609	79	0.474	38
Cooling	0.5	591	296	340	44	0.474	21
Total		1,651	826	949	124		59

 $\Delta kW_{peak} = 0.025 \text{ kW (Philadelphia)}$

Table 2-35: Assumptions and Results of Deemed Savings Calculations (Harrisburg, PA)

	Blower Motor kW	Harrisburg EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,103	552	634	83	0.474	39
Cooling	0.5	551	276	317	41	0.474	20
Total		1,654	827	951	124		59

 $\Delta kW_{peak} = 0.025 \text{ kW (Harrisburg)}$

Table 2-36: Assumptions and Results of Deemed Savings Calculations (Erie, PA)

	Blower Motor kW	Erie EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,349	675	776	101	0.474	48
Cooling	0.5	389	195	224	29	0.474	14
Total		1,739	869	1,000	130		62

 $\Delta kW_{peak} = 0.025 \text{ kW (Erie)}$

Table 2-37: Assumptions and Results of Deemed Savings Calculations (Allentown, PA)

	Blower Motor kW	Allentown EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,193	597	686	89	0.474	42
Cooling	0.5	487	244	280	37	0.474	17
Total		1,681	840	966	126		60

 $\Delta kW_{peak} = 0.024 \text{ kW (Allentown)}$

Table 2-38: Assumptions and Results of Deemed Savings Calculations (Scranton, PA)

	Blower Motor kW	Scranton EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,296	648	745	97	0.474	46
Cooling	0.5	417	208	240	31	0.474	15
Total		1,713	857	985	129		61

 $\Delta kW_{peak} = 0.025 \text{ kW (Scranton)}$

Table 2-39: Assumptions and Results of Deemed Savings Calculations (Williamsport, PA)

	Blower Motor kW	Williamsport EFLH	Clean Annual kWh	Dirty Annual kWh	Furnace Whistle Savings (kWh)	ISR	Estimated Savings (kWh)
Heating	0.5	1,251	625	719	94	0.474	44
Cooling	0.5	422	211	243	32	0.474	15
Total		1,673	836	962	125		59

 $\Delta kW_{peak} = 0.025 \text{ kW (Williamsport)}$

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

 The Sheltair Group HIGH EFFICIENCY FURNACE BLOWER MOTORS MARKET BASELINE ASSESSMENT provided BC Hydro cites Wisconsin Department of Energy [2003] analysis of electricity use from furnaces (see Blower Motor Furnace Study). The Blower Motor Study Table 17 (page 38) shows 505 Watts for PSC motors in space heat

SECTION 2: Residential Measures

- mode; last sentence of the second paragraph on page 38 states: "... multi-speed and single speed furnaces motors drew between 400 and 800 Watts, with 500 being the average value."Submitted to: Fred Liebich BC Hydro Tel. 604 453-6558 Email: fred.liebich@bchydro.com, March 31, 2004.
- FSEC, "Furnace Blower Electricity: National and Regional Savings Potential", page 98 Figure 1 (assumptions provided in Table 2, page 97) for a blower motor applied in
 prototypical 3-Ton HVAC for both PSC and BPM motors, at external static pressure of 0.8
 in. w.g., blower motor Watt requirement is 452 Watts.
- 3. US DOE Office of Energy Efficiency and Renewable Energy "Energy Savers" publication "Clogged air filters will reduce system efficiency by 30% or more." Savings estimates assume the 30% quoted is the worst case and typical households will be at the median or 15% that is assumed to be the efficiency improvement when furnace filters are kept clean.
- 4. The In Service Rate is taken from an SCE Evaluation of 2000-2001 Schools Programs, by Ridge & Associates 8-31-2001, Table 5-19 Installation rates, Air Filter Alarm 47.4%.
- Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941
- 4-6. Energy.gov. "Maintaining Your Air Conditioner". Accessed 7/16/2014. Says that replacing a dirty air filter with a clean one can lower total air conditioner energy consumption by 5-15%. Since the algorithms in this measure only take into account the blower fan energy use, a 15% savings seems reasonable.

2.2.8 PROGRAMMABLE THERMOSTAT

Measure Name	Programmable Thermostat
Target Sector	Residential Establishments
Measure Unit	Programmable Thermostat
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	11years ⁹⁶
Vintage	Retrofit

Programmable thermostats are used to control heating and/or cooling loads in residential buildings by modifying the temperature set-points during specified unoccupied and nighttime hours. These units are expected to replace a manual thermostat and the savings assume an existing ducted HVAC system with electric resistance heating and DX cooling. A standard programmable thermostat installed on a heat pump can have negative energy consequences. However, the option exists to input higher efficiency levels if coupled with a newer unit. The EDCs will strive to educate the customers to use manufacturer default setback and setup settings.

ELIGIBILITY

This measure documents the energy savings resulting from the installation of a programmable thermostat instead to replace an existing standard thermostat. The target sector is primarily residential.

ALGORITHMS

$$\Delta kWh_{cool} = \Delta kWh_{cool} + \Delta kWh_{heat}$$

$$= \frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \frac{1}{SEER \times Eff_{duct}} \times EFLH_{cool} \times ESF_{cool}$$

$$\Delta kWh_{heat} = \frac{CAPY_{heat}}{1000 \frac{W}{kW}} \times \frac{1}{HSPF \times Eff_{duct}} \times EFLH_{heat} \times ESF_{heat}$$

$$\Delta kW_{peak} = 0$$

DEFINITION OF TERMS

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⁹⁶ New York Standard Approach for Estimating Energy Savings from Energy Efficiency Measures in Commercial and Industrial Programs, September 1, 2009, based on DEER.

Table 2-40: Residential Electric HVAC Calculation Assumptions

Component	Type Unit	Value	Sources
CAPY _{COOL} , Capacity of air conditioning unit	Variable Btuh	EDC Data Gathering of Nameplate data	EDC Data Gathering
		Default= 32,000	1
CAPY _{HEAT} , Normal heat capacity of Electric Furnace	Variable Btuh	EDC Data Gathering of Nameplate Data	EDC Data Gathering
		Default= 32,000	1
SEER , Seasonal Energy Efficiency Ratio	$rac{Variable_}{Btu} \ rac{Btu}{W \cdot h}$	EDC Data Gathering of Nameplate data	EDC Data Gathering
		Default= 11.9	1
HSPF , Heating Seasonal Performance Factor of heat pump	$rac{Variable}{Btu}{W\cdot h}$	EDC Data Gathering of Nameplate data	EDC Data Gathering
		Default= 3.413 (equivalent to electric furnace COP of 1)	2
Eff _{duct} , Duct System Efficiency	FixedNone	0.8	3
ESF _{COOL} , Energy Saving Factor for Cooling	FixedNone	<u>0.0</u> 2%	4
ESF _{HEAT} , Energy Saving Factor for Heating	FixedNone	<u>3.6%0.036</u>	5
<i>EFLH_{COOL}</i> , Equivalent Full Load hour for Cooling	hours day Default	Allentown Cooling = 487 Hours Erie Cooling = 389 Hours Harrisburg Cooling = 551 Hours Philadelphia Cooling = 591 Hours Pittsburgh Cooling = 432 Hours Scranton Cooling = 417 Hours Williamsport Cooling = 422 Hours	6
	Optional	Can use the more EDC-specific values in Table 2-12	Alternate EFLH Table 2-12
	Optional	An EDC can estimate it's own EFLH based on customer billing data analysis.	EDC Data Gathering
EFLH _{HEAT} , Full Load Hours for Heating	hours day Default	Allentown Heating = 1,193 Hours Erie Heating = 1,349 Hours Harrisburg Heating = 1,103 Hours Philadelphia Heating = 1,060 Hours Pittsburgh Heating = 1,209 Hours Scranton Heating = 1,296 Hours Williamsport Heating = 1,251 Hours	6
	Optional	An EDC can use the Alternate	Alternate EFLH

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Component	Type Unit	Value	Sources
		EFLH values in Table 2-13	Table 2-13
	Optional	An EDC can estimate it's own EFLH based on customer billing data analysis.	EDC Data Gathering
Measure Life (EUL)	Fixed	11years	7

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Data set from the 2012 Pennsylvania Residential End-Use and Saturation Study submitted to Pennsylvania PUC by GDS Associates, Nexent, and Mondre: http://www.puc.pa.gov/electric/pdf/Act129/PA Residential Baseline Report2012.pdf
- Minimum Federal Standard for new Central Air Conditioners/Heat Pumps between 1990 and 2006.
- 3. New York Standard Approach for Estimating Energy Savings from Energy Efficiency Measures in Commercial and Industrial Programs, September 1, 2009.
- DEER 2005 cooling savings for climate zone 16, assumes a variety of thermostat usage patterns.
- "Programmable Thermostats. Report to KeySpan Energy Delivery on Energy Savings and Cost Effectiveness", GDS Associates, Marietta, GA. 2002. 3.6% factor includes 56% realization rate.
- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners⁹⁷ and 40% oversizing of heat pumps.⁹⁸
- 1 New York Standard Approach for Estimating Energy Savings from Energy Efficiency-Measures in Commercial and Industrial Programs, September 1, 2009, based on DEER.

⁹⁷ Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE, February 1, 1999. Confirmed also by *Central Air Conditioning in Wisconsin, a compilation of recent field research*. Energy Center of Wisconsin. May 2008, emended December 15, 2010, https://ecw.org/sites/default/files/241-1_0.pdf

⁹⁸ ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/wp-content/uploads/2014/01/Manual S verification.pdf.

2.2.9 RESIDENTIAL WHOLE HOUSE FANS

Measure Name	Whole House Fans		
Target Sector	Residential Establishments		
Measure Unit	Whole House Fan		
Unit Energy Savings	Varies by location (187 kWh/yr to 232 kWh/yr)		
Unit Peak Demand Reduction	0 kW		
Measure Life	15 years ⁹⁹		
Vintage	Retrofit		

This measure applies to the installation of a whole house fan. The use of a whole house fan will offset existing central air conditioning loads. Whole house fans operate when the outside temperature is less than the inside temperature, and serve to cool the house by drawing cool air in through open windows and expelling warmer air through attic vents.

The baseline is taken to be an existing home with central air conditioning (CAC) and without a whole house fan.

The retrofit condition for this measure is the installation of a new whole house fan.

ELIGIBILITY

This protocol documents the energy savings for the installation of a whole house fan to be used as a compliement to an existing central HVAC system. The target sector is primarily residential.

ALGORITHMS

The energy savings for this measure result from reduced air conditioning operation. While running, whole house fans can consume up to 90% less power than typical residential central air conditioning units. ¹⁰⁰ Energy savings for this measure are based on whole house fan energy savings values reported by the energy modeling software, REM/Rate. ¹⁰¹

MODEL ASSUMPTIONS

- The savings are reported on a "per house" basis with a modeled baseline cooling provided by a SEER 10 Split A/C unit.
- Savings derived from a comparison between a naturally ventilated home and a home with a whole-house fan.
- 2181 square-foot single-family detached home built over unconditioned basement.

DEER EUL Summary, Database for Energy Efficient Resources, accessed October 2010, http://www.deeresources.com.
 Whole House Fan, Technology Fact Sheet, (March 1999), Department of Energy Building Technologies Program, DOE/GO-10099-745, accessed October 2010 http://www.energysavers.gov/your_home/space_heating_cooling/related.cfm/mytopic=12357
 Architectural Energy Corporation, REM/Rate v12.85.

¹⁰² EIA (2005), Table HC1.1.3: "Housing Unit Characteristics by Average Floorspace",

http://www.eia.doe.gov/emeu/recs/recs2005/hc2005_tables/hcfloorspace/pdf/tablehc1.1.3.pdf Used Single Family Detached "Heated" value for Mid-Atlantic region as representative of the living space cooled by a 10 SEER Split A/C unit. The floorspace recorded for "Cooling" is likely to be affected by Room A/C use.

Table 2-41: Whole House Fan Deemed Energy Savings by PA City

City	Annual Energy Savings (kWh/house)
Allentown	204
Erie	200
Harrisburg	232
Philadelphia	229
Pittsburgh	199
Scranton	187
Williamsport	191

This measure assumes <u>no demand savings</u> as whole house fans are generally only used during milder weather (spring/fall and overnight). Peak 100 hours typically occur during very warm periods when a whole house fan is not likely being used.

MEASURE LIFE

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

2.3 **DOMESTIC HOT WATER**

2.3.1 **EFFICIENT ELECTRIC WATER HEATERS**

Measure Name	Efficient Electric Water Heaters	
Target Sector	Residential Establishments	
Measure Unit	Water Heater	
Unit Energy Savings	Varies with Energy Factor of New Unit	
Unit Peak Demand Reduction	Varies with Energy Factor of New Unit	
Measure Life	14 years ¹⁰³	
Vintage	Replace on Burnout	

Efficient electric water heaters utilize superior insulation to achieve energy factors of 0.93 or above. Standard electric water heaters have energy factors of 0.904.

ELIGIBILITY

This protocol documents the energy savings attributed to electric water heaters with Energy Factor of 0.93 or greater (0.94 or greater for a 30 gallon unit). The target sector primarily consists of single-family residences.

ALGORITHMS

The energy savings calculation utilizes average performance data for available residential efficient and standard water heaters and typical water usage for residential homes. The annual energy savings are obtained through the following formula:

$$\Delta kWh/yr = \frac{\left(\frac{1}{EF_{base}} - \frac{1}{EF_{ee}}\right) \times (HW \times 365 \frac{days}{yr} \times 1 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold})}{3413 \frac{Btu}{kWh}}$$

Demand savings result from reduced hours of operation of the heating element, rather than a reduced connected load. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average demandenergy usage during between 2 PM and to 6 PM on summer weekdays to the total annual energy usage.

 $\underline{\hspace{0.1cm}}= \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} From and Facto ETDFr \times \Delta kWh/yr \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} \underline{\hspace{0.1cm}} ergy Savings$ The Energy to Demand Factor is defined below: Average Demand Usage Summer WD 2-6 PM

EnergyToDemandFactor—ETDF Annual-Energy-Usage

The ratio of the average demand between energy usage during 2 PM andto 6 PM on summer weekdays to the total annual energy usage is taken from an electric water heater metering study performed by BG&E (pg 95 of Source 5).

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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Domestic Hot Water

¹⁰³ DEER values, updated October 10, 2008

load shape data collected for a water heater and HVAC demand response study for PJM104. The factor is constructed as follows:

- 1) Obtain the average kW, as monitored for 82 water heaters in PJM territory105, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, 183 spring/fall days) to obtain annual energy usage.
- 2) Obtain the average kW during 2 PM to 6 PM on summer days from the same data.
- 3) The average 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study106.
- 4) The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number, 0.00008294, is the EnergyToDemandFactor.
- The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted in Figure 2-1 below.

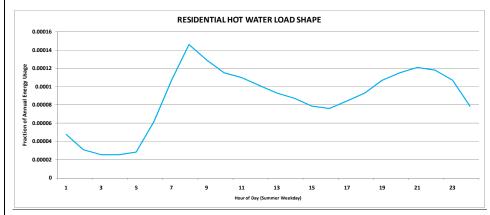


Figure 2-1: Load shapes for hot water in residential buildings taken from a PJM study.

⁴⁰⁴ Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM-Region. Thereport can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx

¹⁰⁵The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5 31 and 5 32

¹⁹⁶The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summerusage to summer weekday usage. The conversion factor is 0.925844. A number smaller than one indicates that for residentialhomes, the hot water usage from 2 PM to 6 PM is slightly higher is the weekends than on weekdays.

DEFINITION OF TERMS

The parameters in the above equation are listed in Table 2-42 below.

Table 2-42: Efficient Electric Water Heater Calculation Assumptions

Component	Type <u>Unit</u>	Values	Source
<i>EF</i> _{base} , Energy Factor of baseline water heater	Fixed None	See Table 2-43 Table 2-43	4 <u>1</u>
EF _{proposedee} , Energy Factor of proposed efficient water heater	Variable<u>None</u>	EDC Data Gathering Default = 0.93 (0.94 for 30 gallon)	Program Design; EDC Data Gathering
HW, Hot water used per day in gallons	Fixed gallon Gallon/day	50-gallon/day	<u>22</u>
T _{hot} , Temperature of hot water	<u>°F</u> Fixed	123 ° F	3
T _{cold} , Temperature of cold water supply	<u>°F</u> Fixed	55 °F	3 4
Energy To Demand Factor ETDF , Energy to Demand Factor (defined above)	Fixed $\frac{kW}{kWh/yr}$	0.000082940.00008047	1-4 <u>5</u>

ENERGY FACTORS BASED ON TANK SIZE

Federal Standards for Energy Factors are equal to 0.97 -0.00132 x Rated Storage in Gallons. The following table shows the Energy Factors for various tank sizes.

Table 2-43: Minimum Baseline Energy Factors based on Tank Size

Tank Size (gallons)	Energy Factor
<u>30</u>	0.9304
40	0.9172
50	0.9040
65	0.8842
80	0.8644
120	0.8116

Note: The new Federal standards that go into effect 4/16/2015 will be incorporated into this measure in the 2016 TRM. These can be viewed at:

http://www1.eere.energy.gov/buildings/appliance standards/product.aspx/productid/27. Do to the increase in baseline efficiency, this measure may no longer provide savings and will be considered for removal during the 2016 TRM development cycle.

DEFAULT SAVINGS

Savings for the installation of efficient electric water heaters are calculated using the formula below:

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$$\Delta kWh/yr = \left(\frac{1}{EF_{Base}} - \frac{1}{EF_{Proposedee}}\right) \times * (3018.0 \, kWh/yr)$$

$$\Delta kW_{peak} = \Delta kWh \times 0.00008294$$

$$\Delta kW_{peak} = \left(\frac{1}{EF_{base}} - \frac{1}{EF_{ee}}\right) \times (0.24286 \, kW)$$

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is 0.904. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30
- Residential Energy Consumption Survey, EIA, 2009.2012 SWE Residential Baseline Study
- 3. 2012 SWE Residential Baseline Study
- 4. Mid-Atlantic TRM Version 3.0, March 2013, footnote #314
- Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal, Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941

2.3.2 HEAT PUMP WATER HEATERS

Measure Name	Heat Pump Water Heaters	
Target Sector	Residential Establishments	
Measure Unit	Water Heater	
Unit Energy Savings	Variable based on energy factosrs	
Unit Peak Demand Reduction	Variable based on energy factors	
Measure Life	14 years 107	
Vintage	Replace on Burnout	

Heat Pump Water Heaters take heat from the surrounding air and transfer it to the water in the tank, unlike conventional water heaters, which use either gas (or sometimes other fuels) burners or electric resistance heating coils to heat the water.

ELIGIBILITY

This protocol documents the energy savings attributed to heat pump water heaters with Energy Factors greater than of 2.0 to 2.3. The target sector primarily consists of single-family residences.

ALGORITHMS

The energy savings calculation utilizes average performance data for available residential heat pump and standard electric resistance water heaters and typical water usage for residential homes. The algorithms take into account interactive effects between the water heater and HVAC system when installed inside conditioned space. The energy savings are obtained through the following formula:

$$\begin{split} \Delta kWh/yr \\ &= \frac{\left(\frac{1}{EF_{base}} - \frac{1}{(EF_{ee} \times F_{derate})}\right) \times HW \times 365 \frac{days}{yr} \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot ^{\circ} F} \times (T_{hot} - T_{cold})}{3413 \frac{Btu}{kWh}} \\ &+ \Delta kWh/yr_{ie,cool} + \Delta kWh/yr_{ie,heat} \end{split}$$

Use below interactive effects calculations only when water heater is installed inside conditioned space. Outside of conditioned space, F_{derate} will account for reduced performance due to cooler annual temperatures.

$$\Delta kWh/yr_{ie,cool} \\ = \frac{HW \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot {}^\circ F} \times (T_{hot} - T_{cold}) \times EFLH_{cool}}{24 \frac{hcs}{dag} \times SEER \times 1000 \frac{W}{kW}} \\ \Delta kWh/yr_{ie,heat} \\ = - \left[\frac{HW \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot {}^\circ F} \times (T_{hot} - T_{cold}) \times EFLH_{heat}}{24 \frac{hcs}{dag} \times HSPF \times 1000 \frac{W}{kW}} \right]$$

http://www.deeresources.com/http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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¹⁰⁷ DEER values, updated October 10, 2008.

For heat pump water heaters, demand savings result primarily from a reduced connected load. However, since the interactive effects during the heating season have no effect on the peak demand, the heating season interactive effects are subtracted from the total kWh savings before the ETDF is applied. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average demandenergy usage between during 2 PM and to 6 PM on summer weekdays to the total annual energy usage.

The ratio of the average <u>demandenergy usage during</u> <u>between 2 PM and</u> to 6 PM on summer weekdays to the total annual energy usage is taken from <u>an electric water heater metering study performed by BG&E (pg 95 of Source 6).load shape data collected for a water heater and HVAC demand response study for PJM ¹⁰⁸. The factor is constructed as follows:</u>

Obtain the average kW, as monitored for 82 water heaters in PJM territory 109, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, and 183 spring/fall days) to obtain annual energy usage.

Obtain the average kW during 2 PM to 6 PM on summer days from the same data.

The average 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study 110.

The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number 0.00008294, is the *EnergyToDemandFactor*.

The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted in Figure 2-2 below.

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¹⁰⁸ Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm_deemed-savings-report.ashx

¹⁰⁹The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32

⁴¹⁰ The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summerusage to summer weekday usage. The conversion factor is 0.925844. A number smaller than one indicates that for residentialhomes, the hot water usage from 2 PM to 6 PM is slightly higher is the weekends than on weekdays.

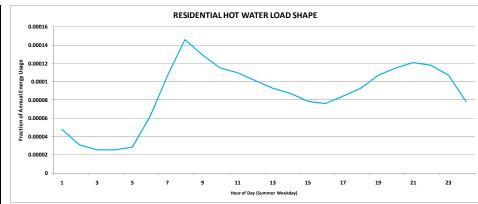


Figure 2-2: Load shapes for hot water in residential buildings taken from a PJM study.

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The parameters in the above equation are listed in Table 2-44.

Table 2-44: Heat Pump Water Heater Calculation Assumptions

Component	Typ e <u>Unit</u>	Values	Source
EFbase EF _{base} , Energy Factor of baseline	FixedNone	See	⁴ 1, 7
water heater		<u>Table 2-47</u>	
		Table 2-47	
		Default= 0.904 (EF for 50 gallon)	
EFproposedEF _{ee} , Energy Factor of	Variable gallons	EDC Data Gathering	Program Design;
proposed efficient water heater		Default : 2.0	EDC Data Gathering
HW , Hot water used per day in gallons	$\frac{gallons}{day}Fixed$	50 gallon/day	⁵ 2
That T_{hot} , Temperature of hot water	<u>°F</u> Fixed	123 <mark>°</mark> ∓	⁶ 3
$rac{Tcold}{T_{cold}}$, $\operatorname{Temperature}$ of cold water supply	<u>°F</u> Fixed	55 ° F	<u>74</u>
FDerateF _{derate} , COP De-rating factor	<u>Fraction</u> Fixed	0.84 <u>Table 2-48</u> <u>Table 2-48</u>	58, and discussion below
EFLH _{cool} , Equivalent Full Load Hours for cooling	$\frac{hours}{yr}$	Table 2-45Table 2-45	<u>6</u>
EFLH _{heat} , Equivalent Full Load Hours for heating	$\frac{hours}{yr}$	Table 2-46Table 2-46	<u>6</u>
HSPF , Heating Seasonal Performance Factor	$rac{Btu}{W\cdot h}$	EDC Data Gathering Default= 7.4	Z
SEER, Seasonal Energy Efficiency Ratio	$\frac{Btu}{W \cdot h}$	EDC Data Gathering Default= 12	7
EnergyToDemandFactorETDF, Energy to Demand Factor (defined above)	Fixed kW kWh/yr	0.000080470.00008294	1-3 8

Table 2-45: Equivalent Full Load Hours for Cooling Season

City	$EFLH_{cool}$
Allentown	<u>487</u>
<u>Erie</u>	<u>389</u>
<u>Harrisburg</u>	<u>551</u>
<u>Philadelphia</u>	<u>591</u>
<u>Pittsburgh</u>	<u>432</u>
<u>Scranton</u>	417
Williamsport	<u>422</u>

Table 2-46: Equivalent Full Load Hours for Heating Season

City	EFLH _{heat}
Allentown	<u>1,193</u>
<u>Erie</u>	<u>1,349</u>
Harrisburg	<u>1,103</u>
<u>Philadelphia</u>	<u>1,060</u>
<u>Pittsburgh</u>	<u>1,209</u>
<u>Scranton</u>	<u>1,296</u>
Williamsport	<u>1,251</u>

ENERGY FACTORS BASED ON TANK SIZE

Federal Standards for <u>electric water heater</u> Energy Factors are equal to 0.97 - 0.00132 x (Rated Storage in Gallons). The following table shows the Energy Factors for various tank sizes.

Table 2-47: Minimum Baseline Energy Factors Based on Tank Size

Tank Size (gallons)	Minimum Energy Factor (EF _{Base})	
40	0.9172	
50	0.9040	
65	0.8842	
80	0.8644	
120	0.8116	

Note: The new Federal standards that go into effect 4/16/2015 will be incorporated into this measure in the 2016 TRM. These can be viewed at:

http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/27

HEAT PUMP WATER HEATER ENERGY FACTOR

The Energy Factors are determined from a DOE testing procedure that is carried out at 67.5°F dry bulb and 56 °F wet bulb temperatures. However, the average dry and wet bulb temperatures in PA areis closer to in the range of 50-56°F DB and 45-50 °F WB¹¹¹. The heat pump performance is temperature and humidity dependent, therefor the location and type of installation is significant—dependent. The plot below shows relative coefficient of performance (COP) compared to the COP at rated conditions 112. According to the linear regression shown on the plot, the COP of a heat pump water heater at 45 °F is 0.84 of the COP at nominal rating conditions. As such, a de rating factor of 0.84 is applied to the nominal Energy Factor of the Heat Pump water heaters. To account for this, an EF de-rating factor (F_{derate}) has been adapted from a 2013 NEEA HPWH field study, Figure 15 (Source 5). The results used are for "Heating Zone 1", which is comprised of Olympia, WA and Portland, OR and have average dry and wet bulb temperatures (51°F DB, 47°F WB and 55°F DB, 49°F WB, respectively) 113 comparable to Pennsylvania.

Table 2-48: EF De-rating Factor for Various Installation Locations

Installation Location	F _{derate} ¹¹⁴
Inside Conditioned Space	<u>0.98</u>
Garage	<u>0.85</u>
<u>Basement</u>	<u>0.72</u>
Default ¹¹⁵	<u>0.87</u>

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¹¹¹ Based on average weather data from weatherbase.com for the 7 Pennsylvania cities referenced elsewhere in this TRM (Allentown, Erie, Harrisburg, Philadelphia, Pittsburgh, Scranton, Williamsport). Based on TMY2 weather files from DOE2.com for Erie, Harrisburg, Pittsburgh, Wilkes-Barre, And Williamsport, the average annual wetbulb temperature is 45 – 1.3 °F. The wetbulb-temperature in garages or attics, where the heat pumps are likely to be installed, are likely to be two or three degrees higher, but for simplicity, 45 °F is assumed to be the annual average wetbulb temperature.

¹¹² The performance curve is adapted from Table 1 in http://wescorhvac.com/HPWH%20design%20details.htm#Single-stage%20HPWHs

The performance curve depends on other factors, such as hot water set point. Our adjustment factor of 0.84 is a first order approximation based on the information available in literature.

¹¹³ Ibid

¹¹⁴ Calculated by dividing the COP in each location from Figure 15 by the rated Energy Factor (2.35) of the unit tested in the study (AirGenerate ATI66).

¹¹⁵ Weighted average of values in Table 107 for water heater locations for all space heating fuel types. Northwest Energy Efficiency Alliance 2011 Residential Building Stock Assessment: Single-Family Characteristics and Energy Use. Published September 18, 2012. Online at http://neea.org/docs/reports/residential-building-stock-assessment-single-family-characteristics-and-energy-use.pdf?sfvrsn=8

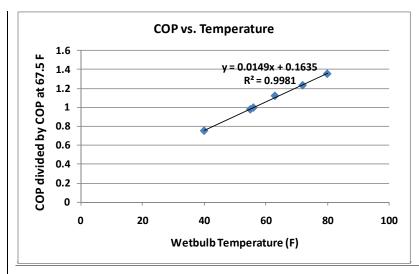


Figure 2-3: Dependence of COP on outdoor wet-bulb temperature

DEFAULT SAVINGS

<u>SDefault savings</u> for the installation of <u>efficient electric heat pump</u> water heaters <u>not located inside conditioned space</u> are calculated using the formulas below.

$$\Delta kWh = \left(\frac{1}{EF_{Base}0.904} - \frac{1}{(EF_{ee} \times F_{derate}) * 0.84}\right) \times * (3018.0)$$

$$\Delta kW_{peak} = \frac{AkWh \times X \ 0.00008294}{\left(\frac{1}{0.904} - \frac{1}{(EF_{ee} \times F_{derate})}\right) \times (0.24286 \ kW)}$$

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, an electric water heater's lifespan is 14 years116.

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with calculation of energy and demand savings using above algorithms.

SOURCES

1. Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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¹¹⁶ DEER values, updated October 10, 2008

Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30

- 2. Residential Energy Consumption Survey, EIA, 2009.
- 3. 2012 SWE Residential Baseline Study
- 4. Mid-Atlantic TRM Version 3.0, March 2013, footnote #314
- NEEA Heat Pump Water Heater Field Study Report. Prepared by Fluid Market Strategies, 2013. http://neea.org/docs/default-source/reports/heat-pump-water-heater-field-study-report.pdf?sfvrsn=5
- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 <u>calculated from kWh consumption for cooling and heating. Models assume 50% over-</u>
 sizing of air conditioners117 and 40% oversizing of heat pumps.¹¹⁸
- 2014 Pennsylvania Residential Baseline Study. Presented to the PUC by GDS Associates.
- 8. Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal, Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941

117 Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE, February 1, 1999. Confirmed also by *Central Air Conditioning in Wisconsin, a compilation of recent field research.* Energy Center of

Wisconsin. May 2008, emended December 15, 2010

118 ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

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Field Code Changed

2.3.3 SOLAR WATER HEATERS

Measure Name	Solar Water Heaters
Target Sector	Residential Establishments
Measure Unit	Water Heater
Default Unit Energy Savings	1,698 kWh
Default Unit Peak Demand Reduction	0.277 kW
Measure Life	15 years ¹¹⁹
Vintage	Retrofit

Solar water heaters utilize solar energy to heat water, which reduces electricity required to heat water

ELIGIBILITY

This protocol documents the energy savings attributed to solar water in PA. The target sector primarily consists of single-family residences.

ALGORITHMS

The energy savings calculation utilizes average performance data for available residential solar and standard water heaters and typical water usage for residential homes. The energy savings are obtained through the following formula:

$$\Delta kWh/yr = \frac{\left(\frac{1}{EF_{base}} - \frac{1}{EF_{ee}}\right) \times HW \times 365 \frac{days}{yr} \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs^{\circ}F} \times (T_{hot} - T_{cold})}{3413 \frac{Btu}{kWh}}$$

The energy factor used in the above equation represents an average energy factor of market available solar water heaters¹²⁰.

The demand reduction is taken as the annual energy usage of the *baseline* water heater multiplied by the ratio of the average <u>demandenergy usage duringbetween</u> 2PM <u>andte</u> 6PM on summer weekdays to the total annual energy usage. Note that this is a different formulation than the demand savings calculations for other water heaters. This modification of the formula reflects the fact that a solar water heater's capacity is subject to seasonal variation, and that during the peak summer season, the water heater is expected to fully supply all domestic hot water needs.

$$\Delta kW_{peak}$$
 = EnergyToDemandFactor ETDF $\times kWh/yr_{base}$ BaseEnergy Usage

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¹¹⁹ ENERGY STAR Solar Water Heater Benefits and Savings. Accessed 8/8/2014.

http://www.energystar.gov/index.cfm?c=solar_wheat.pr_savings_benefits

120 We have taken the average energy factor for all solar water heaters with collector areas of 50 ft2 or smaller from http://www.solar-rating.org/ratings/ratings.htm. As a cross check, we have calculated that the total available solar energy in PA for the same set of solar collectors is about twice as much as the savings claimed herein – that is, there is sufficient solar capacity to actualize an average energy factor of 1.84.

$$kWh/yr_{base} = \frac{\left(\frac{1}{EF_{base}}\right) \times HW \times 365 \frac{days}{yr} \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot F} \times (T_{hot} - T_{cold})}{3413 \frac{Btu}{bWh}}$$

The ETDF (Energy to Demand Factor) is defined below:

EnergyToDemandFactorETDF

 $= \frac{Average \ \frac{Usage}{Demand_{Summer \ WD \ 2 \ PM-6 \ PM}}}{Annual \ Energy \ Usage}$

The ratio of the average <u>demand energy usage betweenduring noon 2 PM</u> and 6 PM on summer weekdays to the total annual energy usage is taken from <u>an electric water heater metering study performed by BG&E (pg 95 of Source 2). lead shape data collected for a water heater and HVAC demand response study for PJM¹²¹. The factor is constructed as follows:</u>

Obtain the average kW, as monitored for 82 water heaters in PJM territory 122, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, and 183 spring/fall days) to obtain annual energy usage.

Obtain the average kW during the 2 PM to 6 PM time period on summer weekdays from the same data. These are the hours that PJM has defined as peak hours ...

The average weekday 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study 123.

The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number, 0.00008294, is the *EnergyToDemandFactor*.

The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted for three business types in Figure 2-6

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¹²¹ Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx—

¹²² The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32

¹²³ The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summerusage to summer weekday usage. The conversion factor is 0.925844. A number smaller than one indicates that for residentialhomes, the hot water usage from 2 PM to 6 PM is slightly higher is the weekends than on weekdays.

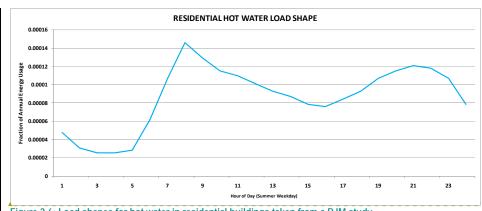


Figure 2-6: Load shapes for hot water in residential buildings taken from a PJM study.

DEFINITION OF TERMS

The parameters in the above equation are listed in Table 2-49.

Table 2-49: Solar Water Heater Calculation Assumptions

Component	TypeUnit	Values	Source
${\it EF}_{\it base}$, Energy Factor of baseline electric water heater	Variable Fraction	See Table 2-50 Table 2-50	<u>3</u> [€]
		Default= 0.904 (50 gallon)	<u>3</u> 6
EF _{ee} , Year-round average Energy Factor of proposed solar water heater	Variable Fraction	EDC Data Gathering	EDC Data Gathering
		Default=1.84	1
HW , Hot water used per day in gallons	gallons day ^{Fixed}	50 gallon/day	<u>4</u> [∓]
T _{hot} , Temperature of hot water	°F Fixed	123	<u>5</u> 8
T _{cold} , Temperature of cold water supply	°F Fixed	55	<u>69</u>
Default Baseline Energy Usage for an electric water heater without a solar water heater (kWh)	Calculated	3,338	
EnergyToDemand FactorETDF , Energy to Demand Factor (defined above)Ratio of average 2 to 6 PM usage during summer peak hours (on- weekdays) to annual energy usage	Fixed $\frac{kW}{kWh/yr}$	0.000082940.00008047	2-5

ENERGY FACTORS BASED ON TANK SIZE

Federal standards for Energy Factors (EF) are equal to 0.97 - (.00132 x Rated Storage in Gallons). The following table shows the baseline Energy Factors for various tank sizes:

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Table 2-50: Minimum Baseline Energy Factors Based on Tank Size

Tank Size (gallons)	Minimum Energy Factors
40	0.9172
50	0.9040
65	0.8842
80	0.8644
120	0.8116

Note: The new Federal standards that go into effect 4/16/2015 will be incorporated into this measure in the 2016 TRM. These can be viewed at:

http://www1.eere.energy.gov/buildings/appliance standards/product.aspx/productid/27

DEFAULT SAVINGS

The partially-deemed algorithm for savings attributable to the installation of a solar water heater is given below.

$$\begin{array}{ll} \Delta kWh/yr & = \left(\frac{1}{EF_{base}} - \frac{1}{EF_{ee}}\right) \times \left(3018 \frac{kWh}{yr}\right) \\ \Delta kW_{peak} & = \left(\frac{1}{EF_{base}}\right) \times \left(0.24286 \ kW\right) \end{array}$$

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific quidelines and requirements for evaluation procedures.

SOURCES

- The average energy factor for all solar water heaters with collector areas of 50 ft² or smaller is from http://www.solar-rating.org/ratings/ratings.htm. As a cross check, we have calculated that the total available solar energy in PA for the same set of solar collectors is about twice as much as the savings claimed herein – that is, there is sufficient solar capacity to actualize an average energy factor of 1.84.
- Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx—
- 3. The average is calculated over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32
- 4. PJM peak load hours are from 2 pm to 6 pm on summer weekdays.

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- The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summer usage to summer weekday usage. The conversion factor is 0.925844.
- Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941
- 6-3. Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50 gallon tank, this is approximately 90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30
- 4. Residential Energy Consumption Survey, EIA, 2009.
- 7.5. Pennsylvania Statewide Residential End-Use and Saturation Study. 2012.
- 8.6. Mid-Atlantic TRM Version 3.0, March 2013, footnote #314

Field Code Changed

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Rev Date: June 2015

Measure Name	Fuel Switching: DHW-Electric Resistance to Fossible Fuel Water Heater
Target Sector	Residential
Measure Unit	Water Heater
Unit Energy Savings	3,338 kWh <u>/vr</u>
Unit Peak Demand Reduction	0.277 <u>0.2687</u> kW
Gas, Fossil Fuel Consumption Increase	Gas: 15.38 MMBtu Propane: 15.38 MMBtu Oil: 20.04 MMBtu
Measure Life	Gas:13 years ¹²⁴ Propane: 13 years ¹²⁵ Oil: 8 years ¹²⁶
<u>Vintage</u>	Replace on Burnout

Natural gas, propane and oil water heaters generally offer the customer lower costs compared to standard electric water heaters. Additionally, they typically see an overall energy savings when looking at the source energy of the electric unit versus the fossil fuel-fired unit. Federal standard electric water heaters have energy factors of 0.904 for a 50 gal unit and an ENERGY STAR gas and propane-fired water heater have an energy factor of 0.67 for a 40gal unit and 0.514 for an oil-fired 40 gal unit.

ELIGIBILITY

This protocol documents the energy savings attributed to converting from a standard electric water heater with Energy Factor of 0.904 or greater to an ENERGY STAR natural gas or propane water heater with Energy Factor of 0.67 or greater and 0.514 for oil water heater. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. The target sector primarily consists of single-family residences.

ALGORITHMS

The energy savings calculation utilizes average performance data for available residential standard electric and fossil fuel-fired water heaters and typical water usage for residential homes. Because there is little electric energy associated with a fossil fuel-fired water heater, the energy savings are the full energy utilization of the electric water heater. The energy savings are obtained through the following formula:

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¹²⁴ RECS 2009 data indicate that the most common size is 31 to 49 gal. An average of 40 gal unit is considered for this protocol. http://www.eia.gov/consumption/residential/data/2009/).

¹²⁵ DEER values, updated October 10, 2008: http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

¹²⁶ ibid.

Although there is a significant electric savings, there is an associated increase in fossil fuel energy consumption. While this fossil fuel consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel usage is obtained through the following formula:

Fuel Consumption (MMBtu/yr) =
$$\frac{\left\{ \left(\frac{1}{\mathsf{EF}_{\mathsf{fuel},\mathsf{inst}}}\right) \times \left(\mathsf{HW} \times 365 \frac{\mathsf{days}}{\mathsf{yr}} \times 1 \frac{BTU}{lb \cdot {}^{\circ}F} \times 8.3 \frac{\mathsf{lb}}{\mathsf{gal}} \times (\mathsf{T}_{\mathsf{hot}} \cdot \mathsf{T}_{\mathsf{cold}})\right) \right\}}{1,000,000 \frac{\mathsf{Btu}}{\mathsf{MMBtu}}}$$

Demand savings result from the removal of the connected load of the electric water heater. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage betweenduring-2 PM andto: 6 PM on summer weekdays to the total annual energy usage.

The ratio of the average energy usage <u>betweenduring 2 PM and</u>te 6 PM on summer weekdays to the total annual energy usage is taken from <u>an electric water heater metering study performed by BG&E (pg 95 of Source 7).</u>

load shape data collected for a water heater and HVAC demand response study for PJM127. The factor is constructed as follows:

Obtain the average kW, as monitored for 82 water heaters in PJM territory128, for each hour of the typical summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, and 183 spring/fall days) to obtain annual energy usage.

Obtain the average kW during 2 PM to 6 PM on summer days from the same data.

The average 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study129.

The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number 0.00008294, is the EnergyToDemandFactor.

The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted in Figure 2-8.

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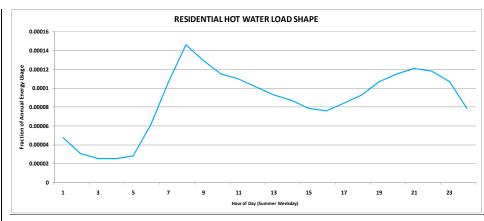


Figure 2-8: Load shapes for hot water in residential buildings taken from a PJM.

DEFINITION OF TERMS

The parameters in the above equation are listed in Table 2-51 below.

 $\textit{Table 2-51: Calculation Assumptions for Fuel Switching } \ \underline{\textit{DHW-}} Electric \underline{\textit{Resistance}} \ to \ Fossil \ Fuel \ Water \ Heater$

Component	TypeUnit	Values	Source
EF _{elec,bl_} , Energy Factor of baseline water heater	<u>Fraction</u> Variable	EDC Data Gathering	<u>1</u> 4
		Default: Table 2-52Table 2-52	
<i>EF_{NG,inst.}</i> , Energy Factor of installed natural gas water heater	<u>Fraction</u> Variable	EDC Data Gathering	<u>2</u> 5
water neater		<u>Default:</u> ≥0.67	
EF _{Propane,inst} , Energy Factor of installed propane	<u>Fraction</u> Variable	EDC Data Gathering	<u>2</u> 5
water heater		<u>Default</u> : ≥0.67	
EF _{Tankless Water Heater} , Energy Factor of installed	<u>Fraction</u> Variable	EDC Data Gathering	<u>2</u> 5
tankless water heater		Default: ≥0.82	
EF _{Oil,inst.} , Energy Factor of installed oil water	<u>Fraction</u> Variable	EDC Data Gathering	<u>3</u> 6
heater*		Default: ≥0.514	
HW_, Hot water used per day in gallons	$\frac{gallons}{day}Fixed$	50 gallon/day	<u>4</u> 7
T _{hot_} , Temperature of hot water	<u>°F</u> Fixed	123	<u>5</u> 8
T _{cold_} , Temperature of cold water supply	<u>°F</u> Fixed	55	<u>6</u> 9
EnergyToDemandFactorETDF _ Energy to Demand Factor (defined above)	Fixed kW kWh/yr	0.000082940.00008047	1-3 <u>7</u>

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Domestic Hot Water

ENERGY FACTORS BASED ON TANK SIZE

Federal Standards for Energy Factors are equal to 0.97 -0.00132 x Rated Storage in Gallons. The following table shows the Energy Factors for various tank sizes.

Table 2-52: Minimum Baseline Energy Factors based on Tank Size

Tank Size (gallons)	Minimum Energy Factors (EF _{elec,bl} EFbase)
40	0.9172
50	0.9040
65	0.8842
80	0.8644
120	0.8116

DEFAULT SAVINGS

The electric savings for the installation of a fossil fuel water heater should be calculated using the partially deemed algorithm below.

$$\begin{split} \Delta kWh/yr & = \left(\frac{1}{EF_{elec,bl}}\right) \times \left(3018\frac{kWh}{yr}\right) \\ \Delta kW_{peak} & = \left(\frac{1}{EF_{elec,bl}}\right) \times \left(0.24286\;kW\right) \end{split}$$

Δ kWh = (1/EFelect, bl) * 3018.0

The default savings for the installation of a natural gas/ propane/oil water heater in place of a standard electric water heater are listed in Table 2-53 below.

Table 2-53: Energy Savings and Demand Reductions for Fuel Switching, Domestic Hot Water Electric to Fossil Fuel

Electric unit Energy Factor	Energy Savings (kWh/ <u>yr</u>)	Demand Reduction (kW)
0.904	3338	<u>0.2687</u> 0.277

The default fossil fuel consumption for the installation of a standard efficiency natural gas/propane/oil water heater in place of a standard electric water heater is listed in

Table 2-54

Table 2-54 below.

Table 2-54: Fuel Consumption for Fuel Switching, Domestic Hot Water Electric to Fossil Fuel

Fuel Type	Energy Factor	Fossil Fuel Consumption (MMBtu)
Gas	0.67	15.37
Propane	0.67	15.37
Oŀįl	0.514	20.04

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Note: 1 MMBtu of propane is equivalent to 10.87 gals of propane, and 1 MMBtu of oil is equivalent to 7.19 gals of oil^{130} .

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, gasand propane water heaters' lifespans are 13 years. An oil water heater's lifespand is 8 years.131.

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

- Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx
- The average is over all 82 water heaters and over all summer, spring/fall, or winter days.
 The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32
- The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summer usage to summer weekday usage. The conversion factor is 0.0008294.
- 4.1. Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is 0.904. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30
- 5-2. Commission Order 132 requires fuel switching to ENERGY STAR measures, not standard efficiency measures. The Energy Factor has therefore been updated to reflect the EnergyStar standard for Gas Storage Water Heaters beginning September 1, 2010. From Residential Water Heaters Key Product Criteria. http://www.energystar.gov/index.cfm?c=water-heat.pr-crit-water-heaters Accessed June 2013
- 6-3. Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons for oil-fired storage water heater. For a 40-gallon tank this 0.514. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 307. "Energy Conservation Program for Consumer Products: Test Procedure for Water Heaters", Federal Register / Vol. 63, No. 90, p. 26005-26006.
- 7.4. "Energy Conservation Program for Consumer Products: Test Procedure for Water Heaters", Federal Register / Vol. 63, No. 90, p. 26005-26006.
- 8-5. Pennsylvania Statewide Residential End-Use and Saturation Study, 2012, page 42.9.

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 $^{^{130}\,\}underline{\text{http://www.energystar.gov/ia/business/industry/industry_challenge/QuickConverter.xls}$

¹³¹_DEER values, updated October 10, 2008: http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xis

¹³² See page 42 of the 2013 TRC Test Final Order

9.6. Mid-Atlantic TRM Version 3.0, March 2013, footnote #314

40.7. Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept, 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941

Field Code Changed

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2.3.5 FUEL SWITCHING: HEAT PUMP WATER HEATER TO FOSSIL FUEL WATER HEATER

Measure Name	Fuel Switching: Heat Pump Water Heater to Fossil Fuel Heater
Target Sector	Residential
Measure Unit	Water Heater
Unit Energy Savings	1,793 - <u>1,734.5</u> kWh (for EF = 2.0)
Unit Peak Demand Reduction	<u>0.150-0.140</u> kW
Gas, Fossil Fuel Consumption Increase	Gas: 15.38 MMBtu Propane: 15.38 MMBtu Oil: 20.04 MMBtu
Measure Life	Gas:13 years ¹³³ Propane: 13 years ¹³⁴ Oil: 8 years ¹³⁵
<u>Vintage</u>	Replace on Burnout

Natural gas, propane and oil water heaters reduce electric energy and demand compared to heat pump water heaters. Standard heat pump water heaters have energy factors of 2.0 and ENERGY STAR gas and propane water heaters have an energy factor of 0.67 for a 40_gal unit and 0.514 for an oil-fired 40 gal unit.

ELIGIBILITY

This protocol documents the energy savings attributed to converting from a standard heat pump water heater with Energy Factor of 2.0 or greater to an ENERGY STAR natural gas or propane water heater with Energy Factor of 0.67 or greater and 0.514 for an oil water heater. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. The target sector primarily consists of single-family residences.

ALGORITHMS

The energy savings calculation utilizes average performance data for available residential standard heat pump water heaters and fossil fuel-fired water heaters and typical water usage for residential homes. Because there is little electric energy associated with a fossil fuel-fired water heater, the energy savings are the full energy utilization of the heat pump water heater. The energy savings are obtained through the following formula:

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¹³³ DEER values, updated October 10, 2008: : http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

¹³⁴ ibid.

¹³⁵ ibid.

$$\varDelta kWh/yr \qquad = \frac{\left\{ \left(\frac{1}{EF_{HP,bl} \times F_{Derate}}\right) \times \left(HW \times 365 \ \frac{days}{yr} \times 1 \ \frac{BTU}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold})\right) \right\}}{3413 \frac{Btu}{kWh}}$$

 $+ \, \Delta kWh/yr_{ie,cool} + \Delta kWh/yr_{ie,heat}$

Use below interactive effects calculations only when water heater is installed inside conditioned space. Outside of conditioned space, F_{derate} will account for reduced performance due to cooler annual temperatures.

$$\Delta kWh/yr_{ie,cool} \\ = \frac{HW \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot F} \times (T_{hot} - T_{cold}) \times EFLH_{cool}}{24 \frac{hrs}{day} \times SEER \times 1000 \frac{W}{kW}} \\ = -\left[\frac{HW \times 8.3 \frac{lbs}{gal} \times 1 \frac{Btu}{lbs \cdot F} \times (T_{hot} - T_{cold}) \times EFLH_{heat}}{24 \frac{hrs}{day} \times HSPF \times 1000 \frac{W}{kW}} \right]$$

Although there is a significant electric savings, there is an associated increase in fossil fuel energy consumption. While this fossil fuel consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel energy is obtained through the following formula:

Fuel Consumption (MMBtu/yr) =
$$\frac{\left\{ \left(\frac{1}{EF_{NG,inst}}\right) \times \left(HW \times 365 \frac{days}{yr} \times 1 \frac{BTU}{lb \cdot °F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold})\right) \right\}}{1,000,000 \frac{Btu}{MMBtu}}$$

Demand savings result from the removal of the connected load of the heat pump water heater. However, since the interactive effects during the heating season have no effect on the peak demand, the heating season interactive effects are subtracted from the total kWh savings before the ETDF is applied. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average demandenergy usage during between 2 PM toand 6 PM on summer weekdays to the total annual energy usage.

The ratio of the average energy usage <u>betweenduring-2 PM to and 6 PM on summer weekdays to the total annual energy usage is taken from an electric water heater metering study performed by BG&E (pg 95 of Source 8). load shape data collected for a water heater and HVAC demand response study for PJM¹³⁸. The factor is constructed as follows:</u>

Obtain the average kW, as monitored for 82 water heaters in PJM territory 137, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter

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⁴³⁶ Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx

¹³⁷The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32

days, and 183 spring/fall days) to obtain annual energy usage. Obtain the average kW during 2 PM to 6 PM on summer days from the same data.

The average 2 PM to 6 PM demand is converted to average *weekday* 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study ¹³⁸.—

The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number, 0.00008294, is the EnergyToDemandFactor.

The load shapes (fractions of annual energy usage that occur within each hour) during summerweek days are plotted in Figure 2-9.

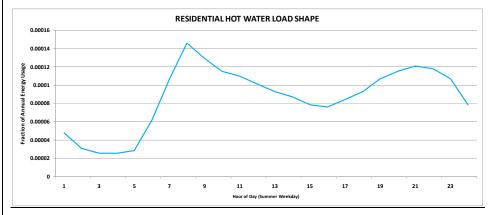


Figure 2-9: Load shapes for hot water in residential buildings taken from a PJM.

DEFINITION OF TERMS

The parameters in the above equation are listed in <u>Table 2-55Table 2-55</u>.

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¹³⁸ The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summerusage to summer weekday usage. The conversion factor is 0.925844. A number smaller than one indicates that for residentialhomes, the hot water usage from noon to 8 PM is slightly higher is the weekends than on weekdays.

Table 2-55: Calculation Assumptions for Heat Pump Water Heater to Fossil Fuel Water Heaters

Component	Type <u>Unit</u>	Values	Source
<i>EF_{HP,bl}</i> , Energy Factor of baseline heat pump water heater	Variable Fraction	Default ≥ 2.0 or EDC Data Gathering	<u>1</u> 4
<i>EF_{NG,inst}</i> . Energy Factor of installed natural gas water heater	Variable Fraction	≥ 0.67 or EDC Data Gathering	<u>2</u> 5
<i>EF</i> _{Propane,inst} , Energy Factor of installed propane water heater	Variable Fraction	>=0.67 or EDC Data Gathering	<u>2</u> 5
<i>EF</i> _{Tankless Water Heater} , Energy Factor of installed tankless water heater	Variable Fraction	>=0.82	<u>2</u> 5
EF _{Oil,inst} , Energy Factor of installed oil water heater	Variable Fraction	>=0.514 or EDC Data Gathering	<u>3</u> 6
HW_, Hot water used per day in gallons	gallons day Fixed	50 gallon/day	<u>4</u> 7
T _{hot} , Temperature of hot water	<u>°F</u> Fixed	123 ° F	<u>5</u> 8
T _{cold} , Temperature of cold water supply	<u>°F</u> Fixed	55 ° ∓	<u>6</u> 9
F _{Derate} , COP De-rating factor	Fixed Fraction	<u>Table 2-58Table</u> 2-580.84	710, and discussion below
EFLH _{cool} , Equivalent Full Load Hours for cooling	hours yr	Table 2-56Table 2-56	8
EFLH _{heat} , Equivalent Full Load Hours for heating	$\frac{hours}{yr}$	Table 2-57Table 2-57	8
HSPF , Heating Seasonal Performance Factor	$\frac{Btu}{W \cdot h}$	EDC Data Gathering Default= 7.4	9
SEER , Seasonal Energy Efficiency Ratio	$\frac{Btu}{W \cdot h}$	EDC Data Gathering Default= 12	9
EnergyToDemandFactorETDF, Average Usage per Average Energy Usage	$\frac{kW}{kWh/yr}$	0.000082940.00008047	1-3 <u>10</u>

Table 2-56: Equivalent Full Load Hours for Cooling Season

City	EFLH_{cool}
<u>Allentown</u>	<u>487</u>
<u>Erie</u>	<u>389</u>
<u>Harrisburg</u>	<u>551</u>
<u>Philadelphia</u>	<u>591</u>
<u>Pittsburgh</u>	<u>432</u>
Scranton	417
Williamsport	<u>422</u>

Table 2-57: Equivalent Full Load Hours for Heating Season

City	$EFLH_{heat}$
<u>Allentown</u>	<u>1,193</u>
<u>Erie</u>	<u>1,349</u>
<u>Harrisburg</u>	<u>1,103</u>
<u>Philadelphia</u>	<u>1,060</u>
<u>Pittsburgh</u>	<u>1,209</u>
<u>Scranton</u>	<u>1,296</u>
Williamsport	<u>1,251</u>

HEAT PUMP WATER HEATER ENERGY FACTOR

The Energy Factors are determined from a DOE testing procedure that is carried out at 67.5°F dry bulb and 56°F wet bulb temperatures. However, the average dry and wet bulb temperatures in PA areis closer to in the range of 50-56°F DB and 45-50°F WB¹³⁹. The heat pump performance is temperature and humidity dependent, therefor the location and type of installation is significant dependent. The plot in Figure 2-10 shows relative coefficient of performance (COP) compared to the COP at rated conditions 1440. According to the linear regression shown on the plot, the COP of a heat pump water heater at 45°F is 0.84 of the COP at nominal rating conditions. As such, a de-rating factor of 0.84 is applied to the nominal Energy Factor of the Heat Pump water heaters. To account for this, an EF de-rating factor (F_{derate}) has been adapted from a 2013 NEEA HPWH field study, Figure 15 (Source 8). The results used are for "Heating Zone 1", which is comprised of Olympia, WA and Portland, OR and have average dry and wet bulb temperatures (51°F DB, 47°F WB and 55°F DB, 49°F WB, respectively) 141°comparable to Pennsylvania.

141 Ibid

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¹³⁹ Based on average weather data from weatherbase.com for the 7 Pennsylvania cities referenced elsewhere in this TRM (Allentown, Erie, Harrisburg, Philadelphia, Pittsburgh, Scranton, Williamsport). Based on TMY2 weather files from DOE2.com for Erie, Harrisburg, Pittsburgh, Wilkes-Barre, And Williamsport, the average annual wetbulb temperature is 45 ± 1.3 °F. The wetbulb-temperature in garages or attics, where the heat pumps are likely to be installed, are likely to be two or three degrees higher, but for simplicity, 45 °F is assumed to be the annual average wetbulb temperature.

¹⁴⁰ The performance curve is adapted from Table 1 in http://wescorhvac.com/HPWH%20design%20details.htm#Single-stage%20HPWHs

The performance curve depends on other factors, such as hot water set point. Our adjustment factor of 0.84 is a first order approximation based on the information available in literature.

<u>Table</u> 2-58: EF De-rating Factor for Various Installation Locations

Installation Location	F _{derate} ¹⁴²
Inside Conditioned Space	<u>0.98</u>
Garage	<u>0.85</u>
<u>Basement</u>	<u>0.72</u>
Default ¹⁴³	0.87

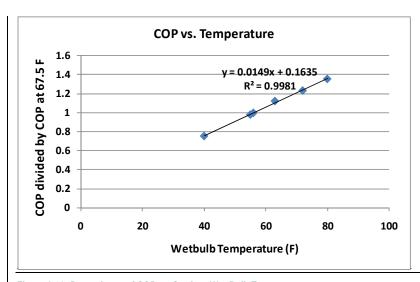


Figure 2-10: Dependence of COP on Outdoor Wet-Bulb Temperature

DEFAULT_SAVINGS

The savings for the installation of a fossil fuel water heater <u>in place of a heat pump water heater</u> <u>not located inside conditioned space</u> should be calculated using the <u>partially</u> deemed algorithm below.

$$\Delta kWh = \frac{1}{[EFHP,Bl*Fderate]} \times 3018.0$$

$$\Delta kWh/yr = \left(\frac{1}{EF_{HP,bl} \times F_{Derate}}\right) \times \left(3018\frac{kWh}{yr}\right)$$

$$\Delta kW_{peak} = \left(\frac{1}{EF_{HP,bl} \times F_{Derate}}\right) \times (0.24286 \, kW)$$

The fossil fuel consumption should be calculated using the <u>partially</u> deemed algorithm below.

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¹⁴² Calculated by dividing the COP in each location from Figure 15 by the rated Energy Factor (2.35) of the unit tested in the study (AirGenerate ATI66).

⁽AirGenerate ATI66).

143 Weighted average of values in Table 107 for water heater locations for all space heating fuel types. Northwest Energy Efficiency
Alliance 2011 Residential Building Stock Assessment: Single-Family Characteristics and Energy Use. Published September 18,
2012. Online at http://neea.org/docs/reports/residential-building-stock-assessment-single-family-characteristics-and-energy-use.pdf?sfvrsn=8

Fossil Fuel Consumption (MMBtu/yr) =
$$\left(\frac{1}{EF_{NG,inst}}\right) \times \left(10.3 \frac{MMBtu}{yr}\right) = \left(\frac{1}{EFNG}, Inst\right) * \times 10.3$$

The default savings for the installation of a fossil fuel-fired water heater in place of a standard heat pump water heater in an unknown, default location are listed in Table 2-59 below.

Table 2-59: Energy Savings and Demand Reductions for Heat Pump Water Heater to Fossil Fuel Water Heater

Heat Pump unit Energy Factor	Energy Savings (kWh)	Demand Reduction (kW)	
2.0	1,796 <u>1,734.5</u>	0.150 <u>0.140</u>	

The default_gas consumption for the installation of an ENERGY STAR natural gas, propane or oil water heater in place of a standard heat pump water heater is listed in Table 2-60

Table 2-60 below.

Table 2-60: Gas, Oil, Propane Consumption for Heat Pump Water Heater to Fossil Fuel Water Heater

Fuel Type	Energy Factor	Gas Consumption (MMBtu)
Gas	0.67	15.37
Propane	0.67	15.37
OII	0.514	20.04

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, gas and propane water heaters' lifespan are 13 years144. An oil water heater's lifespand is 8 years.145

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

- Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx
- 2. The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5-31 and 5-32
- The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summer usage to summer weekday usage. The conversion

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

¹⁴⁵DEER values, updated October 10, 2008: http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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¹⁴⁴ DEER values, updated October 10, 2008

factor is 0.925844. A number smaller than one indicates that for residential homes, the hot water usage from 2 PM to 6 PM is slightly higher is the weekends than on weekdays.

- 4-1. Heat pump water heater efficiencies have not been set in a Federal Standard. However, the Federal Standard for water heaters does refer to a baseline efficiency for heat pump water heaters as EF = 2.0 "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129.
- 5-2. Commission Order 146 requires fuel switching to ENERGY STAR measures, not standard efficiency measures. The Energy Factor has therefore been updated to reflect the EnergyStar standard for Gas Storage Water Heaters beginning September 1, 2010. From Residential Water Heaters Key Product Criteria. http://www.energystar.gov/index.cfm?c=water heat.pr crit water heaters Accessed June 2013 ederal Standards are 0.67 -0.0019 x Rated Storage in Gallons. Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons. For a 40-gallon tank this is 0.594. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30
- 6-3. Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons for oil-fired storage water heater. For a 40-gallon tank this 0.514. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 307. "Energy Conservation Program for Consumer Products: Test Procedure for Water Heaters", Federal Register / Vol. 63, No. 90, p. 26005-26006.
- 7.4. "Energy Conservation Program for Consumer Products: Test Procedure for Water Heaters", Federal Register / Vol. 63, No. 90, p. 26005-26006.
- 8.5. Pennsylvania Statewide Residential End-Use and Saturation Study, 2012, page 42.
- 9-6. Mid-Atlantic TRM Version 3.0. March 2013, footnote #314
- 7. NEEA Heat Pump Water Heater Field Study Report. Prepared by Fluid Market Strategies, 2013.
- Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH
 calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners147 and 40% oversizing of heat pumps.¹⁴⁸
- 40-9. 2014 Pennsylvania Residential Baseline Study. Presented to the PUC by GDS Associates.
- 41.10. Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal, Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941

148 ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files/?id=67.

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¹⁴⁷ Neme, Proctor, Nadal, "National Energy Savings Potential From Addressing Residential HVAC Installation Problems. ACEEE, February 1, 1999. Confirmed also by *Central Air Conditioning in Wisconsin, a compilation of recent field research*. Energy Center of Wisconsin. May 2008, emended December 15, 2010

2.3.6 WATER HEATER TANK WRAP

Measure Name	Water Heater Tank Wrap
Target Sector	Residential
Measure Unit	Tank
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	7 years ¹⁴⁹
Vintage	Retrofit

This measure applies to the installation of an insulated tank wrap or "blanket" to existing residential electric hot water heaters.

The base case for this measure is a standard residential, tank-style, electric water heater with no external insulation wrap.

ELIGIBILITY

This measure documents the energy savings attributed to installing an insulating tank wrap on an existing electric resistance water heater. The target sector is residential.

ALGORITHMS

The annual energy savings for this measure are assumed to be dependent upon decreases in the overall heat transfer coefficient that are achieved by increasing the total R-value of the tank insulation.

$$= \frac{(U_{base}A_{base} - U_{insul}A_{insul}) \times (T_{setpoint} - T_{ambient})}{3412 \times \eta_{Elec}} \times HOU$$

$$\Delta kW_{peak} = \frac{\Delta kWh}{HOU} \times CF$$

DEFINITION OF TERMS

The U.S. Department of Energy recommends adding a water heater wrap of at least R-8 to any water heater with an existing R-value less than R-24. The default inputs for the savings algorithms are given in Table 2-61Table 2-61. Actual tank and blanket U-values can be used in the above algorithms as long as make/model numbers of the tank and blanket are recorded and tracked by the EDC.

Table 2-61: Water Heater Tank Wrap - Default Values

Component	Type <u>Unit</u>	Value	Source
R_{base} , R-value is a measure of resistance to heat flow and is equal to $1/U_{base}$	$\frac{Hr \cdot F \cdot ft^2}{Btu}$ Variable	Default: 8.33 or EDC Data Gathering	1

¹⁴⁹ DEER Version 2008.2.05, December 16, 2008.

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^{150 &}quot;Energy Savers", U.S. Department of Energy, accessed November, 2010 http://www.energysavers.gov/your_home/water_heating/index.cfm/mytopic=13070

Component	Type <u>Unit</u>	Value	Source
<i>R_{insul}</i> , R-value is a measure of resistance to heat flow and is equal to 1/U _{insul}	$\frac{Hr \cdot ^{\circ} \mathbf{F} \cdot ft^2}{Btu}$ Variable	Default: 20 or EDC Data Gathering	2
U_{base} , Overall heat transfer coefficient of water heater prior to adding tank wrap	$\frac{Btu}{Hr \cdot {}^{\circ}\text{F} \cdot ft^2}$	=1/R _{base}	
<i>U_{insul}</i> , Overall heat transfer coefficient of water heater after addition of tank wrap	$\frac{Btu}{Hr \cdot {}^{\circ}\text{F} \cdot ft^2}$	=1/R _{insul}	
A _{base} , Surface area of storage tank prior to adding tank wrap	ft^2	See <u>Table</u> 2-62 Table 2-62	
A _{insul} , Surface area of storage tank after addition of tank wrap	ft^2	See <u>Table</u> 2-62 Table 2-62	
$\eta_{\it Elec}$, Thermal efficiency of electric heater element	FixedNone	0.97	3
T _{setpoint} , Temperature of hot water in tank	Fixed_°F	123	5
T _{ambient} , Temperature of ambient air	Fixed_°E	70	5
HOU , Annual hours of use for water heater tank	Fixed Hours	8760	4
CF , Demand Coincidence Factor	Fixed%	100 <u>%</u>	4

Table 2-62: Deemed savings by water heater capacity

Capacity (gal)	R _{base}	R _{insul}	A _{base} (ft ²) ¹⁵¹	A _{insul} (ft ²) ¹⁵²	ΔkWh	ΔkW
30	8	16	19.16	20.94	143	0.0164
30	10	18	19.16	20.94	100	0.0114
30	12	20	19.16	20.94	73	0.0083
30	8	18	19.16	20.94	163	0.0186
30	10	20	19.16	20.94	115	0.0131
30	12	22	19.16	20.94	85	0.0097
40	8	16	23.18	25.31	174	0.0198
40	10	18	23.18	25.31	120	0.0137
40	12	20	23.18	25.31	88	0.0100
40	8	18	23.18	25.31	197	0.0225
40	10	20	23.18	25.31	139	0.0159
40	12	22	23.18	25.31	103	0.0118
50	8	16	24.99	27.06	190	0.0217
50	10	18	24.99	27.06	131	0.0150
50	12	20	24.99	27.06	97	0.0111
50	8	18	24.99	27.06	214	0.0245

¹⁵¹ Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage.

152 A_{insul} was calculated by assuming that the water heater wrap is a 2" thick fiberglass material.

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Capacity (gal)	R _{base}	R _{insul}	A _{base} (ft ²) ¹⁵¹	A _{insul} (ft ²) ¹⁵²	ΔkWh	ΔkW
50	10	20	24.99	27.06	152	0.0173
50	12	22	24.99	27.06	113	0.0129
80	8	16	31.84	34.14	244	0.0279
80	10	18	31.84	34.14	171	0.0195
80	12	20	31.84	34.14	125	0.0143
80	8	18	31.84	34.14	276	0.0315
80	10	20	31.84	34.14	195	0.0223
80	12	22	31.84	34.14	145	0.0166

MEASURE LIFE

The measure life is 7 years 153.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Results and Methodology of the Engineering Analysis for Residential Water Heater Efficiency Standards, PNNL, 1998.
- 2. The water heater wrap is assumed to be a fiberglass blanket with R-8, increasing the total to R-20.
- 3. New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs. October 15, 2010. Prepared by New York Advisory Contractor Team.
- 4. It is assumed that the tank wrap will insulate the tank during all hours of the year.
- 5. 2012 Residential SWE Baseline Study

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⁴⁵³ DEER Version 2008.2.05, December 16, 2008.

2.3.7 WATER HEATER TEMPERATURE SETBACK

Measure Name	Water Heater Temperature Setback
Target Sector	Residential Establishments
Measure Unit	Water Heater Temperature
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	4 years ¹⁵⁴
Vintage	Retrofit

In homes where the water heater setpoint temperature is set high, savings can be achieved by lowering the setpoint temperature. The recommended lower setpoint is 120°F, but EDCs may substitute another if needed. Savings occur only when the lower temperature of the hot water does not require the use of more hot water. Savings do not occur in applications such as a shower or faucet where the user adjusts the hot water flow to make up for the lower temperature. Clothes washer hot water use and water heater tank losses are included in the savings calculation, but shower, faucet, and dishwasher use are not included due to expected behavioral and automatic (dishwasher) adjustments in response to lower water temperature. It is expected that the net energy use for the dish washer hot water will remain the same after a temperature reduction because dishwashers will adjust hot water temperature to necessary levels using internal heating elements.

ELIGIBILITY

This protocol documents the energy savings attributed to reducing the electric or heat pump water heater temperature setpoint. The target sector primarily consists of single-family residences.

ALGORITHMS

The annual energy savings calculation utilizes average performance data for available residential water heaters and typical water usage for residential homes. The energy savings are obtained through the following formula, where the first term corresponds to tank loss savings and the second to clothes washer savings:

$$\begin{split} \Delta \, kWh/yr & = \frac{A_{tank} \times \left(T_{hot\,i} - T_{hot\,f}\right) \times 8760 \frac{hrs}{yr}}{R_{tank} \times \eta_{elec} \times 3412 \frac{Btu}{kWh}} \\ & + \frac{V_{HW} \times \left(8.3 \frac{lb}{gal}\right) \times \left(365 \frac{days}{yr}\right) \times \left(1 \frac{Btu}{F \cdot lb}\right) \times \left(T_{hot\,i} - T_{hot\,f}\right)}{\left(3412 \frac{Btu}{kWh}\right) \times EF_{WH}} \end{split}$$

Demand savings result from reduced hours of operation of the heating element, rather than a reduced connected load. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average <u>demandenergy usage during between-2 PM andto 6 PM on summer weekdays to the total annual energy usage.</u>

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¹⁵⁴ GDS Associates, Inc., Measure Life Report Prepared for The New England State program Working Group (SPWG), June 2007, http://library.cee1.org/sites/default/files/library/8842/CEE_Eval_MeasureLifeStudyLights%26HVACGDS_1Jun2007.pdf

 ΔkW_{peak}

= EnergyToDemandFactor $ETDF \times \Delta kWh/yr$ Energy Savings

The ETDF (Energy to Demand Factor) is defined below:

EnergyToDemandFactorETDF-

Average DemandUsage Summer WD 2PM- 6 PM Annual Energy Usage

The ratio of the average <u>demandenergy usage during between 2 PM and</u>to 6 PM on summer weekdays to the total annual energy usage is taken from <u>an electric water heater metering study performed by BG&E (pg 95 of Source 2).</u>

load shape data collected for a water heater and HVAC demand response study for PJM155. The factor is constructed as follows:

- 1) Obtain the average kW, as monitored for 82 water heaters in PJM territory156, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, 183 spring/fall days) to obtain annual energy usage.
- 2) Obtain the average kW during 2 PM to 6 PM on summer days from the same data.
- 3) The average 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study157.
- 4) The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number 0.00008294, is the EnergyToDemandFactor.

The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted in Figure 2-11 below.

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⁴⁵⁵_Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. Thereport can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashxhttp://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx

¹⁶⁶The average is over all 82 water heaters and over all summer, spring/fall, or winter days. The load shapes are taken from the fourth columns, labeled "Mean", in tables 14,15, and 16 in pages 5 31 and 5 32

¹⁵⁷.The 5th column, labeled "Mean" of Table 18 in page 5-34 is used to derive an adjustment factor that scales average summerusage to summer weekday usage. The conversion factor is 0.925844. A number smaller than one indicates that for residentialhomes, the hot water usage from noon to 8 PM is slightly higher is the weekends than on weekdays.

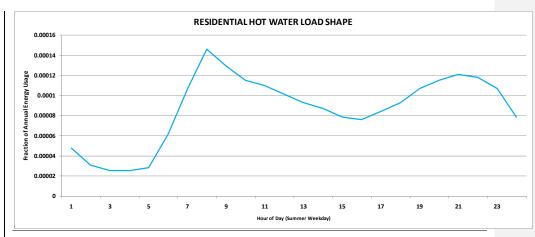


Figure 2-11: Load shapes for hot water in residential buildings taken from a PJM study.

DEFINITION OF TERMS

SECTION 2: Residential Measures

The parameters in the above equation are listed in Table 2-63 below.

Table 2-63: Water Heater Temperature Setback Assumptions

Component	Type <u>Unit</u>	Values	Source
$\mathit{EF}_{\mathit{WH}}$, Energy Factor of water heater	Variable	EDC data collection Default: Electric Storage= 0.904 HPWH= 2.0	1
R_{tank} , R value of water heater tank, $\frac{hr \cdot r \cdot f \cdot f^2}{Btu}$	$\frac{hr.{}^{\circ}\text{F} \cdot ft^2}{Btu}$ Variable	EDC Data Gathering Default: 8.33 ¹⁵⁸	
A_{tank} , Surface Area of water heater tank, ft^2	ft ² Variable	EDC Data Gathering Default: 24.99	50 gal. value in Table 2-119Table 2-119
$\eta_{\it elec^{-1}}$ Thermal efficiency of electric heater element (equiv. to COP for HPWH)	FixedFraction	Electric Storage: 0.97 HPWH: 2.1	2, 3
V_{HW} , Volume of hot water used per day, in gallons	Gallons Fixed	7.32 gallons/day	4, 5, 6, 7, 8
$T_{hot_i.}$, Temperature setpoint of water heater initially	<u>°F Variable</u>	EDC Data Gathering Default: 130	9
T _{hot_f.} , Temperature setpoint water heater after setback	<u>°F_</u> Variable_	EDC data collection Default: 123	10
ETDF , Energy To Demand Factor (defined above)	Fixed $\frac{kW}{kWh/yr}$	0.000082940.00008047	11-13

Note: The new Federal standards that go into effect 4/16/2015 will be incorporated into this measure in the 2016 TRM. These can be viewed at: http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/27

DEEMED SAVINGS

The energy savings and demand reductions are prescriptive according to the above formulae. However, some values for common configurations are provided in

Table 2-64

Table 2-64 below.

Table 2-64: Energy Savings and Demand Reductions

Туре	Tank Size (gallons)	R _{tank}	A _{tank}	$T_{ ext{hot }i} - T_{ ext{hot }f}$ (°F)	$\eta_{ m elec}$	EF _{WH}	Energy Savings (∆ kWh/yr)	Demand Reduction (ΔKW _{peak})
Electric Storage	50	8.33	24.99	10	0.97	0.904	151	0.0125
Electric Storage	50	8.33	24.99	5	0.97	0.904	76	0.0063
HPWH	50	8.33	24.99	10	2.1	2.0	69	0.0057
HPWH	50	8.33	24.99	5	2.1	2.0	35	0.0029

158 Results and Methodology of the Engineering Analysis for Residential Water Heater Efficiency Standards, PNNL, 1998

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MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, an electric water heater's lifespan is 4 years159

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of water heater temperature setpoint coupled with assignment of stipulated energy savings.

SOURCES

- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is 0.904. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30
- New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs. October 15, 2010. Prepared by New York Advisory Contractor Team.
- 3. NEEA Heat Pump Water Heater Field Study Report. Prepared by Fluid Market Strategies. October 22, 2013. http://neea.org/docs/default-source/reports/heat-pump-water-heater-field-study-report.pdf?sfvrsn=5
- 4. Daily Usage based on AWWA Research Foundation, 1998, Residential End Uses of Water, found in EPA's Water Sense guide: http://www.epa.gov/WaterSense/docs/home_suppstat508.pdf Clothes washer hot water use per capita per day adjusted for current water use per load and using PA Census Data. Hot water comprises 28% of total water in clothes washer load. Federal minimum Water Factor standards (9.5) and Energy Star minimum Water Factor standards (6.0) for clothes washers, Section 2.26, "Energy Star Clothes Washers".
- Average capacity of base (3.19 cu. ft.) and energy efficient (3.64 cu. ft.) clothes washers, Table 2-112Table 2-112, Section 2.26.
- Households with Energy Star Clothes Washers 2009 (36%), "Energy Star Product Retrospective: Clothes Washers", 2012. Used to determine current weighted average gallons per load (27.3 gal)
- 7. 2007-2011 U.S. Census Data for Pennsylvania (2.47 persons per household average).
- 8. Engineering assumption
- 9. Pennsylvania Statewide Residential End-Use and Saturation Study, 2012, page 42.9.
- Deemed Savings Estimates for Legacy Air Conditioning and Water Heating Direct Load Control Programs in PJM Region. The report can be accessed online: http://www.pjm.com/~/media/committees-groups/working-groups/lrwg/20070301/20070301-pjm-deemed-savings-report.ashx
- Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal, Aug/Sept. 2011.

SECTION 2: Residential Measures

⁴⁵⁹ GDS Associates, Inc., Measure Life Report Prepared for The New England State program Working Group (SPWG), June 2007

2.3.8 **WATER HEATER PIPE INSULATION**

Measure Name	Electric Water Heater Pipe Insulation
Target Sector	Residential Establishments
Measure Unit	Water Heater
Unit Energy Savings	Default: 10 kWh per foot of installed insulation
Unit Peak Demand Reduction	0.00083 kW per foot of installed insulation
Measure Life	13 years ¹⁶⁰
Vintage	Retrofit

This measure relates to the installation of foam insulation on 10 feet of exposed pipe in unconditioned space, 3/4" thick. The baseline for this measure is a standard efficiency electric water heater (EF=0.904) with an annual energy usage of 3.338 kWh. 161

ELIGIBILITY

This protocol documents the energy savings for an electric water heater attributable to insulating 10 feet of exposed pipe in unconditioned space, 3/4" thick. The target sector primarily consists of residential establishmentsresidences.

ALGORITHMS

The annual energy savings are assumed to be 3% of the annual energy use of an electric water heater (3,338 kWh), or 100.14 kWh based on 10 feet of insulation. This estimate is based on a recent report prepared by the ACEEE for the State of Pennsylvania (Source 1). 462 On a per foot basis, this is equivalent to 10 kWh.

∆kWh/yr = 10 kWh/yr per foot of installed insulation The summer coincident peak kW savings are calculated as follows: ΔkW_{peak} = $\Delta kWh \times * EnergyToDemandFactorETDF$

DEFINITION OF TERMS

Term	Unit	Value	Source
$\Delta kWh/yr$, annual energy savings per foot of installed pipe insulation	$\frac{kWh/yr}{ft}$	10	1

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Domestic Hot Water

 ¹⁶⁰ Efficiency Vermont, Technical Reference User Manual: Measure Savings Algorithms and Cost Assumptions, TRM User Manual
 No. 2008-53, 07/18/08, http://www.veic.org/docs/ResourceLibrary/TRM-User-Manual-Excerpts.pdf.
 161 See "Efficient Electric Water Heater" section Section 2.3 for assumptions used to calculate annual energy usage.

¹⁶² American Council for an Energy-Efficient Economy, Summit Blue Consulting, Vermont Energy Investment Corporation, ICF-International, and Synapse Energy Economics, Potential for Energy Efficiency, Demand Response, and Onsite Solar Energy in Pennsylvania, Report Number E093, April 2009, p. 117.

Term	Unit	Value	Source
ETDF, Energy to Demand Factor	kW kWh/yr	0.00008047	2
ΔkW _{peak} , Summer peak kW savings per foot of installed pipe insulation	$\frac{kW}{ft}$	0.0008047	

The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage during 2 PM to 6 PM on summer weekdays to the total annual energy usage. The Energy to Demand Factor is defined as:

EnergyToDemandFactor ETDF = Average DemandUsage Summer WD 2PM-6PM
Annual Energy Usage

The ratio of the average energy usage <u>betweenduring 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from an electric water heater metering study performed by BG&E (pg 95 of Source 2) from load shape data collected for a water heater and HVAC demand response study for PJM¹⁶³. The factor is constructed as follows:</u>

Obtain the average kW, as monitored for 82 water heaters in PJM territory, for each hour of the typical day summer, winter, and spring/fall days. Weight the results (91 summer days, 91 winter days, and 183 spring/fall days) to obtain annual energy usage.

Obtain the average kW during 2 PM to 6 PM on summer days from the same data.

The average 2 PM to 6 PM demand is converted to average weekday 2 PM to 6 PM demand through comparison of weekday and weekend monitored loads from the same PJM study,

The ratio of the average weekday 2 PM to 6 PM energy demand to the annual energy usage obtained in step 1. The resulting number, 0.00008294, is the Energy to Demand Factor, or Coincidence Factor.

The load shapes (fractions of annual energy usage that occur within each hour) during summer week days are plotted in Figure 2-7.

163 Op. cit.

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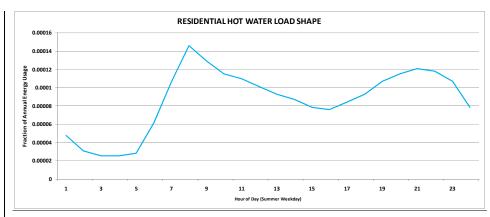


Figure 2-7: Load shapes for hot water in residential buildings taken from a PJM study.

MEASURE LIFE

According to the Efficiency Vermont Technical Reference User Manual (TRM), the expected measure life is 13 years164.

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

- American Council for an Energy-Efficient Economy, Summit Blue Consulting, Vermont Energy Investment Corporation, ICF International, and Synapse Energy Economics, Potential for Energy Efficiency, Demand Response, and Onsite Solar Energy in Pennsylvania, Report Number E093, April 2009, p. 117.
- Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011.

SECTION 2: Residential Measures

¹⁶⁴ Efficiency Vermont, Technical Reference User Manual: Measure Savings Algorithms and Cost Assumptions, TRM User Manual-No. 2008-53, 07/18/08.

2.3.9 Low Flow Faucet Aerators

Measure Name	Low Flow Faucet Aerators
Target Sector	Residential Establishments
Measure Unit	Aerator
Unit Energy Savings	Varies by installation location
Unit Peak Demand Reduction	Varies by installation location
Measure Life	12 years ¹⁶⁵
Vintage	Retrofit

Installation of low-flow faucet aerators is an inexpensive and lasting approach for water conservation. These efficient aerators reduce water consumption and consequently reduce hot water usage and save energy associated with heating the water. This protocol presents the assumptions, analysis and savings from replacing standard flow aerators with low-flow aerators in kitchens and bathrooms.

The low-flow kitchen and bathroom aerators will save on the electric energy usage due to the reduced demand of hot water. The maximum flow rate of qualifying kitchen and bathroom aerators is 1.5 gallons per minute.

ELIGIBILITY

This protocol documents the energy savings attributable to efficient low flow aerators in residential applications. The savings claimed for this measure are attainable in homes with standard resistive water heaters. Homes with non-electric water heaters do not qualify for this measure.

ALGORITHMS

The energy savings and demand reduction are obtained through the following calculations:

```
\frac{AkWh}{Day^{\times}N_{Persons} \times 365 \times (T_{out} - T_{in}) \times U_{H} \times U_{E} \times DF/RE] / (F/home)}{\Delta kWh/yr}
= ISR \times ELEC
\times \begin{bmatrix} (GPM_{base} - GPM_{low}) \times T_{person/day} \times N_{persons} \times 365 \frac{days}{yr} \times DF \times (T_{out} - T_{in}) \times 8.3 \frac{Btu}{gal \cdot F} \\ \#_{faucets} \times 3413 \frac{Btu}{kWh} \times RE \end{bmatrix}
\frac{AkW_{peak}}{\Delta kW_{peak}} = \frac{AkWh}{V} \times F_{ED}
\Delta kW_{peak} = \Delta kWh/yr \times ETDF
```

Where:

ETDF
$$= \frac{CF}{HOU}$$
= [% faucet use during peak 166 \times (T_{Person-Day} \times N_{Person}) /(F/home)] / 240 (minutes in peak period)

165 California's Database of Energy Efficiency Resources (DEER).

SECTION 2: Residential Measures

$$\begin{split} \mathit{CF} & = \frac{\%_{\mathit{faucet use,peak}} \times T_{\mathit{person/day}} \times N_{\mathit{persons}}}{\#_{\mathit{faucets}} \times 240 \frac{\mathit{minutes}}{\mathit{daily peak}}} \\ \\ \mathit{HOU} & = \frac{T_{\mathit{person/day}} \times N_{\mathit{persons}} \times 365 \frac{\mathit{days}}{\mathit{yr}}}{\#_{\mathit{faucets}} \times 60 \frac{\mathit{minutes}}{\mathit{hour}}} \end{split}$$

The ratio of the average energy usage during 2 PM and 6 PM on summer weekdays to the total annual energy usage is taken from average daily load shape data collected for faucets from an Aquacraft, Inc study. 167 The average daily load shapes (percentages of daily energy usage that occur within each hour) are plotted in Figure 2-1 below (symbol FAU represents faucets).

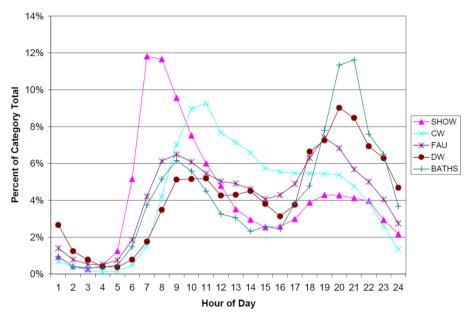


Figure 2-1: Daily Load Shapes for Hot Water Measurers

DEFINITION OF TERMS

The parameters in the above equation are defined in Table 2-65.

SECTION 2: Residential Measures

¹⁶⁶ Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. The percentage of faucet use that takes place during peak hours is determined to be 18.9%. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf

¹⁶⁷ Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf.

Table 2-65: Low Flow Faucet Aerator Calculation Assumptions

<u>Term</u> Description	<u>Unit</u>	Value	Source
GPM_{base} , Average baseline flow rate of aerator (GPM)	Variable gallons minute	Default =2.2 Or EDC Data Gathering	1
$\textit{GPM}_{\textit{low}}$, Average post measure flow rate of aerator (GPM)	w rate of $\frac{\text{Variable}}{\underset{minute}{gallons}}$ Default = 1.5 Or EDC Data Gathering		1
T _{Person-Day} , Average time of hot water usage per person per day (minutes)	Fixed minutes day	Kitchen=4.5 Bathroom=1.6 Unknown=6.1	2
N _{Persons} , Average number of persons per household	Variable persons house	Default SF=2.7 Default MF=1.8 Default Unknown=2.6 Or EDC Data Gathering	3
$\textit{T}_{\textit{out}}$, Average mixed water temperature flowing from the faucet (°F)	Fixed <u>°F</u>	Kitchen=93 Bathroom=86 Unknown= 87.8	4
T_{in} , Average temperature of water entering the house (°F)	Fixed <u>°F</u>	55	5, 6
Unit Conversion: 8.33BTU/(Gallons-°F)	Fixed	8.33	Conventio n
Unit Conversion: 1 kWh/3413 BTU	Fixed	1/3413	Conventio n
RE , Recovery efficiency of electric water heater	FixedFra ction	0.98	7
ETDFF _{ED} -, Energy To Demand Factor	$\frac{\text{Fixed}}{kW} \frac{kW}{kWh/yr}$	0.0001340.00012913	8
Number of days per year	Fixed	365	
# _{faucets} F/home_, Average number of faucets in the home	Fixed faucets house	SF: Kitchen=1.0 Bathroom=3.0 Unknown=4.0 MF: Kitchen=1.0 Bathroom=1.7 Unknown=2.7 Unknown Home Type: Kitchen=1.0 Bathroom=2.8 Unknown=3.8 Or EDC Data Gathering	9
DF , Percentage of water flowing down drain	Fixed%	Kitchen=75% Bathroom=90%	10

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<u>Term</u> Description	<u>Unit</u>	Value	Source
		Unknown=79.5%	
ISR , In Service Rate	Variable <u>%</u>	Variable	EDC Data Gathering
ELEC , Percentage of homes with electric water heat	Variable <u>%</u>	Default=43% Or EDC Data Gathering	11
% _{faucet use,peak} , percentage of daily faucet use during PJM peak period	<u>%</u>	19.5%	8

Sources:

For example, a direct installed (ISR=1) kitchen low flow faucet aerator in a single family electric DHW home:

For example, a direct installed (ISR=1) low flow faucet aerator in unknown faucet in an unknown family type electric DHW home:

$$\Delta kWh = 1.0 * 1.0 * (((2.2 - 1.5) * 6.1 * 2.6 * 365 * (87.8 - 55) * 8.33 * (1/3413) * 0.795 / 0.98) / 4.0) = 65.8 kWh per faucet$$

DEFAULT SAVINGS

Housing Type	Faucet Location	Unit Energy Savings (kWh)	Unit Demand Savings (kW)
	Kitchen	220.3	0.0284 0.02952
Single Family	Bathroom	25.6	0.0033 0.00343
	Unknown	68.3	0.0088 0.009152
	Kitchen	146.9	0.0190 <u>0.01969</u>
Multifamily	Bathroom	30.1	0.0039 <u>0.004033</u>
	Unknown	67.5	0.0087 0.009045
Statewide	Kitchen	212.2	0.02740.028435
(Unknown	Bathroom	26.4	0.00340.003538
Housing Type)	Unknown	65.8	0.00850.008817

2.8.5 MEASURE LIFE

The measure life is 12 years, according to California's Database of Energy Efficiency Resources (DEER).

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

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SOURCES

- 1. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Baseline GPM of replaced aerators is set to the federal minimum GPM of 2.2. The GPM of new aerators is set to the typical rated GPM value of 1.5 GPM. Discounted GPM flow rates were not applied because the "throttle factor" adjustment was found to have been already accounted for in the mixed water temperature variable. Additionally, the GPM_{Base} was set to a default value of 2.2 due to the inability to verify what the GPM flow rate was of the replaced faucet.
- Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. If aerator location is known, use the corresponding kitchen/bathroom value. If unknown, use 6.1 min/person/day as the average length of use value, which is the total for the household: kitchen (4.5 min/person/day) + bathroom (1.6 min/person/day) = 6.1 min/person/day.
- 3. Table 4-8, section 4.2.4. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- 4. Table 7. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. The study finds that the average mixed water temperature flowing from the kitchen and bathroom faucets is 93°F and 86°F, respectively. If the faucet location is unknown, 87.8°F is the corresponding value to be used, which was calculated by taking a weighted average of faucet type (using the statewide values): ((1*93)+(3*86))/(1+3) = 87.8.
- Table 9. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Inlet water temperatures were measured and a weighted average based upon city populations was used to calculate the value of 55°F.
- 6. A good approximation of annual average water main temperature is the average annual ambient air temperature. Average water main temperature = 55° F based on: http://lwf.ncdc.noaa.gov/img/documentlibrary/clim81supp3/tempnormal_hires.jpg
- 7. Mid Atlantic TRM Version 2.0 (updated July 2011) and Ohio TRM updated August 2010.
- Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf. The statewide values were used for inputs in the F_{ED} algorithm components. The CF for faucets is found to be 0.003280.00339: [% faucet use during peak × (T_{Person-Day}× N_{Person}) /(F/home)] / 240 (minutes in peak period) = [19.518.9% × (6.1 x 2.6 / 3.8)] / 240 = 0.003280.00339. The Hours for faucets is found to be 25.4: (T_{Person-Day}× N_{Person}× 365) /(F/home) / 60 = (6.1 x 2.6 x 365) / 3.8 / 60 = 25.4. The resulting F_{ED} is calculated to be-0.0001340.00012913: CF / Hours = 0.00328 / 25.4 = 0.000129130.000134.
- 9. Table 4-68, section 4.6.3. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- 10. Illinois TRM Effective June 1, 2013. Faucet usages are at times dictated by volume, only "directly down the drain" usage will provide savings. Due to the lack of a metering study that has determined this specific factor, the Illinois Technical Advisory Group has deemed these values to be 75% for the kitchen and 90% for the bathroom. If the aerator location is unknown an average of 79.5% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*0.75)+(0.3*0.9)=0.795.

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11. Table 4-61, Section 4.6.1 GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission. This study finds that only 43% of households statewide have an electric water heater. As such, if the proportion of households with electric water heaters is unknown, deemed savings should only be applied to 43% of the study group.

2.3.10 Low Flow Showerheads

Measure Name	Low Flow Showerheads
Target Sector	Residential Establishments
Measure Unit	Water Heater
Unit Energy Savings	Partially Deemed
Unit Peak Demand Reduction	Partially Deemed
Measure Life	9 years ¹⁶⁸
<u>Vintage</u>	Retrofit

This measure relates to the installation of a low flow (generally 1.5 GPM) showerhead in bathrooms in homes with electric water heater. The baseline is a standard showerhead using 2.5 GPM

ELIGIBILITY

This protocol documents the energy savings attributable to replacing a standard showerhead with an energy efficient low flow showerhead for electric water heaters. The target sector primarily consists of residential residences establishments.

ALGORITHMS

The annual energy savings are obtained through the following formula:

```
∆kWh
                                                  = ISR \times ELEC \times [(GPMBase - GPMLow) \times TPerson - Day \times
                                                 NPersons \times NShowers – Day \times 365 \times (Tout – Tin) \times
                                                 UH \times UE / RE / (S/home)
\Delta kWh/yr
= ISR \times ELEC
   \left[\frac{(GPM_{base}-GPM_{low})\times T_{person/day}\times N_{persons}\times N_{showers/day}\times 365\frac{days}{yr}\times (T_{out}-T_{in})\times 8.3\frac{Btu}{gal\cdot °F}}{\#_{showers}\times 3413\frac{Btu}{kWh}\times RE}\right]
<u>∆kW<sub>peak</sub></u>
                                                  = AkWh × F<sub>ED</sub>
\Delta kW_{peak}
                                    = \Delta kWh/yr \times ETDF
Where:
                                          CF
ETDF
                                        HOU
                                                 = [% showerhead use during peak<sup>169</sup>-× (T<sub>Person-Day</sub>: N<sub>Person-</sub>×
CF-
                                                 N<sub>Showers Day</sub> /(S/home)] / 240 (minutes in peak period)
                                        \%_{shower use,peak} \times T_{person/day} \times N_{persons} \times N_{showers/day}
                        CF
```

 $\#_{\text{showers}} \times 240 \frac{\text{minutes}}{\text{daily peak}}$

http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf

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¹⁶⁸ Efficiency Vermont, Technical Reference User Manual: Measure Savings Algorithms and Cost Assumptions, TRM User Manual No. 2008-53, 07/18/08, http://www.veic.org/docs/ResourceLibrary/TRM-User-Manual-Excerpts.pdf.

¹⁶⁹ Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis.-2001. The percentage of showerhead use that takes place during peak hours is determined to be 11.7%. <a href="http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-

$$HOU = \frac{T_{\text{person/day}} \times N_{\text{persons}} \times N_{\text{showers/day}} \times 365 \frac{days}{yr}}{\#_{\text{showers}} \times 60 \frac{minutes}{hour}}$$

The ratio of the average energy usage during 2 PM and 6 PM on summer weekdays to the total annual energy usage is taken from average daily load shape data collected for showerheads from an Aquacraft, Inc study.¹⁷⁰ The average daily load shapes (percentages of daily energy usage that occur within each hour) during are plotted in Figure 2-2 below (symbol SHOW represents showerheads).

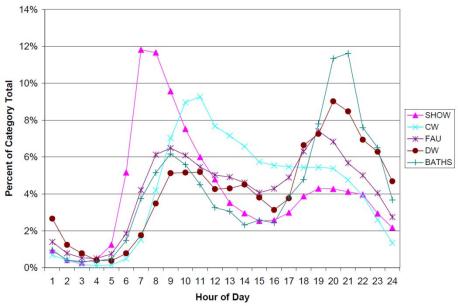


Figure 2-2: Daily Load Shapes for Hot Water Measures

DEFINITION OF TERMS

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¹⁷⁰ Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf.

Table 2-66: Low Flow Showerhead Calculation Assumptions

<u>Term</u> Description	Type <u>Unit</u>	Value	Source
<i>GPM</i> _{base} , Gallons per minute of baseline showerhead	Fixed gallons minute	Default value = 2.5	1
<i>GPM</i> _{low} , Gallons per minute of low flow showerhead	Variable gallons minute	Default value = 1.5 or EDC Data Gathering	2
$T_{person/day}$, Average time of shower usage per person (minutes)	Fixed minutes day	7.8	3
$N_{\it persons}$, Average number of persons per household	Variable persons house	Default SF=2.7 Default MF=1.8 Default unknown=2.6 Or EDC Data Gathering	4
$N_{ m showers/day}$, Average number of showers per person per day	Fixed showers/per day	0.6	5
Number of days per year	Fixed	365	
$\#_{showers}$, Average number of showers in the home	Variable showers house	Or EDC Data Gathering Default SF=1.7 (89.6% of homes) Default MF=1.3 (10.4% of homes) Default unknown = 1.6	6
T_{out} , Assumed temperature of water used by showerhead	Fixed° F	101° - F	7
T _{in} , Assumed temperature of water entering house	Fixed° F	55 <u>°</u> F	7,8
Unit Conversion: 8.33BTU/(Gallons-°F)	Fixed	8.33	Convention
Unit Conversion: 1 kWh/3413 BTU	Fixed	1/3413	Convention n
RE , Recovery efficiency of electric water heater	FixedFracti on	0.98	9
F _{ED} - <u>ETDF</u> , Energy To Demand Factor	Fixed $\frac{kW}{kWh/yr}$	0.00008013	10
ISR , In Service Rate	Variable %	Variable	EDC Data Gathering
<u>ELEC</u> .Percentage of homes with electric water heat	Variable %	Default=43% Or EDC Data Gathering	11
$\%_{shower\ use,peak}$, percentage of daily shower use during PJM peak period	<u>%</u>	11.7%	<u>10</u>

For example, a direct-installed (ISR=1) 1.5 GPM low flow showerhead in a single family electric DHW home where the number of showers is not known:

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For example, a direct-installed (*ISR=1*) 1.5 GPM low flow showerhead in an unknown family type home with electric DHW where the number of showers is not known:

DEFAULT SAVINGS

Housing Type	Low Flow Rate (gpm)	Unit Energy Savings (kWh)	Unit Demand Savings (kW)
	2.0	155	0.0124
Single Family	1.75	233	0.0187
	1.5	311	0.0249
	2.0	135	0.0108
Multifamily	1.75	203	0.0163
	1.5	271	0.0217
Statewide	2.0	159	0.0127
(Unknown	1.75	239	0.0192
Housing Type)	1.5	318	0.0255

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

SOURCES

- Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Uses the federal minimum GPM allowed as the baseline for the replaced showerheads, corresponding to 2.5 GPM.
- 2. Illinois TRM Effective June 1, 2013. Allows for varying flow rate of the low-flow showerhead, most notably values of 2.0 GPM, 1.75 GPM and 1.5 GPM. Custom or actual values are also allowed for.
- 3. Table 6. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. The study compared shower length by single-family and multifamily populations, finding no statistical difference in showering times. For the energy-saving analysis, the study used the combined single-family and multifamily average shower length of 7.8 minutes.
- 4. Table 4-8, section 4.2.4. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- 5. Table 8. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. For each shower fixture metered, the evaluation team knew the total number of showers taken, duration of time meters remained in each home, and total occupants reported to live in

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the home. From these values average showers taken per day, per person was calculated. The study compared showers per day, per person by single-family and multifamily populations, finding no statistical difference in the values. For the energy-saving analysis, the study used the combined single-family and multifamily average showers per day, per person of 0.6.

- 6. Table 4-69, section 4.6.3. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- 7. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Temperature sensors provided the mixed water temperature readings resulting in an average of 101°F. Inlet water temperatures were measured and a weighted average based upon city populations was used to calculate the value of 55°F.
- 8. A good approximation of annual average water main temperature is the average annual ambient air temperature. Average water main temperature = 55° F based on: http://lwf.ncdc.noaa.gov/img/documentlibrary/clim81supp3/tempnormal hires.jpg
- 9. Mid Atlantic TRM Version 2.0 (updated July 2011) and Ohio TRM updated August 2010.
- 10. Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow analysis. 2001 trace http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf. The statewide values were used for inputs in the F_{FD} algorithm components. The CF for showerheads is found to be 0.00371: [% showerhead use during peak × (T_{Person-Day}× N_{Person}) /(S/home)] / 240 (minutes in peak period) = $[11.7\% \times (7.8 \times 2.6 \times 0.6 / 1.6)] / 240 = 0.00371$. The Hours for showerheads is found to be 46.3: $(T_{Person-Day} \times N_{Persons} \times 365) / (S/home) / 60 = (7.8 \times 2.6)$ \times 0.6 \times 365) / 1.6 / 60 = 46.3. The resulting F_{ED} is calculated to be 0.00008013: CF / Hours = 0.00371 / 46.3 = 0.00008013.
- 11. Table 4-61, Section 4.6.1 of the 2012 Pennsylvania Statewide Residential End-Use and Saturation Study. This study finds that only 43% of households statewide have an electric water heater. As such, if the proportion of households with electric water heaters is unknown, deemed savings should only be applied to 43% of the study group.

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2.3.11 THERMOSTATIC SHOWER RESTRICTION VALVE

Measure Name	Thermostatic Shower Restriction Valve	
Target Sector	Residential Establishments	
Measure Unit	Water Heater	
Unit Energy Savings	Partially Deemed	
Unit Peak Demand Reduction	Partially Deemed	
Measure Life	10 years ¹⁷¹	

This measure relates to the installation of a device that reduces hot water usage during shower warm-up by way of a thermostatic shower restriction valve, reducing hot water waste during shower warm-up.

ELIGIBILITY

This protocol documents the energy savings attributable to installing a thermostatic restriction valve, device, or equivalent product on an existing showerhead. Only homes with electric water heaters are eligible, and the savings associated with this measure may be combined with a low flow showerhead. The target sector primarily consists of residences.

ALGORITHMS

 ΔkW_{peak}

The annual energy savings are obtained through the following formula:

 $=\Delta kWh \times ETDF$

$$\Delta kWh/yr \\ = \frac{ISR \times ELEC \times GPM_{base}}{60 \frac{sec}{mln}} \times UH \times UE \times (T_{out} - T_{in}) \times \frac{\left(N_{persons} - N_{showers-day}\right)}{S/_{home}} \\ \times \frac{BehavioralWasteSeconds}{RE} \times 365 \frac{days}{yr}$$

The ratio of the average energy usage during 2 PM and 6 PM on summer weekdays to the total annual energy usage is taken from average daily load shape data collected for showerheads from an Aquacraft, Inc study. 172 The average daily load shapes (percentages of daily energy usage that occur within each hour) during are plotted in Figure 2-3Figure 2-3 below (symbol SHOW represents showerheads).

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 $^{^{\}rm 171}$ Alignment with New York and Michigan TRM.

¹⁷² Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf.

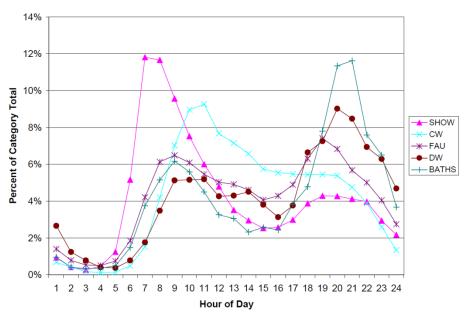


Figure 2-3: Daily Load Shapes for Hot Water Measures

DEFINITION OF TERMS

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Table 2-67: Assumptions for Thermostatic Shower Restriction Valve

Parameter	Unit	Value	Source
GPM _{Base} , Gallons per minute of baseline showerhead	gallons min	Default value = 2.5 or EDC Data Gathering	1
T _{Person-Day} , Average length of shower	Fixed	7.8	2
N _{persons} , Average number of persons per household	persons household	Default SF=2.6 Default MF=1.9 Default unknown=2.4 Or EDC Data Gathering	3
N _{Showers-Day} , Average number of showers per person per day	showers day	0.6	4
days/year	$\frac{days}{yr}$	365	
S/home, Average number of showerhead fixtures in the home	None	Default SF=1.3 (76.3% of homes) Default MF=1.1 (18.2% of homes) Default unknown = 1.2 Or EDC Data Gathering	5
T _{out} , Assumed temperature of water used by showerhead	°F	101	6
T _{in} , Assumed temperature of water entering house	°F	55	7,8
U _H , Unit Conversion	Fixed	8.33	Convention
U _E , Unit Conversion	Fixed	1/3413	Convention
RE, Recovery efficiency of electric water heater	Fixed	0.98	8
ETDF, Energy To Demand Factor	kW kWh/yr	0.00008013	9
ISR, In Service Rate	%	Variable	EDC Data Gathering
ELEC, Percentage of homes with electric water heat	%	Default=43% Or EDC Data Gathering	10
BehavioralWasteSeconds, Time	sec	Default = 55 or EDC Data Gathering	11

DEFAULT SAVINGS

Table 2-68: Restriction Valve Calculation Assumptions

Application	Baseline Flowrate (GPM)	Energy Savings (kWh/yr)	Peak Demand Reduction (kW)	Therm Savings
	2.5	39.1	0.0031	4
Single Family	2	31.3	0.0025	3.2
	1.5	23.5	0.0019	2.4
	2.5	34.1	0.0027	3.5
Multifamily	2	27.3	0.0022	2.8
	1.5	20.5	0.0016	2.1
/5 (); ; ;	2.5	40.0	0.0032	4.1
Unknown / Default Housing Type	2	32.0	0.0026	3.3
Турс	1.5	24.0	0.0019	2.5

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

SOURCES

- Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Uses the federal minimum GPM allowed as the baseline for the replaced showerheads, corresponding to 2.5 GPM.
- 2. Table 6. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. The study compared shower length by single-family and multifamily populations, finding no statistical difference in showering times. For the energy-saving analysis, the study used the combined single-family and multifamily average shower length of 7.8 minutes.
- Table 4-8, section 4.2.4. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- 4. Table 8. Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. For each shower fixture metered, the evaluation team knew the total number of showers taken, duration of time meters remained in each home, and total occupants reported to live in the home. From these values average showers taken per day, per person was calculated. The study compared showers per day, per person by single-family and multifamily populations, finding no statistical difference in the values. For the energy-saving analysis, the study used the combined single-family and multifamily average showers per day, per person of 0.6.
- 5. Table 4-69, section 4.6.3. GDS Associates, Inc. Pennsylvania Statewide Residential End-Use Saturation Study, 2012. For The Pennsylvania Public Utility Commission.
- Cadmus and Opinion Dynamics Evaluation Team. Showerhead and Faucet Aerator Meter Study. For Michigan Evaluation Working Group. June 2013. Temperature sensors provided the mixed water temperature readings resulting in an average of 101°F. Inlet

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water temperatures were measured and a weighted average based upon city populations was used to calculate the value of 55°F.

- 7. A good approximation of annual average water main temperature is the average annual ambient air temperature. Average water main temperature = 55° F based on: http://lwf.ncdc.noaa.gov/img/documentlibrary/clim81supp3/tempnormal_hires.jpg
- 8. Mid Atlantic TRM Version 2.0 (updated July 2011) and Ohio TRM updated August 2010.
- 9. Aquacraft, Inc., Water Engineering and Management. The end use of hot water in single family homes from flow trace analysis. 2001. http://www.aquacraft.com/sites/default/files/pub/DeOreo-(2001)-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf. The statewide values were used for inputs in the F_{ED} algorithm components. The CF for showerheads is found to be 0.00371: [% showerhead use during peak × (T_{Person-Day}× N_{Person} / (S/home)] / 240 (minutes in peak period) = [11.7% × (7.8 x 2.6 x 0.6 / 1.6)] / 240 = 0.00371. The Hours for showerheads is found to be 46.3: (T_{Person-Day}× N_{Persons}× 365) /(S/home) / 60 = (7.8 x 2.6 x 0.6 x 365) / 1.6 / 60 = 46.3. The resulting F_{ED} is calculated to be 0.00008013: CF / Hours = 0.00371 / 46.3 = 0.00008013.
- 10. Table 4-61, Section 4.6.1 of the 2012 Pennsylvania Statewide Residential End-Use and Saturation Study. This study finds that only 43% of households statewide have an electric water heater. As such, if the proportion of households with electric water heaters is unknown, deemed savings should only be applied to 43% of the study group.
- Estimate based on ShowerStart™Pilot Project White Paper 2008, City of San Diego and the Pennsylvania Power and Electric Pilot Study, 2014.

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2.4 APPLIANCES

2.4.1 ENERGY STAR REFRIGERATORS

Measure Name	Refrigerators
Target Sector	Residential Establishments
Measure Unit	Refrigerator
Unit Energy Savings	Varies by Configuration
Unit Peak Demand Reduction	Varies by Configuration
Measure Life	12 years ¹⁷³
Vintage	Replace on Burnout

ELIGIBILITY

This measure is for the purchase and installation of a new refrigerator meeting ENERGY STAR or ENERGY STAR Most Efficient criteria. An ENERGY STAR refrigerator must be at least 20 percent more efficient than the minimum federal government standard. The ENERGY STAR Most Efficient is a new certification that identifies the most efficient products among those that qualify for ENERGY STAR. ENERGY STAR Most Efficient refrigerators must be at least 30 percent more efficient than the minimum federal standard.

ALGORITHMS

The general form of the equation for the ENERGY STAR Refrigerator measure savings algorithm is:

Total Savings = Number of Refrigerators \times Savings per Refrigerator

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of refrigerators. The number of refrigerators will be determined using market assessments and market tracking.

If the volume and configuration of the refrigerator is known, the baseline model's annual energy consumption (kWh_{base}) may be determined using Table 2-70Table 2-70Table 2-53.

The efficient model's annual energy consumption (kWh_{ee} or kWh_{me}) may be determined using manufacturers' test data for the given model. Where test data is not available the algorithms in Table 2-70Table 2-53 and Table 2-72Table 2-55 for "ENERGY STAR and ENERGY STAR Most Efficient maximum energy usage in kWh/year" may be used to determine the efficient energy consumption for a conservative savings estimate.

ENERGY STAR Refrigerator

$$\Delta kWh/yr$$
 = $kWh_{base} - kWh_{ee}$
 ΔkW_{peak} = $(kWh_{base} - kWh_{ee}) \times ETDF/Hours \times *CF$

¹⁷³ ENERGY STAR Appliances.November 2013. U.S. Environmental Protection Agency and U.S. Department of Enegy. ENERGY STAR. http://www.energystar.gov/.

ENERGY STAR Most Efficient Refrigerator

 $\Delta kWh/yr$ = $kWh_{base} - kWh_{me}$

 ΔkW_{peak} = $(kWh_{base} - kWh_{me}) \times ETDF/Hours * CF$

DEFINITION OF TERMS

Table 2-69: Assumptions for ENERGY STAR Refrigerators

Term	Unit	Value	Source
kWh _{base} , Annual energy consumption of baseline unit	kWh/yr	<u>Table 2-70</u>	1
kWh _{ee} , Annual energy consumption of ENERGY STAR qualified unit	kWh/yr	EDC Data Gathering Default= Table 2-70 Table 2-70	2
kWh _{me} , Annual energy consumption of ENERGY STAR Most Efficient qualified unit	kWh/yr	EDC Data Gathering Default = Table 2-72	3
ETDF , Energy to Demand Factor	kW kWh/yr	<u>0.0001119</u>	4

Refrigerator energy use is characterized by configuration (top freezer, bottom freezer, etc.), volume, whether defrost is manual or automatic and whether there is through-the-door ice. If this information is known, annual energy consumption (kWh_{base}) of the federal standard model may be determined using Table 2-70 Table 2-70. The efficient model's annual energy consumption (kWh_{ee} or kWh_{me}) may be determined using manufacturer's test data for the given model. Where test data is not available, the algorithms in Table 2-70 Table 2-70 and Table 2-72 Table 2-72 for "ENERGY STAR and ENERGY STAR Most Efficient maximum energy usage in kWh/year" may be used to determine efficient energy consumption for a conservative savings estimate. The term "AV" in the equations refers to "Adjusted Volume;" in ft³, where which is AV = (Fresh Volume) + 1.63 x (Freezer Volume).

<u>Table 2-70:</u> Federal Standard and ENERGY STAR Refrigerators Maximum Annual Energy Consumption if Configuration and Volume Known¹⁷⁴

Refrigerator Category	Federal Standard Maximum Usage in kWh/yr	ENERGY STAR Maximum Energy Usage in kWh/yr
Standard Size Models: 7.75	cubic feet or greater	

¹⁷⁴ ENERGY STAR Refrigerators and Freezers Key Product Criteria. http://www.energystar.gov/index.cfm?c=refrig.pr_crit_refrigerators Access July 2013

Refrigerator Category	Federal Standard Maximum Usage in kWh/yr	ENERGY STAR Maximum Energy Usage in kWh/yr
Refrigerator-freezers and refrigerators other than all- refrigerators with manual defrost.	7.99AV + 225.0	7.19 * AV + 202.5
1A. All-refrigerators—manual defrost.	6.79AV + 193.6	6.11 * AV + 174.2
2. Refrigerator-freezers—partial automatic defrost	7.99AV + 225.0	7.19 * AV + 202.5
Refrigerator-freezers—automatic defrost with top- mounted freezer without an automatic icemaker.	8.07AV + 233.7	7.26 * AV + 210.3
3-BI. Built-in refrigerator-freezer—automatic defrost with top-mounted freezer without an automatic icemaker.	9.15AV + 264.9	8.24 * AV + 238.4
3I. Refrigerator-freezers—automatic defrost with top- mounted freezer with an automatic icemaker without through-the-door ice service.	8.07AV + 317.7	7.26 * AV + 294.3
3I-BI. Built-in refrigerator-freezers—automatic defrost with top-mounted freezer with an automatic icemaker without through-the-door ice service.	9.15AV + 348.9	8.24 * AV + 322.4
3A. All-refrigerators—automatic defrost.	7.07AV + 201.6	6.36 * AV + 181.4
3A-BI. Built-in All-refrigerators—automatic defrost.	8.02AV + 228.5	7.22 * AV + 205.7
4. Refrigerator-freezers—automatic defrost with side- mounted freezer without an automatic icemaker.	8.51AV + 297.8	7.66 * AV + 268.0
4-BI. Built-In Refrigerator-freezers—automatic defrost with side-mounted freezer without an automatic icemaker.	10.22AV + 357.4	9.20 * AV + 321.7
4I. Refrigerator-freezers—automatic defrost with side- mounted freezer with an automatic icemaker without through-the-door ice service.	8.51AV + 381.8	7.66 * AV + 352.0
4I-BI. Built-In Refrigerator-freezers—automatic defrost with side-mounted freezer with an automatic icemaker without through-the-door ice service.	10.22AV + 441.4	9.20 * AV + 405.7
5. Refrigerator-freezers—automatic defrost with bottom-mounted freezer without an automatic icemaker.	8.85AV + 317.0	7.97 * AV + 285.3
5-BI. Built-In Refrigerator-freezers—automatic defrost with bottom-mounted freezer without an automatic icemaker.	9.40AV + 336.9	8.46 * AV + 303.2
51. Refrigerator-freezers—automatic defrost with bottom-mounted freezer with an automatic icemaker without through-the-door ice service.	8.85AV + 401.0	7.97 * AV + 369.3
5I-BI. Built-In Refrigerator-freezers—automatic defrost with bottom-mounted freezer with an automatic icemaker without through-the-door ice service.	9.40AV + 420.9	8.46 * AV + 387.2
5A. Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service.	9.25AV + 475.4	8.33 * AV + 436.3
5A-BI. Built-in refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service.	9.83AV + 499.9	8.85 * AV + 458.3
6. Refrigerator-freezers—automatic defrost with top- mounted freezer with through-the-door ice service.	8.40AV + 385.4	7.56 * AV + 355.3
7. Refrigerator-freezers—automatic defrost with side- mounted freezer with through-the-door ice service.	8.54AV + 432.8	7.69 * AV + 397.9
7-BI. Built-In Refrigerator-freezers—automatic defrost with side-mounted freezer with through-the-door ice service.	10.25AV + 502.6	9.23 * AV + 460.7

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	Refrigerator Category	Federal Standard Maximum Usage in kWh/yr	ENERGY STAR Maximum Energy Usage in kWh/yr
	Compact Size Models: Less than 7.75 cubic	feet and 36 inches or l	ess in height
	11. Compact refrigerator-freezers and refrigerators other than all-refrigerators with manual defrost.	9.03AV + 252.3	8.13 * AV + 227.1
l	11A.Compact all-refrigerators—manual defrost.	7.84AV + 219.1	7.06 * AV + 197.2
	12. Compact refrigerator-freezers—partial automatic defrost	<u>5.91AV + 335.8</u>	5.32 * AV + 302.2
	13. Compact refrigerator-freezers—automatic defrost with top-mounted freezer.	11.80AV + 339.2	10.62 * AV + 305.3
	131. Compact refrigerator-freezers—automatic defrost with top-mounted freezer with an automatic icemaker.	11.80AV + 423.2	10.62 * AV + 389.3
	13A. Compact all-refrigerators—automatic defrost.	9.17AV + 259.3	8.25 * AV + 233.4
	14. Compact refrigerator-freezers—automatic defrost with side-mounted freezer.	6.82AV + 456.9	6.14 * AV + 411.2
	14l. Compact refrigerator-freezers—automatic defrost with side-mounted freezer with an automatic icemaker.	6.82AV + 540.9	6.14 * AV + 495.2
	15. Compact refrigerator-freezers—automatic defrost with bottom-mounted freezer.	11.80AV + 339.2	10.62 * AV + 305.3
	15I. Compact refrigerator-freezers—automatic defrost with bottom-mounted freezer with an automatic icemaker.	11.80AV + 423.2	10.62 * AV + 389.3

The default values for each configuration are given in

Table 2-71
Table 2-71

Table 2-71: Default Savings Values for ENERGY STAR Refrigerators¹⁷⁵

Refrigerator Category	Assumed Volume of Unit (cubic feet) ¹⁷⁶	Convention al Unit Energy Usage in kWh/yr ⁴⁷⁷	ENERGY STAR Energy Usage in kWh/yr ⁴⁷⁸	ΔkWh/yr	ΔkW _{peak}
1A. All-refrigerators—manual defrost.Manual Defrost and Partial Automatic Defrost	12.2	311 276	230 249	<u>28</u> 80	0.0031 _{0.00} 92
2. Refrigerator-freezers—partial automatic defrost	12.2	<u>322</u>	<u>290</u>	<u>32</u>	0.0036
31. Refrigerator-freezers— automatic defrost with top- mounted freezer with an automatic icemaker without through-the-door ice service. Top- mount freezer without door ice	17.9	4 75 462	369 424	111 38	0.00420.01 26
4I. Refrigerator-freezers— automatic defrost with side- mounted freezer with an	22.7	713 <u>575</u>	<u>526</u> 557	<u>49</u> 156	0.00550.01 78

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¹⁷⁶ ENERGY STAR Appliances Calculator. Accessed November 2013.

477-ENERGY STAR Residential Refrigerators Qualified Products List. July 5, 2013. Average federal standard consumption of allqualifying models by configuration.

178 Ibid. Average consumption of all ENERGY STAR qualifying models by configuration.

Refrigerator Category	Assumed Volume of Unit (cubic feet) ¹⁷⁶	Convention al Unit Energy Usage in kWh/yr ⁴⁷⁷	ENERGY STAR Energy Usage in kWh/yr ⁴⁷⁸	ΔkWh/yr	ΔkW _{peak}
automatic icemaker without through-the-door ice service.Side- mount freezer without door ice					
51. Refrigerator-freezers— automatic defrost with bottom- mounted freezer with an automatic icemaker without through-the-door ice service.Bottom mount freezer- without door ice	20.0	692<u>578</u>	<u>529</u> 538	<u>49</u> 154	0.00550.01 76
7. Refrigerator-freezers— automatic defrost with side- mounted freezer with through- the-door ice service. Side mount- freezer with door ice	24.6	629 643	<u>587</u> 4 90	<u>56</u> 139	0.0062 0.01 59
5A. Refrigerator-freezer— automatic defrost with bottom- mounted freezer with through- the-door ice service.Bottom- mount freezer with door ice	25.4	569 710	<u>648</u> 447	<u>62</u> 122	0.0070 0.01 39
3A. All-refrigerators—automatic defrost.Refrigerator only single door without ice	12.2	381 288	<u>259</u> 292	<u>29</u> 89	0.0032 0.01
Refrigerator/Freezer – single-door	12.2	450	348	102	0.0116
Compact Size Models: Less than	7.75 cubic fe	et and 36 inch	es or less in l	neight	
11A.Compact all-refrigerators— manual defrost.Manual Defrost- and Partial Automatic Defrost	3.3	362 245	276 220	<u>24</u> 86	0.00270.00 98
12. Compact refrigerator- freezers—partial automatic defrost	3.3	<u>355</u>	<u>320</u>	<u>36</u>	0.0040
13. Compact refrigerator- freezers—automatic defrost with top-mounted freezer. Top Mount- and Refrigerator Only	4.5	<u>392</u> 417	<u>353</u> 306	<u>39</u> 111	0.00440.01 26
15. Compact refrigerator- freezers—automatic defrost with bottom-mounted freezer_Bottom- mount freezer	5.1	452 399	<u>359</u> 362	<u>40</u> 90	0.0045 03

ENERGY STAR Most Efficient annual energy consumption (kWh_{me}) may be determined using manufacturer's test data for the given model. Where test data is not available, the algorithms in Table 2-72Table 2-55 for "ENERGY STAR Most Efficient maximum energy usage in kWh/year" may be used to determine efficient energy consumption for a conservative savings estimate. Baseline annual energy usage consumption (kWh_{base}) of the federal standard model may be determined using $\frac{\text{Table 2-70Table 2-70Table 2-53}}{\text{Table 2-70Table 2-70Table 2-53}}$

Table 2-55: ENERGY STAR Most Efficient Annual Energy Usage if Configuration and Volume Known

Refrigerator Category	ENERGY STAR Most Efficient Maximum— Energy Usage in kWh/yr
Manual Defrost and Automatic Defrost	AV ≤ 49.8, Eann ≤ 6.17*AV + 173.9 AV > 49.8, Eann ≤ 481

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Top mount freezer without door ice	AV ≤ 42.0, Eann ≤ 6.86*AV + 193.2 AV > 42.0, Eann ≤ 481
Side mount freezer without door ice	AV ≤ 36.5, Eann ≤ 3.44*AV + 355.3 AV > 36.5, Eann ≤ 481
Bottom mount freezer without door ice	AV ≤ 49.6, Eann ≤ 3.22*AV + 321.3 AV > 49.6, Eann ≤ 481
Bottom mount freezer with door ice	AV ≤ 29.6, Eann ≤ 3.50*AV + 377.3 AV > 29.6, Eann ≤ 481
Top mount freezer with door ice	AV ≤ 32.5, Eann ≤ 7.14*AV + 249.2 AV > 32.5, Eann ≤ 481
Side mount freezer with door ice	AV ≤ 27.8, Eann ≤ 7.07*AV + 284.2 AV > 27.8, Eann ≤ 481

Table 2-72: ENERGY STAR Most Efficient Annual Energy Usage if Configuration and Volume Known

Refrigerator Category	ENERGY STAR Most Efficient Maximum Annual Energy Usage in kWh/yr
Refrigerator-freezers and refrigerators other than all-refrigerators with manual defrost.	<u>AV ≤ 65.6, Eann ≤ 6.79*AV + 191.3</u> <u>AV > 65.6, Eann ≤ 637</u>
2. Refrigerator-freezers—partial automatic defrost	$AV \le 65.6$, Eann $\le 6.79*AV + 191.3$ $AV > 65.6$, Eann ≤ 637
3. Refrigerator-freezers—automatic defrost with top-mounted freezer without an automatic icemaker.	AV ≤ 63.9, Eann ≤ 6.86*AV + 198.6 AV > 63.9, Eann ≤ 637
3-BI. Built-in refrigerator-freezer—automatic defrost with top-mounted freezer without an automatic icemaker.	$AV \le 63.9$, Eann $\le 6.86*AV + 198.6$ $AV > 63.9$, Eann ≤ 637
31. Refrigerator-freezers—automatic defrost with top-mounted freezer with an automatic icemaker without through-the-door ice service.	AV ≤ 51.6, Eann ≤ 6.86*AV + 282.6 AV > 51.6, Eann ≤ 637
3I-BI. Built-in refrigerator-freezers—automatic defrost with top-mounted freezer with an automatic icemaker without through-the-door ice service.	$AV \le 51.6$, Eann $\le 6.86*AV + 282.6$ $AV > 51.6$, Eann ≤ 637
4. Refrigerator-freezers—automatic defrost with side-mounted freezer without an automatic icemaker.	$AV \le 53.0$, Eann $\le 7.23*AV + 253.1$ $AV > 53.0$, Eann ≤ 637
4-Bl. Built-In Refrigerator-freezers—automatic defrost with side-mounted freezer without an automatic icemaker.	AV ≤ 53.0, Eann ≤ 7.23*AV + 253.1 AV > 53.0, Eann ≤ 637
41. Refrigerator-freezers—automatic defrost with side-mounted freezer with an automatic icemaker without through-the-door ice service.	AV ≤ 41.4, Eann ≤ 7.23*AV + 337.1 AV > 41.4, Eann ≤ 637
4I-BI. Built-In Refrigerator-freezers— automatic defrost with side-mounted freezer with an automatic icemaker without through- the-door ice service.	$AV \le 41.4$, Eann $\le 7.23*AV + 337.1$ $AV > 41.4$, Eann ≤ 637
Refrigerator-freezers—automatic defrost with bottom-mounted freezer without an automatic icemaker.	AV ≤ 48.8, Eann ≤ 7.52*AV + 269.5 AV > 48.8, Eann ≤ 637
5-BI. Built-In Refrigerator-freezers—automatic defrost with bottom-mounted freezer without an automatic icemaker.	AV ≤ 48.8, Eann ≤ 7.52*AV + 269.5 AV > 48.8, Eann ≤ 637

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Refrigerator Category	ENERGY STAR Most Efficient Maximum Annual Energy Usage in kWh/yr
51. Refrigerator-freezers—automatic defrost with bottom-mounted freezer with an automatic icemaker without through-the-door ice service.	$AV \le 37.7$, Eann $\le 7.52*AV + 353.5$ $AV > 37.7$, Eann ≤ 637
51-BI. Built-In Refrigerator-freezers— automatic defrost with bottom-mounted freezer with an automatic icemaker without through-the-door ice service.	$AV \le 37.7$, Eann $\le 7.52*AV + 353.5$ $AV > 37.7$, Eann ≤ 637
5A. Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service.	AV ≤ 28.0, Eann ≤ 7.86*AV + 416.7 AV > 28.0, Eann ≤ 637
5A-BI. Built-in refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service.	AV ≤ 28.0, Eann ≤ 7.86*AV + 416.7 AV > 28.0, Eann ≤ 637
6. Refrigerator-freezers—automatic defrost with top-mounted freezer with through-the-door ice service.	AV < 41.5, Eann ≤ 7.14*AV + 340.2 AV > 41.5, Eann ≤ 637
7. Refrigerator-freezers—automatic defrost with side-mounted freezer with through-the-door ice service.	AV ≤ 35.3, Eann ≤ 7.26*AV + 380.5 AV > 35.3, Eann ≤ 637
7-BI. Built-In Refrigerator-freezers—automatic defrost with side-mounted freezer with through-the-door ice service.	$AV \le 35.3$, Eann $\le 7.26*AV + 380.5$ $AV > 35.3$, Eann ≤ 637

DEEMED SAVINGS

The default values for each ENERGY STAR Most Efficient configuration are given in Table 2-73.

Table 2-73: Default Savings Values for ENERGY STAR Most Efficient Refrigerators¹⁷⁹

Refrigerator Category	Assumed Volume of Unit (cubic feet) ¹⁸⁰	Conventional Unit Energy Usage in kWh/yr ¹⁸¹	ENERGY STAR Most Efficient Consumption in kWh/yr ¹⁸²	ΔkWh/yr	ΔkW _{peak}
Top mount freezer without door ice		475	350	125	0.0149
51. Refrigerator-freezers—automatic defrost with bottom-mounted freezer with an automatic icemaker without through-the-door ice service.Bottom-mount freezer without door ice	24.6	692 <u>619</u>	4 03 486	228 133	0.014940.0192
5A. Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service.Bottom mount freezer with door ice	<u>32.1</u>	569 <u>772</u>	502 631	<u>141</u> 167	<u>0.0158</u> 0.026

¹⁷⁹ Configurations of qualified models as of July 5, 2013.

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¹⁸⁰ Average Adjusted Volume of qualified models within the category.

181 ENERGY STAR Residential Refrigerators Qualified Products List. August July 5, 20143. Average federal standard consumption of all qualifying models by configuration.

¹⁸² Average consumption of all qualified units as of <u>AugustJuly</u> 5, 201<u>4</u>3. Qualified units list from

http://www.energystar.gov/index.cfm?c=most_efficient.me_medium_fridges_freezers_https://data.energystar.gov/Active-Specifications/ENERGY-STAR-Certified-Residential-Refrigerators/dgpf-upjt?

State of Pennsylvania Technical Reference Manual Rev Date: June 2015

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FUTURE STANDARDS CHANGES

As of September 15, 2014 new federal minimum efficiency standards for refrigerators and refrigerators-freezers will take effect. The maximum allowable energy usage by refrigerator configuration is listed in Table 2-57. These standards will take effect beginning with the 2015 TRM.

Table 2-57: Federal Refrigerator Standards Effective as of the 2015 TRM¹⁸⁴

Refrigerator Category	Federal Standard Maximim Usage in- kWh/year (Effective 2015 TRM)
Standard Size Models: 7.75 cubic feet or	greater
Refrigerators-freezers and refrigerators other than all- refrigerators with manual defrost (including partial- automatic defrost)	7.99*AV + 225.0
All-refrigerators – manual defrost	6.79*AV + 193.6
Automatic defrost with top-mounted freezer without- through-the-door ice service-	8.07*AV + 233.7
Automatic defrost with side-mounted freezer without- through-the-door ice service	8.51*AV + 297.8
Automatic defrost with bottom-mounted freezer without- through-the-door ice service	8.85*AV + 317.0
Automatic defrost with top-mounted freezer with through- the-door ice service	8.40*AV + 385.4
Automatic defrost with side-mounted freezer with through- the-door ice service	8.54*AV + 432.8
Compact Size Models: Less than 7.75 cubic feet and height	36 inches or less in
Compact refrigerator-freezers and refrigerators other than- all-refrigerators with manual defrost-	9.03*AV + 252.3
Compact refrigerator-freezers – manual defrost	7.84*AV + 219.1
Compact refrigerator-freezer – partial automatic defrost	5.91*AV + 335.8
Compact refrigerator-freezersautomatic defrost with top- mounted freezer-	11.80*AV + 339.2
Compact Refrigerator-Freezersautomatic defrost with- side-mounted freezer	6.82*AV + 456.9
Compact Refrigerator-Freezers—automatic defrost with- bottom-mounted freezer	11.80*AV + 339.2

EVALUATION PROTOCOLS

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¹⁸³ ENERGY STAR Appliances.November 2013. U.S. Environmental Protection Agency and U.S. Department of Energy. ENERGY STAR. http://www.energystar.gov/.

184 U.S. Department of Energy. Federal Register. 179th ed. Vol. 76. September 15, 2011.

https://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrig_finalrule_frnotice.pdf

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

State of Pennsylvania

- Federal Standards for Residential Refrigerators and Freezers, Effective 9/14/2014. http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43
- 2. ENERGY STAR Program Requirements Product Specifications for Residential Refrigerators and Freezers Version 5.0. Effective 9/15/2014. http://www.energystar.gov/products/specs/system/files/ENERGY%20STAR%20Final%20 Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Program%20 Requirements.pdf
- 2.3. ENERGY STAR Recognition Criteria for Most Efficient Refrigerator-Freezers. Table 2. http://www.energystar.gov/ia/partners/downloads/most_efficient/final_criteria/Refrigerator-Freezers.pdf?6a37-2bde
- 4. Assessment of Energy and Capacity Savings Potential In Iowa. Quantec in collaboration with Summit Blue Consulting, Nexant, Inc., A-TEC Energy Corporation, and Britt/Makela Group, prepared for the Iowa utility Association, February 2008. http://plainsjustice.org/files/EEP-08-1/Quantec/QuantecReportVol1.pdf

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2.4.2 ENERGY STAR FREEZERS

Measure Name	Freezers
Target Sector	Residential Establishments
Measure Unit	Freezer
Unit Energy Savings	Varies by Configuration
Unit Peak Demand Reduction	Varies by Configuration
Measure Life	12 years ¹⁸⁵
<u>Vintage</u>	Replace on Burnout

ELIGIBILITY

This measure is for the purchase and installation of a new freezer meeting ENERGY STAR criteria. An ENERGY STAR freezer must be at least 10 percent more efficient than the minimum federal government standard.

ALGORITHMS

The general form of the equation for the ENERGY STAR Freezer measure savings algorithm is:

Total Savings = Number of Freezers \times Savings per Freezer

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of freezers. The number of freezers will be determined using market assessments and market tracking.

If the volume and configuration of the freezer is known, the baseline model's annual energy consumption (kWh $_{\rm base}$) may be are determined using

Table 2-74

 $\overline{\text{Lable 2-74}}$. The efficient model's annual energy consumption (kWh_{ee}) may be determined using manufacturer's test data for the given model. Where test data is not available the algorithms in

Table 2-75

Table 2-75 for "ENERGY STAR Maximum Energy Usage in kWh/year" may be used to determine the efficient energy consumption for a conservative savings estimate

ENERGY STAR Freezer

 $\Delta kWh/yr = kWh_{base} - kWh_{ee}$ $\Delta kW_{peak} = (kWh_{base} - kWh_{ee}) \times ETDF/Hours * CF$

DEFINITION OF TERMS

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¹⁸⁵ ENERGY STAR Appliances Savings Calculator. Accessed July 10, 2013.

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Term	Unit	Value	Source
kWh _{base} , Annual energy consumption of baseline unit	kWh/yr	<u>Table 2-74</u>	1
kWh _{ee} , Annual energy consumption of ENERGY STAR qualified unit	kWh/yr	EDC Data Gathering Default= Table 2-74 Table 2-74	2
HOU, Hours of operation per year		EDC Data Gathering Default = 8,760	
CF, Demand coincidence factor	%	EDC Data Gathering Default= 1	
ETDF , Energy to Demand Factor	kW kWh/yr	0.0001119	<u>3</u>

Freezer energy use is characterized by configuration (upright, chest or compact), volume and whether defrost is manual or automatic. If this information is known, annual energy consumption of the federal minimum efficiency standard model may be determined using

Table 2-74

Table 2-74. The efficient model's annual energy consumption (kWh_{ee}) may be determined using manufacturers' test data for the given model. Where test data is not available, the algorithms in Table 2-75

Table 2-75 for "ENERGY STAR maximum energy usage in kWh/year" may be used to determine efficient energy consumption for a conservative savings estimate. The term "AV" in the equations refers to "Adjusted Volume," which is AV = 1.73 x Total Volume. Note this table is also provided for planning purposes to compare to the changing federal standards detailed in Table 2-60.

Table 2-58: Federal Standard and ENERGY STAR Freezers Maximum Annual Energy Consumption if Configuration and Volume Known¹⁸⁶

Freezer Category	Federal Standard Maximim Usage in kWh/year	ENERGY STAR Maximum Energy Usage in kWh/year		
Upright with manual defrost	7.55*AV+258.3	< 6.795*AV + 232.47		
Upright with automatic defrost	12.43*AV+326.1	< 11.187*AV + 293.49		
Chest Freezer	9.88*AV+143.7	< 8.892*AV + 129.33		
Compact Upright with manual defrost	9.78*AV+250.8	< 7.824*AV + 200.64		
Compact Upright with automatic defrost	11.40*AV+391	< 9.120*AV + 312.8		
Compact Chest Freezer	10.45*AV+152	< 8.360*AV + 121.6		

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¹⁸⁶ ENERGY STAR Refrigerators and Freezers Key Product Criteria. http://www.energystar.gov/index.cfm?c=refrig.pr_crit_refrigerators

Table 2-74: Federal Standard and ENERGY STAR Freezers Maximum Annual Energy Consumption if Configuration and Volume Known¹⁸⁷

Freezer Category	Federal Standard Maximum Usage in kWh/year	ENERGY STAR Maximum Energy Usage in kWh/year
8. Upright freezers with manual defrost.	<u>5.57AV + 193.7</u>	5.01 * AV + 174.3
9. Upright freezers with automatic defrost without an automatic icemaker.	8.62AV + 228.3	7.76 * AV + 205.5
91. Upright freezers with automatic defrost with an automatic icemaker.	8.62AV + 312.3	7.76 * AV + 289.5
9-BI. Built-In Upright freezers with automatic defrost without an automatic icemaker.	9.86AV + 260.9	8.87 * AV + 234.8
9I-BI. Built-in upright freezers with automatic defrost with an automatic icemaker.	9.86AV + 344.9	8.87 * AV + 318.8
10. Chest freezers and all other freezers except compact freezers.	<u>7.29AV + 107.8</u>	6.56 * AV + 97.0
10A. Chest freezers with automatic defrost.	10.24AV + 148.1	9.22 * AV + 133.3
16. Compact upright freezers with manual defrost.	8.65AV + 225.7	7.79 * AV + 203.1
17. Compact upright freezers with automatic defrost.	10.17AV + 351.9	9.15 * AV + 316.7
18. Compact chest freezers.	9.25AV + 136.8	8.33 * AV + 123.1

The default values for each configuration are given in

<u>Table 2-75</u>

Table 2-75. Note that a compact freezer is defined as a freezer that has a volume less than 7.75 cubic feet and is 36 inches or less in height.

DEFAULTEMED SAVINGS

Table 2-75: Default Savings Values for ENERGY STAR Freezers

Freezer Category	Average Adjusted Volume of Qualified Units in ft3	Conventional Unit Energy Usage in kWh/yr ¹⁸⁸	ENERGY STAR Energy Usage in kWh/yr ¹⁸⁹	ΔkWh/yr	∆kW _{peak}
8. Upright freezers with manual defrost. Upright with manual defrost	Currently no qualified	405	347	58	0.0066

¹⁸⁷ ENERGY STAR Refrigerators and Freezers Key Product Criteria. http://www.energystar.gov/index.cfm?c=refrig.pr crit refrigerators

¹⁸⁹ Ibid. Average ENERGY STAR consumption of all qualifying models by configuration.

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Field Code Changed

¹⁸⁸ ENERGY STAR Residential Freezers Qualified Products List. <u>August July</u> <u>510</u>, 201<u>5</u>3. Average federal standard consumption of all qualifying models by configuration.

http://www.energystar.gov/products/specs/system/files/Refrigerators_and_Freezers_Program_Requirements%20v4_1.pdf_ https://data.energystar.gov/Active-Specifications/ENERGY-STAR-Certified-Residential-Freezers/2wah-sixr?

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Freezer Category	Average Adjusted Volume of Qualified Units in ft3	Conventional Unit Energy Usage in kWh/yr ¹⁸⁸	ENERGY STAR Energy Usage in kWh/yr ¹⁸⁹	ΔkWh/yr	∆kW _{peak}
	<u>units</u>				
Upright freezers with automatic defrost without an automatic icemaker. Upright- with automatic defrost	<u>24.7</u>	<u>441681</u>	<u>419</u> 599	<u>2282</u>	0.00250.0094
10. Chest freezers and all other freezers except compact freezers. Chest Freezer	<u>18.5</u>	<u>243392</u>	<u>215</u> 351	<u>28</u> 41	<u>0.0031</u> 0.0047
16. Compact upright freezers with manual defrost. Compact- Upright with manual defrost	3.7	<u>258</u> 301	<u>232</u> 235	<u>26</u> 66	0.00290.0076
17. Compact upright freezers with automatic defrost. Compact Upright with automatic defrost	<u>7.7</u>	<u>430</u> 4 95	<u>367</u> 355	<u>63</u> 140	0.00710.0160
18. Compact chest freezers. Compact Chest Freezer	<u>8.9</u>	<u>219</u> 257	<u>177</u> 204	<u>42</u> 54	0.00470.0061

MEASURE LIFE

ENERGY STAR Freezers: Measure Life = 12 years. 490

FUTURE STANDARDS CHANGES

As of September 15, 2014 new federal minimum efficiency standards for freezers will take effect. The maximum allowable energy usage by freezer configuration is listed in Table 2-60. These standards will take effect beginning with the 2015 TRM.

Table 2-60: Federal Freezer Standards Effective as of the 2015 TRM¹⁹¹-

Freezer Category	Federal Standard Maximim Usage in kWh/year (Effective 2015 TRM)
Upright with manual defrost	5.57*AV + 193.7
Upright with automatic defrost	8.62*AV + 228.3
Chest Freezer	7.29*AV + 107.8
Compact Upright with manual defrost	8.65*AV + 225.7
Compact Upright with automatic defrost	10.17*AV + 351.9
Compact Chest Freezer	9.25*AV + 136.8

¹⁹⁰ ENERGY STAR Appliances Savings Calculator. Accessed July 10, 2013.

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¹⁹¹ U.S. Department of Energy. Federal Register. 179th ed. Vol. 76. September 15, 2011. https://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrig_finalrule_fmotice.pdf

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EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Federal Standards for Residential Refrigerators and Freezers, Effective 9/14/2014. http://www1.eere.energy.gov/buildings/appliance standards/product.aspx/productid/43
- 2. ENERGY STAR Program Requirements Product Specifications for Residential Refrigerators and Freezers Version 5.0. Effective 9/15/2014. http://www.energystar.gov/products/specs/system/files/ENERGY%20STAR%20Final%20 Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Program%20 Requirements.pdf
- 4-3. Assessment of Energy and Capacity Savings Potential In Iowa. Quantec in collaboration with Summit Blue Consulting, Nexant, Inc., A-TEC Energy Corporation, and Britt/Makela Group, prepared for the Iowa utility Association, February 2008. http://plainsjustice.org/files/EEP-08-1/Quantec/QuantecReportVol1.pdf

Field Code Changed

2.4.3 REFRIGERATOR / FREEZER RECYCLING WITH AND WITHOUT REPLACEMENT

Measure Name	Refrigerator/Freezer Recycling and Replacement	
Target Sector	Residential Establishments	
Measure Unit	Refrigerator or Freezer	
Default Unit Annual Energy Savings- Refrigerators	Varies by EDC	
Default Unit Peak Demand Reduction- Refrigerators	Varies by EDC	
Default Unit Annual Energy Savings- Freezers	Varies by EDC	
Default Unit Peak Demand Reduction- Freezers	Varies by EDC	
Measure Life (no replacement)	8 years ¹⁹²	
Measure Life (with replacement)	7 years (see measure life discussion below)	
<u>Vintage</u>	Early Retirement, Early Replacement	

ELIGIBILITY

This measure is (1) the retirement of a refrigerator or freezer with no replacement or (2) the recycling and replacement before end of life of an existing refrigerator or freezer with a new refrigerator or freezer. This protocol quantifies savings where the replacement refrigerator or freezer is ENERGY STAR or non-ENERGY STAR qualified. This protocol applies to both residential and non-residential sectors, as refrigerator and freezer usage and energy usage are assumed to be independent of customer rate class¹⁹³. The partially deemed savings algorithms are based on regression analysis of metered data on kWh consumption from other States. The particallypartially deemed savings algorithms for this measure can be applied to refrigerator and freezer retirements or early replacements meeting the following criteria:

- 1. Existing, working refrigerator or freezer 10-30 cubic feet in size (savings do not apply if unit is not working)
- 2. Unit is a primary or secondary unit

EDCs can use the default savings values listed for each EDC in this protocol or an EDC can calculate program savings using the partially deemed savings algorithms, the deemed regression equation coefficients, and actual program year recycled refrigerator/freezer data. An EDC's use of actual program year data can provide a more accurate annual ex ante savings estimate due to the changing mix of recycled appliance models from year-to-year.

PARTIALLY DEEMED SAVINGS ALGORITHMS

Equation 1:

DEEMED kWhsaved Per Unit = EXISTING UEC * PART USE

Equation 2:

NET_kWhsaved Per Unit = DEEMED kWhsaved Per Unit - (REPLACEMENTUEC * PART_USE)

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¹⁹² Vermont Energy Investment Corporation (VEIC) for NEEP, Mid Atlantic TRM Version 2.0. July 2011. Pg.36.

¹⁹³ For example, non-residential rate class usage cases include residential dwellings that are master-metered, usage in offices or any other applications that involve typical refrigerator usage.

DEFINITION OF TERMS

Table 2-76: Calculation Assumptions and Definitions for Refrigerator and Freezer Recycling

Component	Unit	Values	Source
DEFAULT_kWhsaved , Annual electricity savings measured in kilowatt hours.	kWh/yr		
EXISTING_UEC , The average annual unit energy consumption of participating refrigerators and freezers for Program year 4. Table 2-77 to Table 2-80 Table 2-80 Table 2-80 below provide the equation inputs needed to calculate the UEC for removed refrigerators and freezers respectively as well as the calculation of the default Unit Energy Consumption value for refrigerators or freezers for each EDC.	kWh/yr	Table 2-77 Table 2-77 through Table 2-80 Table 2-80	
PART_USE, The portion of the year the average refrigerator or freezer would likely have operated if not recycled through the program	%	EDC Data Gathering Default: Refrigerator= 96.9% Freezer= 98.5%	7
REPLACEMENTUEC, The annual unit energy consumption of the average replacement unit	kWh/yr	Standard Refrigerator: 602537 kWh/yr 194 ENERGY STAR Refrigerator: 482417-kWh/yr 195 Standard Freezer: 527510- kWh/yr 196 ENERGY STAR Freezer: 474423 kWh/yr 197	8
ETDF , Energy to Demand Factor	$\frac{kW}{kWh/yr}$	0.0001119	<u>9</u>

DEFAULT SAVINGS CALCULATIONS

For removed refrigerators, the annual Unit Energy Consumption (UEC) is based upon regression analyses of data from refrigerators metered and recycled through five utilities. The UEC for removed refrigerators was calculated specifically for each utility using data collected from each utility's Program Year Four (PY4) Appliance Removal programs. Therefore, each UEC represents the average ages, sizes, etc of the fleet of refrigerators removed in Program Year Four.

SECTION 2: Residential Measures

 $[\]underline{^{194}\,\text{Average of the 6 Non-ENERGY STAR Refrigerator options in the ENERGY STAR Calculator}}$

¹⁹⁵ Average of the 6 ENERGY STAR Refrigerator options in the ENERGY STAR Calculator

¹⁹⁶ Average of the 6 Non-ENERGY STAR Freezer options in the ENERGY STAR Calculator

¹⁹⁷ Average of the 6 ENERGY STAR Freezer options in the ENERGY STAR Calculator

Existing Refrigerator UEC

- = 365.25 * (0.582 + 0.027 * (average age of appliance) + 1.055
- * (% of appliances manufactured before 1990) + 0.067
- * $(number\ of\ cubic\ feet)$ 1.977 * $(\%\ of\ single\ door\ units)$ + 1.071
- *(% of side by side) + 0.605 *(% of primary usage) + 0.02
- * $(unconditioned\ space\ CDDs) 0.045* (unconditioned\ HDDs)) = kWh$

Source for refrigerator UEC equation: US DOE Uniform Method Project, Savings Protocol for Refrigerator Retirement, April 2013

Refrigerator Unit Energy Consumption Equation					
Equation Intercept and Independent Variables	Estimate Coefficient (Daily kWh)				
Intercept	0.582				
Appliance Age (years)	0.027				
Durmy: Manufactured Pre-1990	1.055				
Appliance Size (square feet)	0.067				
Durmy: Single-Door Configuration	-1.977				
Dummy: Side-by-Side Configuration	1.071				
Dummy: Primary Usage Type (in absence of the program)	0.6054				
Interaction: Located in Unconditioned Space x CDDs	0.02				
Interaction: Located in Unconditioned Space x HDDs	-0.045				

Refrigerator Savings - No Replacement:

DEFAULT_kWhsaved Per Unit NO REP = EXISTING_UEC * PART_USE = kWh

Refrigerator Savings - Replacement with Energy Star Unit:

DEFAULT_kWhsaved Per Unit with ES REP = DEFAULT kWhsaved Per Unit NO REP - (REPLACEMENTUEC ES * PART_USE) = kWh

Refrigerator Savings - Replacement with non-Energy Star Unit:

 $DEFAULT_kWhsaved \ Per \ Unit_{\textit{NOTREP}} = kWhsaved \ Per \ Unit_{\textit{NOTREP}} - \\ (REPLACEMENTUEC_{\textit{NONES}} * PART_USE) = kWh$

Existing Freezer UEC

- = 365.25 days
- *(-2.297 + 0.067 * [average age of appliance] + 0.401
- * $[8\% \ of \ appliances \ manufactured \ pre 1993] + 0.150$
- $* \left[average \ number \ of \ cubic \ feet\right] + 0.854$
- * [% of appliances that are chest freezers] + 0.1046 * [CDDs]) = kWh

Source for freezer UEC equation: Cadmus memo to Michigan Public Service Commission (August 2012)

Freezer Unit Energy Consumption Equation

SECTION 2: Residential Measures

Equation Intercept and Independent Variables	Estimate Coefficient (Daily kWh)
Intercept	-2.297
Appliance Age (years)	0.401
Dummy: Manufactured Pre-1993	0.067
Appliance Size (cubic feet)	0.15
% of appliances that are chest freezers	0.854
Cooling Degree Days (CDD)	0.1046

Freezer Savings - No Replacement:

DEFAULT_kWhsaved Per Unit NO REP = EXISTING_UEC * PART_USE = kWh

Freezer Savings - Replacement with Energy Star Unit:

DEFAULT_kWhsaved Per Unit NO REP = DEFAULT kWhsaved Per Unit NO REP = (REPLACEMENTUEC ES * PART_USE) = kWh

Freezer Savings - Replacement with non-Energy Star Unit:

DEFAULT_kWhsaved Per Unitwith NONES REP = kWhsaved Per Unit NO REP - (REPLACEMENTUEC NONES * PART_USE) = kWh

The Commission has computed the EDC-specific values that are needed for input to the regression equations for determining the Unit Energy Consumption based on Act 129 PY4 data provided by each EDC for refrigerators and freezers removed in PY4. Once these input values were determined, they were substituted into the above equation in order to estimate the UEC for removed refrigerators and freezers for each EDC.

Table 2-77 Table 2-77 to

<u>Table 2-80</u>Table 2-80 below provide the equation inputs needed to calculate the UEC for removed refrigerators and freezers respectively as well as the calculation of the default Unit Energy Consumption value for refrigerators or freezers for each EDC. Note that equation inputs in <u>Table 2-77</u>Table 2-77 to

<u>Table 2-80</u> are provided for each appliance (refrigerators, then freezers) depending on whether the units were or were not replaced with a new unit.

Table 2-77: Default values for Residential Refrigerator Recycling Without Replacement With a New Refrigerator

Variable Name	Duquesne Light	Met Ed	PECO	PennElec	PennPower	PPL	West Penn Power
Age of appliance	32.67	27.96	25.55	22.88	29.06	30.06	30.94
% manufactured pre 1990	68.18%	61.83%	49.86%	61.84%	60.56%	60.43%	64.98%
Appliance Size (volume in cubic							
feet)	16.03	18.47	18.65	16.11	16.25	18.07	16.57
Dummy: percent that are single-							
door appliances	12.13%	4.39%	8.12%	8.56%	7.71%	7.03%	8.27%
Dummy: percent that are side-							
by-side configuration	12.40%	21.43%	19.89%	10.78%	12.96%	18.31%	13.71%
Dummy: primary usage type (in							
absence of the program)	18.45%	10.12%	24.71%	17.22%	16.76%	26.01%	14.03%
Located in unconditioned space							
X CDD	0.98	1.78	2.47	1.07	1.22	0.92	1.20
Located in unconditioned space							
X HDD	7.90	10.86	9.51	9.93	10.17	9.85	9.42
ESTIMATED UEC (Annual kWh							
per year) for a removed							
refrigerator that is not replaced	1068.9	1087.3	1048.5	933.8	1003.9	1107.2	1052.4

Table 2-78: Default values for Residential Freezer Recycling Without Replacement With a New Freezer

Variable Name	Duquesne Light	Met Ed	PECO	PennElec	PennPower	PPL	West Penn Power
Age of appliance	35.51	32.21	29.05	32.88	34.17	32.68	33.21
% manufactured pre 1990	89.41%	87.03%	76.60%	86.94%	88.55%	88.20%	86.85%
Appliance Size (volume in cubic							
feet)	15.40	15.44	15.59	15.70	15.80	15.65	15.74
Dummy: percent that are chest							
appliances	27.45%	36.47%	22.70%	34.83%	37.89%	30.14%	33.5%
Located in unconditioned space X							
CDD	0.78	1.55	1.97	0.88	0.99	0.88	1.08
ESTIMATED UEC (Annual kWh per							
year) for a removed freezer that is							
not replaced	1120.3	1095.4	983.9	1095.6	1148.7	1074.9	1109.1

When calculating default per unit kWh savings for a removed refrigerator or freezer, it is necessary to calculate and apply a "Part-Use" factor. "Part-use" is an appliance recycling-specific adjustment factor used to convert the UEC (determined through the methods detailed above) into an average per-unit deemed savings value. The UEC itself is not equal to the default savings value, because: (1) the UEC model yields an estimate of annual consumption, and (2) not all recycled refrigerators and freezers would have operated year-round had they not been decommissioned through the program.

In Program Year 3, the Commission determined that the average removed refrigerator was plugged in and used 96.9% of the year and the average freezer was plugged in and used 98.5% of the year. Thus, the default value for the part-use factor is 96.9% (and 98.5%) based on program year 3 data for all EDCs. EDCs may elect to calculate an EDC specific part-use factor for a specific program year. In the event an EDC desires to calculate an EDC specific part-use factor, EDCs should use the following methodology. Using participant surveys, evaluators should determine the amount of time a removed refrigerator is plugged in.

Table 2-79 Table 2-79 to

Table 2-80 Table 2-80 below shows the basis for the calculation of default per unit savings for refrigerators and freezers that are removed but then replaced.

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Table 2-79: Default values for Residential Refrigerator Recycling With Replacement With a New Energy Star Refrigerator¹⁹⁸

	Duquesne						West Penn
Variable Name	Light	Met Ed	PECO	PennElec	PennPower	PPL	Power
Age of Appliance	25.97	19.39	22.47	22.54	23.21	30.06	25.5
% manufactured pre 1990	44.70%	30.88%	37.85%	41.57%	41.52%	60.43%	42.54%
Appliance Size (volume in cubic							
feet)	18.39	20.65	19.57	18.45	18.73	18.07	18.82
Dummy: % that are single-door							
appliances	2.62%	0.61%	3.72%	3.01%	2.29%	7.03%	2.25%
Dummy: % that are side-by-side							
configuration	15.44%	35.79%	24.51%	19.83%	18.67%	18.31%	22.15%
Dummy: primary usage type (in							
absence of the program)	58.73%	88.55%	43.53%	67.13%	62.86%	26.01%	61.83%
Located in unconditioned space X							
CCD	0.55	0.18	2.4	0.47	0.54	0.92	0.52
Located in unconditioned space X							
HDD	4.41	1.11	9.25	4.15	4.5	9.85	4.27
ESTIMATED UEC Savings (Annual							
kWh per year) for a removed							
refrigerator that is replaced	11193.70	1342.5	1089.5	1186	1185.1	1107.2	1229.
kWh use of new refrigerator	482.00	482.00	482.00	482.00	482.00	482.00	482.0

 $\label{thm:condition} \textbf{Table 2-80: Default values for Residential Freezer Recycling With Replacement With a New Energy Star Freezer^{199} \\$

	Duquesne						West Penn
Variable Name	Light	Met Ed	PECO	PennElec	PennPower	PPL	Power
Age of Appliance	31.28	28.2	28.23	29.22	33.21	32.68	31.15
% manufactured pre 1990	80.90%	86.67%	70.42%	80.60%	91.43%	88.20%	77.95%
Appliance Size (volume in cubic							
feet)	16.12	16.67	16.15	16.41	16.29	15.65	16.97
Dummy: % that are chest appliances	31.46%	33.33%	14.79%	35.78%	42.86%	30.14%	41.00%
Located in unconditioned space X							
CCD	0.87	0.64	2.33	0.69	0.95	0.88	0.72
ESTIMATED UEC Savings (Annual							
kWh per year) for a removed							
refrigerator that is replaced	1059.80	1019.6	974.9	1031.1	1169.8	1074.9	1123.1
kWh use of new freezer	474.00	474.00	474.00	474.00	474.00	474.00	474.00

Per unit kW demand savings are based upon annual hours of use of 5,000 and a peak-coincidence factor of 62%.

Peak Demand Savings

Use the below algorithm to calculate the peak demand savings for any Refrigerator wither with or without replacement. Multiply the annual kWh savings by an Energy to Demand Factor (ETDF), which is supplied in Table 2-49 above.



¹⁹⁸ kWh use of new refrigerator is average consumption of all ENERGY STAR qualifying models by configuration from ENERGY STAR Residential Refrigerators Qualified Products List. July 5, 2013.

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¹⁹⁹ kWh use of new freezer is average consumption of all ENERGY STAR qualifying models by configuration from ENERGY STAR Residential Freezer Qualified Products List. July 5, 2013.

MEASURE LIFE

Refrigerator/Freezer Replacement programs: Measure Life = 7 yrs

Measure Life Rationale

The 2010 PA TRM specifies a Measure Life of 13 years for refrigerator replacement and 8 years for refrigerator retirement (Appendix A). It is assumed that the TRM listed measure life is either an Effective Useful Life (EUL) or Remaining Useful Life (RUL), as appropriate to the measure. Survey results from a study of the low-income program for SDG&E (2006)²⁰⁰ found that among the program's target population, refrigerators are likely to be replaced less frequently than among average customers. Southern California Edison uses an EUL of 18 years for its Low-Income Refrigerator Replacement measure which reflects the less frequent replacement cycle among low-income households. The PA TRM limits measure savings to a maximum of 15 yrs.

Due to the nature of a Refrigerator/Freezer Early Replacement Program, measure savings should be calculated over the life of the ENERGY STAR replacement unit. These savings should be calculated over two periods, the RUL of the existing unit, and the remainder of the measure life beyond the RUL. For the RUL of the existing unit, the energy savings would be equal to the full savings difference between the existing baseline unit and the ENERGY STAR unit, and for the remainder of the measure life the savings would be equal to the difference between a Federal Standard unit and the ENERGY STAR unit. The RUL can be assumed to be 1/3 of the measure EUL.

As an example, Low-Income programs use a measure life of 18 years and an RUL of 6 yrs (1/3*18). The measure savings for the RUL of 6 yrs would be equal to the full savings. The savings for the remainder of 12 years would reflect savings from normal replacement of an ENERGY STAR refrigerator over a Federal Standard baseline, as defined in the TRM.

Example Measure savings over lifetime

= 1205 kWh/yr * 6 yrs + 100 kWh/yr (ES side mount freezer w/door ice) * 12 yrs = 8430 kWh/measure lifetime

For non-Low-Income specific programs, the measure life would be 13 years and an RUL of 4 yrs (1/3*13). The measure savings for the RUL of 4 yrs would be equal to the full savings. The savings for the remainder of 9 years would reflect savings from normal replacement of an ENERGY STAR refrigerator over a Federal Standard baseline, as defined in the TRM.

Example Measure savings over lifetime

= 1205 kWh/yr * 4 yrs + 100 kWh/yr (ES side mount freezer w/door ice) * 9 yrs = 5720 kWh/measure lifetime

To simplify the programs and remove the need to calculate two different savings, a compromise value for measure life of 7 years for both Low-Income specific and non-Low Income specific programs can be used with full savings over this entire period. This provides an equivalent savings as the Low-Income specific dual period methodology for an EUL of 18 yrs and a RUL of 6 yrs.

Example Measure savings over lifetime

= 1205 kWh/yr * 7 yrs = 8435 kWh/measure lifetime

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²⁰⁰ 2004 - 2005 Final Report: A Measurement and Evaluation Study of the 2004-2005 Limited Income Refrigerator Replacement & Lighting Program, Prepared for: San Diego Gas & Electric, July 31, 2006

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- U.S. Department of Energy, draft Uniform Methods Project protocol titled "Refrigerator Recycling Evaluation Protocol", prepared by Doug Bruchs of the Cadmus Group, July 2012
- 2. Cadmus Memo August 20, 2012 Technical Memo from the Cadmus Group to the Michigan Evaluation Working Group on the topic of Appliance Recycling Measure Savings Study. This memo summarizes research on the energy savings of recycled refrigerators and freezers conducted by The Cadmus Group, Inc. and Opinion Dynamics (together known as the evaluation team) on behalf Consumers Energy (Consumers) and DTE Energy (DTE). This memo provides an overview of the research conducted and Cadmus' recommendations for deemed per-unit energy and demand savings values for affected measures in the Michigan Energy Measures Database (MEMD).
- 2009-2010 Pacific Power/Rocky Mountain Power Impact Evaluations PacifiCorp has impact evaluations for CA, ID, UT, WA, and WY that contain an earlier version of the multi-state Appliance Recycling Program regression models for both refrigerators and freezers. The Statewide Evaluator reviewed the report for the State of Washington, but all states include the same models and are publicly available online. The model coefficients can be found on pages 16 and 17 of the Washington document. http://www.pacificorp.com/content/dam/pacificorp/doc/Energy_Sources/Demand_Side_M anagement/WA 2011 SYLR Final Report.pdf
- 2010 Ontario Power Authority Impact Evaluation This evaluation report contains a regression equation for annual consumption for refrigerators only (the freezer sample was too small). That equation can be found on page 10 of the OPA evaluation report. See
 - http://www.powerauthority.on.ca/sites/default/files/new_files/2010/2010%20Residential%20Great%20Refrigerator%20Roundup%20Program%20Evaluation.pdf
- Efficiency Vermont; Technical Reference User Manual (TRM). 2008. TRM User Manual No. 2008-53. Burlington, VT 05401. July 18, 2008.
- Mid Atlantic TRM Version 2.0. July 2011. Prepared by Vermont Energy Investment Corporation. Facilitated and managed by Northeast Energy Efficiency Partnerships.
- 7. Based on program year 3 data for all EDCs.
- 8. ENERGY STAR Appliance Savings Calculator. Updated May 2014.
- 6-9. Assessment of Energy and Capacity Savings Potential In Iowa. Quantec in collaboration with Summit Blue Consulting, Nexant, Inc., A-TEC Energy Corporation, and Britt/Makela Group, prepared for the Iowa utility Association, February 2008. http://plainsjustice.org/files/EEP-08-1/Quantec/QuantecReportVol1.pdf

Field Code Changed

2.4.4 ENERGY STAR CLOTHES WASHERS

Measure Name	Clothes Washers
Target Sector	Residential Establishments
Measure Unit	Clothes Washer
Unit Energy Savings	Varies by Fuel Mix
Unit Peak Demand Reduction	0.0147 kWVaries by Fuel Mix
Measure Life	11 years ²⁰¹
Vintage	Replace on Burnout

This measure is for the purchase and installation of a clothes washer meeting ENERGY STAR eligibility criteria. ENERGY STAR clothes washers use less energy and hot water than non-qualified models.

ELIGIBILITY

This protocol documents the energy savings attributed to purchasing an ENERGY STAR clothes washer instead of a standard one. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. The target sector is residential.

ALGORITHMS

The general form of the equation for the ENERGY STAR Clothes Washer measure savings algorithm is:

Total Savings = Number of Clothes Washers \times Savings per Clothes Washer

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of clothes washers. The number of clothes washers will be determined using market assessments and market tracking.

Per unit energy and demand savings are given by the following algorithms:

$$\begin{split} \Delta kWh/yr &= \\ Cycles \times \left(\left[\frac{CAPY_{base}}{MEF_{base}} \times \left(CW_{base} + (DHW_{base} \times \%_{ElectricDWH}) + \left(Dryer_{base} \times \%_{ElectricDryer} \times \%_{dry/wash} \right) \right) \right] \\ &- \left[\frac{CAPY_{ee}}{MEF_{ee}} \times \left(CW_{ee} + (DHW_{ee} \times \%_{ElectricDWH}) + \left(Dryer_{ee} \times \%_{ElectricDryer} \times \%_{dry/wash} \right) \right) \right] \right) \end{split}$$

$$\begin{array}{ll} \Delta kW_{peak} & = DSav_{cw} \times CF \\ \Delta kW_{peak} & = \frac{\Delta kWh/yr}{Cycles \times time_{cycle}} \times CF \end{array}$$

Where MEF is the Modified Energy Factor, which is the energy performance metric for clothes washers. MEF is defined as:

²⁰¹ ENERGY STAR Calculator. Accessed July 10, 2013.

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MEF is the quotient of the capacity of the clothes container, C, divided by the total clothes washer energy consumption per cycle, with such energy consumption expressed as the sum of the machine electrical energy consumption, M, the hot water energy consumption, E, and the energy required for removal of the remaining moisture in the wash load, D. The higher the value, the more efficient the clothes washer is. 202

$$MEF = \frac{C}{(M+E+D)}$$

Note: As of March 7, 2015, new Federal Standards and ENERGY STAR specifications will become effective that use "IMEF" which incorporates energy used during low power modes. These new standards and specifications will be incorporated into the 2016 TRM. In the current (2015) TRM, clothes washers carrying an IMEF rating can be substituted in the above algorithms in place of MEF when using the EDC data gathering option for MEF ee.

DEFINITION OF TERMS

As of February 1, 2013 a clothes washer must have a MEF \geq 2.0 and a WF \leq 6.0 to meet ENERGY STAR standards. WF is the Water Factor, which is the measure of water efficiency of a clothes washer, expressed in gallons per cubic feet. WF is the quotient of the total weighted percycle water consumption divided by the capacity of the clothes washer. 203

The federal standard for a clothes washer must have a MEF \geq 1.26 and WF \leq 9.5. ²⁰⁴

The default values for the terms in the algorithms are listed in <u>Table 2-81Table 2-81</u>. If unit information is known (such as capacity, MEF, fuel mix) then actual values should be used.

Table 2-81: ENERGY STAR Clothes Washers - References

	Term	Type <u>Unit</u>	Value	Source
	<i>CAPY</i> _{base} , Capacity of baseline clothes washer	Fixed ft ³	3.10	1
	CAPY _{EE} , Capacity of ENERGY STAR clothes washer	ft³ Variable	EDC Data Gathering	EDC Data Gathering
•			Default: 3.10	2
	<i>MEF</i> _{base} , Modified Energy Factor of baseline clothes washer_	$\frac{ft^3}{(kWh/cycle)}$	1.26	1
	MEF _{EE} , Modified Energy Factor of ENERGY STAR clothes washer (can also	Variable ft3	EDC Data Gathering	EDC Data Gathering
	use IMEF, which has same units)	(kWh/cycle)	Default: 2.00	2

²⁰² Definition provided on ENERGY STAR Clothes Washers Key Product Criteria website:

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http://www.energystar.gov/index.cfm?c=clotheswash.pr_crit_clothes_washers.

²⁰³ Based on ENERGY STAR Version 6.1 requirements, *ENERGY STAR Program Requirements Product Specification for Clothes Washers, Eligibility Criteria Version 6.1.* Updated February 15,2013.

http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/commercial_clothes_washers/ENERGY_STAR_Final_Version_6_Clothes_Washer_Specification.pdf

²⁰⁴ U.S. Department of Energy, Energy Efficiency and Renewable Energy Building Technologies Office, Residential Clothes Washers. Accessed November 2013. http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/39

Term	Type <u>Unit</u>	Value	Source
Cycles , Number of clothes washer cycles per year	cycles yr ^{Fixed}	265 250	3
CW _{base} , % of total energy consumption for baseline clothes washer mechanical operation	<u>%</u> Fixed	9%	4
CW _{EE} , % of total energy consumption for ENERGY STAR clothes washer_mechanical operation	<u>%</u> Fixed	9%	4
DHW _{base} , % of total energy consumption attributed tofer baseline clothes washer water heating	<u>%</u> Fixed	37%	4
DHW _{EE} , % of Ttotal energy consumption attributed tofer ENERGY STAR clothes washer water heating	<u>%</u> Fixed	22%	4
% _{ElectricDWH} , % of ∓total energy consumption <u>attributed tofer</u> ENERGY	Variable %	EDC Data Gathering	Appliance Saturation Studies
STAR clothes washer water heating		Default: 43%	3
Dryer _{base} , <u>% of</u> <u>T</u> total energy consumption for <u>baseline clothes washer</u> dryer operation <u>with baseline clothes</u> <u>washer</u>	Fixed%	54%	4
Dryer _{EE} , % of Ttotal energy consumption for ENERGY STAR clothes washer dryer operation with ENERGY STAR clothes washer	Fixed%	69%	4
% _{ElectricDryer} , Percentage of dryers that are electric	<u>%Variable</u>	EDC Data Gathering	Appliance Saturation Studies
are creente		Default: <u>76</u> 64%	<u>3</u> 5
% _{dry/wash} . Percentage of homes with a dryer that use the dryer every time clothes are washed	<u>%</u>	Default= 95% Or EDC data gathering	<u>5</u>
time _{cycle} , average duration of a clothes washer cycle	hours	1	<u>6</u>

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Term	Typ e <u>Unit</u>	Value	Source
DSav _{GW-T} Summer demand savings per- purchased ENERGY STAR clothes- washer. ²⁰⁵	FixedNone_	0.0147	6
CF , Demand Coincidence Factor. The coincidence of average clothes washer demand to summer system peak	<u>%</u> Fixed	4 <u>2.9%</u>	7

DEFAULT SAVINGS

The default values for various fuel mixes are given in

Table 2-82

Table 2-82

Table 2-82: Default Clothes Washer Savings

Fuel Mix	∆kWh/yr ∆kWh	ΔkW_{peak}
Electric DHW/Electric Dryer	215 237.8	0.02602
Electric DHW/Gas Dryer	159 <u>172.6</u>	<u>0.01889</u>
Gas DHW/Electric Dryer	55 86.9	0.00951
Gas DHW/Gas Dryer	19 21.7	0.00238
Default (4317% Electric DHW 7664% Electric Dryer)	79 <u>136.2</u>	0.01490

MEASURE LIFE

ENERGY STAR Clothes Washer: Measure Life = 11 years.206

FUTURE STANDARDS CHANGES

As of March 7, 2015 new federal minimum efficiency standards for clothes washers will take effect. Further efficiency standards for top-loading clothes washers go into effect beginning January 1, 2018. The 2015 efficiency standards for front-loading clothes washers will continue to be effective in 2018. The efficiency standards and the effective TRM in which these standards become the baseline are detailed in Table 2-83Table 2-83.

Note that the current standards are based on the MEF and WF, but beginning_in 3/7/2015 the standards will be based on the Integrated Modified Energy Factor (IMEF) and Integrated Water Factor (IWF). The IMEF incorporates energy use in standby and off modes and includes updates to the provisions of per-cycle measurements. The IWF more accurately represents consumer usage patterns as compared to the current metric. ²⁰⁷

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²⁰⁵ Further research to update this value and CF is planned for 2015 TRM update.

ENERGY STAR Calculator . Accessed July 10, 2013.

²⁰⁷ Ibid.

These standards are effective for both compact—and standard-size clothes washers. A compact clothes washer is defined to have a capacity of less than 1.6 ft3 and a standard-size clothes washer has a capacity of 1.6 ft3 or greater.²⁰⁸

The corresponding ENERGY STAR updates do not include Compact washers, so these will not be included in the measure.

Table 2-83: Future Federal Standards for Clothes Washers²⁰⁹

l		2016 2015 TRM		2018	TRM
		Minimum IMEF	Maximum IWF	Minimum IMEF	Maximum IWF
	Top-loading, Compact	0.86	14.4	1.15	12.0
	Top-loading, Standard	1.29	8.4	1.57	6.5
1	Front-loading, Compact	1.13	8.3	N/A	A
	Front-loading, Standard	1.84	4.7	N/.	A

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Energy Star Calculator, EPA research on available models. Accessed June 2013
- Energy_Star Calculator, Average MEF and capacity of all ENERGY STAR qualified clothes washers. Accessed June 2013
- 3. Statewide average for all housing types from Pennsylvania Statewide Residential BaselineEnd-Use and Saturation-Study, 20142.
- 4. The percentage of total consumption that is used for the machine, water heating and dryer varies with efficiency. Percentages were developed using the above parameters and using the U.S. Department of Energy's Life-Cycle Cost and Payback Period tool, available at: http://www1.eere.energy.gov/buildings/appliance_standards/residential/clothes_washers_support_stakeholder_negotiations.html
- EIA 2009 Residential Energy Consumption Survey (RECS) appliance data for the state of Pennsylvania. http://www.eia.gov/consumption/residential/index.cfm
- 6. Energy and water savings based on Consortium for Energy Efficiency estimates. Assumes 75% of participants have gas water heating and 60% have gas drying (the balance being electric). Demand savings derived using NEEP screening clothes washer load shape.
- 2011-04 Technical Support Document: Energy Efficiency Program for Consumer Products and Commercial and Industrial Equipment. Residential Clothes Dryers and

²⁰⁸ U.S. Department of Energy. 10 CFR Parts 429 and 430. Energy Conservation Program: Energy Conservation Standards for Residential Clothes Washers. Direct Final Rule.

²⁰⁹ U.S. Department of Energy. 10 CFR Parts 429 and 430. *Energy Conservation Program: Energy Conservation Standards for Residential Clothes Washers*. Direct Final Rule.

Room Air Conditioners, Chapter 7. Clothes Dryer Frequency from Table 7.3.3 for Electric Standard.

http://www.regulations.gov/contentStreamer?objectId=0900006480c8ee11&disposition=a ttachment&contentType=pdf

- Assumption used in 2014 Illinois TRM and 2014 Mid Atlantic TRM.
- 7. Value from Clothes Washer Measure, Mid Atlantic TRM 2014. Metered data from Navigant Consulting "EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program." March 21, 2014, page 36.

Field Code Changed

Rev Date: June 2015

Appliances

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2.4.5 ENERGY STAR DRYERS

Measure Name	ENERGY STAR Clothes Dryers
Target Sector	Residential
Measure Unit	Clothes Dryer
Unit Energy Savings	Varies by Dryer type
Unit Peak Demand Reduction	Varies by Dryer type
Measure Life	13 years ²¹⁰

ENERGY STAR Clothes Dryers are more efficient than standard ones, and thus save energy. They have a higher CEF (Combined Energy Factor) and may incorporate a moisture sensor to reduce excessive drying of clothes and prolonged drying cycles.

ELIGIBILITY

This protocol documents the energy savings attributed to purchasing a vented ENERGY STAR Dryer that meets or exceeds the CEF_{ee} requirement in Table 2-84 Table 2-84 Table X-2 instead of a standard dryer. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. 211 The target sector is residential.

ALGORITHMS

The energy savings are obtained through the following formulas:

$$\Delta kWh/yr = Cycles_{wash} \times \%_{dry/wash} \times Load_{avg} \times \left(\frac{1}{CEF_{base}} - \frac{1}{CEF_{ee}}\right)$$

$$= \frac{\left(\frac{1}{CEF_{base}} - \frac{1}{CEF_{ee}}\right) \times Load_{avg}}{time_{cycle}} \times CF$$

DEFINITION OF TERMS

The parameters in the above equation are listed in <u>Table 2-84</u>Table 2-84.

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²¹⁰ From National Association of Home Builders Study Table 1, 2007. http://www.nahb.org/fileUpload_details.aspx?contentID=99359

²¹¹ The Pennsylvania SWE and PUC TUS staff added this condition relating to certification that has been applied for but not yet received at the request of several of the Pennsylvania EDCs. EDCs will be responsible for tracking whether certification is granted.

Table 2-84: Calculation Assumptions for ENERGY STAR Clothes Dryers

(Component	Unit	Values	Source
	Cycles _{wash} , Number of washing machine cycles per year	cycles/yr	250 cycles/year ²¹²	1
	$Load_{avg}$, Weight of average dryer load, in pounds per load	lbs/load	Standard Dryer: 8.45 lbs/load ²¹³ Compact Dryer: 3.0 lbs/load ²¹⁴	2, 3
t	$k_{ m dry/wash}$, Percentage of homes with a dryer hat use the dryer every time clothes are washed	%	95% Or EDC data gathering	3
	CEF _{base} , Combined Energy Factor of baseline dryer, in lbs/kWh	lbs/kWh	Table 2-85Table 2-85	4
	CEF _{ee} , Combined Energy Factor of ENERGY STAR dryer, in lbs/kWh	lbs/kWh	Table 2-85Table 2-85 or EDC Data Gathering	5
	ime_{cycle} , Duration of average drying cycle in nours	hours	Default: 1 hour ²¹⁵ or EDC Data Gathering	Assumption
(CF , Coincidence Factor	%	4.2%	6

Table 2-85: Combined Energy Factor for baseline and ENERGY STAR units

Product Type	CEF _{base} (lbs/kWh)	CEF _{ee} (lbs/kWh)
Vented Electric, Standard (4.4 ft³ or greater capacity)	3.73	3.93
Vented Electric, Compact (120V) (less than 4.4 ft³ capacity)	3.61	3.80
Vented Electric, Compact (240V) (less than 4.4 ft³ capacity)	3.27	3.45

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Appliances

²¹² Note: this matches the number of cycles for ENERGY STAR clothes washers ²¹³ This figure is from the updated DOE Appendix D2 test procedure and is used in absence of a source of real world behavioral

²¹⁵ This figure is used in absence of real world behavioral data.

DEFAULT SAVINGS

Table 2-86: Energy Savings and Demand Reductions for ENERGY STAR Clothes Dryers

Product Type	Energy Savings (kWh/yr)	Demand Reduction (kW)
Vented Electric, Standard (4.4 ft³ or greater capacity)	25.05	0.0048
Vented Electric, Compact (120V) (less than 4.4 ft³ capacity)	9.03	0.0017
Vented Electric, Compact (240V) (less than 4.4 ft³ capacity)	10.4	0.0020

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with calculation of energy and demand savings using above algorithms.

SOURCES

- Statewide average for all housing types from Pennsylvania Statewide Residential Baseline, 2014.
- Test Loads for Compact and Standard Dryer in Appendix D2 to Subpart B of Part 430— Uniform Test Method for Measuring the Energy Consumption of Clothes Dryers. http://www.ecfr.gov/cgi-bin/text-idx?SID=9d051184ada3b0d0b5b553f624e0ab05&node=10:3.0.1.4.18.2.9.6.14&rgn=div9
- 3. 2011-04 Technical Support Document: Energy Efficiency Program for Consumer Products and Commercial and Industrial Equipment. Residential Clothes Dryers and Room Air Conditioners, Chapter 7. Clothes Dryer Frequency from Table 7.3.3 for Electric Standard. http://www.regulations.gov/contentStreamer?objectId=0900006480c8ee11&disposition=attachment&contentType=pdf
- Federal Standard for Clothes Dryers, Effective January 1, 2015 http://www1.eere.energy.gov/buildings/appliance-standards/product.aspx/productid/36
- ENERGY STAR Specification for Clothes Dryers Version 1.0, Effective January 1, 2015. http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Draft%20Version%201.0%20Clothes%20Dryers%20Specification 0.pdf
- Central Maine Power Company. "Residential End-Use Metering Project". 1988. Using 8760 data for electric clothes dryers, calculating the CF according to the PJM peak definition.

2.4.6 ELECTRIC CLOTHES DRYER WITH MOISTURE SENSOR

Measure Name	Electric Clothes Dryer with Moisture Sensor
Target Sector	Residential Establishments
Measure Unit	Clothes Dryer
Unit Energy Savings	136 kWh
Unit Peak Demand Reduction	0.047 kW
Measure Life	13 years [±]
Vintage	Removed

Clothes dryers with drum moisture sensors and associated moisture-sensing controls achieve energy savings over clothes dryers that do not have moisture sensors.

ELIGIBILITY

This measure requires the purchase of an electric clothes dryer with a drum moisture sensor and associated moisture-sensing controls. ENERGY STAR currently does not rate or certify electric clothes dryers.

The TRM does not provide energy and demand savings for electric clothes dryers. The following sections detail how this measure's energy and demand savings were determined.

ALGORITHMS

Energy Savings

The annual energy savings of this measure was determined to be **136 kWh**. This value was based on the difference between the annu²¹⁶al estimated consumption of a standard unit without a moisture sensor as compared to a standard unit with a moisture sensor. This calculation is shown below:

$$4kWh$$
 = 905 - 769 = 136 kWh

The annual consumption of a standard unit without a moisture sensor (905 kWh) was based on 2008 estimates from Natural Resources Canada. 217

The annual consumption of a standard unit with a moisture sensor (769 kWh) was based on estimates from EPRI²¹⁸ and the Consumer Energy Center²¹⁹ that units equipped with moisture sensors (and energy efficient motors, EPRI) are about 15% more efficient than units without.

$$4kWh$$
 = 905 - (905 ×* 0.15) = 769 kWh

DEMAND SAVINGS

The demand savings of this measure was determined to be 0.346 kW. This value was based on the estimated energy savings divided by the estimated annual hours of use. The estimated

http://www.nahb.org/fileUpload_details.aspx?contentID=99359.

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²⁴⁶ NAHB, Study of Life Expectancy of Home Components, Feb. 2007, p. 7,

²¹⁷ Natural Resources Canada Report.pdf

²⁴⁸-EPRI Electric Clothes Dryer Report.pdf

²¹⁹ Natural Living Guide.pdf

annual hours of use was based on 392²²⁰ loads per year with a 1 hour dry cycle. This calculation is shown below:

$$\frac{kW_{avg}}{kW_{avg}} = \frac{\frac{136^{\frac{kWh}{yr}}}{yr}}{\frac{392^{\frac{hrs}{yr}}}{yr}} \frac{136^{\frac{kWh}{yr}}}{yr} / \frac{392^{\frac{hrs}{yr}}}{yr} = 0.346 \frac{kW}{yr}$$

The demand coincidence factor of this measure was determined to be **0.136**. This value was-based on the assumption that 5 of 7 loads are run on peak days, 5 of 7 days the peak can occur on, 1.07 loads per day (7.5 per week, Reference #4), 45 minutes loads, and 3 available daily peak-hours. This calculation is shown below:

CF =
$$(\frac{5}{2}5/7) \times (\frac{5}{2}5/7) \times (1.07) \times (0.75) \times (\frac{1}{2}1/3) = 0.136$$

The resulting demand savings based on this coincidence factor was determined to be **0.047 kW**. This calculation is shown below:

$$4kW_{peak} = 0.346 \times 0.136 = 0.047 \, kW$$

The assumptions used to determine this measure's net demand value are listed below:

Summer Annual Hours of Operation Assumption = 37.3% (May 2009 TRM)

MEASURE LIFE

Based on information developed by the National Association of Homebuilders and ENERGY-STAR, we have assumed the measure life of an electric clothes dryer to be 13 years.

EVALUATION PROTOCOL

The most appropriate evaluation protocol for this measure is verification of installation coupledwith assignment of stipulated energy savings.

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²²⁰ Energy Star Clothes Washer Calculator Assumptions.pdf

⁵ NAHB, Study of Life Expectancy of Home Components, Feb. 2007, p. 7.

⁶ ENERGY STAR Market & Industry Scoping Report – Residential Clothes Dryers, Nov. 2011, p. 4_

http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf.

2.4.72.4.6 FUEL SWITCHING: ELECTRIC CLOTHES DRYER TO GAS CLOTHES DRYER

Measure Name	Fuel Switch: Electric Clothes Dryer to Gas Clothes Dryer	
Target Sector	Residential Establishments	
Measure Unit	Fuel Switch: Electric Clothes Dryer to Gas Clothes Dryer	
Unit Energy Savings	875 kWh -2.99 MMBtu (increase in gas consumption)	
Unit Peak Demand Reduction	0.149 kW	
Measure Life	14 years ²²¹	

This protocol outlines the savings associated to purchasing a gas clothes dryers to replace an electric dryer. The measure characterization and savings estimates are based on average usage per person and average number of people per household. Therefore, this is a deemed measure with identical savings applied to all installation instances, applicable across all housing types.

ELIGIBILITY

This measure is targeted to residential customers that purchase a gas clothes dryer rather than an electric dryer.

ALGORITHMS

²²¹ DOE life-cycle cost and payback period Excel-based calculator.

http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/rcw_dfr_lcc_standard.xlsm

DEFINITION OF TERMS

Table 2-87 Electric Clothes Dryer to Gas Clothes Dryer - Values and Resources

Term	Unit	Values	Source
ΔkWh , Annual electricity savings, deemed	$\frac{kWh}{yr}$	EDC Data Gathering Default = 875	Calculated
kWh _{base} , Baseline annual electricity consumption of electric dryer, deemed	$\frac{kWh}{yr}$	EDC Data Gathering Default = 905	1
kWh_{gas} , Annual electricity consumption of gas dryer, deemed	kWh yr	EDC Data Gathering Default = 30	2
ΔMMBtu, Weighted average gas fuel increase	MMBtu	EDC Data Gathering Default = -2.99	Calculated, 3
0.003413, Conversion factor	MMBtu kWh	EDC Data Gathering Default = 0.003413	None
Cycles _{wash} , Number of washing machine cycles per year	cycles/yr	260	4
% _{dry/wash} , Percentage of homes with a dryer that use the dryer every time clothes are washed	%	95%	5
time _{cycle} , Duration of average drying cycle in hours	hours	EDC Data Gathering Default= 1	Assumption
CF, Coincidence Factor	%	EDC Data Gathering Default = 4.2 %	6

DEFAULT SAVINGS

Savings estimates for this measure are fully deemed and may be claimed using the algorithms above and the deemed variable inputs.

EVALUATION PROTOCOLS

The appropriate evaluation protocol is to verify installation and proper selection of deemed values.

SOURCES

- 1. Average annual dryer kWh without moisture sensor per 2014 PA TRM protocol 2.2 *Electric Clothes Dryer with Moisture Sensor*.
- 2011-04 Technical Support Document: Energy Efficiency Program for Consumer Products and Commercial and Industrial Equipment. Residential Clothes Dryers and

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Room Air Conditioners, Chapter 7. Median annual electricity consumption of gas dryers from Table 7.3.4: Electric Standard and Gas Clothes Dryer: Average Annual Energy Consumption

Levels by Efficiency http://www.regulations.gov/#!documentDetail;D=EERE-2007-BT-STD-0010-0053

- Negative gas fuel savings indicate increase in fuel consumption. It is assumed that gas
 and electric dryers have similar efficiencies. All heated air passes through the clothes and
 contributes to drying.
- 4. Statewide average for all housing types from Pennsylvania Statewide Residential End-Use and Saturation Study, 2014.
- 2011-04 Technical Support Document: Energy Efficiency Program for Consumer Products and Commercial and Industrial Equipment. Residential Clothes Dryers and Room Air Conditioners, Chapter 7. Clothes Dryer Frequency from Table 7.3.3 for Electric Standard.
 - http://www.regulations.gov/contentStreamer?objectId=0900006480c8ee11&disposition=a ttachment&contentType=pdf
- Central Maine Power Company. "Residential End-Use Metering Project". 1988. Using 8760 data for electric clothes dryers, calculating the CF according to the PJM peak definition.

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2.4.82.4.7 ENERGY STAR DISHWASHERS

Measure Name	Dishwashers	
Target Sector	Residential Establishments	
Measure Unit Dishwasher		
Unit Energy Savings	Varies by Water Heating Fuel Mix	
Unit Peak Demand Reduction	0.0225 kW Varies by Water Heating Fuel Mix	
Measure Life	10 years ²²²	
Vintage Replace on Burnout		

ELIGIBILITY

This measure is for the purchase and installation of a dishwasher meeting ENERGY STAR eligibility criteria. ENERGY STAR dishwashers use less energy and hot water than non-qualified models.

ALGORITHMS

The general form of the equation for the ENERGY STAR Dishwasher measure savings algorithm is:

Total Savings
$$=$$
 Number of Dishwashers \times Savings per Dishwasher

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of dishwashers. The number of dishwashers will be determined using market assessments and market tracking.

Per unit energy and demand savings algorithms for dishwashers utilizing electrically heated hot water:

$$\Delta \, kWh/yr = \, \left(kWh_{base} - kWh_{ee}\right) \, \times \, \left[\%kWh_{OP} + \left(\%kWh_{heat} \times \%Electric_{DHW}\right)\right]$$

$$\frac{\Delta kW}{\Delta kW_{peak}} = \frac{DSavDW \times CF}{HOU} \times CF$$

DEFINITION OF TERMS

Table 2-88: ENERGY STAR Dishwashers - References

Component	Unit	Value	Source
kWh _{base} , Annual energy consumption of baseline dishwasher	kWh/yr	355	1
kWh _{ee} , Annual energy consumption of ENERGY STAR qualified unit	kWh/yr	295	1

222 EnergyStar Calculator. Accessed July 2013.

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Component	Unit	Value	Source
%kWh _{op} , Percentage of unit dishwasher energy consumption used for operation	%	44%	1
%kWh _{heat} , Percentage of dishwasher unit energy consumption used for water heating	%	56%	1
%Electric _{DW} , Percentage of dishwashers assumed to utilize electrically heated hot water	%	EDC Data Gathering Default = 43%	2
HOU , Hours of use per year	hours/yr	234	3 ²²³
CF, Demand Coincidence Factor. The coincidence of average dishwasher demand to summer system peak	<u>%</u>	2.6%	4

ENERGY STAR qualified dishwashers must use less than or equal to the water and energy consumption values given in Table 2-89. Note, as of May 30, 2013, ENERGY STAR compact dishwashers have the same maximum water and energy consumption requirements as the federal standard and therefore are not included in the TRM since there is not energy savings to be calculated for installation of an ENERGY STAR compact dishwasher. A standard sized dishwasher is defined as any dishwasher that can hold 8 or more place settings and at least six serving pieces. 224

Table 2-89: Federal Standard and ENERGY STAR v 5.0 Residential Dishwasher Standard

	Federal St	andard ²²⁵	ENERGY S	STAR v 5.0 ²²⁶
Product Type	Water (gallons per cycle)	Energy (kWh per year)	Water (gallons per cycle)	Energy (kWh per year)
Standard	≤ 6.50	≤ 355	≤ 4.25	≤ 295

The default values for electric and non-electric water heating and the default fuel mix from

Table 2-88 are given in Table 2-90.

Table 2-90: Default Dishwasher Energy Savings

Water Heating	∆kWh/yr	ΔkW_{peak}
Electric (%Electric _{DHW} = 100%)	60.0	<u>0.00667</u>
Non-Electric (%Electric _{DHW} = 0%)	26.4	0.00293
Default Fuel Mix (%Electric _{DHW} = 43%)	40.8	0.00453

²²³ HOU=(3 loads/week)*(52 weeks/yr)*(1.5 hours/load). 3 load/week comes from 2014 Baseline study. 1.5 hours/load is assumption

Appliances

used by Efficiency Vermont and Illinois Statewide TRMs

224 Dishwashers Key Product Criteria. http://www.energystar.gov/index.cfm?c=dishwash.pr_crit_dishwashers

²²⁶ ENERGY STAR Program Requirements Product Specification for Residential Dishwashers.

http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/res_dishwashers/ES_V5_Dishwashers_Specification. pdf

MEASURE LIFE

ENERGY STAR Dishwashers: Measure Life = 10 years²²⁷

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. ENERGY STAR Appliances Calculator. Accessed July 2013.
- Statewide average for all housing types from Pennsylvania Statewide Residential <u>Baseline StudyEnd-Use and Saturation Study</u>, 201<u>4</u>2, <u>Demand savings derived using dishwasher load shape.</u>
- 3. Coincidence factor already embedded in summer peak demand reduction estimate
- 3. 2014 Pennsylvania Residential Baseline Study. Submitted by GDS Associates, April 2014.
- CF value for ENERGY STAR Dishwashers from Illinois Statewide TRM Version 2.0, June 2013.

²²⁷ EnergyStar Calculator. Accessed July 2013.

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2.4.92.4.8 ENERGY STAR DEHUMIDIFIERSS

Measure Name	Dehumidifiers
Target Sector	Residential Establishments
Measure Unit	Dehumidifier
Unit Energy Savings	Varies based on capacity
Unit Peak Demand Reduction	0.0098 kW
Measure Life	12 years ²²⁸
<u>Vintage</u>	Replace on Burnout

ENERGY STAR qualified dehumidifiers are 15 percent more efficient than non-qualified models due to more efficient refrigeration coils, compressors and fans.

ELIGIBILITY

State of Pennsylvania

This protocol documents the energy and demand savings attributed to purchasing an ENERGY STAR dehumidifier instead of a standard one. Dehumidifiers must meet ENERGY STAR Version 3.0 Product Specifications to qualify. The target sector is residential.

ALGORITHMS

The general form of the equation for the ENERGY STAR Dehumidifier measure savings algorithm is:

Total Savings
$$=$$
 Number of Dehumidifiers \times Savings per Dehumidifier

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of dehumidifiers. The number of dehumidifiers will be determined using market assessments and market tracking.

Per unit energy and demand savings algorithms:

$$\Delta kWh/yr = \left(\frac{CAPY \times 0.437 \frac{liters}{pint}}{24 \frac{hours}{day}}\right) \times HOU \times \left(\frac{1}{L/kWh_{base}} - \frac{1}{L/kWh_{ee}}\right)$$

$$\Delta kW_{peak} = DSavDH \times CF$$

$$\Delta kW_{peak} = \frac{\Delta kWh/yr}{HOU} \times CF$$

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²²⁸ EnergyS tar Calculator Accessed July 2013 using ENERGY STAR Appliances. February 2008. U.S. Environmental Protection Agency and U.S. Department of Energy. ENERGY STAR. http://www.energystar.gov/.

DEFINITION OF TERMS

Component	Unit	Value	Sources
CAPY , Average capacity of the unit	pints day	EDC Data Gathering	
HOU , Annual hours of operation	$\frac{hours}{yr}$	1632	1
$_{L/kWh_{base}}$, Baseline unit liters of water per kWh consumed	liters kWh	Table 2-91 Table 2-91, Federal Standard	<u>2</u>
L/kWh_{ee} , ENERGY STAR qualified unit liters of water per kWh consumed	liters kWh	EDC Data Gathering Default : Table 2-91 Table 2-91, ENERGY STAR	<u>3</u>
CF , Demand Coincidence Factor	%	<u>40.5%</u>	4

Table 2-91

Table 2.91 shows the federal standard minimum efficiency and ENERGY STAR standards, effective October 1, 2012. Federal standards do not limit residential dehumidifier capacity, but since ENERGY STAR standards do limit the capacity to 185 pints per day,

Table 2-91

Table 2-91_Table 2-59 only presents standards for the range of dehumidifier capacities that savings can be claimed.

Table 2-91: Dehumidifier Minimum Federal Efficiency and ENERGY STAR Standards

Capacity (pints/day)	Federal Standard (L/kWh_{base})	ENERGY STAR 230 (L/kWh_{ee})
≤ 35	1.35	
> 35 ≤ 45	1.50	≥ 1.85
>45 ≤ 54	1.60	2 1.00
>54 < 75	1.70	
75 ≤ 185	2.5	≥ 2.80

DEFAULT SAVINGS

The annual energy usage and savings of an ENERGY STAR unit over the federal minimum standard are presented in Table 2-92 for each capacity range.

SECTION 2: Residential Measures

Appliances

²²⁹ EnergyStar Calculator Accessed July 2013 using U.S. Department of Energy. Federal Register. 66th ed. Vol. 74. April 8, 2009. https://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/r24fr16040.pdf;

230 EnergyS tar Calculator Accessed July 2013 using ENERGY STAR Program Requirements Product Specification for

Dehumidifiers, Eligibility Criteria Version 3.0.

http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/dehumid/ES_Dehumidifiers_Final_V3.0_Eligibility_Cri teria.pdf?3cbf-7a48;

Table 2-92: Dehumidifier Default Energy Savings²³¹

Capacity Range (pints/day)	Default Capacity (pints/day)	Federal Standard (kWh/yr)	ENERGY STAR (kWh/yr)	ΔkWh/yr	ΔkW _{peak}
≤ 35	35	834	609	225	0.05584
> 35 ≤ 45	45	965	782	183	0.04541
>45 ≤ 54	54	1086	939	147	0.03648
>54 < 75	74	1,400	1,287	113	0.02804
75 ≤ 185	130	1.673	1.493	180	0.04467

MEASURE LIFE

ENERGY STAR Dehumidifiers: Measure Life = 12 years.232

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. ENERGY STAR Appliance Savings Calculator. Updated August, 2013.
- US Department of ENERGY Website. Appliance and Equipment Standards. Accessed June 2014. http://www1.eere.energy.gov/buildings/appliance standards/product.aspx/productid/55
- 3. ENERGY STAR Program Requirements Product Specification for Dehumidifiers, Eligibility Criteria Version 3.0. http://www.energystar.gov/ia/partners/prod/development/revisions/downloads/dehumid/E S Dehumidifiers Final V3.0 Eligibility Criteria.pdf?3cbf-7a48
- 4. Dehumidifier Metering in PA and Ohio by ADM from 7/17/2013 to 9/22/2013. 31 Units metered. Assumes all non-coincident peaks occur within window and that the average load during this window is representative of the June PJM days as well.

²³¹ Derived from equations in section 2.27, matching values generated by Energy Star Appliance Savings Calculator: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx

²³² EnergyS tar Calculator Accessed July 2013 using ENERGY STAR Appliances. February 2008. U.S. Environmental Protection-Agency and U.S. Department of Enegy. <u>ENERGY STAR</u>, http://www.energystar.gov/.

2.4.102.4.9 ENERGY STAR WATER COOLERS

Measure Name	ENERGY STAR Water Coolers
Target Sector	Residential Establishments
Measure Unit	Water Cooler
Unit Energy Savings	Cold Water Only: 47 kWh Hot/Cold Water: 361 kWh
Unit Peak Demand Reduction	0.0232 kW
Measure Life	10 years ²³³
Vintage	Replace on Burnout

This protocol estimates savings for installing ENERGY STAR Water Coolers compared to standard efficiency equipment in residential applications. The measurement of energy and demand savings is based on a deemed savings value multiplied by the quantity of the measure.

ELIGIBILITY

In order for this measure protocol to apply, the high-efficiency equipment must meet the ENERGY STAR_2.0 efficiency criteria: Cold Only & Cook & Cold Units ≤0.16 kWh_-hours/day, Hot & Cold Storage Units ≤0.871.20 kWh_hours/day, and Hot & Cold On-Demand ≤0.18 kWh/day.

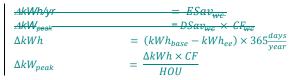
ALGORITHMS

The general form of the equation for the ENERGY STAR Water Coolers measure savings algorithms is:

Total Savings = Number of Water Coolers \times Savings per Water Cooler

To determine resource savings, the per unit estimates in the algorithms will be multiplied by the number of water coolers. Per unit savings are primarily derived from the May 2012 release of the ENERGY STAR calculator for water coolers.

Per unit energy and demand savings algorithms:



²³³ ENERGY STAR Water Coolers Savings Calculator (Calculator updated: May 2012)

DEFINITION OF TERMS

Table 2-93: ENERGY STAR Water Coolers - References

Component	Туре	Value	Sources
kWh _{base} , Energy use of baseline water cooler, kWh/day	kWh/day	Cold Only: 0.29 Hot & Cold: 2.19	1
kWh _{ee} , Energy use of ENERGY STAR water cooler, kWh/day	kWh/day	Cold Only: 0.16 Hot & Cold: 1.20 Hot & Cold Storage: 0.87 Hot & Cold On-Demand: 0.18 or EDC Data Gathering	2
HOU, Annual hours of use	Hours/year	8760	3
CF , Coincidence Factor	%	100%	3

DEFAULT SAVINGS

Table 2-94: Default Savings for ENERGY STAR Water Coolers

Cooler Type	ΔkWh	$\Delta oldsymbol{kW}_{peak}$
Cold Only	<u>47.5 kWh</u>	<u>0.0054 kW</u>
Hot & Cold Storage	481.8 kWh	<u>0.055 kW</u>
Hot & Cold On-Demand	733.65 kWh	0.0838 kW

MEASURE LIFE

The measure life for an ENERGY STAR Water Coolers is 10 years 234.

SOURCES

- ENERGY STAR Water Coolers Savings Calculator (Calculator updated: May 20132). Default values were used.
- 2. Average demand savings are estimated by taking an average of the energy savings values for both types of water coolers and dividing it by 8,760 hours usage. ENERGY STAR Product Specifications for Water Coolers Version 2.0. http://www.energystar.gov/products/specs/system/files/WC%20V2%200%20Final%20Program%20Requirements.pdf
- 2-3. Engineering Assumption used in 2014 Pennsylvania TRM for Refrigerator and Freezer $\frac{kW_{peak}}{k}$

SECTION 2: Residential Measures

²³⁴ ENERGY STAR Water Coolers Savings Calculator (Calculator updated: May 2012)

2.4.112.4.10 ENERGY STAR CEILING FANS

Measure Name	ENERGY STAR Ceiling Fans
Target Sector	Residential
Measure Unit	Ceiling Fan Unit
Unit Energy Savings	Varies by Ceiling Fan Type
Unit Peak Demand Reduction	Varies by Ceiling Fan Type
Measure Life	20 years for fan ²³⁵ , See Section 2.292.1 for lighting

ENERGY STAR ceiling fans require a more efficient CFM/Watt rating at the low, medium, and high settings than standard ceiling fans as well ENERGY STAR qualified lighting for those with light kits included. Both of these features save energy compared to standard ceiling fans.

ELIGIBILITY

This protocol documents the energy savings attributed to installing an ENERGY STAR Version 3.0 ceiling fan (with or without a lighting kit) in lieu of a standard efficiency ceiling fan. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. The target sector primarily consists of single-family residences.

ALGORITHMS

The total energy savings is equal to the savings contribution of the fan plus the savings contribution of the lighting, if applicable. If the ENERGY STAR fan does not include a lighting kit, then $\Delta kWh_{\rm lighting}=0$. These algorithms do not seek to estimate the behavioral change attributable to the use of a ceiling fan vs. a lower AC setting.

The energy savings are obtained through the following formula:

$$\begin{array}{ll} \varDelta kWh/yr_{total} &= \varDelta kWh_{fan} + \varDelta kWh_{lighting} \\ = \left[\left(\%_{low} \times (Low_{base} - Low_{ee})\right) + \left(\%_{med} \times (Med_{base} - Med_{ee})\right) \right. \\ & \left. + \left(\%_{high} \times (High_{base} - High_{ee})\right)\right] \times \frac{1\ kW}{1000\ W} \times HOU_{fan} \times 365\ \frac{days}{yr} \\ = \varDelta kWh_{lighting} &= \varDelta kWh\ \text{from Section 2.1: Ceiling Fan with ENERGY STAR Light} \end{array}$$

Fixture

Demand savings result from the lower connected load of the ENERGY STAR fan and ENERGY STAR lighting. Peak demand savings are estimated using a Coincidence Factor (CF).

$$\Delta kW_{peak,total}$$
 = $\Delta kW_{peak,fan} + \Delta kW_{peak,lighting}$

SECTION 2: Residential Measures

 $^{^{\}rm 235}$ Residential and C&I Lighting and HVAC Report Prepared for SPWG, 2007. Pg C-2.

²³⁶ The Pennsylvania SWE and PUC TUS staff added this condition relating to certification that has been applied for but not yet received at the request of several of the Pennsylvania EDCs. EDCs will be responsible for tracking whether certification is granted.

$$\begin{split} \Delta kW_{peak,fan} \\ &= \left[\left(\%_{low} \times (Low_{base} - Low_{ee}) \right) + \left(\%_{med} \times (Med_{base} - Med_{ee}) \right) \right. \\ &\left. + \left(\%_{high} \times (High_{base} - High_{ee}) \right) \right] \times \frac{1 \ kW}{1000 \ W} \times CF_{fan} \end{split}$$

 $\Delta kW_{peak,lighting} = \Delta kW_{peak}$ from Section 2.1: Ceiling Fan with ENERGY STAR Light Fixture

DEFINITION OF TERMS

The parameters in the above equations are listed in Table 2-95.

Table 2-95: Calculation Assumptions for ENERGY STAR Ceiling Fans

Component	Unit	Values	Source
$\%_{low}$, percentage of low setting use	%	40%	1
$\%_{med}$, percentage of medium setting use	%	40%	1
$\%_{high}$, percentage of high setting use	%	20%	1
Low _{base} , Wattage of low setting, baseline	Watts	15 Watts	1
$\mathit{Med}_{\mathit{base}}$, Wattage of medium setting, baseline	Watts	34 Watts	1
High _{base} , Wattage of high setting, baseline	Watts	67 Watts	1
Low_{ee} , Wattage of low setting, ENERGY STAR	Variable	EDC Data Gathering Default: 4.8 Watts ²³⁷	2, 3
Med_{ee} , Wattage of medium setting, ENERGY STAR	Variable	EDC Data Gathering Default: 18.2 Watts ²³⁸	2, 3
High_{ee} , Wattage of high setting, ENERGY STAR	Variable	EDC Data Gathering Default: 45.9 Watts ²³⁹	2, 3
$\mathit{HOU}_{\mathit{fan}}$, fan daily hours of use	Variable	EDC Data Gathering Default: 3.0 hours/day ²⁴⁰	1
CF_{fan} , Demand Coincidence Factor	Variable	EDC Data Gathering Default: 9.1% ²⁴¹	4
CF _{lighting} , Demand Coincidence Factor	Fixed	See Section 2.1	4

DEFAULT SAVINGS

SECTION 2: Residential Measures

²³⁷ Equals the minimum 'Low' CFM required by ENERGY STAR divided by the average CFM/Watt on 'Low' for all fans on ES qualified products list
²³⁸ Equals the minimum 'Medium' CFM required by ENERGY STAR divided by the average CFM/Watt on 'Medium' for all fans on

²³⁸ Equals the minimum 'Medium' CFM required by ENERGY STAR divided by the average CFM/Watt on 'Medium' for all fans or ES qualified products list

²³⁹ Equals the minimum 'High' CFM required by ENERGY STAR divided by the average CFM/Watt on 'High' for all fans on ES qualified products list

 $^{^{240}}$ The 3 hour/day for a ceiling fan is assumed here to be the same hours of use as a typical residential lightbulb, in absence of better data.. EDCs are allowed to do research on hours of use for ceiling fans in lieu of using the 3 hours/day figure. It is likely that the hours of use for ceiling fans is different than that of residential lighting.

²⁴¹ Assumed same usage characteristics as lighting, same assumption as Minnesota TRM. EDCs are allowed to do research on Coincidence Factor for ceiling fans in lieu of using the 3 hours/day figure. It is likely that the hours of use for ceiling fans is different than that of residential lighting.

Table 2-96: Energy Savings and Demand Reductions for ENERGY STAR Ceiling Fans

Product Type	Energy Savings (kWh)	Demand Reduction (kW)
Fan Only	16.0	0.00132

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with calculation of energy and demand savings using above algorithms.

SOURCES

- 1. ENERGY STAR Lighting Fixture and Ceiling Fan Calculator. Updated September, 2013.
- 2. ENERGY STAR Ceiling Requirements Version 3.0
- 3. ENERGY STAR Certified Ceiling Fan List, Accessed April 3, 2014.
- 4. EmPOWER Maryland 2012 Final Evaluation Report: Residential Lighting Program, Prepared by Navigant Consulting and the Cadmus Group, Inc., March 2013, Table 50.

2.5 CONSUMER ELECTRONICS

2.5.1 ENERGY STAR TELEVISIONS

Measure Name	ENERGY STAR Televisions
Target Sector	Residential Establishments
Measure Unit	Television Unit
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	6 years ²⁴²
Vintage	Replace on Burnout

ENERGY STAR certified televisions are on average over 25 percent more energy efficient than conventional models, saving energy in all usage modes: sleep, idle, and on.

ELIGIBILITY

This measure applies to the purchase of an ENERGY STAR TV meeting Version 6.0 standards. Version 6.0 standards are effective as of June 1, 2013. Additionally, in 2012 ENERGY STAR introduced the ENERGY STAR Most Efficient designation, which recognizes the most efficient of the ENERGY STAR qualified televisions.

The baseline equipment is a TV meeting ENERGY STAR Version 5.3 requirements.

ALGORITHMS

Energy Savings (per TV):

$$\Delta kWh/yr = \left[\frac{\left(W_{base, \, active} - W_{ee, \, active}\right)}{1000\frac{W}{kW}} \times HOU_{active} \times 365\frac{days}{yr}\right]$$

Coincident Demand Savings (per TV):

$$\Delta k W_{peak} = \frac{\sqrt{(W_{base,active} - W_{ee,active})}}{1000 \frac{W}{kW}} \times CF$$

Savings calculations are based on power consumption while the TV is in active mode only, as requirements for standby power are the same for both baseline and new units.

SECTION 2: Residential Measures

²⁴² ENERGY STAR Program Requirements for Televisions, Partner Commitments Versions 6.0, accessed June 2013, http://www.energystar.gov/products/specs/system/files/Final%20Version%206%200%20TV%20Program%20Requirements.pdf.

DEFINITION OF TERMS

Table 2-97: ENERGY STAR TVs - References

Component	TypeUnit	Value	Source
$HOURS_{ort}HOURS_{active}$, number of hours per day that a typical TV is on (active mode turned on and in use	$\frac{hours}{day}$ Fixed	5	<u>1</u> 2
Wbase.activeen, power use (in Watts) of baseline TV while in on mode (i.e. active mode turned on and operating).	<u>Watts</u>	See Table 2-98 Table 2-98 2-98	2
Wee, active.on, Power use of ENERGY STAR Version 6.0 or ENERGY STAR Most Efficient TV while in on mode (i.e. active mode turned on and operating)	<u>W</u> atts	See Table 2-98Table 2-98	<u>3</u>
CF, Demand Coincidence Factor	Fixed %	0.28 17%	<u>4</u>

ON MODE POWER CONSUMPTION REQUIREMENTS

$$P_{on\ max} = 100 \times \{TANH[0.00085(A - 140) + 0.052]\} + 14.1$$

Where:

- P_{on_max} is the maximum allowable On Mode Power consumption in Watts. All ENERGY STAR Televisions must use 1.0 watts or less while in Sleep Mode (i.e. standby mode).
- A is the viewable screen area of the product in sq. inches, calculated by multiplying the viewable image width by the viewable image height
- tanh is the hyperbolic tangent function

ENERGY STAR Most Efficient Televisions must meet all of the program requirements of ENERGY STAR Version 6.0 as well as the following additional requirement: 245

 $P_{ON MAX} = 82 \times TANH[0.00084(A - 150) + 0.05] + 12.75$

Where TANH is the hyperbolic tangent function.

²⁻Based on ENERGY STAR Version 5.3 requirements, from ENERGY STAR Program Requirements for Televisions, Partner Commitments, accessed November 2013.

http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/television/V5.3_Program_Requirements.pdf?db43-0cc6

 ²⁴⁴ ENERGY STAR Program Requirements for Televisions, Partner Commitments Versions 6.0, accessed June 2013.
 http://www.energystar.gov/products/specs/system/files/Final%20Version%206%200%20TV%20Program%20Requirements.pdf.
 http://www.energystar.gov/ia/partners/downloads/Televisions_Criteria_ME_2012.pdf

Table 2-98: TV power consumption

Diagonal Screen Size (inches) ²⁴⁶	Baseline Active Power Consumption [W _{base,active}] ²⁴⁷	ENERGY STAR V. 6.0 Active Power Consumption [W _{ES,active}] ²⁴⁸	ENERGY STAR Most Efficient Power Consumption [W _{ES,active}]
< 20	17	16	13
20 < 30	40	30	20
30 < 40	62	50	31
40 < 50	91	72	43
50 < 60	108*	92	54
≥ 60	108*	99	58

DEEMED SAVINGS

Deemed annual energy savings for ENERGY STAR 6.0 and ENERGY STAR Most Efficient TVs are given in <u>Table 2-99Table 2-99</u>.

Table 2-99: Deemed energy savings for ENERGY STAR Version 6.0and ENERGY STAR Most Efficient TVs.

Diagonal Screen Size (inches) ²⁴⁹	Energy Savings ENERGY STAR V. 6.0 TVs (kWh/year)	Energy Savings ENERGY STAR Most Efficient TVs (kWh/yr)
< 20	2	7
20 < 30	18	37
30 < 40	22	57
40 < 50	35	88
50 < 60	29	99
≥ 60	16	91

Coincident demand savings are given in the <u>Table 2-100Table 2-100Table 2-100</u>: <u>Deemed coincident demand savings for ENERGY STAR Version 6.0 and ENERGY STAR Most Efficient TVsTable 2-100</u>: <u>Deemed coincident demand savings for ENERGY STAR Version 6.0 and ENERGY STAR Most Efficient TVs</u>

SECTION 2: Residential Measures

²⁴⁶ Calculations are based on TV dimensions at the midpoint of the specified range. For example, a diagonal of 25" was used to compute values for the range of 20"-30". 15" was used to compute the value for sizes < 20".</p>
²⁴⁷ Based on ENERGY STAR Version 5.3 requirements, from ENERGY STAR Program Requirements for Televisions, Partner

²⁴⁷ Based on ENERGY STAR Version 5.3 requirements, from ENERGY STAR Program Requirements for Televisions, Partner Commitments, accessed November 2013,

http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/television/V5.3_Program_Requirements.pdf?db43-0c-6

²⁴⁸ Based on ENERGY STAR Version 6.0 requirements, from *ENERGY STAR Program Requirements for Televisions, Partner Commitments*, accessed November 2013,

 $[\]label{lem:http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/television/Fina_IDraft_Version_6_TVs_Specification.pdf?94ce-893a$

²⁴⁹ Calculations are based on TV dimensions at the midpoint of the specified range. For example, a diagonal of 25" was used to compute values for the range of 20"-30". 15" was used to compute the value for sizes < 20". 60" was used to compute the value for sizes ≥ 60"</p>

Table 2-100: Deemed coincident demand savings for ENERGY STAR Version 6.0 and ENERGY STAR Most Efficient TVs

Diagonal Screen Size (inches) ²⁵⁰	Coincident Demand Savings ENERGY STAR V. 6.0 (kW)	Coincident Demand Savings ENERGY STAR Most Efficient (kW)
< 20	0.00028 <u>0.00017</u>	0.0011 <u>0.00068</u>
20 < 30	0.0028 <u>0.0017</u>	0.0056 <u>0.00340</u>
30 < 40	<u>0.0034</u> 0.00204	0.0087 <u>0.00527</u>
40 < 50	0.00530.00323	0.013 4 <u>0.00816</u>
50 < 60	0.00450.00272	0.0151 <u>0.00918</u>
≥ 60	0.0025 <u>0.00153</u>	0.0140 <u>0.00850</u>

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Calculations assume TV is in on mode (or turned on) for 5 hours per day and sleep/standby mode for 19 hours per day. Based on assumptions from ENERGY STAR Calculator, 'EPA Research on Available Models, 2012, accessed June 2013, http://www.energystar.gov/ia/business/bulk purchasing/bpsavings calc/Calc Televisions Bulk xls
 - http://www.google.com/url?sa=t&rct=j&g=&esrc=s&source=web&cd=1&cad=rja&ved=0C DAQFjAA&url=http%3A%2F%2Fwww.energystar.gov%2Fia%2Fbusiness%2Fbulk_purch asing%2Fbpsavings_calc%2FConsumer_Electronics_Calculator.xlsx&ei=bzWyUbb0H4X x0wHw4oBw&usg=AFQjCNGPH4-NaXM_-1IM4J29of6Plpx5g&sig2=Xau5mB6YjLf3r81hOgmWAQ&bvm=bv.47534661,d.dmQ
- Based on ENERGY STAR Version 5.3 requirements, from ENERGY STAR Program
 Requirements for Televisions, Partner Commitments, accessed November 2013,
 http://www.energystar.gov/ia/partners/prod development/revisions/downloads/television/
 V5.3 Program Requirements.pdf?db43-0cc6
- 3. Based on ENERGY STAR Version 6.0 requirements, from ENERGY STAR Program Requirements for Televisions, Partner Commitments, accessed November 2013, http://www.energystar.gov/ia/partners/prod development/revisions/downloads/television/Fina IDraft Version 6 TVs Specification.pdf?94ce-893a
- 4.4.CF Value for Efficient Televisions in Efficiency Vermont TRM, 2013. The Efficiency Vermont Peak definition is June-August, 1-5PM non-holiday weekdays, close to the PJM peak definition.

SECTION 2: Residential Measures

²⁵⁰ Hbid. Calculations are based on TV dimensions at the midpoint of the specified range. For example, a diagonal of 25" was used to compute values for the range of 20"-30". 15" was used to compute the value for sizes < 20". 60" was used to compute the value for sizes ≥ 60"

2.5.2 ENERGY STAR OFFICE EQUIPMENT

Measure Name	ENERGY STAR Office Equipment
Target Sector	Residential Establishments
Measure Unit	Office Equipment Device
Unit Energy Savings	Table 2-102
Unit Peak Demand Reduction	Table 2-102
Measure Life	Computer: 4 years Monitor: 4 years Fax: 4 years Printer: 5 years Copier: 6 years Multifunction Device: 6 years
<u>Vintage</u>	Replace on Burnout

ELIGIBILITY

This protocol estimates savings for installing ENERGY STAR office equipment compared to standard efficiency equipment in residential applications. The measurement of energy and demand savings is based on a deemed savings value multiplied by the quantity of the measure. The target sector is primarily residential.

ALGORITHMS

The general form of the equation for the ENERGY STAR Office Equipment measure savings is:

Total Savings = Number of Units \times Savings per Unit

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of units. Per unit savings are primarily derived from the December 2010 release of the ENERGY STAR calculator for office equipment.

ENERGY STAR Computer

 $\Delta kWh/yr$ = $ESav_{COM}$ ΔkW_{peak} = $DSav_{COM}$

ENERGY STAR Fax Machine

 $\triangle kWh/yr$ = $ESav_{FAX}$ $\triangle kW_{peak}$ = $DSav_{FAX}$

ENERGY STAR Copier

 $\Delta kWh/yr$ = $ESav_{COP}$ ΔkW_{peak} = $DSav_{COP}$

SECTION 2: Residential Measures

ENERGY STAR Printer

 $\Delta kWh/yr$ = $ESav_{PRI}$ ΔkW_{peak} = $DSav_{PRI}$

ENERGY STAR Multifunction

 $\Delta kWh/yr$ = $ESav_{MUL}$ ΔkW_{peak} = $DSav_{MUL}$

ENERGY STAR Monitor

 $\triangle kWh/yr$ = $ESav_{MON}$ $\triangle kW_{peak}$ = $DSav_{MON}$

DEFINITION OF TERMS

Table 2-101: ENERGY STAR Office Equipment - References

Component	TypeUnit	Value	Sources
ESav _{COM} , Electricity savings per purchased ENERGY STAR computer.	FixedkWh/ yr	See Table 2-102	1
ESav _{FAX} , Electricity savings per purchased ENERGY STAR Fax Machine			
ESav _{COP} , Electricity savings per purchased ENERGY STAR Copier			
ESav _{PRI} , Electricity savings per purchased ENERGY STAR Printer			
ESav _{MUL} , Electricity savings per purchased ENERGY STAR Multifunction Machine			
ESav _{MON} , Electricity savings per purchased ENERGY STAR Monitor			
DSav _{COM} , Summer demand savings per purchased ENERGY STAR computer.	FixedkW/y	See Table 2-102	2
DSav _{FAX} , Summer demand savings per purchased ENERGY STAR Fax Machine			
DSav _{COP} , Summer demand savings per purchased ENERGY STAR Copier			
DSav _{PRI} , Summer demand savings per purchased ENERGY STAR Printer			
DSav _{MUL} , Summer demand savings per purchased ENERGY STAR Multifunction Machine			
DSav _{MON} , Monitor			
CF _{COM} : CF _{FAX} : CF _{COP} : CF _{PRI} : CF _{MUL} : CF _{MON} : Demand-Coincidence Factor (See Section 1.5). The-coincidence of average office equipment demand-to-summer system peak equals 1.18 for demand-impacts for all office equipment reflecting-embedded coincidence in the DSay factor.	Fixed <u>kW</u>	1.18	3

SECTION 2: Residential Measures

DEFAULT SAVINGS

Table 2-102: ENERGY STAR Office Equipment Energy and Demand Savings Values

Measure	Energy Savings (ESav)	Summer Peak Demand Savings (DSav)	Source
Computer	133 kWh/yr	0.018 kW	1
Fax Machine (laser)	78 kWh/yr	0.0105 kW	1
Copier (monochrome)			
1-25 images/min	73 kWh/yr	0.0098 kW	1
26-50 images/min	151 kWh/yr	0.0203 kW	
51+ images/min	162 kWh/yr	0.0218 kW	
Printer (laser, monochrome)			
1-10 images/min	26 kWh/yr	0.0035 kW	
11-20 images/min	73 kWh/yr	0.0098 kW	1
21-30 images/min	104 kWh/yr	0.0140 kW	
31-40 images/min	156 kWh/yr	0.0210 kW	
41-50 images/min	133 kWh/yr	0.0179 kW	
51+ images/min	329 kWh/yr	0.0443 kW	
Multifunction (laser, monochrome)			
1-10 images/min	78 kWh/yr	0.0105 kW	
11-20 images/min	147 kWh/yr	0.0198 kW	1
21-44 images/min	253 kWh/yr	0.0341 kW	
45-99 images/min	422 kWh/yr	0.0569 kW	
100+ images/min	730 kWh/yr	0.0984 kW	
Monitor	15 kWh/yr	0.0020 kW	1

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

1. ENERGY_STAR Office Equipment Calculator (Referenced latest version released in MayCalculator updated: December 2010). Default values were used. —Using a commercial office equipment load shape, the percentage of total savings that occur during the PJM peak demand period was calculated and multiplied by the energy savings.

SECTION 2: Residential Measures

2.5.3 SMART STRIP PLUG OUTLETS

Measure Name	Smart Strip Plug Outlets
Target Sector	Residential
Measure Unit	Per Smart Strip
Unit Energy Savings	48.9 kWh (5-plug, unspecified use or multiple purchased) 58.7 kWh (7-plug, unspecified use or multiple purchased) 62.1 kWh (5-plug, Entertainment Center) 74.5 kWh (7-plug, Entertainment Center)
Unit Peak Demand Reduction	0.005 <u>6</u> 4 kW (5-plug, unspecified use or multiple purchased) 0.006 <u>7</u> 4 kW (7-plug, unspecified use or multiple purchased) 0.00 <u>77</u> 68 kW (5-plug, Entertainment Center) 0.00 <u>92</u> 82 kW (7-plug, Entertainment Center)
Measure Life	4 years ²⁵¹
<u>Vintage</u>	Retrofit

Smart Strips are power strips that contain a number of controlled sockets with at least one uncontrolled socket. When the appliance that is plugged into the uncontrolled socket is turned off, the power strips then shuts off the items plugged into the controlled sockets.

ELIGIBILITY

This protocol documents the energy savings attributed to the installation of smart strip plugs. The most likely area of application is within residential spaces, i.e. single family and multifamily homes. The two areas of usage considered are home office systems and home entertainment systems. Power strips used with entertainment systems typically save more energy than power strips used with home office components. It is expected that approximately three to five items will be plugged into each 5-plug power strip, and that five to six items will be plugged into a 7-plug power strip.

ALGORITHMS

The energy savings and demand reduction were obtained through the following calculations using standard standby or low power wattages for typical entertainment center and home office components. If the intended use of the power strip is not specified, or if multiple power strips are purchased, the algorithm for "unspecified use should be applied". If it is known that the power strip is intended to be used for an entertainment center, the "entertainment center" algorithm should be applied:

```
      \Delta kWh/yr \ unspecified \ use \qquad = \frac{(kW_{comp} \times HOU_{comp}) + (kW_{TV} \times HOU_{Tv})}{2} \times 365 \frac{days}{yr} \qquad = 48.9 \ kWh \ (5-plug); \ 58.7  kWh \ (7-plug)
```

$$\Delta kWh/yr$$
 entertainment center = $kW_{TV} \times HOU_{Tv} \times 365 \frac{days}{yr}$ = 62.1 kWh (5-plug); 74.5 kWh (7-plug)

$$\Delta kW_{peak} \, unspecified \, use \qquad = \frac{CF \times (kW_{comp} + kW_{TV})}{2} \qquad = 0.00564 \, kW \, \, (5\text{-plug}); \, 0.006\underline{7}4 \, kW \, \, (7\text{-plug}) \\ \Delta kW_{peak} \, entertainment \, center \qquad = CF \times kW_{TV} \quad = 0.0077\underline{6}8 \, kW \, \, (5\text{-plug}); \, 0.00\underline{92}82 \, kW \, \, (7\text{-plug})$$

²⁵¹ Smart Strip Electrical Savings and Usability", David Rogers, Power Smart Engineering, October 2008.

SECTION 2: Residential Measures

DEFINITION OF TERMS

The parameters in the above equation are listed in

Table 2-103
Table 2-103

Table 2-103: Smart Strip Plug Outlet Calculation Assumptions

Parameter	Type <u>Unit</u>	Value	Source
<i>kW_{comp}</i> , Idle kW of computer system	Fixed <u>kW</u>	0.0049 (5-plug) 0.00588 (7-plug)	1,2,4
HOU_{comp} , Daily hours of Computer idle time	$\frac{hours}{day}$ Fixed	20	1
kW_{TV} , Idle kW of TV system	Fixed <u>kW</u>	0.0085 (5-plug) 0.0102 (7-plug)	1,4
HOU_{TV} , Daily hours of TV idle time	hours Fixed	20	1
CF , Coincidence Factor	Fixed%	0.80Entertainment Center: 90.0% Unspecified Use ²⁵² : 83.2%	3

DEEMED SAVINGS

∆kWh	= 48.9 kWh (5-plug power strip, unspecified use or multiple purchsed)
	58.7 kWh (7-plug power strip, unspecified use or multiple purchased)
	62.1 kWh (5-plug power strip, entertainment center)
1	74.5 kWh (7-plug power strip, entertainment center)
ΔkW_{peak}	= 0.005 <u>6</u> 4 kW (5-plug power strip, unspecified use or multiple purchase)
	0.006 <u>7</u> 4 kW (7-plug power strip, unspecified use, or multiple purchased)
1	0.007768 kW (5-plug power strip, entertainment center)
1	0.009282 kW (7 plug power strip, entertainment center)

MEASURE LIFE

EVALUATION PROTOCOLS

 $\underline{^{252}\,\text{CF for "unspecified use" is the average of the standby losses CF for Entertainment Center (90.0\%) and Home Office (76.3\%)}$ form Efficiency Vermont TRM.
"Smart Strip Electrical Savings and Usability", David Rogers, Power Smart Engineering, October 2008.

SECTION 2: Residential Measures

Consumer Electronics

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

- "Electricity Savings Opportunities for Home Electronics and Other Plug-In Devices in Minnesota Homes", Energy Center of Wisconsin, May 2010.
- 2. "Smart Plug Strips", ECOS, July 2009.
- CF Values of Standby Losses for Entertainment Center and Home Office in Efficiency Vermont TRM, 2013 Efficiency Vermont coincidence factor for smart strip measure—in the absence of empirical evaluation data, this was based on the assumptions of the typical run pattern for televisions and computers in homes.
- 4. "Advanced Power Strip Research Report", NYSERDA, August 2011.

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2.6 BUILDING SHELL

2.6.1 CEILING / ATTIC AND WALL INSULATION

Measure Name	Ceiling/Attic and Wall Insulation
Target Sector	Residential Establishments
Measure Unit	Insulation Addition
Unit Energy Savings	<u>Varies</u>
Unit Peak Demand Reduction	<u>Varies</u>
Measure Life	25 years ²⁵⁴
<u>Vintage</u>	Retrofit

ELIGIBILITY

This measure applies to installation/retrofit of new or additional insulation in a ceiling/attic, or walls of existing residential homes or apartment units in multifamily complexes with a primary electric heating and/or cooling source. The installation must achieve a finished ceiling/attic insulation rating of R-38 or higher, and/or must add wall insulation of at least an R-6 or greater rating.

The baseline for this measure is an existing residential home with a ceiling/attic insulation R-value less than or equal to R-30, and wall insulation R-value less than or equal to R-11, with an electric primary heating source and/or cooling source.

ALGORITHMS

The savings values are based on the following algorithms.

Cooling savings with central A/C:

$$\Delta kWh/yr_{CAC} = \frac{CDD \times 24 \frac{hr}{day} \times DUA}{SEER_{CAC} \times 1000 \frac{W}{kW}} \times \left[AHF \times A_{roof} \left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}}\right) + A_{wall} \left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}}\right)\right]$$

$$\Delta kW_{peak\text{-CAC}} = \frac{\Delta kWh_{CAC}}{EFLH_{cool}} \times CF_{CAC}$$

Cooling savings with room A/C:

$$\Delta kWh/yr_{RAC} = \frac{CDD \times 24 \frac{hr}{day} \times DUA \times F_{Room AC}}{\overline{EER}_{RAC} \times 1000 \frac{W}{kW}} \times \left[AHF \times A_{roof} \left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}}\right) + A_{wall} \left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}}\right)\right]$$

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²⁵⁴ Massachusetts Statewide Technical Reference Manual for Estimating Savings from Energy Efficiency Measures, Version 1.0, accessed August 2010 at http://www.ma-eeac.org/docs/091023-MA-TRMdraft.pdf. Note that PA Act 129 savings can be claimed for no more than 15 years.

$$\Delta kW_{peak\text{-RAC}}$$

$$\Delta kW_{peak\text{-}RAC} \qquad \qquad = \frac{\Delta kWh_{RAC}}{EFLH_{cool\,RAC}} \times CF_{RAC}$$

Cooling savings with electric air-to-air heat pump:

$$\Delta kWh/yr_{ASHP cool} = \frac{CDD \times 24 \frac{hr}{day}}{SFER \dots \times 1}$$

$$\Delta kWh/yr_{ASHP\;cool} = \frac{CDD \times 24 \frac{hr}{day} \times DUA}{SEER_{ASHP} \times 1000 \frac{W}{kW}} \times \left[AHF \times A_{roof} \left(\frac{1}{R_{roof,b\;l}} - \frac{1}{R_{roof,ee}}\right) + A_{wall} \left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}}\right)\right]$$

$$\Delta kW_{peak ext{-ASHP cool}}$$

$$= \frac{\Delta kWh_{ASHP\ cool}}{EFLH_{cool}} \times CF_{ASHP}$$

Cooling savings with electric ground source heat pump:

$$\begin{split} \Delta kWh/yr_{GSHP\;cool} &= \frac{CDD\times24\frac{hr}{day}\times DUA}{EER_{GSHP}\times GSHPDF\times GSER\times 1000\frac{W}{kW}}\times \left[AHF\right. \\ &\times A_{roof}\left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}}\right) + A_{wall}\left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}}\right) \right] \end{split}$$

$$\Delta kW_{peak ext{-}GSHP\ cool}$$

$$\Delta kW_{peak\text{-}GSHP\;cool} = \frac{\Delta kWh_{GSHP\;cool}}{EFLH_{cool}} \times CF_{GSHP}$$

Heating savings with electric ground source heat pump:

$$\Delta kWh/yr_{GSHP\ heat} = \frac{HDD\times24\frac{hr}{day}}{COP_{GSHP}\times GSHPDF\times GSOP\times1000\frac{W}{kW}} \times \left[A_{roof}\left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}}\right) + A_{wall}\left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}}\right)\right]$$

Heating savings with electric air-to-air heat pump:

$$\Delta kWh/yr_{ASHP\;heat} = \frac{HDD \times 24 \frac{hr}{day}}{HSPF_{ASHP} \times 1000 \frac{W}{klW}} \times \left[A_{roof} \left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}} \right) + A_{wall} \left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}} \right) \right]$$

$$\Delta kW_{peak-ASHP\ heat} = 0$$

Heating savings with electric baseboard or electric furnace heat (assumes 100% efficiency):

$$\Delta kWh/yr_{elec\ heat} = \frac{HDD \times 24 \frac{hr}{day}}{3412 \frac{Btu}{kWh}} \times \left[A_{roof} \left(\frac{1}{R_{roof,bl}} - \frac{1}{R_{roof,ee}} \right) + A_{wall} \left(\frac{1}{R_{wall,bl}} - \frac{1}{R_{wall,ee}} \right) \right]$$

$$\Delta kW_{\text{max}} = 0$$

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Building Shell

DEFINITION OF TERMS

The default values for each term are shown in $\frac{\text{Table 2-104}\text{Table 2-104}}{\text{Table 2-104}}$. The default values for heating and cooling days and hours are given in $\frac{\text{Table 2-104}\text{Table 2-104}}{\text{Table 2-104}}$.

Table 2-104: Default values for algorithm terms, Ceiling/Attic and Wall Insulation

Term	<u>Unit</u> Type	Value	Source
A _{roof} , Area of the ceiling/attic with upgraded insulation	Variableft ²	Varies	EDC Data Gathering
A _{wall} , Area of the wall with upgraded insulation	Variable ft ²	Varies	EDC Data Gathering
DUA, Discretionary Use Adjustment to account for the fact that people do not always operate their air conditioning system when the outside temperature is greater than 65F.	FixedNone	0.75	OH TRM ²⁵⁵ 1
AHF, Attic Heating Factor increases cooling load to home due to attic temperatures being warmer than ambient outdoor air temperature on sunny days.	FixedNone	1.056	13,14 <u>2, 3</u>
R _{roof,bl} ²⁵⁶ , Assembly R-value of	$\frac{{}^{\circ}F \cdot ft^2 \cdot hr}{Rtu} Variable$	5	Un-insulated attic
ceiling/attic before retrofit	вій	16	4.5" (R-13) of existing attic insulation
		22	6" (R-19) of existing attic insulation
		30	10" (R-30) of existing attic insulation
		Existing Assembly R-value	EDC Data Gathering
R _{roof,ee} ²⁵⁷ , Assembly R-value of ceiling/attic after retrofit	$\frac{{}^\circ\!F \cdot ft^2 \cdot hr}{Btu}$ Variable	38	Retrofit to R-38 total attic insulation
		49	Retrofit to R-49 total attic insulation
		Retrofit Assembly R- value	EDC Data Gathering
$R_{\it wall, bl}^{\it 258}$, Assembly R-value of wall before retrofit	**Frft**-hr Variable	Default = 5.0	15 Assumes existing, un-insulated wall with 2x4 studs @ 16" o.c.,

²⁶⁵ "State of Ohio Energy Efficiency Technical Reference Manual," prepared for the Public Utilities Commission of Ohio by Vermont Energy Investment Corporation. August 6, 2010.

SECTION 2: Residential Measures

²⁵⁶ Used eQuest 3.64 to derive roof assembly R-values. When insulation is added between the joists as in most insulation up to R-30 (10"), the assembly R-value is based on a parallel heat transfer calculation of the insulation and joists, rather than a series heat transfer.

transfer.

257 Generally as insulation is added beyond R-30 (10"), the insulation has cleared the joists and the R-value of the insulation above the joists can be added as a series heat transfer rather than a parallel heat transfer condition. Therefore, above R-30 insulation levels, the additional R-value can be added directly to the assembly value of R-30 insulation.

²⁵⁸ Used eQuest 6.64 to derive wall assembly R-values.

Term	<u>Unit</u> Type	Value	Source
			w/ wood/vinyl siding
		Existing Assembly R-value	EDC Data Gathering
$R_{wall,ee}^{}$, Assembly R-value of wall after retrofit	**Frft**hr Variable** **Btu** **Table** Btu** **Table** **Table	Default = 11.0	Assumes adding R-6 per DOE recommendations ²⁶⁰
		Retrofit Assembly R- value	EDC Data Gathering
SEER _{CAC} , Seasonal Energy Efficiency Ratio of existing home central air conditioner	$\frac{Btu}{W \cdot hr}$ Variable	Default for equipment installed before 1/23/2006 = 10	Minimum Federal Standard for new- Central Air- Conditioners/Heat-
		Default for equipment installed after 1/23/2006 = 13	Pumps between 1990- and 2006 ASHRAE 90.1-20074
		Nameplate	EDC Data Gathering
$\overline{\it EER}_{\it RAC}$, Average Energy Efficiency Ratio of existing room air conditioner	$\frac{Btu}{W \cdot hr}$ Variable	Default = 9.8	DOE Federal Test Procedure 10 CFR 430, Appendix F (Used in ES Calculator for baseline)
		Nameplate	EDC Data Gathering
SEER _{ASHP} , Seasonal Energy Efficiency Ratio of existing home air source heat pump	$\frac{Btu}{W \cdot hr}$ Variable	Default for equipment installed before 1/23/2006 = 10	Minimum Federal- Standard for new- Central Air-
		Default for equipment installed after 1/23/2006 = 13	Conditioners/Heat Pumps between 1990 and 2006
		Default for equipment installed after 6/1/2015 = 14	ASHRAE 90.1-20074
		Nameplate	EDC Data Gathering
HSPF _{ASHP} , Heating Seasonal Performance Factor for existing home heat pump	$\frac{Btu}{W \cdot hr}$ Variable	Default for equipment installed before 1/23/2006 = 6.8	4Minimum Federal- Standard for new- Central Air-
		Default for equipment installed after 1/23/2006 = 7.7	Conditioners/Heat- Pumps between 1990- and 2006
		Default for equipment installed after 6/1/2015 = 8.2	ASHRAE 90.1-20074
		Nameplate	EDC Data Gathering

 $^{^{259}}$ Used eQuest 6.64 to derive wall assembly R-values. It is coincidence that adding R-6 to a 2x4 stud wall essentially yields R-9 assembly value even though this was done using a parallel heat transfer calculation. This was due to rounding. The defaults are

SECTION 2: Residential Measures

Building Shell

based on conservative assumptions of wall construction.

260 DOE recommendation on ENERGY STAR website for adding wall insulation to existing homes in Zones 5-8. Insulation may be loose fill in stud cavities or board insulation beneath siding. http://www.energystar.gov/index.cfm?c=home_sealing.hm_improvement_insulation_table

Term	<u>Unit</u> Type	Value	Source
of existing home ground source heat pump		Source Heat Pump = 13.4 Default for Groundwater Source Heat Pump = 16.2	
GSER , Factor to determine the	None	Nameplate 1.02	EDC Gathering 6
SEER of a GSHP based on its EER	<u>None</u>	1.02	<u> </u>
COP _{GSHP} , Coefficient of Performance for existing home ground source heat pump	<u>None</u>	Default for Ground Source Heat Pump = 3.1 Default for Groundwater Source Heat Pump = 3.6	5
		<u>Nameplate</u>	EDC Gathering
GSOP , Factor to determine the HSPF of a GSHP based on its COP	$\frac{Btu}{W \cdot hr}$	3.413	7
GSHPDF , Ground Source Heat Pump De-rate Factor	<u>None</u>	0.885	(Engineering Estimate - See 2.2.1)
CF _{CAC} , Demand Coincidence Factor (See Section 1.4) for central AC systems	Fixed%	0.7064.7%	<u>8</u> Table 2-1
CF _{RAC} , Demand Coincidence Factor (See Section 1.4) for Room AC systems	<u>%</u> Fixed	0.58 30%	9See Section 2.29
CF _{ASHP} , Demand Coincidence Factor (See Section 1.5) for ASHP systems	<u>%</u> Fixed	0.7064.7%	8Table 2-1
<u>CF_{GSHP}</u> , <u>Demand Coincidence</u> <u>Factor for GSHP systems</u>	<u>%</u>	64.7%	8
F _{Room,AC} , Adjustment factor to relate insulated area to area served by Room AC units	Fixed None	0.38	Calculated ²⁶¹
<u>CDD</u> , Cooling Degree Days	°F · Days	Table 2-105 Table 2-105	10
HDD , Heating Degree Days	°F · Days	<u>Table 2-105</u> <u>Table 2-105</u>	10
EFLH _{cool} , Equivalent Full Load Cooling hours for Room AC	hours year	<u>Table 2-105</u> Table 2-105	11

From PECO baseline study, average home size = 2323 ft², average number of room AC units per home = 2.1. Average Room AC capacity = 10,000 BtuH per ENERGY STAR Room AC Calculator, which serves 425 ft² (average between 400 and 450 ft² for 10,000 BtuH unit per ENERGY STAR Room AC sizing chart). $F_{Room,AC} = (425 \text{ ft}^2 * 2.1)/(2323 \text{ ft}^2) = 0.38$

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Term	<u>Unit</u> Type	Value	Source
EFLH _{COOL RAC} , Equivalent Full Load Cooling hours for Central AC and ASHP	hours year	<u>Table 2-105</u> <u>Table 2-105</u>	12

Table 2-105: EFLH, CDD and HDD by City

City	EFLH _{cool} (Hours) ²⁶²	EFLH _{cool RAC} (Hours) ²⁶³	CDD (Base 65) ²⁶⁴	HDD (Base 65) ²⁶⁵
Allentown	487	243	787	5830
Erie	389	149	620	6243
Harrisburg	551	288	955	5201
Philadelphia	591	320	1235	4759
Pittsburgh	432	228	726	5829
Scranton	417	193	611	6234
Williamsport	422	204	709	6063

Alternate EFLH values from Table 2-12 and Table 2-13 in Section 2.1 may also be used for central airc conditioners and air source heat pumps. The tables show cooling EFLH and heating EFLH, respectively, by city and for each EDC's housing demographics. EFLH values are only shown for cities that are close to customers in each EDC's service territory. In order to determine the most appropriate EFLH value to use for a project, first select the appropriate EDC, then, from that column, pick the closest city to the project location. The value shown in that cell will be the EFLH value to use for the project.

MEASURE LIFE

Measure life = 25 years 266.

ATTIC HEATING EFFECT ON COOLING LOADS

On sunny days, attic temperatures can be 20%-35% higher than ambient outdoor air temperatures during the 7 hours between 9 AM and 4 PM and 6%-8% higher for the 4 hours from 7 AM to 9 AM and 4 PM to 6 PM.¹³ The remaining 13 hours of the day there was no significant

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²⁶² PA TRM Table 2-1.

²⁶³ PA TRM Section 2.12 Room AC Retirement

²⁶⁴ Climatography of the United States No. 81. Monthly Station Normals of Temperature, Precipitation, and Heating and Cooling-Degree Days 1971-2000, 36 Pennsylvania. NOAA. http://cdo.ncdc.noaa.gov/climatenormals/clim81/PAnorm.pdf
²⁶⁵ Ibid.

²⁶⁶ Massachusetts Statewide Technical Reference Manual for Estimating Savings from Energy Efficiency Measures, Version 1.0, accessed August 2010 at http://www.ma-eeac.org/decs/091023-MA-

TRMdraft.pdfnttps://www.nationalgridus.com/non html/eer/ma/10 MA E EEAR Pt 3.pdf. Note that PA Act 129 savings can be claimed for no more than 15 years.

⁴³"Improving Attic Thermal Performance", Home Energy, November 2004.

¹⁴ NOAA Climatic Data for Pennsylvania cities-Cloudiness (mean number of days Sunny, Partly Cloudy, and Cloudy), http://ols.nndc.noaa.gov/plolstore/plsgl/olstore.prodspecific?prodnum=C00095-PUB-A0001.

¹⁵ 2013 Illinois Statewide TRM - An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL's Building Energy Simulation Test for Existing Homes (BESTEST-EX).

http://ilsagfiles.org/SAG files/Technical Reference Manual/Version 2/Illinois Statewide TRM Version 2.0.pdf.

difference seen between attic temperature and outdoor air temperature; this results in an average hourly temperature difference between the attic and outdoor air of approximately +9% over the course of a 24 hour period, but only on sunny days. According to NOAA climatic data for Pennsylvania cities (Allentown, Erie, Harrisburg, Philadelphia, and Pitsburgh) for June through August, it is sunny or partly cloudy an average of 62% of the days. ¹⁴ It is assumed that there is an attic heating effect on both sunny and partly cloudy days, but not on cloudy days; therefore, an appropriate attic heating factor would be 1.056 based on the fact that the average hourly difference between attic temperature and outdoor air temperature is approximately +5.6% (9% x 62%).

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- "State of Ohio Energy Efficiency Technical Reference Manual," prepared for the Public Utilities Commission of Ohio by Vermont Energy Investment Corporation. August 6, 2010.
- 2. "Improving Attic Thermal Performance", Home Energy, November 2004.
- NOAA Climatic Data for Pennsylvania cities- Cloudiness (mean number of days Sunny, Partly Cloudy, and Cloudy), http://ols.nndc.noaa.gov/plolstore/plsql/olstore.prodspecific?prodnum=C00095-PUB-A0001.
- US DOE Federal Standards for Central Air Conditioners and Heat Pumps. http://www1.eere.energy.gov/buildings/appliance standards/product.aspx/productid/75
- Minimum efficiency standards for Ground and Groundwater Source Heat Pumps. IECC 2009.
- 6. VEIC estimate. Extrapolation of manufacturer data.
- 7. Engineering calculation, HSPF/COP=3.413
- Straub, Mary and Switzer, Sheldon. "Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941
- Consistent with CFs found in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008.²⁶⁷
- Climatography of the United States No. 81. Monthly Station Normals of Temperature, Precipitation, and Heating and Cooling Degree Days 1971-2000, 36 Pennsylvania. NOAA. http://cdo.ncdc.noaa.gov/climatenormals/clim81/PAnorm.pdf
- 11. Based on REM/Rate modeling using models from the PA 2012 Potential Study. EFLH calculated from kWh consumption for cooling and heating. Models assume 50% oversizing of air conditioners and 40% oversizing of heat pumps.²⁶⁸
- 12. 2014 PA TRM Section 2.2.4 Room AC Retirement

²⁶⁷ In the absence of better, Pennsylvania-specific data, this is the same source and value as the Mid-Atlantic and Illinois TRMs. ²⁶⁸ ACCA, "Verifying ACCA Manual S Procedures," http://www.acca.org/Files//?id=67.

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2.6.2 AIR SEALING

Measure Name	Residential Air Sealing
Target Sector	Residential Establishments
Measure Unit	Residential Air Sealing
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ²⁶⁹

Technical Reference Manual

Thermal shell air leaks are sealed through strategic use and installation of air-tight materials. Leaks are detected and leakage rates measured with the assistance of a blower-door test. This measure applies to the sealing of thermal shell air leaks in existing residential homes or apartment units in multifamily complexes with a primary electric heating and/or cooling source.

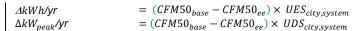
ELIGIBILITY

The baseline for this measure is the existing air leakage as determined through approved and appropriate test methods using a blower door. The baseline condition of a building upon first inspection significantly impacts the opportunity for cost-effective energy savings through air-sealing.

Air sealing materials and diagnostic testing should meet all qualification criteria for program eligibility. The initial and final tested leakage rates should be performed in such a manner that the identified reductions can be properly discerned, particularly in situations where multiple building envelope measures may be implemented simultaneously.

For example, if air sealing, duct sealing and insulation are all installed as a whole home retrofit, efforts should be made to isolate the CFM reductions from each measure individually. This may require performance of a blower door test between each measure installation. Alternatively, the baseline blower door test may be performed after the duct sealing is completed, then air sealing measures installed and the retrofit blower door test completed prior to installation of the new insulation.

ALGORITHMS



²⁶⁹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

DEFINITION OF TERMS

Table 2-106: Residential Air Sealing – Values and References

Term	Unit	Values	Source
$\mathit{CFM50}_{base}$, Baseline infiltration at 50 Pa	CFM	Measured, EDC Data Gathering	
CFM50 _{ee} , Infiltration at 50 Pa post air sealing	CFM	Measured, EDC Data Gathering	
UES _{city,system} , Energy savings per CFM50 of air leakage reduction	kWh CFM	See Table 2-107	1
UDS _{city,system} , Demand savings per CFM50 of air leakage reduction	kW CFM	See Table 2-108	2

DEFAULT SAVINGS

Table 2-107: Default Unit Energy Savings per Reduced CFM50 for Air Sealing

	kWh savings per reduction in air leakage (CFM50)					
City	Air Source Heat Pump		Electric Resistance		Ground Source Heat Pump	
3,	HP Cooling	Heating	Central AC Cooling	Heating	Central AC Cooling	Heating
Allentown	0.035	0.978	0.031	1.784	0.008	0.637
Erie	0.015	1.242	0.013	2.039	0.000	0.734
Harrisburg	0.074	1.108	0.068	2.000	0.027	0.717
Philadelphia	0.065	0.697	0.059	1.445	0.024	0.521
Pittsburgh	0.031	1.057	0.028	1.778	0.004	0.637
Scranton	0.020	1.235	0.017	1.852	0.000	0.668
Williamsport	0.031	1.113	0.028	1.789	0.004	0.643

Table 2-108: Default Unit Coincident Peak Demand Savings per Reduced CFM50 for Air Sealing

	Peak kW savings per reduction in air leakage (CFM50)				
City	Air Source Heat Pump Cooling	Central AC Cooling	Ground Source Heat Pump Cooling		
Allentown	0.010	0.009	0.003		
Erie	0.005	0.005	0.001		
Harrisburg	0.019	0.017	0.008		
Philadelphia	0.018	0.017	0.008		
Pittsburgh	0.011	0.010	0.003		
Scranton	0.007	0.007	0.001		

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	Peak kW savings per reduction in air leakage (CFM50)					
City	Air Source Heat Pump Cooling	Central AC Cooling Ground Source He Pump Cooling				
Williamsport	0.011	0.010	0.003			

EVALUATION PROTOCOLS

SECTION 2: Residential Measures

The appropriate evaluation protocol for this measure is verification of pre/post blower door test results and proper application of the TRM protocol.

SOURCES

- Based on BEopt Modeling with EnergyPlus performed by Navigant on behalf of PECO. See Navigant Residential Air Sealing review memorandum dated September 3, 2013. UES are were calculated using EnergyPlus models and the TMY3 weather file for each city. EDC's may use UES determined through EDC specific studies.
- 2. Based on BEopt Modeling with EnergyPlus performed by Navigant on behalf of PECO. See Navigant Residential Air Sealing review memorandum dated September 3, 2013. UDS are were calculated using EnergyPlus models and the TMY2 weather file for each city and based on the average demand reductions for the PJM peak period. UDS represent coincident peak demand savings, thus the coincidence factor is already incorporated into the values. EDC's may use UDS determined through EDC specific studies.

2.6.3 ENERGY STAR WINDOWS

Measure Name	ENERGY STAR Windows	
Target Sector	Residential Establishments	
Measure Unit	Window Area	
Unit Energy Savings	Varies	
Unit Peak Demand Reduction	Varies	
Measure Life (15 max, but 20 for TRC) years ²⁷⁰		
Vintage	Replace on Burnout	

ELIGIBILITY

This protocol documents the energy savings for replacing existing windows in a residence with ENERGY STAR certified windows. The target sector is primarily residential.

ALGORITHMS

The general form of the equation for the ENERGY STAR or other high-efficiency windows energy savings' algorithms is:

Total Savings = Area of Window
$$ft^2 \times \frac{Savings}{ft^2}$$

To determine resource savings, the per-square-foot estimates in the algorithms will be multiplied by the number of square feet of window area. The number of square feet of window area will be determined using market assessments and market tracking. Some of these market tracking mechanisms are under development. The per-unit energy and demand savings estimates are based on prior building simulations of windows.

Savings' estimates for ENERGY STAR Windows are based on modeling a typical 2,500 square foot home using REM Rate, the home energy rating tool.²⁷¹ Savings are per square foot of qualifying window area. Savings will vary based on heating and cooling system type and fuel.

These fuel and HVAC system market shares will need to be estimated from prior market research efforts or from future program evaluation results.

Heat Pump HVAC System:

$$\triangle kWh/yr$$
 = $ESav_{HP}$
 $\triangle kW_{peak}$ = $DSav_{HP} \times CF$

Electric Heat/Central Air Conditioning:

$$\triangle kWh/yr$$
 = $ESav_{RES}$
 $\triangle kW_{peak}$ = $DSav_{CAC} \times CF$

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²⁷⁰ Capped based on the requirements of the Pennsylvania Technical Reference Manual (June 2010)

²⁷¹ Energy Information Administration. *Residential Energy Consumption Survey.* 2005. http://www.eia.doe.gov/emeu/recs/recs2005/hc2005_tables/detailed_tables2005.html

Rev Date: June 2015

Electric Heat/No Central Air Conditioning:

 $\Delta kWh/yr$ = $ESav_{ResNoCAC}$ ΔkW_{peak} = $DSav_{NOCAC} \times CF$

DEFINITION OF TERMS

Table 2-109: ENERGY STAR Windows - References

Component	TypeUnit	Value	Sources
$\textit{ESav}_{\textit{HP}}$, Electricity savings (heating and cooling) with heat pump installed	$\frac{kWh}{ft^2}Fixed$	2.2395 kWh/ft²	1
HP Time Period Allocation Factors	FixedNone	Summer/On-Peak 10% Summer/Off-Peak 7% Winter/On-Peak 40% Winter/Off-Peak 44%	2
ESav _{RES/CAC} , Electricity savings with electric resistance heating and central AC installed.	kWh Fixed	4.0 kWh/ft²	1
Res/CAC Time Period Allocation Factors	FixedNone	Summer/On-Peak 10% Summer/Off-Peak 7% Winter/On-Peak 40% Winter/Off-Peak 44%	2
ESav _{RES/NOCAC} , Electricity savings with electric resistance heating and no central AC installed	$\frac{kWh}{ft^2}$ Fixed	3.97 kWh/ft²	1
Res/No CAC Time Period Allocation Factors	Fixed None	Summer/On-Peak 3% Summer/Off-Peak 3% Winter/On-Peak 45% Winter/Off-Peak 49%	2
<i>DSav_{HP}</i> , Summer demand savings with heat pump installed.	$\frac{kW}{ft^2}$ Fixed	0.000602 <mark>kW/ft²</mark>	1
DSav _{CAC} , Summer demand savings with central AC installed.	$\frac{kW}{ft^2}$ Fixed	0.000602 <mark>kW/ft²</mark>	1
DSav _{NOCAC} , Summer demand savings with no central AC installed.	$\frac{kW}{ft^2}$ Fixed	0.00 kW/ft²	1
CF , Demand Coincidence Factor (See Section 1.5)	Fixed%	0.75 <u>64.7%</u>	3

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

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State of Pennsylvania – Technical Reference Manual – Rev Date: June 2015

SOURCES:

- From REMRATE Modeling of a typical 2,500 sq. ft. NJ home. Savings expressed on a per-square-foot of window area basis. New Brunswick climate data.
- 2. Time period allocation factors used in cost-effectiveness analysis.
- 3. Based on reduction in peak cooling load. Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs".

 Study by BG&E. The Electricity Journal. Aug/Sept. 2011. http://www.sciencedirect.com/science/article/pii/S1040619011001941

4. Prorated based on 12% of the annual degree days falling in the summer period and 88% of the annual degree days falling in the winter period.

Field Code Changed

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2.6.4 RESIDENTIAL NEW CONSTRUCTION

Measure Name	Residential New Construction
Target Sector	Residential Establishments
Measure Unit	Multiple
Unit Energy Savings	<u>Varies</u>
Unit Peak Demand Reduction	<u>Varies</u>
Measure Life	<u>Varies years</u>
Vintage	New Construction

ELIGIBILITY

This protocol documents the energy savings attributed to improvements to the construction of residential homes above the baseline home as calculated by the appropriate energy modeling software or as determined by deemed savings values.

ALGORITHMS

Insulation Up-Grades, Efficient Windows, Air Sealing, Efficient HVAC Equipment and Duct Sealing (Weather-Sensitive Measures):

Energy and peak demand savings due to improvements in the above mentioned measures in Residential New Construction programs will be a direct output of accredited Home Energy Ratings (HERS) software that meets the applicable Mortgage Industry National Home Energy Rating System Standards. REM/Rate²⁷² is cited here as an example of an accredited software which can be used to estimate savings for this program. REM/Rate has a module that compares the energy characteristics of the energy efficient home to the baseline/reference home and calculates savings. For residential new construction, the baseline building thermal envelope and/or system characteristics shall be based on the current state adopted 2009 International Residential Code (IRC 2009).

The energy savings for weather-sensitive measures will be calculated from the software output using the following algorithm:

Energy savings of the qualified home (kWh)

=
$$(Heating \, kWh_{base} - Heating \, kWh_a) + (Cooling \, kWh_{base} - Cooling \, kWh_a)$$

The system peak electric demand savings for weather-sensitive measures will be calculated from the software output with the following algorithm, which is based on compliance and certification of the energy efficient home to the EPA's ENERGY STAR for New Homes' program standard:

Peak demand of the baseline home

$$= \frac{PL_{base}}{EER_{base}}$$

Peak demand of the qualifying home

$$= \frac{PL_q}{EER_q}$$

(http://apps1.eere.energy.gov/buildings/tools_directory/http://apps1.eere.energy.gov/buildings/tools_directory/software)

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²⁷² DoE's Building Energy Software Tools Directory

Coincident system peak electric demand savings

= (Peak demand of the baseline home - Peak demand of the qualifying home) \times CF

Hot Water, Lighting, and Appliances (Non-Weather-Sensitive Measures):

Quantification of additional energy and peak demand savings due to the installation of high-efficiency electric water heaters, lighting and other appliances will be based on the algorithms presented for these measures in Section 1 (Residential Measures) of this Manual. Where the TRM algorithms involve deemed savings, e.g. lighting, the savings in the baseline and qualifying homes should be compared to determine the actual savings of the qualifying home above the baseline.

In instances where REM/Rate calculated parameters or model inputs do not match TRM algorithm inputs, additional data collection is necessary to use the TRM algorithms. One such example is lighting. REM/Rate requires an input of percent of lighting fixtures that are energy efficient whereas the TRM requires an exact fixture count. Another example is refrigerators, where REM/Rate requires projected kWh consumed and the TRM deems savings based on the type of refrigerator.

It is also possible to have increases in consumption or coincident peak demand instead of savings for some non-weather sensitive measures. For example, if the amount of efficient lighting in a new home is less than the amount assumed in the baseline (IRC 2009), the home will have higher energy consumption and coincident peak demand for lighting, even though it still qualifies for the program.

According to Architectural Energy Corporation, the developer of the REM/Rate model, this model does account for the interaction of energy savings due to the installation of high efficiency lighting or appliances with the energy used in a home for space conditioning. Architectural Energy Corporation staff explained to the Statewide Evaluator that lighting and appliance energy usage is accounted for in the REM/Rate model, and the model does adjust energy use due to the installation of high efficiency lighting and appliances. ²⁷³

DEFINITION OF TERMS

A summary of the input values and their data sources follows:

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²⁷³ Email from V. Robert Salcido, P.E., LEED AP, Director of Products at Architectural Energy Corporation to Josh Duckwall, Project Manager at GDS Associates, November 21, 2013.

Table 2-110: Residential New Construction – References

Component	Type <u>Unit</u>	Value	Sources
Heating $kWh_{\textit{base}}$, Annual heating energy consumption of the baseline home in kWh, from software.	Variable<u>kWh</u>	Software Calculated	1
Heating kWh_q , Annual heating energy consumption of the qualifying home in kWh, from software.	<u>kWh</u> Variable	Software Calculated	2
Cooling kWh_{base} , Annual cooling energy consumption of the baseline home in kWh , from software.	<u>kWh</u> Variable	Software Calculated	1
Cooling kWh_q , Annual cooling energy consumption of the qualifying home in kWh, from software.	<u>kWh</u> Variable	Software Calculated	2
<i>PL</i> _{base} , Estimated peak cooling load of the baseline home in kbtuh, from software.	Variable<u>kBtuh</u>	Software Calculated	3
EER _{base} . Energy Efficiency Ratio of the baseline unit.	$\frac{Btu}{W \cdot h}$ Variable	EDC Data Gathering or SEER _b * BLEER	4
\textit{EER}_q , Energy Efficiency Ratio of the qualifying unit.	$\frac{Btu}{W \cdot h}$ Variable	EDC Data Gathering or SEER _q * BLEER	4
SEER _{base} , Seasonal Energy Efficiency Ratio of the baseline unit.	$\frac{Btu}{W \cdot h}$ Fixed	Default = 13	5
BLEER, Factor to convert baseline SEER _b to EER _b .	$\frac{Btu}{W \cdot h}$ Fixed	Default =0.87 (11.3/13)	6
PL_q , Estimated peak cooling loadfor the qualifying home constructed, in kbtuh, from software.	Variable kBtuh	Software Calculated	7
$SEER_q$, SEER associated with the HVAC system in the qualifying home.	$\frac{Btu}{W \cdot h}$ Variable	EDC Data Gathering	8
CF , Demand Coincidence Factor (See Section 1.5)	Fixed%	Default = 64.7 % 0.70	9

The following table lists the building envelope characteristics of the baseline reference home based on IRC 2009 for the three climate zones in Pennsylvania.

Table 2-111: Baseline Insulation and Fenestration Requirements by Component (Equivalent U-Factors)

Climate Zone	Fenestration U-Factor	Skylight U-Factor	Ceiling U- Factor	Frame Wall U- Factor	Mass Wall U-Factor	U-	Basement Wall U-Factor	Slab R-Value &Depth	Crawl Space Wall U- Factor
4A	0.35	0.60	0.030	0.082	0.141	0.047	0.059	10, 2 ft	0.065
5A	0.35	0.60	0.030	0.060	0.082	0.033	0.059	10, 2 ft	0.065
6A	0.35	0.60	0.026	0.060	0.060	0.033	0.059	10, 4 ft	0.065

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Table 2-112: Energy Star Homes - User Defined Reference Home

Data Point	Value ²⁷⁴	Source
Air Infiltration Rate	0.30 ACH for windows, skylights, sliding glass doors 0.50 ACH for swinging doors	1 <u>3</u>
Duct Leakage	12 cfm25 (12 cubic feet per minute per 100 square feet of conditioned space when tested at 25 pascals)	1 <u>3</u>
Duct Insulation	Supply ducts in attics shall be insulated to a minimum of R-8. All other ducts insulated to a minimum of R-6.	1 <u>0</u>
Duct Location	50% in conditioned space, 50% unconditioned space	Program Design
Mechanical Ventilation	None	1 <u>0</u>
Lighting Systems	Minimum 50% of permanent installed fixtures to be high- efficacy lamps	1 <u>0</u>
Appliances	Use Default	
Setback Thermostat	Maintain zone temperature down to 55 °F (13 °C) or up to 85 °F (29 °C)	1 <u>0</u>
Temperature Set Points	Heating: 70°F Cooling: 78°F	1 <u>0</u>
Heating Efficiency		
Furnace	80% AFUE	<u>11</u> 2
Boiler	80% AFUE	<u>11</u> 2
Combo Water Heater	76% AFUE (recovery efficiency)	<u>11</u> 2
Air Source Heat Pump	7.7 <u>8.2</u> HSPF	1 <u>0</u>
Geothermal Heat Pump	7.7 HSPF	1 <u>0</u>
PTAC / PTHP	Not differentiated from air source HP	1 <u>0</u>
Cooling Efficiency		
Central Air Conditioning	13.0 SEER	1 <u>0</u>
Air Source Heat Pump	1 <mark>3.4.</mark> 0 SEER	1 <u>0</u>
Geothermal Heat Pump	13 SEER (11.2 EER)	1 <u>0</u>
PTAC / PTHP	Not differentiated from central AC	1 <u>0</u>
Window Air Conditioners	Not differentiated from central AC	1 <u>0</u>
Domestic WH Efficiency		
Electric	EF = 0.97 - (0.00132 * gallons)	<u>12</u> 3
Natural Gas	EF = 0.67 - (0.0019 * gallons)	<u>12</u> 3
Additional Water Heater Tank Insulation	None	_

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 $^{^{\}rm 274}$ Single and multiple family as noted.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Calculation of annual energy consumption of a baseline home from the home energy rating tool based on the reference home energy characteristics.
- 2. Calculation of annual energy consumption of an energy efficient home from the home energy rating tool based on the qualifying home energy characteristics
- Calculation of peak load of baseline home from the home energy rating tool based on the reference home energy characteristics.
- If the EER of the unit is know, use the EER. If only the SEER is known, then use SEER * BLEER to estimate the EER.
- Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.
- 6. Ratio to calculate EER from SEER based average EER for SEER 13 units.
- 7. Calculation of peak load of energy efficient home from the home energy rating tool based on the qualifying home energy characteristics.
- 8. SEER of HVAC unit in energy efficient qualifying home.
- 9. Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011. Found at http://www.sciencedirect.com/science/article/pii/S1040619011001941 Based on an analysis of six different utilities by Proctor Engineering.
- 10. 2009 International Residential Code (IRC 2009, Sections N1102 N1104)
- 11. Federal Register / Vol. 73, No. 145 / Monday, July 28, 2008 / Rules and Regulations, p. 43611-43613, 10 CFR Part 430, "Energy Conservation Program for Consumer Products: Energy Conservation Standards for Residential Furnaces and Boilers."
- 12. Federal Register / Vol. 75, No. 73 / Friday, April 16, 2010 / Rules and Regulations, p. 20112-20236, 10 CFR Part 430, "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters; Final Rule."
- 42-13. 2009 International Residential Code Table N1102.1.2. Table N1102.1.2 Equivalent U-Factors presents the R-Value requirements of Table N1102.1.1 in an equivalent U-Factor format. Users may choose to follow Table N1102.1.1 instead. IRC 2009 supersedes this table in case of discrepancy. Additional requirements per Section N1102 of IRC 2009 must be followed even if not listed here.

2.6.5 ENERGY STAR AUDIT

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ALGORITHMS

No algorithm was developed to measure energy savings for this program. The purpose of the program is to provide information and tools that residential customers can use to make decisions about what actions to take to improve energy efficiency in their homes. Many measure installations that are likely to produce significant energy savings are covered in other programs. These savings are captured in the measured savings for those programs. The savings produced by this program that are not captured in other programs would be difficult to isolate and relatively expensive to measure.

2.6.62.6.5 HOME PERFORMANCE WITH ENERGY STAR

Measure Name	Home Performance with ENERGY STAR
Target Sector	Residential Establishments
Measure Unit	Multiple
Unit Energy Savings	Varies
Unit Peak Demand Reduction	Varies
Measure Life	Years
<u>Vintage</u>	Retrofit

In order to implement Home Performance with ENERGY STAR, there are various standards a program implementer must adhere to in order to deliver the program. These standards, along with operational guidelines on how to navigate through the HPwES program can be found on the ENERGY STAR website. Minimum requirements, Sponsor requirements, reporting requirements, and descriptions of the performance and prescriptive based options can be found in the v. 1.5 Reference Manual. The program implementer must use software that meets a national standard for savings calculations from whole-house approaches such as home performance. The software program implementer must adhere to at least one of the following standards:

- A software tool whose performance has passed testing according to the National Renewable Energy Laboratory's HERS BESTEST software energy simulation testing protocol.²⁷⁶
- Software approved by the US Department of Energy's Weatherization Assistance Program.²⁷⁷
- RESNET approved rating software.²⁷⁸

There are numerous software packages that comply with these standards. Some examples of the software packages are REM/Rate, EnergyGauge, TREAT, and HomeCheck. <u>These examples are not meant to be an exhaustive list of software approved by the bodies mentioned above —</u>

HomeCheck Software ExampleThe HomeCheck software is described below as an example of a software that can be used to determine if a home qualifies for Home Performance with ENERGY STAR.

ELIGIBILITY

The efficient condition is the performance of the residential home as modeled in the approved software after home performance improvements have been made. The baseline condition is the same home modeled prior to any energy efficiency improvements.

ALGORITHMS

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²⁷⁵ The HPwES Reference Manual may be found at

https://www.energystar.gov/ia/home_improvement/downloads/HPwES_Sponsor_Guide_v1-5.pdf?07e7-3320

²⁷⁶ A new standard for BESTEST-EX for existing homes is currently being developed - status is found at http://www.nrel.gov/buildings/bestest_Ex.html. The existing 1995 standard can be found at http://www.nrel.gov/docs/legosti/fy96/7332a.pdf.

²⁷⁷ A listing of the approved software available at http://www.waptac.org/si.asp?id=736.

²⁷⁸ A listing of the approved software available at http://resnet.us.

There are no algorithms associated with this measure as the energy savings are shown through modeling software. For modeling software that provides 8760 energy consumption data, the following algorithm may be used as guidance to determine demand savings:

$$\Delta kW_{peak}$$
 = $\left(Average\ kW_{PJM\ PEAK}\right)_{hase} - \left(Average\ kW_{PJM\ PEAK}\right)_{ee}$

DEFINITION OF TERMS

Table 2-113: Home Performance with ENERGY STAR - References

Component	<u>TypeUnit</u>	<u>Values</u>	<u>Source</u>
Average kW _{PJM PEAK} , Average demand during the PJM Peak Period	<u>FixedkW</u>	EDC Data Gathering	<u>11</u>

Interactivity

Non-interacted first year savings are calculated for each individual measure.

Non-interacted SIR (RawSIR) is calculated for each measure.

Measures are ranked in descending order of RawSIR,

Starting with the most cost effective measure (as defined by RawSIR), first year savings are adjusted for each measure as follows:

Mechanical measures (such as thermostats, HVAC system upgrades or distribution system upgrades) are adjusted to account for the load reduction from measures with a higher RawSIR.

Architectural measures are adjusted to account for overall HVAC system efficiency changes and thermostat load reduction changes. Architectural measures with a higher RawSIR than that of HVAC system measures are calculated using the existing efficiencies. Those with RawSIR's lower than that of heating equipment use the new heating efficiencies.

Interacted SIR is then calculated for each measure, along with cumulative SIR for the entire job.

All measures are then re-ranked in descending order of SIR.

The process is repeated, replacing RawSIR with SIR until the order of measures does not chan

Lighting

Quantification of additional savings due to the addition of high efficiency lighting will be based onthe applicable algorithms presented for these appliances in the ENERGY STAR Lighting-Algorithms section found in ENERGY STAR Products.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

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 The coincident summer peak period is defined as the period between the hour ending 15:00 Eastern Prevailing Time²⁷⁹ (EPT) and the hour ending 18:00 EPT during all days from June 1 through August 31, inclusive, that is not a weekend or federal holiday²⁸⁰.

²⁷⁹ This is same as the Daylight Savings Time (DST) ²⁸⁰ PJM Manual 18B for Energy Efficiency Measurement & Verification

SECTION 2: Residential Measures

2.6.72.6.6 ENERGY STAR MANUFACTURED HOMES

Measure Name	ENERGY STAR® Manufactured Homes
Target Sector	Residential Establishments
Measure Unit	Variable
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 Years ²⁸¹
Vintage	New Construction

ELIGIBILITY

This measure applies to ENERGY STAR Manufactured Homes.

ALGORITHMS

Insulation Upgrades, Efficient Windows, Air Sealing, Efficient HVAC Equipment and Duct Sealing (Weather-Sensitive Measures):

Energy and peak demand savings due to improvements in the above measures in ENERGY STAR Manufactured Homes programs will be a direct output of accredited Home Energy Ratings (HERS) software that meets the applicable Mortgage Industry National Home Energy Rating System Standards. REM/Rate²⁸² is cited here as an example of an accredited software which can be used to estimate savings for this program. REM/Rate has a module that compares the energy characteristics of the energy efficient home to the baseline/reference home and calculates savings. For ENERGY STAR Manufactured Homes, the baseline building thermal envelope and/or system characteristics shall be based on the current Manufactured Homes Construction and Safety Standards (HUD Code). For this measure a manufactured home "means a structure, transportable in one or more sections, which in the traveling mode, is eight body feet or more in width or forty body feet or more in length, or, when erected on site, is three hundred twenty or more square feet, and which is built on a permanent chassis and designed to be used as a dwelling with or without a permanent foundation when connected to the required utilities, and includes the plumbing, heating, air conditioning, and electrical systems contained therein."

The energy savings for weather-sensitive measures will be calculated from the software output using the following algorithm:

Energy savings of the qualified home (kWh/yr)

 Δ kWh = (Heating kWh_{base}- Heating kWh_{ee}) + (Cooling kWh_{base}- Cooling kWh_{ee})

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²⁸¹ NREL, Northwest Energy Efficient Manufactured Housing Program Specification Development, T.Huges, B. Peeks February 2013 (http://www.nrel.gov/docs/fy13osti/56761.pdf)

²⁸² DoE's Building Energy Software Tools Directory (http://apps1.eere.energy.gov/buildings/tools_directory/software).
²⁸³ 24 CFR Part 3280-MANUFACTURED HOMES CONSTRUCTION AND SAFETY

STANDARD(http://www.gpo.gov/fdsys/pkg/CFR-2013-title24-vol5/pdf/CFR-2013-title24-vol5-part3280.pdf)

The system peak electric demand savings for weather-sensitive measures will be calculated from the software output with the following algorithm, which is based on compliance and certification of the energy efficient home to the EPA's ENERGY STAR Manufactured Home' program standard:

Peak demand of the baseline home

$$=\frac{PL_b}{EER_b}$$

Peak demand of the qualifying home

$$=\frac{PL_q}{EER_q}$$

Coincident system peak electric demand savings (kW)

$$\Delta kW_{peak}$$
 = (Peak demand of the baseline home - Peak demand of the qualifying home) \times CF

Hot Water, Lighting, and Appliances (Non-Weather-Sensitive Measures):

Quantification of additional energy and peak demand savings due to the installation of highefficiency electric water heaters, lighting and other appliances will be based on the algorithms presented for these measures in Section 1 (Residential Measures) of this Manual. Where the TRM algorithms involve deemed savings, e.g. lighting, the savings in the baseline and qualifying homes should be compared to determine the actual savings of the qualifying home above the baseline.

In instances where REM/Rate calculated parameters or model inputs do not match TRM algorithm inputs, additional data collection is necessary to use the TRM algorithms. One such example is lighting. REM/Rate requires an input of percent of lighting fixtures that are energy efficient whereas the TRM requires an exact fixture count. Another example is refrigerators, where REM/Rate requires projected kWh consumed and the TRM deems savings based on the type of refrigerator.

According to Architectural Energy Corporation, the developer of the REM/Rate model, this model does account for the interaction of energy savings due to the installation of high efficiency lighting or appliances with the energy used in a home for space conditioning. Architectural Energy Corporation staff explained to the Statewide Evaluator that lighting and appliance energy usage is accounted for in the REM/Rate model, and the model does adjust energy use due to the installation of high efficiency lighting and appliances. ²⁸⁴ It was verified in the RESNET® Standard that lighting and appliances are account for as internal gains and will represent an interaction with the HVAC systems. ²⁸⁵

²⁸⁴ Email from V. Robert Salcido, P.E., LEED AP, Director of Products at Architectural Energy Corporation to Josh Duckwall, Project Manager at GDS Associates, November 21, 2013.

²⁸⁵ http://www.resnet.us/standards/PropStdsRevision-01-11_Revised_FINAL.pdf

DEFINITION OF TERMS

A summary of the input values and their data sources follows: Table 2-114: ENERGY STAR Manufactured Homes– References

Component	Unit	Value	Sources
Heating kWh _{base} , Annual heating energy consumption of the baseline home	kWh	Software Calculated	1
Heating kWh _{ee} , Annual heating energy consumption of the qualifying home	kWh	Software Calculated	1
Cooling kWh _{base} , Annual cooling energy consumption of the baseline home	kWh	Software Calculated	1
Cooling kWh _{ee} , Annual cooling energy consumption of the qualifying home	kWh	Software Calculated	1
PL _b , Estimated peak cooling load of the baseline home	kBTU/h	Software Calculated	1
EER _b , Energy Efficiency Ratio of the baseline unit.	$\frac{Btu}{W \cdot h}$	EDC Data Gathering or SEER _b * BLEER	2
EER _q , Energy Efficiency Ratio of the qualifying unit.	$\frac{Btu}{W \cdot h}$	EDC Data Gathering or SEER _q * BLEER	2
SEER _b , Seasonal Energy Efficiency Ratio of the baseline unit.	$\frac{Btu}{W \cdot h}$	13	4
BLEER, Factor to convert baseline SEERb to EERb.	$\frac{Btu}{W \cdot h}$	EDC Data Gathering Default = $\frac{11.3}{13}$	3
PL _q , Estimated peak cooling load for the qualifying home constructed, in kbtuh, from software.	kBTU/h	Software Calculated	1
$\ensuremath{SEER}_q, \ensuremath{SEER}$ associated with the HVAC system in the qualifying home.	$\frac{Btu}{W \cdot h}$	EDC Data Gathering	5
CF, Demand Coincidence Factor (See Section 1.5)	%	EDC Data Gathering Default = 64.7%	6

The HUD Code defines required insulation levels as an average envelope Uo value per zone. In Pennsylvania zone 3 requirements apply with a required Uo value of 0.079. This value cannot be directly used to define a baseline envelope R-values because the Uo value is dependent on both the size of the manufactured homes and insulating levels together. However because manufactured homes are typically built to standard dimensions baseline U-values can be estimated with reasonable accuracy.

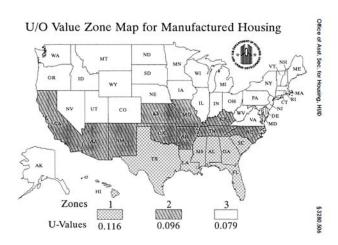


Figure 2-4: Uo Baseline Requirements²⁸⁶

The HUD Code required insulation levels can be expressed as a set of estimated envelope parameters to be used in REM/Rate's user defined reference home function. Using typical manufactured home sizes these values are expressed below along with federal standard baseline parameters below in Table 2-115 Table 2-115.

Table 2-115: ENERGY STAR Manufactured Homes - User Defined Reference Home

Data Point	Value ²⁸⁷	Source
Walls	U-value 0.090	7, 8
Ceilings	U-value 0.045	7, 8
Floor	U-value 0.045	7, 8
Windows	U-value 0.59	7, 8
Doors	U-Value 0.33	7, 8
Air Infiltration Rate	10 ACH50	7
Duct Leakage	RESNET/HERS default	7
Duct Insulation	RESNET/HERS default	7
Duct Location	Supply 100% manufactured home belly, Return 100% conditioned space	9
Mechanical Ventilation	0.035 CFM/sqft Exhaust	8
Lighting Systems	0% CFL 10% pin based (Default assumption)	10
Appliances	Use Default	7

²⁸⁶ 24 CFR Part 3280-MANUFACTURED HOMES CONSTRUCTION AND SAFETY

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Building Shell

 $STANDARD(http://www.gpo.gov/fdsys/pkg/CFR-2013-title24-vol5/pdf/CFR-2013-title24-vol5-part3280.pdf) \\ ^{287} Single and multiple family as noted.$

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering.

SOURCES

- Calculation of annual energy and peak load consumption of a baseline home from the home energy rating tool based on the reference home energy characteristics.
- 2. If the EER of the unit is known, use the EER. If only the SEER is known, then use SEER * BLEER to estimate the EER.
- 3. Ratio to calculate EER from SEER based average EER for SEER 13 units.
- Federal Register / October 31, 2011 / Rules and Regulations, 10 CFR Part 430, "2011-10-31 Energy Conservation Program: Energy Conservation Standards for Residential Furnaces and Residential Central Air Conditioners and Heat Pumps; Notice of effective date and compliance dates for direct final rule." http://www.regulations.gov/#!documentDetail;D=EERE-2011-BT-STD-0011-0058
 - http://www.regulations.gov/#!documentDetail;D=EERE-2011-BT-STD-0011-0058
- 5. SEER of HVAC unit in energy efficient qualifying home.

SECTION 2: Residential Measures

Building Shell

- Straub, Mary and Switzer, Sheldon."Using Available Information for Efficient Evaluation of Demand Side Management Programs". Study by BG&E. The Electricity Journal. Aug/Sept. 2011.
- ENERGY STAR QUALIFIED MANUFACTURED HOMES-Guide for Retailers with instructions for installers and HVAC contractors / June 2007 / (http://www.research-alliance.org/pages/es_retail.htm)
- 8. 24 CFR Part 3280-MANUFACTURED HOMES CONSTRUCTION AND SAFETY STANDARD(http://www.gpo.gov/fdsys/pkg/CFR-2013-title24-vol5/pdf/CFR-2013-title24-vol5-part3280.pdf)
- 9. Standard manufactured home construction
- 10. Not a requirement of the HUD Code.
- 11. 2009 International Residential Code (IRC2009, Sections N1102-N1104)
- 12. Federal Register / Vol. 73, No. 145 / Monday, July 28, 2008 / Rules and Regulations, p. 43611-43613, 10 CFR Part 430, "Energy Conservation Program for Consumer Products: Energy Conservation Standards for Residential Furnaces and Boilers."
- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 40-gallon tank this is 0.9172. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30
- 14. Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons. For a 40-gallon tank this is 0.9172. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 30
- 15. No regirement in code or federal regulation.

SECTION 2: Residential Measures

2.7 MISCELLANEOUS

2.7.1 POOL PUMP LOAD SHIFTING

Measure Name	Pool Pump Load Shifting
Target Sector	Residential Establishments
Measure Unit	Pool Pump Load Shifting
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	1 year ²⁸⁸⁴
<u>Vintage</u>	Retrofit

Residential pool pumps can be scheduled to avoid the 2 PM to 6 PM peak period.

ELIGIBILITY

This protocol documents the energy savings attributed to schedule residential single speed pool pumps to avoid run during the peak hours from 2 PM to 6 PM. The target sector primarily consists of single-family residences. This measure is intended to be implemented by trade allies that participate in in-home audits, or by pool maintenance professionals.

ALGORITHMS

The residential pool pump r ²⁸⁰eschedule measure is intended to produce demand savings, but if the final daily hours of operation are different than the initial daily hours of operation, an energy savings (or increase) may result. The demand savings result from not running pool pumps during the peak hours of 2 PM to 6 PM.

The peak coincident factor, CF, is defined as the average coincident factor during 2 PM to 6 PM on summer weekdays. Ideally, the demand coincidence factor for the supplanted single-speed pump can be obtained from the pump's time clock. The coincidence factor is equal to the number of hours that the pump was set to run between 2 PM and 6 PM, divided by 4.

SECTION 2: Residential Measures

²⁸⁸ The measure life is initially assumed to be one year. If there is significant uptake of this measure then a retention study may be warranted

²⁸⁹The measure life is initially assumed to be one year. If there is significant uptake of this measure then a retention study may be warranted.

DEFINITION OF TERMS

Table 2-116: Pool Pump Load Shifting Assumptions

Component	Type Unit	Value	Source
△hours/day , The change in daily operating hours.	hours Fixed	0	2
kW _{pump} , Electric demand of single speed pump at a given flow rate. This quantity should be measured or taken from Table 2-116	Fixed <u>kW</u>	1.364 kW or See Table 2-117	Table 2-117
CF_{pre} , Peak coincident factor of single speed pump from 2 PM to 6 PM in summer weekday prior to pump rescheduling. This quantity should be inferred from the timer settings	Fixed <u>%</u>	<u>30.6%</u> 0.235	3
<i>CF</i> _{post} , Peak coincident factor of single speed pump from 2 PM to 6 PM in summer weekday after pump rescheduling. This quantity should be inferred from the new timer settings.	Fixed <u>%</u>	0%	2
Days _{operating} , Days per year pump is in operation. This quantity should be recorded by applicant.	days yr Fixed	100	1

Average Single Speed Pump Electric Demand

Since this measure involves functional pool pumps, actual measurements of pump demand are encouraged. If this is not possible, then the pool pump power can be inferred from the nameplate horsepower. Table 2-117 shows the average service factor (over-sizing factor), motor efficiency, and electrical power demand per pump size based on California Energy Commission (CEC) appliance database for single speed pool pump²⁹⁰. Note that the power to horsepower ratios appear high because many pumps, in particular those under 2 HP, have high 'service factors'. The true motor capacity is the product of the nameplate horsepower and the service factor.

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^{290 &}quot;CEC Appliances Database – Pool Pumps." California Energy Commission. Updated Feb 2008. Accessed March 2008. http://www.energy.ca.gov/appliances/database/historical_excel_files/2009-03-01-excel_based_files/Pool_Products/Pool_Pumps.zip

Pump Horse Power (HP)	Average Pump Service Factor*	Average Pump Motor Efficiency*	Average Pump Power (W)*
0.50	1.62	0.66	946
0.75	1.29	0.65	1,081
1.00	1.28	0.70	1,306
1.50	1.19	0.75	1,512
2.00	1.20	0.78	2,040
2.50	1.11	0.77	2,182
3.00	1.21	0.79	2,666

Technical Reference Manual

MEASURE LIFE

The measure life is initially assumed to be one year. If there is significant uptake of this measure then a retention study may be warranted.

EVALUATION PROTOCOL

The most appropriate evaluation protocol for this measure is verification of pool pump run time as well as verification of hours of operation coincident with peak demand.

SOURCES

- Mid-Atlantic TRM, version 2.0. Prepared by Vermont Energy Investment Corporation. Facilitated and managed by the Northeast Energy Efficiency Partnerships. July 2011.
- 2. Program is designed to shift load to off-peak hours, not necessarily to reduce load.
- 3. Derived from Pool Pump and Demand Response Potential, DR 07.01 Report, SCE Design and Engineering, Table 16. Statewide value calculated using the non-weather dependent coincident peak demand calculator with inland valley data. Calculated using the average of the 3 regions. The pool pump operating schedule is not weather dependant, but operator dependant. This is noted on page 22, paragraph 2 of the source. http://www.etcc-ca.com/sites/default/files/OLD/images/stories/pdf/ETCC Report 473.pdf

Field Code Changed

SECTION 2: Residential Measures

²⁹¹ Averaged for all listed single-speed pumps in the last available version of the CEC appliance efficiency database. The powers are for 'CEC Curve A' which represents hydraulic properties of 2" PVC pipes rather than older, 1.5" copper pipes.

2.7.2 VARIABLE SPEED POOL PUMPS (WITH LOAD SHIFTING OPTION)

Measure Name	Residential VSD Pool Pumps
Target Sector	Residential Establishments
Measure Unit	VFD Pool Pumps
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	10 years ^{292±}
<u>Vintage</u>	Replace on Burnout

This measure has two potential components. First, a variable speed pool pump must be purchased and installed on a residential pool to replace an existing constant speed pool pump. Second, the variable speed pool pump may be commissioned such that it does not operate in the 2 PM to 6 PM period (on weekdays). This second, optional step is referred to as *load shifting*. Residential variable frequency drive pool pumps can be adjusted so that the minimal required flow is achieved for each application. Reducing the flow rate results in significant energy savings because pump power and pump energy usage scale with the cubic and quadratic powers the flow rate respectively. Additional savings are achieved because the VSD pool pumps typically employ premium efficiency motors. Since the only difference between the VSD pool pump without load shifting and VSD pool pump with load shifting measures pertains to the pool pump operation schedule, this protocol is written in such that it may support both measures at once.

ELIGIBILITY

To qualify for the load shifting rebate, the pumps are required to be off during the hours of 2 PM to 6 PM weekdays. This practice results in additional demand reductions.

ALGORITHMS

This protocol documents the energy savings attributed to variable frequency drive pool pumps in various pool sizes. The target sector primarily consists of single-family residences.

```
 \begin{array}{lll} \Delta kWh/yr & = kWh_{base} - kWh_{VFD} \\ = kWh/yr_{base} & = HOU_{ss} \times kW_{ss} \times Days \\ = (H_{VFD} \times kW_{VFD}) \times Days/year \\ = \left[ \left( HOU_{VFD,clean} \times kW_{VFD,clean} \right) + \left( HOU_{VFD,filter} \times kW_{VFD,filter} \right) \right] \times Days \\ \end{array}
```

The demand reductions are obtained through the following formula:

```
 \Delta kW_{peak} = kW_{basepeak} - kW_{VFDpeak} 
= (CF_{SS} \times kW_{SS}) 
kW_{VFDpeak} 
= \frac{\left[ (HOU_{peak,clean} \times kW_{VFD,clean}) + (HOU_{peak,filter} \times kW_{VFD,filter}) \right]}{4 \ hours} \times CF_{VFD}
```

```
<sup>292</sup> DEER values, updated October 10, 2008
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http://www.deeresources.com/

193 DEER values, updated October 10, 2008

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xlshttp://www.deeresources.com/

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Rev Date: June 2015

The peak coincidencet factor, CF, is defined as the average coincidencet factor during 2 PM to 6 PM on summer weekdays. Ideally, the demand coincidence factor for the supplanted single-speed pump can be obtained from the pump's time clock. The coincidence factor is equal to the number of hours that the pump was set to run between 2 PM to 6 PM, divided by 4. If this information is not available, the recommended daily hours of operation to use are 5.18 and the demand coincidence factor is 30.6%0.27. These operation parameters are derived from the 2011 Mid Atlantic TRM.

DEFINITION OF TERMS

The parameters in the above equation are listed below.

Table 2-118: Residential VFD Pool Pumps Calculations Assumptions

	Component	Type <u>Unit</u>	Values	Source
	HOUSS, Hours of operation per day for Single Speed Pump. This quantity should be recorded by the applicant	hours Variable	EDC Data Gathering Default= 5.18	2
	HOUVFD_filter , Hours of operation per day for Variable Frequency Drive Pump on filtration mode. This quantity should be recorded by the applicant.	hours Variable	EDC Data Gathering Default = 13.00	2
	<u>HOU_{VFD,clean}</u> , Hours of operation per day for Variable Frequency Drive Pump on cleaning mode. This quantity should be recorded by the applicant.	hours day	EDC Data Gathering Default = 0	
	Days, Pool pump days of operation per year.	$\frac{days}{yr}$ Fixed	100	2
	kW _{SS} , Electric demand of single speed pump at a given flow rate. This quantity should be recorded by the applicant or looked up through the horsepower in <u>Table 2-119</u> Table 2-119.	Variable Kilowatts	EDC Data Gathering Default =;-1.364 W or See Table 2-119	1 and Table 2-117 or Table 2-119
	kW _{VFD_filter} , Electric demand of variable frequency drive pump <u>during filtration</u> mode at a given flow rate. This quantity should be measured and recorded by the applicant.	<u>Kilowatts</u>	EDC Data Gathering	
	kW _{VFD, clean} , Electric demand of variable frequency drive pump during cleaning mode. This quantity should be measured and recorded by the applicant.	<u>Kilowatts</u>	EDC Data Gathering	
	HOU _{peak,filter_} , Average daily hours of operation during peak period (between 2pm and 6pm) for Variable Frequency Drive Pump on filtration mode. This quantity should be recorded by the applicant.	hours day	EDC Data Gathering Default = 4	
	HOU _{peak.clean} , Average daily hours of operation during peak period (between 2pm and 6pm) for Variable Frequency Drive	hours day	EDC Data Gathering	

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Component	Type Unit	Values	Source
Pump on cleaning mode. This quantity should be recorded by the applicant.		Default = 0	
CF _{SS} , Peak coincident factor of single speed pump from 2 PM to 6 PM in summer weekday. This quantity can be deduced from the pool pump timer settings for the old pump.	Variable%	EDC Data Gathering Default = Default: 0.23530.6%	3
CF _{VFD} , Peak coincident factor of VFD pump from 2 PM to 6 PM in summer weekday. This quantity should be inferred from the new timer settings.	Fixed%	0EDC Data Gathering	Program- Design

Average Single Speed Pump Electric Demand

Since this measure involves functional pool pumps, actual measurements of pump demand are encouraged. If this is not possible, then the pool pump power can be inferred from the nameplate horsepower. Table 2-119 Table 2-119 shows the average service factor (over-sizing factor), motor efficiency, and electrical power demand per pump size based on California Energy Commission (CEC) appliance database for single speed pool pump²⁹⁴. Note that the power to horsepower ratios appear high because many pumps, in particular those under 2 HP, have high 'service factors'. The true motor capacity is the product of the nameplate horsepower and the service factor.

Table 2-119: Single Speed Pool Pump Specification²⁹⁵

Pump Horse Power (HP)	Average Pump Service Factor	Average Pump Motor Efficiency	Average Pump Power (W)
0.50	1.62	0.66	946
0.75	1.29	0.65	1,081
1.00	1.28	0.70	1,306
1.50	1.19	0.75	1,512
2.00	1.20	0.78	2,040
2.50	1.11	0.77	2,182
3.00	1.21	0.79	2,666

Electric Demand and Pump Flow Rate

The electric demand on a pump is related to pump flow rate, pool hydraulic properties, and the pump motor efficiency. For VFD pumps that have premium efficiency (92%) motors, a regression is used to relate electric demand and pump flow rates using the data from Southern California Edison's Innovative Designs for Energy Efficiency (InDEE) Program. This regression reflects the hydraulic properties of pools that are retrofitted with VSD pool pumps. The regression is:

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²⁹⁴ "CEC Appliances Database – Pool Pumps." *California Energy Commission.* Updated Feb 2008. Accessed March 2008. http://www.energy.ca.gov/appliances/database/historical_excel_files/2009-03-01-excel_based_files/Pool_Products/Pool_Pumps.zip

²⁹⁵ Averaged for all listed single-speed pumps in the last available version of the CEC appliance efficiency database. The powers are for 'CEC Curve A' which represents hydraulic properties of 2" PVC pipes rather than older, 1.5" copper pipes.

Demand (W) = $= 0.0978f^2 + 10.989f + 10.281$

Where f is the pump flow rate in gallons per minute.

This regression can be used if the flow rate is known but the wattage is unknown. However, most VFD pool pumps can display instantaneous flow and power. Power measurements or readings in the final flow configuration are encouraged.

DEFAULT SAVINGS

The energy savings and demand reductions are prescriptive according to the above formulae. All other factors held constant, the sole difference between quantifying demand reductions for the *VSD Pool Pump* and the *VSD Pool Pump with Load Shifting* measures resides in the value of the parameter **CF**_{VFD}

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources²⁹⁶, a variable speed drive's lifespan is 10 years.

EVALUATION PROTOCOL

The most appropriate evaluation protocol for this measure is verification of installation coupled with survey on run time and speed settings. It may be helpful to work with pool service professionals in addition to surveying customers to obtain pump settings, as some customers may not be comfortable operating their pump controls. Working with a pool service professional may enable the evaluator to obtain more data points and more accurate data.

SOURCES

- "CEC Appliances Database Pool Pumps." California Energy Commission. Updated Feb 2008. Accessed March 2008. http://www.energy.ca.gov/appliances/database/historical_excel_files/2009-03-01_excel_based_files/Pool_Products/Pool_Pumps.zip
- 2. Mid-Atlantic TRM, version 2.0. Prepared by Vermont Energy Investment Corporation. Facilitated and managed by the Northeast Energy Efficiency Partnerships. July 2011.
- 3. Derived from Pool Pump and Demand Response Potential, DR 07.01 Report, SCE Design and Engineering, Table 16. Statewide value calculated using the non-weather dependent coincident peak demand calculator with inland valley data. Calculated using the average of the 3 regions. The pool pump operating schedule is not weather dependent, but operator dependent. This is noted on page 22, paragraph 2 of the source. http://www.etcc-ca.com/sites/default/files/OLD/images/stories/pdf/ETCC Report 473.pdf

Field Code Changed

²⁹⁶ DEER values, updated October 10, 2008

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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3 COMMERCIAL AND INDUSTRIAL MEASURES

The following section of the TRM contains savings protocols for commercial and industrial measures.

3.1 LIGHTING

3.1.1 LIGHTING FIXTURE IMPROVEMENTS

Measure Name	<u>Lighting Fixture Improvements</u>
Target Sector	Commercial and Industrial Establishments
Measure Unit	Lighting Equipment
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	13 years ²⁹⁷
Measure Vintage	Early Replacement

ELIGIBILITY

<u>Prescriptive Lighting Lighting Fixture</u> Improvements include fixture or lamp and ballast replacement in existing commercial and industrial customers' facilities.

Note that the Energy Policy Act of 2005 ("EPACT 2005") and Energy Independence and Security Act ("EISA") 2007 standards introduced new efficacy standards for linear fluorescent bulbs and ballasts, effectively phasing out magnetic ballasts (effective October 1, 2010) and most T-12 bulbs (effective July 14, 2012). This induces a shift in what a participant would have purchased in the absence of the program because T-12 bulbs on magnetic ballasts are no longer viable options and, therefore, adjusts the baseline assumption. The baseline for a lighting retrofit project will continue to be the existing lighting system (fixtures, lamps, ballast) for the entirety of Phase II298. This is to reflect the time required for the market to adjust to the new code standards, taking into account the fact that end-users may have an existing stock of T-12 lamps and do not need to purchase new replacement lamps for several years.

With this understanding, these new code standards will not impact the EDCs' first year savings (which will be used to determine EDC compliance). However, these regulatory changes affect the TRC Test valuation for T-12 replacements as the energy savings and useful life are reduced each year due to the changing lighting baseline values as such lighting becomes unavailable. This section describes a methodology to calculate lifetime savings for linear fluorescent measures that replace T-12s in Program Year s 6 (June 1, 2014 — May 31, 2015) and 7 (June 1, 2015 – May 31, 2016) (PY6 and PY7). Standard T-8s become the baseline for all T-12 linear fluorescent retrofits beginning June 1, 2016, should the Commission implement a Phase III of the Act 129 EE&C

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²⁹⁷ Measure Life Study, prepared for the Massachusetts Joint Utilities by ERS. October, 10, 2005. http://rtf.nwcouncil.org/subcommittees/nonreslighting/Measure%20Life%20Study_MA%20Joint%20Utilities_2005_ERS-1.pdf

²⁹⁸ The PEG will continue to monitor the market's activity and that the baseline for lighting retrofit projects be reconsidered during the first TRM update that would be effective during a potential Phase III.

Programs. Therefore, measures installed in PY6 and PY7 will claim full savings until June 1, 2016. Savings adjustment factors²⁹⁹ would be applied to the full savings for savings starting June 1, 2016, and for the remainder of the measure life. Savings adjustments are developed for different combinations of retrofits from T-12s to T-8 or T-5 lighting. In TRC Test calculations, the EDCs may adjust lifetime savings either by applying savings adjustment factors or by reducing the effective useful life³⁰⁰ (EUL) to adjust lifetime savings. Savings adjustment factors and reduced EULs for HPT8 and T5 measures are in

<u>Table 3-2</u>

Table 3-2 and

Table 3-3

Table 3-3 below.

ALGORITHMS

For all lighting fixture improvements (without control improvements), the following algorithms apply:

$$\Delta kWh = (kW_{base} - kW_{ee}) \times [HOU \times (1 - SVG_{base}) \times (1 + IF_{energy})]$$

 $\Delta kW_{peak} = (kW_{base} - kW_{ee}) \times [CF \times (1 - SVG_{base}) \times (1 + IF_{demand})]$

DEFINITION OF TERMS

Table 3-1: Variables for Retrofit Lighting

	<u>Term</u>	<u>Unit</u>	<u>Values</u>	<u>Source</u>
	kW _{base} ,Connected load of the baseline lighting as defined by project classification	<u>kW</u>	See Standard Wattage Table in Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool	Appendix C: Lighting Audit and Design ToolAppendix C:-Lighting Audit and Design Tool
	kW _{ee} , Connected load of the post-retrofit or energy–efficient lgithing system	<u>kW</u>	See Standard Wattage Table in Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool	Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool
	SVG_{base} . Savings factor for existing lighting control (percent of time the lights are off)	None	EDC Data Gathering Default: See Table 3-4	EDC Data Gathering See Table 3-4
İ	CF, Demand Coincidence Factor	None	EDC Data	EDC Data

²⁹⁹ Savings adjustment is defined as the ratio between the wattage reduction from the T-8 baseline to HPT-8 or T-5 lighting and the wattage reduction from the T-12 fixture

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Lighting Page 2

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wattage reduction from the T-12 fixture.

300 EUL adjustments are calculated by applying the savings adjustment factor to the remaining useful life of the measure and reducing the EUL accordingly. The savings adjustment factor methodology and the adjusted EUL methodology will produce the same lifetime savings.

Term	<u>Unit</u>	<u>Values</u>	Source
HOU. Hours of Use – the average annual operating		Gathering Default: See Table 3-5 Table 3-5 EDC Data Gathering	Gathering See Table 3-5 Table 3-5 EDC Data Gathering
hours of the baseline lighting equipment, which if applied to full connected load will yield annual energy use.	<u>Hours</u>	Default: See Table 3-5 Table 3-5	See Table 3-5 Table 3-5
IF _{energy} . Interactive Energy Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary energy savings in cooling required which results from decreased indoor lighting wattage.	None	Default: See Table 3-6 Table 3-6	See Table 3-6 Table 3-6
IF _{demand} , Interactive Demand Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary demand savings in cooling required which results from decreased indoor lighting wattage.	None	Default: See Table 3-6 Table 3-6	See Table 3-6 Table 3-6

Table 3-2: 2016 Savings Adjustment Factors and Adjusted EULs for HPT8 Measures³⁰¹

	Savings Adj	ustment Fact	or	Adjusted EUL		
Fixture Type	T12 EEmag ballast and 34 w lamps	T12 EEmag ballast and 40 w lamps	T12 mag ballast and 40 w lamps	T12 EEmag ballast and 34 w lamps	T12 EEmag ballast and 40 w lamps	T12 mag ballast and 40 w lamps
1-Lamp Relamp/Reballast	53%	70%	80%	<u>7.4</u> 8.9	<u>9.411.1</u>	<u>10.6</u> 12.4
2-Lamp Relamp/Reballast	47%	70%	78%	<u>6.6</u> 8.1	<u>9.4</u> 11.1	<u>10.4</u> 12.1
3-Lamp Relamp/Reballast	58%	62%	79%	<u>8.0</u> 9.5	<u>8.4</u> 10.1	10.512.3
4-Lamp Relamp/Reballast	56%	71%	77%	<u>7.7</u> 9.3	<u>9.5</u> 11.2	<u>10.2</u> 12.0

Table 3-3: 2016 Savings Adjustment Factors and Adjusted EULs for T5 Measures

	Savings Adju	stment Factor	t Factor Adjusted EUL			
Fixture Type	T12 EEmag ballast and 34 w lamps	T12 EEmag ballast and 40 w lamps	T12 mag ballast and 40 w lamps	T12 EEmag ballast and 34 w lamps	T12 EEmag ballast and 40 w lamps	T12 mag ballast and 40 w lamps
1-Lamp T5 Industrial/Strip	58%	71%	76%	<u>8.0</u> 9.5	<u>9.5</u> 11.2	<u>10.1</u> 11.9

^{| 301} The savings adjustment is equal to the ratio between wattage reduction from T8 baseline to HPT8 or T5 lighting and wattage reduction from T12 fixture. These factors are taken from the 2013 IL TRM.

SECTION 3: Commercial and Industrial Measures

2-Lamp T5 Industrial/Strip	39%	60%	66%	<u>5.7</u> 7.1	<u>8.2</u> 9.8	<u>8.910.6</u>
3-Lamp T5 Industrial/Strip	49%	60%	69%	<u>6.9</u> 8.4	<u>8.29.8</u>	<u>9.311.0</u>
4-Lamp T5 Industrial/Strip	40%	59%	49%	<u>5.8</u> 7.2	<u>8.1</u> 9.7	<u>6.9</u> 8.4

Other factors required to calculate savings are shown in Table 3-4Table 3-4.

Table 3-5

Table 3-5, and

Table 3-6

Table 3.6. Note that if HOU is stated and verified by logging lighting hours of use groupings, actual hours should be applied. In addition, the site-specific CF must also be used to calculate savings if actual hours are used. The IF factors shown in

Table 3-6

Table 3-6 are to be used only when the facilities are air conditioned and only for fixtures in conditioned or refrigerated space. The HOU for refrigerated spaces are to be estimated or logged separately. To the extent that operating schedules are known based on metered data, site-specific coincidence factors may be calculated in place of the default coincidence factors provided in

Table 3-5 Table 3-5.

Table 3-4: Savings Control Factors Assumptions302

Strategy	Definition	Technology	Savings %	Sources
Switch	Manual On/Off Switch	Light Switch	0%	
Occupancy	Adjusting light levels according to the presence of occupants	Occupancy Sensors	24%	
	the presence of occupants	Time Clocks	24%	
		Energy Management System	24%	
Daylighting	ng Adjusting light levels automatically in response to the	Photosensors	28%	
	presence of natural light	Time Clocks	28%	
Personal Tuning	Adjusting individual light levels by occupants according to their	Dimmers	31%	
runing	personal preferences; applies, for	Wireless on-off switches	31%	
1	example, to private offices, workstation-specific lighting in	Bi-level switches	31%	1,2,3
1	open-plan offices, and	Computer based controls	31%	1,2,0
	classrooms	Pre-set scene selection	31%	
Institutional Tuning	Adjustment of light levels through commissioning and technology to	Dimmable ballasts	36%	
Turning	meet location specific needs or building policies; or provision of switches or controls for areas or groups of occupants; examples of the former include high-end trim dimming (also known as ballast tuning or reduction of ballast factor), task tuning and lumen maintenance	On-off or dimmer switches for non- personal tuning	36%	

³⁰² Subject to verification by EDC Evaluation or SWE

Γ	Multiple	Includes combination of any of			
	Types	the types described above.	Occupancy and personal		
	**	Occupancy and personal tuning,	tuning/		
		daylighting and occupancy are	daylighting and		
		most common.	occupancy	38%	

Table 3-5: Lighting HOU and CF by Building Type or Function

Building Type	HOU	CF303	Source
Auto Related	4,056	0.62*	<u>10</u> 7
Daycare	2,590	0.62*	<u>11</u> 8
Dusk-to-Dawn / Exterior Lighting	3,833	0.00	<u>5</u> 2
Education – School	1,632	0.31	<u>4</u> 1
Education – College/University	2,348	0.76	<u>4</u> 1
Grocery	4,660	0.87	<u>4</u> 1
Health/Medical – Clinic	3,213	0.73	<u>4</u> 1
Hospitals	5,182	0.80	<u>4</u> 1
Industrial Manufacturing – 1 Shift	2,857	0.57	<u>9</u> 6
Industrial Manufacturing – 2 Shift	4,730	0.57	<u>9</u> 6
Industrial Manufacturing – 3 Shift	6,631	0.57	<u>9</u> 6
Libraries	2,566	0.62*	<u>12</u> 9
Lodging – Guest Rooms	914	0.09	<u>4</u> 1
Lodging – Common Spaces	7,884	0.90	<u>4</u> 1
Multi-Family (Common Areas) - High- rise & Low-rise	5,950	0.62*	<u>6</u> 3
Nursing Home	4,160	0.62*	<u>7</u> 4
Office	2,567	0.61	<u>4</u> 1
Parking Garages	6,552	0.62*	1 <u>3</u> 0
Public Order and Safety	5,366	0.62*	1 <u>4</u> 1
Public Assembly (one shift)	2,610	0.62*	<u>7</u> 4
Public Services (nonfood)	3,425	0.62*	<u>8</u> 5
Restaurant	3,613	0.65	<u>4</u> 1
Retail	2,829	0.73	<u>4</u> 1
Religious Worship/Church	1,810	0.62*	1 <u>5</u> 2
Storage Conditioned/Unconditioned	3,420	0.62*	<u>7</u> 4
Warehouse	2,316	0.54	<u>4</u> 1
24/7 Facilities or Spaces	8,760	1.00	N/A

³⁰³ Coincidence Factor values are taken from the 2011 Mid-Atlantic TRM. For the building types where CF values are not available in the Mid-Atlantic TRM, an average of CF values available for all building types in the Mid-Atlantic TRM is reported. Subject to revision based on detailed measurement or additional research in subsequent TRM Updates.

Building Type	HOU	CF303	Source
Other304	Varies	Varies	<u>4</u> 1

^{* 0.62} represents the simple average of all coincidence factors listed in the 2011 Mid-Atlantic TRM

Table 3-6: Interactive Factors and Other Lighting Variables

1	Component Term	Type Unit	Value <u>s</u>	Source
l	1F _{demand}	<u>None</u> Fixed	Cooled space (60 °F $-$ 79 °F) = 0.34 Freezer spaces (-35 °F $-$ 20 °F) = 0.50 Medium-temperature refrigerated spaces (20 °F $-$ 40 °F) = 0.29 High-temperature refrigerated spaces (40 °F $-$ 60 °F) = 0.18 Un-cooled space = 0	<u>3</u> 1
l	1F _{energy}	<u>None</u> Fixed	Cooled space (60 °F $-$ 79 °F) = 0.12 Freezer spaces (-35 °F $-$ 20 °F) = 0.50 Medium-temperature refrigerated spaces (20 °F $-$ 40 °F) = 0.29 High-temperature refrigerated spaces (40 °F $-$ 60 °F) = 0.18 Un-cooled space = 0	<u>3</u> 4
	kW _{base}	Variable <u>kW</u>	See Standard Wattage Table in Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool	Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool
	kW _{inst}	Variable<u>kW</u>	See Standard Wattage Table in Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool	Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool

Baseline Assumptions Default Savings

There are no default savings associated with this measure.

EVALUATION PROTOCOLS

Methods for Determining Baseline Conditions

The following are acceptable methods for determining baseline conditions when verification by direct inspection is not possible as may occur in a rebate program where customers submit an

³⁰⁴ To be used only when no other category is applicable. Hours of operation must be documented by facility staff interviews, posted schedules, or metered data.

application and equipment receipts only after installing efficient lighting equipment, or for a retroactive project as allowed by Act 129. In order of preference:

- Examination of replaced lighting equipment that is still on site waiting to be recycled or otherwise disposed of-
- Examination of replacement lamp and ballast inventories where the customer has replacement equipment for the retrofitted fixtures in stock. The inventory must be under the control of the customer or customer's agent—
- Interviews with and written statements from customers, facility managers, building engineers or others with firsthand knowledge about purchasing and operating practices at the affected site(s) identifying the lamp and ballast configuration(s) of the baseline condition.
- Interviews with and written statements from the project's lighting contractor or the customer's project coordinator identifying the lamp and ballast configuration(s) of the baseline equipment

- Detailed Inventory Form

For lighting improvement projects, savings are generally proportional to the number of fixtures installed or replaced. The method of savings verification will vary depending on the size of the project because fixtures can be hand-counted to a reasonable degree to a limit.

Projects with connected load savings less than 20 kW

For projects having less than 20_kW in connected load savings, a detailed inventory is not required but information sufficient to validate savings according to the algorithm in 3.1.1Section 4.1.20 must be included in the documentation. This includes identification of baseline equipment utilized for quantifying kW_{base} . Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design ToolContains a prescriptive lighting table, which can estimate savings for small, simple projects under 20 kW in savings provided that the user self-certifies the baseline condition, and information on pre-installation conditions include, at a minimum, lamp type, lamp wattage, ballast type and fixture configuration (2 lamp, 4 lamp, etc.).

Projects with connected load savings of 20 kW or higher

For projects having a connected load savings of 20 kW or higher, a detailed inventory is required. Using the algorithms in this measureSection 1.1.20, ΔkW values will be multiplied by the number of fixtures installed. The total ΔkW savings is derived by summing the total ΔkW for each installed measure.

Within a single project, to the extent there are different control strategies (SVG), hours of use (HOU), coincidence factors (CF) or interactive factors (IF), the ΔkW will be broken out to account for these different factors. This will be accomplished using Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool, a Microsoft Excel inventory form that specifies the lamp and ballast configuration using the Standard Wattage Table and SVG, HOU, CF and IF values for each line entry. The inventory will also specify the location and number of fixtures for reference and validation.

Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool was developed to automate the calculation of energy and demand impacts for retrofit lighting projects, based on a series of entries by the user defining key characteristics of the retrofit project. The main sheet, "Lighting Form", is a detailed line-by-line inventory incorporating variables required to calculate savings. Each line item represents a specific area with common baseline fixtures, retrofit fixtures, controls strategy, space cooling, and space usage.

Baseline and retrofit fixture wattages are determined by selecting the appropriate fixture code from the "Wattage Table" sheet. The "Fixture Code Locator" sheet can be used to find the appropriate

SECTION 3: Commercial and Industrial Measures

code for a particular lamp-ballast combination 305. Actual wattages of fixtures determined by manufacturer's equipment specification sheets or other independent sources may not be used unless (1) the manufacturer's cut sheet indicates that the difference in delta-watts of fixture wattages (i.e. difference in delta watts of baseline and "actual" installed efficient fixture wattage and delta watts of baseline and nearest matching efficient fixture in standard wattage table of appendix C) is more than 10% 306 or (2) the corresponding fixture code is not listed in the Standard Wattage Table. In these cases, alternate wattages for lamp-ballast combinations can be inputted using the "User Input" sheet of Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Desig

Some lighting contractors may have developed in-house lighting inventory forms that are used to determine preliminary estimates of projects. In order to ensure standardization of all lighting projects, <a href="Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool must still be used. However, if a third-party lighting inventory form is provided, entries to <a href="Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool may be condensed into groups sharing common baseline fixtures, retrofit fixtures, space type, building type, and controls. Whereas <a href="Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Desig

Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool will be updated periodically to include new fixtures and technologies available as may be appropriate. Additional guidance can be found in the "Manual" sheet of Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool.

Quantifying Annual Hours of Operation

- Usage Groups and Annual Hours of Use

Projects with connected load savings less than 20 kW

For whole facility lighting projects with connected load savings less than 20 kW, apply stipulated whole building hours shown in <u>Table 3-5-Table 3-5</u>. If the project cannot be described by the categories listed in <u>Table 3-5-Table 3-5</u> or the project retrofitted only a portion of a facility's lighting system for which whole building hours of use would not be appropriate, select the "other" category and determine hours using facility staff interviews, posted schedules, or metered data.

For whole facility lighting projects where the facility's actual lighting hours deviates by more than 10% from Table 3-5Table 3-5

Table 3-2

Table 3.2 hours for the appropriate building type, use of the "other" category may be used at the discretion of the EDCs"s implementation and evaluation contractors. If this option is chosen, EDC implementation and evaluation contractors should apply this methodology consistently throughout a program year for all projects to which it pertains.

For projects using the "other" category, "usage groups" should be considered and used at the discretion of the EDCs' implementation and evaluation contractors in place of stipulated whole

Lighting Page 24-

³⁰⁵ The Locator is intended to assist users locate codes in the Standard Wattage Table. It does not generate new codes or wattages. In a few cases, the fixture code noted in the Standard Wattage Table may not use standard notation. Therefore, these fixtures may not be able to be found using the Locator and a manual search may be necessary to locate the code.

³⁰⁶ This value was agreed upon by the Technical Working Group convened to discuss updates to the TRM. This value is subject to adjustment based on implementation feedback during PY3 and PY4.

building hours, but are not required. Where usage groups are used, fixtures should be separated into "usage groups" that exhibit similar usage patterns. Use of usage groups may be subject to SWE review. Annual hours of use values should be estimated for each group using facility staff interviews, posted schedules, building monitoring system (BMS), or metered data.

Projects with connected load savings of 20 kW or higher

For projects with connected load savings of 20 kW or higher, "usage groups" must be considered and used in place of stipulated whole building hours where possible. Fixtures should be separated into "usage groups" that exhibit similar usage patterns. Annual hours of use values should be estimated for each group using facility staff interviews, posted schedules, building monitoring system (BMS), or metered data.

For all projects, annual hours are subject to adjustment by EDC evaluators or SWE.

- Metering³⁰⁷

Projects with savings below 500,000 kWh

Metering is encouraged for projects with expected savings below 500,000 kWh but have high uncertainty, i.e. where hours are unknown, variable, or difficult to verify. Exact conditions of "high uncertainty" are to be determined by the EDC evaluation contractors to appropriately manage variance. Metering completed by the implementation contractor maybe leveraged by the evaluation contractor, subject to a reasonableness review. Sampling methodologies within a site are to be either discerned by the EDC evaluation contractor based on the characteristics of the facility in question or performed consistent with guidance the EDC EM&V contractor provides.

Projects with savings of 500,000 kWh or higher

For projects with expected savings of 500,000 kWh or higher, metering is required 308 but trend data from BMS is an acceptable substitute. Metering completed by the implementation contractor maybe leveraged by the evaluation contractor, subject to a reasonableness review. Sampling methodologies within a site are to be either discerned by the EDC evaluation contractor and communicated to implementation contractors based on the characteristics of the facility in question or performed consistent with guidance the EDC EM&V contractor provides.

When BMS data is used as a method of obtaining customer-specific data in lieu of metering, the following guidelines should be followed:

- Care should be taken with respect to BMS data, since the programmed schedule may not reflect regular hours of long unscheduled overrides of the lighting system, such as nightly cleaning in office buildings, and may not reflect how the lights were actually used, but only the times of day the common area lighting is commanded on and off by the BMS.
- The BMS trends should represent the actual status of the lights (not just the command sent to the lights), and the ICSP and EC are required to demonstrate that the BMS system is functioning as expected, prior to relying on the data for evaluation purposes.
- The BMS data utilized should be specific to the lighting systems, and should be required to be representative of the building areas included in the lighting project.

SOURCES

 Williams, A., Atkinson, B., Garbesi, K., Rubinstein, F., "A Meta-Analysis of Energy Savings from Lighting Controls in Commercial Buildings", Lawrence Berkeley National Laboratory, September 2011. http://eetd.lbl.gov/sites/all/files/a_meta-analysis_of_energy_savings_from_lighting_controls_in_commercial_buildings_lbnl-5095e.pdf

 ³⁰⁷ The exact variables that should be determined using metering areis shown in Table 3-14 Table 3-14 of the 2015 TRM.
 308 The Commission allows the EDCs to use alternative methods for obtaining customer-sepcific data where customer processes do not support metering. The EDCs are required to provide supporting documentation to the SWE for review if there are any such exceptions.

- Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs, Incremental Cost Study, KEMA, October 28, 2009.
 - https://focusonenergy.com/sites/default/files/bpincrementalcoststudyfinal_evaluationreport.pdf
- 3. 2011 Efficiency Vermont TRM
- The Mid-Atlantic TRM Northeast Energy Efficiency Partnerships, Mid-Atlantic Technical Reference Manual, Version 2.0, submitted by Vermont Energy Investment Corporation, July, 2011.
 - a. Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010.
 - California Public Utility Commission. Database for Energy Efficiency Resources, 2008
 - Small Commercial Contract Group Direct Impact Evaluation Report prepared by Itron for the California Public Utilities Commission Energy Division, February 9, 2010
- State of Ohio Energy Efficiency Technical Reference Manual, Vermont Energy Investment Corporation, August 6, 2010. Exterior lighting 3,833 hours per year assumes 10.5 hours per day; typical average for photocell control.
- Illinois Energy Efficiency Technical Reference Manual, Vermont Energy Investment Corporation, 2012. Multi-family common area value based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010.
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- 9. UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating Company, September 2011. http://www.energizect.com/sites/default/files/2012%20CT%20Program%20Savings%20D ocumentation%20FINAL.pdf
- California Public Utility Commission. Database for Energy Efficiency Resources, 2011; available at www.deeresources.com
- 11. Analysis of 3-"Kinder Care" daycare centers serving 150-160 children per day average 9,175 ft2; 4.9 Watts per ft2; load factor 23.1% estimate 2,208 hours per year. Given an operating assumption of five days per week, 12 hours per day (6:00AM to 6:00 PM) closed weekends (260 days); Closed on 6 NERC holidays that fall on weekdays (2002, 2008 and 2013) deduct 144 hours: (260 X 12)-144 = 2,976 hours per year; assumption adopts an average of measured and operational bases or 2,592 hours per year.
- 12. Southern California Edison Company, Design & Engineering Services, Work Paper WPSCNRMI0054, Revision 0, September 17, 2007, Ventura County Partnership Program, Fillmore Public Library (Ventura County); Two 8-Foot T8 Lamp and Electronic Ballast to Four 4-Foot T8 Lamps and Premium Electronic Ballast. Reference: "The Los Angeles County building study was used to determine the lighting operating hours for this work paper. At Case Site #19A (L.A. County Montebello Public Library), the lights were at full-load during work hours and at zero-load during non-work hours. This and the L.A. County Claremont Library (also referenced in the Los Angeles County building study) are small library branches similar to those of this work paper's library (Ventura County's Fillmore Library). As such, the three locations have the same lighting profile. Therefore,

the lighting operating hour value of 1,664 hours/year stated above is reasonably accurate." Duquesne Light customer data on 29 libraries (SIC 8231) reflects an average load factor 26.4% equivalent to 2285 hours per year. Connecticut Light and Power and United Illuminating Company (CL&P and UI) program savings documentation for 2008 Program Year Table 2.0.0 C&I Hours, page 246 - Libraries 3,748 hours. An average of the three references is 2,566 hours.

- 13. CL&P and UI 2008 program documentation (referenced above) cites an estimated 4,368 hours, only 68 hours greater than dusk to down operating hours. ESNA RP-20-98; Lighting for Parking Facilities acknowledges "Garages usually require supplemental daytime luminance in above-ground facilities, and full day and night lighting for underground facilities." Emphasis added. The adopted assumption of 6,552 increases the CL&P and UI value by 50% (suggest data logging to document greater hours i.e., 8760 hours per year).
- 14. DOE 2003 Commercial Building Energy Survey (CBECS), Table B1. Summary Table: Total and Means of Floor space, Number of Workers, and Hours of Operation for Non-Mall Buildings, Released: June 2006 103 Mean Hours per Week for 71,000 Building Type: "Public Order and Safety" 32 X 52 weeks = 5,366 hour per year. http://www.eia.gov/consumption/commercial/data/2003/pdf/b1rse-b46rse.pdf
- 15. DOE 2003 Commercial Building Energy Survey (CBECS), Table B1. Summary Table: Total and Means of Floor space, Number of Workers, and Hours of Operation for Non-Mall Buildings, Released: June 2006 32 Mean Hours per Week for 370,000 Building Type: "Religious Worship" 32 X 52 weeks = 1,664 hour per year. http://www.eia.gov/consumption/commercial/data/2003/pdf/b1rse-b46rse.pdf

3.1.2 New Construction Lighting

Measure Name	New Construction Lighting
Target Sector	Commercial and Industrial Establishments
Measure Unit	Lighting Equipment
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ³⁰⁹
Measure Vintage	New Construction

New Construction and Major Renovation incentives are intended to encourage decision-makers in new construction and major renovation projects to incorporate greater energy efficiency into their building design and construction practices that will result in a permanent reduction in electrical (kWh) usage above baseline practices,

ELIGIBILITY REQUIREMENTS

New construction applies to new building projects wherein no structure or site footprint presently exists, addition or expansion of an existing building or site footprint, or major tenant improvements that change the use of the space. Eligible lighting equipment and fixture/lamp types include fluorescent fixtures (lamps and ballasts), compact fluorescent lamps, LED exit signs, high intensity discharge (HID) lamps, interior and exterior LED lamps and fixtures, cold-cathode fluorescent lamps (CCFL), induction lamps, and lighting controls. The baseline demand (kW_{base}) for calculating savings is determined using one of the two methods detailed in ASHRAE 90.1-2007. The interior lighting baseline is calculated using the more conservative of the Building Area Method³¹⁰ as shown in Table 3-8Table 3-8Table 1-2 or the Space-by-Space Method³¹¹ as shown in Table 3-9Table 1-3. For exterior lighting, the baseline is calculated using the Baseline Exterior Lighting Power Densities³¹² as shown in Table 3-10Table 3-10Table 1-4. The post-installation demand is calculated based on the installed fixtures using the "06 Wattage Table" sheet in Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Industrial New Construction Projects.

The calculation of annualized energy savings and peak demand savings is based on algorithms through the stipulation of key variables (i.e. Coincidence Factor, Interactive Factor and Hours of Use) and through end-use metering referenced in historical studies or measured, as may be required, at the project level.

For eligibility requirements of solid state lighting products, see Appendix F: Eligibility Requirements for Solid State Lighting Products in Commercial and Industrial Applications Appendix F.

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³⁰⁹ Measure Life Study, prepared for the Massachusetts Joint Utilities by ERS. October, 10, 2005.
http://rtf.nwcouncil.org/subcommittees/nonreslighting/Measure%20Life%20Study MA%20Joint%20Utilities 2005 ERS-1.pdf
310 ASHRAE 90.1-2007, Table 9.5.1 – Building Area Method, https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf
311 ASHRAE 90.1-2007, Table 9.6.1 – Space-by-Space Method, https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf
312 ASHRAE 90.1-2007, Table 9.4.5 – Baseline Exterior Lighting Power Densities.
https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf

ALGORITHMS

For all new construction projects analyzed using the ASHRAE 90.1-2007 **Building Area Method**, the following algorithms apply:

For all new construction projects analyzed using the AHRAE 90.1-2007 **Space-by-Space Method**, the following algorithms apply:

Where n is the number of spaces and:

$$\begin{split} \Delta kWh_1 & = \left(kW_{base,1} - kW_{ee,1}\right) \times \left[HOU_1 \times (1 - SVG_1) \times \left(1 + IF_{energy,1}\right)\right] \\ \Delta kW_{p1} & = \left(kW_{base,1} - kW_{ee,1}\right) \times \left[CF_1 \times \left(1 + IF_{demand,1}\right)\right] \end{split}$$

DEFINITION OF TERMS

Table 3-7: Variables for New Construction Lighting

Term	<u>Unit</u>	<u>Values</u>	Source
kW _{base} . The baseline space or building connected load as calculated by multiplying the space or building area by the appropriate Lighting Power Density (LPD) values specified in either Table 3-8 ar Table 3-99	<u>kW</u>	Calculated based on space or building type and size.	<u>Calculated Value</u>
kW _{ee} . The calculated connected load of the energy efficient lighting	<u>kW</u>	Calculated based on specifications of installed equipment using Appendix E: Lighting Audit and Design Tool for C&I New Construction ProjectsAppendix E: Lighting-Audit and Design Tool for C&I New Construction-ProjectsAppendix E	<u>Calculated Value</u>
SVG, Savings factor for the new lighting control (percent of time the lights are off).	None	Based on Metering Default: See Table 3-13 Table 3-1313	EDC Data Gathering 14
CF, Demand Coincidence Factor	None	Based on Metering ³¹³ Default: See Table 3-11Table-3-11	EDC Data Gathering See Table 3-11Table 3-11
HOU, Hours of Use – the average annual operating hours of the facility.	<u>Hours</u>	Based on Metering ³¹⁴ Default: See Table 3-11Table-3-11	EDC Data Gathering See Table 3-11Table 3-11
IF, Interactive Factor	None	Vary based on building type and space cooling details.	See Table 3-12 Table 3-12

Table 1-2: Lighting Measure Lives by End_Use

Lighting End-Use	
Lighting (Non-SSL) — New/Remodel/Replacement	15
Lighting (SSL - 25,000 hours) New/Remodel/Replacement	6
Lighting (SSL - 30,000 hours) - New/Remodel/Replacement	7
Lighting (SSL – 35,000 hours) — New/Remodel/Replacement	8
Lighting (SSL – 40,000 hours) — New/Remodel/Replacement	10
Lighting (SSL – 45,000 hours) — New/Remodel/Replacement	11

³¹³ It is noted that if site-specific data is used to determine HOU, then the same data must be used to determine the site-specific CF. Similarly, if the default TRM HOU is used, then the default TRM CF must also be used in the savings calculations.

³¹⁴ Ibid.

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Lighting (SSL – 50,000 hours) — New/Remodel/Replacement	12
Lighting (SSL - 55,000 hours) - New/Remodel/Replacement	13
Lighting (SSL - 60,000 hours) - New/Remodel/Replacement	14
Lighting (SSL - ≥60,000 hours) - New/Remodel/Replacement	15*

^{*}For the purpose of calculating the total Resource Cost Test for Act 129, measure cannot claim savings for more than fifteen years.

Table 3-8: Lighting Power Densities from ASHRAE 90.1-2007 Building Area Method315

Building Area Type ³¹⁶	LPD (W/ft2)	Building Area Type	LPD (W/ft2)
Automotive facility	0.9	Multifamily	0.7
Convention center	1.2	Museum	1.1
Courthouse	1.2	Office	1.0
Dining: bar lounge/leisure	1.3	Parking garage	0.3
Dining: cafeteria/fast food	1.4	Penitentiary	1.0
Dining: family	1.6	Performing arts theater	1.6
Dormitory	1.0	Police/fire station	1.0
Exercise center	1.0	Post office	1.1
Gymnasium	1.1	Religious building	1.3
Health-care clinic	1.0	Retail	1.5
Hospital	1.2	School/university	1.2
Hotel	1.0	Sports arena	1.1
Library	1.3	Town hall	1.1
Manufacturing facility	1.3	Transportation	1.0
Motel	1.0	Warehouse	0.8
Motion picture theater	1.2	Workshop	1.4

Lighting

vLivesing End-Use Measureings for this measure. Commercial and Industrial New Construction

³¹⁵ ASHRAE 90.1-2007, "Table 9.5.1 Lighting Power Densities Using the Building Area Method." https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf
³¹⁶ In cases where both a common space type and a building specific type are listed, the building specific space type shall apply.

Table 3-9: Lighting Power Densities from ASHRAE 90.1-2007 Space-by-Space Method317

Common Space Type ³¹⁸	LPD (W/ft2)	Building Specific Space Types	LPD (W/ft2)
Office-Enclosed	1.1	Gymnasium/Exercise Center	
Office-Open Plan	1.1	Playing Area	1.4
Conference/Meeting/Multipurpose	1.3	Exercise Area	0.9
Classroom/Lecture/Training	1.4	Courthouse/Police Station/Penitentia	ry
For Penitentiary	1.3	Courtroom	1.9
Lobby	1.3	Confinement Cells	0.9
For Hotel	1.1	Judges Chambers	1.3
For Performing Arts Theater	3.3	Fire Stations	
For Motion Picture Theater	1.1	Fire Station Engine Room	0.8
Audience/Seating Area	0.9	Sleeping Quarters	0.3
For Gymnasium	0.4	Post Office-Sorting Area	1.2
For Exercise Center	0.3	Convention Center-Exhibit Space	1.3
For Convention Center	0.7	Library	
For Penitentiary	0.7	Card File and Cataloging	1.1
For Religious Buildings	1.7	Stacks	1.7
For Sports Arena	0.4	Reading Area	1.2
For Performing Arts Theater	2.6	Hospital	
For Motion Picture Theater	1.2	Emergency	2.7
For Transportation	0.5	Recovery	0.8
Atrium—First Three Floors	0.6	Nurse Station	1.0
Atrium—Each Additional Floor	0.2	Exam/Treatment	1.5
Lounge/Recreation	1.2	Pharmacy	1.2
For Hospital	0.8	Patient Room	0.7
Dining Area	0.9	Operating Room	2.2
For Penitentiary	1.3	Nursery	0.6
For Hotel	1.3	Medical Supply	1.4
For Motel	1.2	Physical Therapy	0.9
For Bar Lounge/Leisure Dining	1.4	Radiology	0.4

³¹⁷ ASHRAE 90.1-2007, "Table 9.6.1 Lighting Power Densities Using the Space-by-Space Method."

https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf

318 In cases where both a common space type and a building specific type are listed, the building specific space type shall apply.

Lighting

Common Space Type ³¹⁸	LPD (W/ft2)	Building Specific Space Types	LPD (W/ft2)
For Family Dining	2.1	Laundry—Washing	0.6
Food Preparation	1.2	Automotive—Service/Repair	0.7
Laboratory	1.4	Manufacturing	
Restrooms	0.9	Low (<25 ft Floor to Ceiling Height)	1.2
Dressing/Locker/Fitting Room	0.6	High (>25 ft Floor to Ceiling Height)	1.7
Corridor/Transition	0.5	Detailed Manufacturing	2.1
For Hospital	1.0	Equipment Room	1.2
For Manufacturing	0.5	Control Room	0.5
Stairs—Active	0.6	Hotel/Motel Guest Rooms	1.1
Active Storage	0.8	Dormitory—Living Quarters	1.1
For Hospital	0.9	Museum	
Inactive Storage	0.3	General Exhibition	1.0
For Museum	0.8	Restoration	1.7
Electrical/Mechanical	1.5	Bank/Office—Banking Activity Area	1.5
Workshop	1.9	Religious Buildings	
Sales Area	1.7	Worship Pulpit, Choir	2.4
		Fellowship Hall	0.9
		Retail	
		Sales Area [For accent lighting, see 9.3.1.2.1(c)]	1.7
		Mall Concourse	1.7
		Sports Arena	
		Ring Sports Area	2.7
		Court Sports Area	2.3
		Indoor Playing Field Area	1.4
		Warehouse	
		Fine Material Storage	1.4
		Medium/Bulky Material Storage	0.9
		Parking Garage—Garage Area 0.2	
		Transportation	
		Airport—Concourse	0.6
		Air/Train/Bus—Baggage Area	1.0

SECTION 3: Commercial and Industrial Measures

Common Space Type ³¹⁸	LPD (W/ft2)	Building Specific Space Types	LPD (W/ft2)
		Terminal—Ticket Counter	1.5

Table 3-10: Baseline Exterior Lighting Power Densities³¹⁹

Building Exterior	Space Description	LPD
Uncovered Parking Area	Parking Lots and Drives	0.15 W/ft ²
Building Grounds	Walkways less than 10 ft wide	1.0 W/linear foot
	Walkways 10 ft wide or greater	0.2 W/ft ²
	Plaza areas	
	Special feature areas	
	Stairways	1.0 W/ft ²
Building Entrances and Exits	Main entries	30 W/linear foot of door width
	Other doors	20 W/linear foot of door width
Canopies and Overhangs	Free standing and attached and overhangs	1.25 W/ft ²
Outdoor sales	Open areas (including vehicle sales lots)	0.5 W/ft ²
	Street frontage for vehicle sales lots in addition to "open area" allowance	20 W/linear foot
Building facades		0.2 W/ft² for each illuminated wall or surface or 5.0 W/linear foot for each illuminated wall or surface length
Automated teller machines and night depositories		270 W per location plus 90 W per additional ATM per location
Entrances and gatehouse inspection stations at guarded facilities		1.25 W/ft ² of uncovered area
Loading areas for law enforcement, fire, ambulance, and other emergency service vehicles		0.5 W/ft ² of uncovered area
Drive-through windows at fast food restaurants		400 W per drive-through
Parking near 24-hour retail entrances		800 W per main entry

SECTION 3: Commercial and Industrial Measures

³¹⁹ ASHRAE 90.1-2007 Table 9.4.5. https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf

Table 3-11: Lighting HOU and CF by Building Type or Function for New Construction Lighting 320

Building Type	HOU	CF ³²¹	Source
Auto Related	<u>3.083</u> 4,056	0.47* 0.62*	7
Daycare	<u>1,968</u> 2,590	<u>0.47*</u> 0.62*	8
Dusk-to-Dawn / Exterior Lighting	3,833	0.00	2
Education – School	<u>1,240</u> 1,632	<u>0.24</u> 0.31	1
Education – College/University	<u>1,784</u> 2,348	<u>0.58</u> 0.76	1
Grocery	<u>3,542</u> 4,660	<u>0.66</u> 0.87	1
Health/Medical – Clinic	<u>2,442</u> 3,213	<u>0.55</u> 0.73	1
Hospitals	<u>3,938</u> 5,182	0.610.80	1
Industrial Manufacturing – 1 Shift	<u>2,171</u> 2,857	<u>0.43</u> 0.57	6
Industrial Manufacturing – 2 Shift	<u>3,595</u> 4, 730	<u>0.43</u> 0.57	6
Industrial Manufacturing – 3 Shift	<u>6,631</u> 6,631	<u>0.43</u> 0.57	6
Libraries	<u>1,950</u> 2,566	0.47* 0.62*	9
Lodging – Guest Rooms	<u>695</u> 914	<u>0.07</u> 0.09	1
Lodging – Common Spaces	<u>5,992</u> 7,884	0.680.90	1
Multi-Family (Common Areas) - High- rise & Low-rise	<u>4,522</u> 5,950	<u>0.47*0.62*</u>	3
Nursing Home	<u>3,162</u> 4,160	<u>0.47*</u> 0.62*	4
Office	<u>1,951</u> 2,567	0.460.61	1
Parking Garages	<u>4,980</u> 6,552	<u>0.47*</u> 0.62*	10
Public Order and Safety	<u>4,078</u> 5,366	<u>0.47*</u> 0.62*	11
Public Assembly (one shift)	<u>1,984</u> 2,610	<u>0.47*</u> 0.62*	4
Public Services (nonfood)	<u>2,603</u> 3,425	0.47* 0.62*	5
Restaurant	<u>2,746</u> 3,613	<u>0.49</u> 0.65	1
Retail	<u>2,150</u> 2,829	0.550.73	1
Religious Worship/Church	<u>1,376</u> 1,810	<u>0.47*</u> 0.62*	12
Storage Conditioned/Unconditioned	<u>2.599</u> 3,420	0.47* 0.62*	4
Warehouse	<u>1,760</u> 2,316	<u>0.41</u> 0.54	1
24/7 Facilities or Spaces	8,760	1.00	N/A
Other ³²²	Varies	Varies	1

³²⁰ The default HOU and CF values, with the exception of Dusk-to-Dawn/Exterior Lighting and 24/7 Facilities or Spaces, are reduced by 24% to account for the savings associated with installing required controls in new construction buildings.

321 Pre-reduction Coincidence Factor values are taken from the 2011 Mid-Atlantic TRM. For the building types where CF values are

^{321 &}lt;u>Pre-reduction</u> Coincidence Factor values are taken from the 2011 Mid-Atlantic TRM. For the building types where CF values are not available in the Mid-Atlantic TRM, an average of CF values available for all building types in the Mid-Atlantic TRM is reported. <u>Subject to revision based on detailed measurement or additional research in subsequent TRM Updates</u>.

³²² To be used only when no other category is applicable. Hours of operation must be documented by facility staff interviews, posted schedules, or metered data.

Table 3-12: Interactive Factors

	Component <u>Term</u>	Type Unit	Value <u>s</u>	Source
			Cooled space (60 °F – 79 °F) = 0.34	
			Freezer spaces (-35 °F – 20 °F) = 0.50	
	IF_{demand}	FixedNone	Medium-temperature refrigerated spaces (20 $^{\circ}$ F – 40 $^{\circ}$ F) = 0.29	<u>13</u>
-			High-temperature refrigerated spaces (40 $^{\circ}$ F – 60 $^{\circ}$ F) = 0.18	
			Un-cooled space = 0	
			Cooled space (60 °F – 79 °F) = 0.12	
			Freezer spaces (-35 °F $-$ 20 °F) = 0.50	
	IF_{energy}	FixedNone	Medium-temperature refrigerated spaces (20 $^{\circ}$ F – 40 $^{\circ}$ F) = 0.29	<u>13</u>
•			High-temperature refrigerated spaces (40 $^{\circ}$ F – 60 $^{\circ}$ F) = 0.18	
			Un-cooled space = 0	

Table 3-13: Savings Control Factors

Control Strategy	Definition	Technology	SVG ³²³	Sources
Switch	Manual On/Off Switch	Light Switch	0%	
Occupancy	Adjusting light levels according to	Occupancy Sensors	24% 0%	
	the presence of occupants	Time Clocks	24% 0%	
		Energy Management System	24% 0%	14
Daylighting	Adjusting light levels automatically in	Photosensors	28% 10%	
	response to the presence of natural light	Time Clocks	28% 10%	

BASELINE ASSUMPTIONS DEFAULT SAVINGS

There are no default savings associated with this measure.

The following are acceptable methods for determining baseline conditions when verification by direct inspection is not possible as may occur in a rebate program where customers submit an application and equipment receipts only after installing efficient lighting equipment, or for a retroactive project as allowed by Act 129. In order of preference:

Interviews with and written statements from the project's lighting contractor or the customer's project coordinator identifying the lamp and ballast configuration(s) of the baseline equipment

³²³ According to Table G3.2 of the ASHRAE 90.1 standard, a savings factor of 10% is applied to new construction controls above code (dimmers, wireless on-off switches, bi-level switches, etc).

EVALUATION PROTOCOLS

For lighting improvement projects, savings are generally proportional to the number of fixtures installed or replaced. The method of savings verification will vary depending on the size of the project because fixtures can be hand counted to a reasonable degree to a limit.

Projects with connected load savings less than 20 kW

For projects having less than 20kW in connected load savings, a detailed inventory is not required but information sufficient to validate savings according to the algorithm in Section 1.1.2 must be included in the documentation. This includes identification of baseline equipment utilized for quantifying kW base. Appendix C contains a prescriptive lighting table, which can estimate savings for small, simple projects under 20kW in savings provided that the user self-certifies the baseline condition, and information on pre-installation conditions include, at a minimum, lamp type, lamp wattage, ballast type and fixture configuration (2 lamp, 4 lamp, etc.).

Projects with connected load savings of 20 kW or higher

- Detailed Inventory Form

A detailed inventory of all installed fixtures contributing to general light requirements is mandatory for participation in this measure. Lighting that need not be included in the inventory is as follows:

- 1. Display or accent lighting in galleries, museums, and monuments
- 2. Lighting that is integral to:

Equipment or instrumentation and installed by its manufacturer,

Refrigerator and freezer cases (both open and glass-enclosed),

Equipment used for food warming and food preparation,

Medical equipment, or

Advertising or directional signage

- 3. Lighting specifically designed only for use during medical procedures
- 4. Lighting used for plant growth or maintenance
- Lighting used in spaces designed specifically for occupants with special lighting needs
- 6. Lighting in retail display windows that are enclosed by ceiling height partitions.

Within a single project, to the extent that there are different control strategies (SVG), hours of use (HOU), coincidence factors (CF) or interactive factors (IF), the $\triangle kW$ will be broken out to account for these different factors. This will be accomplished using Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects, a Microsoft Excel inventory form that specifies the lamp and ballast configuration using the Standard Wattage Table and SVG, HOU, CF and IF values for each line entry. The inventory will also specify the location and number of fixtures for reference and validation

Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects was developed to automate the calculation of energy and demand impacts for retrofit—New Construction lighting projects, based on a series of entries by the user defining key characteristics of the retrofit project. The main sheet, "Interior Lighting Form", is a detailed line-by-line inventory incorporating variables required to calculate savings. Each line item represents a specific area with common baseline fixtures, retrofitinstalled fixtures, controls strategy, space cooling, and space usage.

Baseline and retrofitInstalled fixture wattages are determined by selecting the appropriate fixture code from the "06 Wattage Table" sheet. The "08 Fixture Code Locator" sheet can be used to find the appropriate code for a particular lamp-ballast combination³²⁴. Actual wattages of fixtures determined by manufacturer's equipment specification sheets or other independent sources may not be used unless (1) the manufacturer's cut sheet indicates that the difference in delta-watts of fixture wattages (i.e. difference in delta watts of baseline and "actual" installed efficient fixture wattage and delta watts of baseline and nearest matching efficient fixture in standard wattage table of Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects E: Lighting Audit and Design Tool for C&I New Construction Projects is more than 10% 325 or (2) the corresponding fixture code is not listed in the Standard Wattage Table. In these cases, alternate wattages for lamp-ballast combinations can be inputted using the "02 Interior User Input" or the "04 Exterior User Input" sheets of Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New onstruction ProjectsC. Documentation supporting the alternate wattages must be provided in the form of manufacturer provided specification sheets or other industry accepted sources (e.g. ENERGY STAR listing, Design Lights Consortium listing). It must cite test data performed under standard ANSI procedures. These exceptions will be used as the basis for periodically updating the Standard Wattage Table to better reflect market conditions and more accurately represent savings.

Some lighting contractors may have developed in-house lighting inventory forms that are used to determine preliminary estimates of projects. In order to ensure standardization of all New Construction lighting projects, <a href="Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Must still be used. However, if a third-party lighting inventory form is provided, entries to <a href="Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects May be condensed into groups sharing common baseline fixtures, retrofitinstalled fixtures, space type, building type, and controls. Whereas <a href="Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects Appendix E: Lighting Audit and Design Tool for C&I New Construction Projects separates fixtures by location to facilitate evaluation and audit activities, third-party forms can serve that specific function if provided.

Appendix E: Lighting Audit and Design Tool for C&I New Construction ProjectsAppendix E: Lighting Audit and Design Tool for C&I New Construction Projects will be updated periodically to include new fixtures and technologies available as may be appropriate. Additional guidance can be found in the "Manual" sheet of Appendix E: Lighting Audit and Design Tool for C&I New Construction ProjectsAppendix E: Lighting Audit and Design Tool for C&I New Construction Projects.

Quantifying Annual Hours of Operation

Usage Groups and Annual Hours of Use

Projects with connected load savings less than 20 kW

For whole facility lighting projects with connected load savings less than 20 kW, apply stipulated whole building hours shown in Table 1-6. If the project cannot be described by the categories listed in Table 1-6, or the project retrofitted only a portion of a facility's lighting system for which

³²⁴ The Locator is intended to assist users locate codes in the Standard Wattage Table. It does not generate new codes or wattages. In a few cases, the fixture code noted in the Standard Wattage Table may not use standard notation. Therefore, these fixtures may not be able to be found using the Locator and a manual search may be necessary to locate the code.

³²⁵ This value was agreed upon by the Technical Working Group convened to discuss updates to the TRM. This value is subject to adjustment based on implementation feedback during PY3 and PY4.

whole building hours of use would not be appropriate, select the "other" category and determine hours using facility staff interviews, posted schedules, or metered data.

For whole facility lighting projects where the facility's actual lighting hours deviates by more than 10% from Table 1-4 hours for the appropriate building type, use of the "other" category may be used at the discretion of the EDC's implementation and evaluation contractors. If this option is chosen, EDC implementation and evaluation contractors should apply this methodology consistently throughout a program year for all projects to which it pertains.

For projects using the "other" category, "usage groups" should be considered and used at the discretion of the EDCs' implementation and evaluation contractors in place of stipulated whole building hours, but are not required. Where usage groups are used, fixtures should be separated into "usage groups" that exhibit similar usage patterns. Use of usage groups may be subject to SWE review. Annual hours of use values should be estimated for each group using facility staff interviews, posted schedules, building monitoring system (BMS), or metered data.

Projects with connected load savings of 20 kW or higher

For all projects, annual hours are subject to adjustment by EDC evaluators or SWE.

- Metering³²⁶

Projects with savings below 500,000 kWh

Metering is encouraged for projects with expected savings below 500,000 kWh but have high uncertainty, i.e. where hours are unknown, variable, or difficult to verify. Exact conditions of "high uncertainty" are to be determined by the EDC evaluation contractors to appropriately manage variance. Metering completed by the implementation contractor maybe leveraged by the evaluation contractor, subject to a reasonableness review. Sampling methodologies within a site are to be either discerned by the EDC evaluation contractor based on the characteristics of the facility in question or performed consistent with guidance the EDC EM&V contractor provides.

Projects with savings of 500,000 kWh or higher

For projects with expected savings of 500,000 kWh or higher, metering is required 327 but trend data from BMS is an acceptable substitute. Metering completed by the implementation contractor maybe leveraged by the evaluation contractor, subject to a reasonableness review. Sampling methodologies within a site are to be either discerned by the EDC evaluation contractor and communicated to implementation contractors based on the characteristics of the facility in question or performed consistent with guidance the EDC EM&V contractor provides.-

When BMS data is used as a method of obtaining customer-specific data in lieu of metering, the following guidelines should be followed:

- Care should be taken with respect to BMS data, since the programmed schedule may not
 reflect regular hours of long unscheduled overrides of the lighting system, such as nightly
 cleaning in office buildings, and may not reflect how the lights were actually used, but
 only the times of day the common area lighting is commanded on and off by the BMS.
- The BMS trends should represent the actual status of the lights (not just the command sent to the lights), and the ICSP and EC are required to demonstrate that the BMS system is functioning as expected, prior to relying on the data for evaluation purposes.
- The BMS data utilized should be specific to the lighting systems, and should be required to be representative of the building areas included in the lighting project.

³²⁶ The exact variables that should be determined using metering is shown in Table 3-8 of this 2014 TRM.

³²⁷ The Commission allows the EDCs to use alternative methods for obtaining customer-sepcific data where customer processes do not support metering. The EDCs are required to provide supporting documentation to the SWE for review if there are any such exceptions.

SOURCES

- The Mid-Atlantic TRM Northeast Energy Efficiency Partnerships, Mid-Atlantic Technical Reference Manual, Version 2.0, submitted by Vermont Energy Investment Corporation, July, 2011.
 - Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010.

Rev Date: June 2015

- California Public Utility Commission. Database for Energy Efficiency Resources, 2008
- Small Commercial Contract Group Direct Impact Evaluation Report prepared by Itron for the California Public Utilities Commission Energy Division, February 9, 2010
- 2. State of Ohio Energy Efficiency Technical Reference Manual, Vermont Energy Investment Corporation, August 6, 2010. Exterior lighting 3,833 hours per year assumes 10.5 hours per day; typical average for photocell control.
- 3. Illinois Energy Efficiency Technical Reference Manual, Vermont Energy Investment Corporation, 2012. Multi-family common area value based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010.
- California Public Utility Commission. Database for Energy Efficiency Resources, 2011. www.deeresources.com
- State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Deemed Savings Manual V1.0", KEMA, March, 2010. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf
- 6. UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating Company, September 2011. http://www.energizect.com/sites/default/files/2012%20CT%20Program%20Savings%20Documentation%20FINAL.pdf
- California Public Utility Commission. Database for Energy Efficiency Resources, 2011; available at www.deeresources.com
- 8. Analysis of 3-"Kinder Care" daycare centers serving 150-160 children per day average 9,175 ft2; 4.9 Watts per ft2; load factor 23.1% estimate 2,208 hours per year. Given an operating assumption of five days per week, 12 hours per day (6:00AM to 6:00 PM) closed weekends (260 days); Closed on 6 NERC holidays that fall on weekdays (2002, 2008 and 2013) deduct 144 hours: (260 X 12)-144 = 2,976 hours per year; assumption adopts an average of measured and operational bases or 2,592 hours per year.
- 9. Southern California Edison Company, Design & Engineering Services, Work Paper WPSCNRMI0054, Revision 0, September 17, 2007, Ventura County Partnership Program, Fillmore Public Library (Ventura County); Two 8-Foot T8 Lamp and Electronic Ballast to Four 4-Foot T8 Lamps and Premium Electronic Ballast. Reference: "The Los Angeles County building study was used to determine the lighting operating hours for this work paper. At Case Site #19A (L.A. County Montebello Public Library), the lights were at full-load during work hours and at zero-load during non-work hours. This and the L.A. County Claremont Library (also referenced in the Los Angeles County building study) are small library branches similar to those of this work paper's library (Ventura County's Fillmore Library). As such, the three locations have the same lighting profile. Therefore, the lighting operating hour value of 1,664 hours/year stated above is reasonably accurate." Duquesne Light customer data on 29 libraries (SIC 8231) reflects an average

load factor 26.4% equivalent to 2285 hours per year. Connecticut Light and Power and United Illuminating Company (CL&P and UI) program savings documentation for 2008 Program Year Table 2.0.0 C&I Hours, page 246 - Libraries 3,748 hours. An average of the three references is 2,566 hours.

- 10. CL&P and UI 2008 program documentation (referenced above) cites an estimated 4,368 hours, only 68 hours greater than dusk to down operating hours. ESNA RP-20-98; Lighting for Parking Facilities acknowledges "Garages usually require supplemental daytime luminance in above-ground facilities, and full day and night lighting for underground facilities." Emphasis added. The adopted assumption of 6,552 increases the CL&P and UI value by 50% (suggest data logging to document greater hours i.e., 8760 hours per year).
- 11. DOE 2003 Commercial Building Energy Survey (CBECS), Table B1. Summary Table: Total and Means of Floor space, Number of Workers, and Hours of Operation for Non-Mall Buildings, Released: June 2006 103 Mean Hours per Week for 71,000 Building Type: "Public Order and Safety" 32 X 52 weeks = 5,366 hour per year. http://www.eia.gov/consumption/commercial/data/2003/pdf/b1rse-b46rse.pdf
- DOE 2003 Commercial Building Energy Survey (CBECS), Table B1. Summary Table:
 Total and Means of Floor space, Number of Workers, and Hours of Operation for Non-Mall Buildings, Released: June 2006 32 Mean Hours per Week for 370,000 Building Type: "Religious Worship" 32 X 52 weeks = 1,664 hour per year.
 http://www.eia.gov/consumption/commercial/data/2003/pdf/b1rse-b46rse.pdf
- 13. VA TRM, Efficiency Vermont. Technical Reference User Manual: Measure Savings Algorithms and Cost Assumptions (July 2008).
- 14. ASHRAE 90.1-2007 Appendix G, Table G3.2 Power Adjustments Percentages for Automatic Lighting Controls for new construction projects. https://law.resource.org/pub/us/code/ibr/ashrae.90.1.2007.pdf

3.1.3 LIGHTING CONTROLS

Measure Name	Lighting Controls
Target Sector	Commercial and Industrial Establishments
Measure Unit	Wattage Controlled
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	8 years 328
Measure Vintage	Retrofit

ELIGIBILITY

Lighting controls turn lights on and off automatically, which are activated by time, light, motion, or sound. The measurement of energy savings is based on algorithms with key variables (e.g. coincidence factor (CF), hours of use (HOU)) provided through existing end-use metering of a sample of facilities or from other utility programs with experience with these measures (i.e., % of annual lighting energy saved by lighting control). These key variables are listed in Table 3-14 Table 3-14.

If a lighting improvement consists of solely lighting controls, the lighting fixture baseline is the existing fixtures with the existing lamps and ballasts or, if retrofitted, new fixtures with new lamps and ballasts as defined in Lighting Audit and Design Tool shown in Appendix C: Lighting Audit and Design Tool Shown in Appendix C: Lighting Audit and Design Tool. In either case, the $\frac{kW_{ee}}{kW_{ee}}$ for the purpose of the algorithm is set to $\frac{kW_{hase}}{kW_{ee}}$.

ALGORITHMS

 $\triangle kWh$ = $kW_{controlled} \times HOU \times (SVG_{ee} - SVG_{base}) \times (1 + IF_{energy})$

 ΔkW_{veak} = $kW_{controlled} \times (SVG_{ee} - SVG_{base}) \times (1 + IF_{demand}) \times CF$

DEFINITION OF TERMS

328 DEER Effective Useful Life. October 10, 2008.

Table 3-14: Lighting Controls Assumptions

ĺ	Component-Term	<u>Unit</u> Type	Value <u>s</u>	Source
Ī	kW _{controlled} , Total lighting load connected to the new control in kilowatts. Savings are per control. The total connected load per control should be collected from the customer or the default values shown in Table 3-14Table 3-14Table 1-1 should be used.	Lighting Audit and Design Tool in Appendix C: Lighting Audit and Design ToolAppendix C: Lighting Audit and Design Tool		EDC Data Gathering
		Variable- <u>kW</u>	Default: Wall mounted occupancy sensor: 0.350 Remote mounted occupancy sensor: 0.587 Fixture mounted occupancy sensor: 0.073	2, 3
Ĩ	SVG_{base} and SVG_{ee} , Savings factor for baseline lighting and new lighting control (percent of time the lights are off), typically manual switch.	Variable <u>None</u>	Based on metering	EDC Data Gathering
			Default: See <u>Table 3-4</u> Table 3-44	1
			Based on metering ³²⁹	EDC Data Gathering
	CF, Demand Coincidence Factor (See Section 1.5)	Variable None	By building type and size	EDC Data Gathering 1 EDC Data Gathering 1 EDC Data Gathering See Table 3-5: Lighting HOU and CF by Building Type or Function EDC Data Gathering See Table 3-5: Lighting HOU and CF- by Building Type or Function EDC Data Gathering See Table 3-5: Lighting HOU and CF by Building Type
			Based on metering ³³⁰	
	HOU, Hours of Use – the average annual operating hours of the baseline lighting equipment (before the lighting controls are in place), which if applied to full connected load will yield annual energy use.	Variable <u>Hours</u>	By building type and size	Lighting HOU and CF by

³²⁹ It is noted that if site-specific data is used to determine HOU, then the same data must be used to determine the site-specific CF. Similarly, if the default TRM HOU is used, then the default TRM CF must also be used in the savings calculations.

330 Ibid.

			by Building Type or Function
IF, Interactive Factor	Variable None	By building type and size	See Table 3-6: Interactive Factors and Other Lighting Variables Table 3-6: Interactive Factors and Other Lighting Variables

Table 3-15: Savings Control Factors Assumptions³³¹

Strategy	Definition	Technology	Savings %
Switch	Manual On/Off Switch	Light Switch	0%
Occupancy	Adjusting light levels according to the presence of occupants	Occupancy Sensors	24%
	occupants	Time Clocks	24%
		Energy Management System	24%
Daylighting	Adjusting light levels automatically in response to-	Photosensors	28%
	the presence of natural light	Time Clocks	28%
Personal Tuning	Adjusting individual light levels by occupants according to their personal preferences; applies, for	Dimmers	31%
Turning	example, to private offices, workstation-specific-lighting in open-plan offices, and classrooms	Wireless on-off- switches	31%
		Bi-level switches	31%
		Computer based controls	31%
		Pre-set scene-selection	31%
Institutional- Tuning	Adjustment of light levels through commissioning and technology to meet location specific needs or	Dimmable ballasts	36%
+umy	building policies; or provision of switches or- controls for areas or groups of occupants; examples of the former include high-end trim- dimming (also known as ballast tuning or reduction- of ballast factor), task tuning and lumen- maintenance	On-off or dimmer- switches for non- personal tuning	36%
Multiple- Types	Includes combination of any of the types described- above. Occupancy and personal tuning, daylighting and occupancy are most common.	Occupancy and- personal tuning/- daylighting and- occupancy	38%

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate

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³³¹ Subject to verification by EDC Evaluation or SWE

<u>evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables.</u>

It is noted that if site-specific data is used to determine HOU, then the same data must be used to determine the site-specific CF. Similarly, if the default TRM HOU is used, then the default TRM CF must also be used in the savings calculations. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Williams, A., Atkinson, B., Garbesi, K., Rubinstein, F., "A Meta-Analysis of Energy Savings from Lighting Controls in Commercial Buildings", Lawrence Berkeley National Laboratory, September 2011. http://eetd.lbl.gov/sites/all/files/a_meta-analysis of energy savings from lighting controls in commercial buildings lbnl-5095e.pdf
- Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs, Incremental Cost Study, KEMA, October 28, 2009. _ https://focusonenergy.com/sites/default/files/bpincrementalcoststudyfinal_evaluationrepor
 - https://iocusorienergy.com/sites/derauli/iiles/opincrementalcoststudylinal_evaluationreport.pdf
- 3. 2011 Efficiency Vermont TRM. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf

3.1.4 **TRAFFIC LIGHTS**

Measure Name	Traffic Lights
Target Sector	Government, Non-Profit and Institutional
Measure Unit	Traffic Light
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	10 years ³³² -
Measure Vintage	Early Replacement

ELIGIBILITY

This protocol applies to the early replacement of existing incandescent traffic lights and pedestrian signals with LEDs. New LED traffic signals must comply with ENERGY STAR requirements. 333

ALGORITHMS

AkWh	$= (kW_{base} - kW_{ee}) \times HOU$
ΔkW_{peak}	$= (kW_{base} - kW_{ee}) \times GF$
∆kWh	$= (kW_{base} - kW_{ee}) \times HOU$
ΔkW_{peak}	$= (kW_{base} - kW_{ee}) \times CF$

Lighting

³³² The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007. http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf
333ENERGY STAR Program Requirements for Traffic Signals.
https://www.energystar.gov/ia/partners/product_specs/eligibility/traffic_elig.pdf?98bf-1786

DEFINITION OF TERMS

Table 3-15: Assumptions for LED Traffic Signals

Term	Unit	Values	Source
kW _{base} , The connected load of the baseline lighting as defined by project classification.	kW	Vary based on fixture details, See <u>Table 3-16</u> Table 3-16	2, 3, 4, 5
kW _{ee} , The connected load of the post-retrofit or energy-efficient lighting system.	kW	Vary based on fixture details, See <u>Table 3-16</u> Table 3-16	2, 3, 4, 5
CF, Demand Coincidence Factor	None	Default: Red Round: 55% Yellow Round: 2% Round Green: 43% Red Arrow: 86% Yellow Arrow: 8% Green Arrow: 8% Pedestrian: 100%	1
HOU, Hours of use	Hours	Default: Round Red: 4,818 Round Yellow: 175 Round Green: 3,767 Red Arrow: 7,358 Yellow Arrow: 701 Green Arrow: 701 Pedestrian: 8,760	1

DEFAULT SAVINGS

Table 3-16: Default Values for Traffic Signal and Pedestrian Signage Upgrades

Fixture Type	% Burn	kW _{base}	kW _{ee}	ΔkW_{peak}	ΔkWh	Sources
Round Traffic Signals						
8" Red	55%	0.069	0.006	0.035	304	
8" Yellow	2%	0.069	0.007	0.001	11	5, 2
8" Green	43%	0.069	0.008	0.026	230	
12" Red	55%	0.150	0.006	0.079	694	
12" Yellow	2%	0.150	0.012	0.003	24	5, 2
12" Green	43%	0.150	0.007	0.061	539	
Turn Arrows				•		
8" Red	84%	0.116	0.005	0.093	817	
8" Yellow	8%	0.116	0.014	0.008	71	5, 3
8" Green	8%	0.116	0.006	0.009	77	
12" Red	84%	0.116	0.006	0.092	809	
12" Yellow	8%	0.116	0.006	0.009	77	5, 2
12" Green	8%	0.116	0.006	0.009	77	
Pedestrian Signs (All Bur	n 100%)					
9" Hand Only		0.116	0.008	0.108	946	
9" Pedestrian Only		0.116	0.006	0.110	964	
12" Hand Only		0.116	0.008	0.108	946	
12" Pedestrian Only		0.116	0.007	0.109	955	
12" Countdown Only		0.116	0.005	0.111	972	5, 2
12" Pedestrian and Hand Overlay		0.116	0.007	0.109	955	
16" Pedestrian and Hand Side by Side		0.116	0.008	0.108	946	
16" Pedestrian and Hand Overlay		0.116	0.007	0.109	955	
16" Hand with Countdown Side-by-side		0.116	0.010	0.106	929	
16" Pedestrian and Hand w Countdown Overlay	ith	0.116	0.008	0.108	946	5, 4

Notes:

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. PECO Comments on the PA TRM, received March 30, 2009.
- 2. ITE Compliant LED Signal Modules Catalog by Dialight. http://www.dialight.com

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¹⁾ Energy Savings (kWh) are annual per lamp.

²⁾ Demand Savings (kW_{peak}) listed are per lamp.

- 3. RX11 LED Signal Modules Spec Sheet by GE Lighting Solutions, http://www.gelighting.com
- 4. LED Countdown Pedestrian Signals Spec Sheet by GE Lighting Solutions, http://www.gelighting.com
- 5. GE Lighting Product Catalog by GE Lighting Solutions. http://genet.gelighting.com

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3.1.5 LED EXIT SIGNS

Measure Name	LED Exit Signs
Target Sector	Commercial and Industrial Establishments
Measure Unit	LED Exit Sign
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	16 years ³³⁴
Measure Vintage	Early Replacement

ELIGIBILITY

This measure includes the early replacement of existing incandescent or fluorescent exit signs with a new LED exit sign. If the exit signs match those listed in Table 3-17 Table 3-17, the default savings value for LED exit signs installed cooled spaces can be used without completing Appendix C: Lighting Audit and Design Tool Appendix C: Lighting Audit and Design Tool.

ALGORITHMS

 $\Delta kWh = (kW_{base} - kW_{ee}) \times [HOU \times (1 + IF_{energy})]$ $\Delta kW_{peak} = (kW_{base} - kW_{ee}) \times [CF \times (1 + IF_{demand})]_{-}$

³³⁴ DEER Effective Useful Life. October 10, 2008. For the purpose of calculating the total Resource Cost Test for Act 129. measure cannot claim savings for more than 15 years.

DEFINITION OF TERMS

Table 3-17: LED Exit Signs Calculation Assumptions

Term	Unit	Values	Source	
		Actual Wattage	EDC Data Gathering	
kW_{base} , Connected load of baseline lighting as defined by project classification		Single-Sided Incandescent: 0.020 kW		
	kW	kW Dual-Sided Incandescent: 0.040 kW Single-Sided Fluorescent: 0.009 kW Append Audit Tool		
			1001	
		Dual-Sided Fluorescent: 0.020 kW		
		Actual Wattage	EDC Data Gathering	
kW _{ee} . Connected load of the post-retrofit or energy-efficient lighting	kW	Single-Sided: 0.002 kW Dual-Sided: 0.004 kW	Appendix C: Lighting Audit and Design Tool	
CF, Demand Coincidence Factor	None	1.0	1	
HOU, Hours of Use – the average annual operating hours of the baseline lighting equipment.	Hours	8,760	1	
IF_{energy} , Interactive HVAC Energy Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary energy savings in cooling required which results from decreased indoor lighting wattage.	None	See: Table 3-6Table 3-6	Table 3-6 Table 3-6	
IF_{demand} . Interactive HVAC Demand Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary demand savings in cooling required which results from decreased indoor lighting wattage.	None	See: Error! Reference source not found. Table 3-6.	Table 3-6 Table 3-6	

DEFAULT SAVINGS

Single-Sided LED Exit Signs replacing Incandescent Exit Signs in Cooled Spaces

 ΔkWh = 176 kWh

 ΔkW_{peak} = 0.024 kW

Dual-Sided LED Exit Signs replacing Incandescent Exit Signs in Cooled Spaces

 ΔkWh = 353 kWh

 $\Delta kW_{peak} = 0.048 \; kW$

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Single-Sided LED Exit Signs replacing Fluorescent Exit Signs in Cooled Spaces

 ΔkWh = 69 kWh

 $\Delta kW_{peak} = 0.009 \ kW$

Dual-Sided LED Exit Signs replacing Fluorescent Exit Signs in Cooled Spaces

 ΔkWh = 157 kWh

 ΔkW_{peak} = 0.021 kW

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

WI Focus on Energy, "Business Programs: Deemed Savings Manual V1.0." Update
Date: March 22, 2010. LED Exit Lighting.
https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf

3.1.6 **LED CHANNEL SIGNAGE**

Measure Name	LED Channel Signage
Target Sector	Commercial and Industrial Establishments
Measure Unit	LED Channel Signage
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	<u>15 years³³⁵</u>
Measure Vintage	Early Replacement

Rev Date: June 2015

Channel signage refers to the illuminated signs found inside and outside shopping malls to identify store names. Typically these signs are constructed from sheet metal sides forming the shape of letters and a translucent plastic lens. Luminance is most commonly provided by single or double strip neon lamps, powered by neon sign transformers. Retrofit kits are available to upgrade existing signage from neon to LED light sources, substantially reducing the electrical power and energy required for equivalent sign luminance. Red, green, blue, yellow, and white LEDs are available, but at a higher cost than red LEDs. Red is the most common color and the most cost-effective to retrofit, currently comprising approximately 80% of the market.

ELIGIBILITY

This measure must replace inefficient argon-mercury or neon channel letter signs with efficient LED channel letter signs. Retrofit kits or complete replacement LED signs are eligible. Replacement signs cannot use more than $20\%^{336}$ of the actual input power of the sign that is replaced. Measure the length of the sign as follows:

- Measure the length of each individual letter at the centerline. Do not measure the distance between letters.
- Add up the measurements of each individual letter to get the length of the entire sign being replaced.

ALGORITHMS

The savings are calculated using the equations below and the assumptions in Table 3-18 Table

Indoor applications:

Lighting

³³⁵ Southern California Edison Company, LED Channel Letter Signage (Red), Work

Paper SCE13LG052, Revision, April 6, 2012. DEER only includes an LED Exit Sign measure which was used to estimate the effective useful life of the LED Channel Letter Signage. The Work Paper assumes 16 years for interior and exterior applications. The measure life is capped at 15 years per Act 129.

336 http://www.aepohio.com/global/utilities/lib/docs/save/programs/Application_Steps_Incentive_Process.pdf

$$\Delta kWh = \begin{bmatrix} kW_{base} \times \left(1 + IF_{energy}\right) \times HOU \times (1 - SVG_{base}) \end{bmatrix} \\ - \begin{bmatrix} kW_{ee} \times \left(1 + IF_{energy}\right) \times HOU \times (1 - SVG_{ee}) \end{bmatrix} \\ = \begin{bmatrix} kW_{base} \times \left(1 + IF_{demand}\right) \times CF \times (1 - SVG_{base}) \end{bmatrix} \\ - \begin{bmatrix} kW_{ee} \times \left(1 + IF_{demand}\right) \times CF \times (1 - SVG_{ee}) \end{bmatrix}$$

Outdoor applications:

```
 \begin{split} &= [kW_{base} \times HOU \times (1 - SVG_{base})] - [kW_{ee} \times HOU \times (1 - SVG_{ee})] \\ &= [kW_{base} \times CF \times (1 - SVG_{base})] - [kW_{ee} \times CF \times (1 - SVG_{ee})] \end{split} 
\Delta kWh^{337}
\Delta kW_{peak}
```

DEFINITION OF TERMS

Table 3-18: LED Channel Signage Calculation Assumptions

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
kW_{base} , kW of baseline (preretrofit) lighting	<u>kW</u> Variable	EDC Data Gathering Default: See Table 3-1913338	EDC Data Gathering
kW_{ee} , kW of post-retrofit or energy-efficient lighting system (LED) lighting per letter	<u>kW</u> Variable	EDC Data Gathering Default: See Table 3-1913339	EDC Data Gathering
CF, Demand Coincidence Factor (See Section 1.5)	<u>None</u> Variable	EDC Data Gathering Default for Indoor Applications: See Table 3-5: Lighting HOU and CF by Building Type or FunctionTable 3-5: Lighting HOU and CF by- Building Type or Function Default for Outdoor Applications: 0 ³⁴⁰	EDC Data Gathering Table 3-5: Lighting HOU and CF by Building Type or Function Table 3-5: Lighting HOU and CF by Building Type or Function
HOU, Hours of Use	<u>Hours</u> Variable	EDC Data Gathering Default: See <u>Table 3-5:</u> Lighting HOU and CF by Building Type or <u>FunctionTable 3-5:</u> Lighting HOU and CF by Building Type or <u>Function</u> Table 3-5: Lighting HOU and CF by Building Type or <u>Function</u>	EDC Data Gathering Table 3-5: Lighting HOU and CF by Building Type or FunctionTable 3-5: Lighting HOU and CF by Building Type or Function

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Lighting

 ³³⁷ For exterior measures, energy interactive effects are not included in the energy savings calculations.
 338 Average values were estimated based on wattages data obtained from major channel letter lighting product manufacturers. San Diego Gas & Electric, LED Channel Letter Signs, Work Paper WPSDGENRLG0021, Revision #1, August 25, 2010.

³⁴⁰ The peak demand reduction is zero, as the exterior lighting applications are assumed to be in operation during off-peak hours and have a peak coincidence factor of 0.0.

Field Code Changed

Field Code Changed
Field Code Changed

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
IF _{demand} , Interactive HVAC Demand Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary demand savings in cooling required which results from decreased indoor lighting wattage.	FixedNone	See Table 3-6: Interactive Factors and Other Lighting VariablesTable 3-6: Interactive Factors and Other Lighting VariablesTable (Interactive Factors and Other Lighting Variables)	Table 3-71
IF _{energy} , Interactive HVAC Energy Factor – applies to C&I interior lighting in space that has air conditioning or refrigeration only. This represents the secondary energy savings in cooling required which results from decreased indoor lighting wattage.	FixedNone	See_Table 3-6: Interactive Factors and Other Lighting VariablesTable (Interactive Factors and Other Lighting Variables)	Table 3-71
SVG_{base} , Savings factor for existing lighting control (percent of time the lights are off), typically manual switch.	Variable <u>None</u>	Default: See Table (Savings Control Factor Assumptions)	Table (Savings Control Factor Assumptions)
SVG_{ee} . Savings factor for new lighting control (percent of time the lights are off).	Variable None	Default: See Table (Savings Control Factor Assumptions)	Table (Savings- Control Factor- Assumptions)

Table 3-19: Power demand of baseline (neon and argon-mercury) and energy-efficient (LED) signs

	Power Demand (kW/letter)		Power Demand (kW/letter)	
Sign Height	Neon	Red LED	Argon-mercury	White LED
≤ 2 f<u>ff</u> .	0.043	0.006	0.034	0.004
> 2 fl[t] .	0.108	0.014	0.086	0.008

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOL

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables.

It is noted that if site-specific data is used to determine HOU, then the same data must be used to determine the site-specific CF. Similarly, if the default TRM HOU is used, then the default TRM CF must also be used in the savings calculations. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

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SOURCES

 Efficiency Vermont. Technical Reference User Manual: Measure Savings Algorithms and Cost Assumptions (July 2008).

SECTION 3: Commercial and Industrial Measures

3.1.7 LED REFRIGERATION DISPLAY CASE LIGHTING

Measure Name	LED Refrigeration Display Case Lighting	
Target Sector	Commercial and Industrial Establishments	
Measure Unit	Refrigeration Display Case Lighting	
Unit Energy Savings	Variable	
Unit Peak Demand Reduction	Variable	
Measure Life	8 years ³⁴¹	
Measure Vintage	Early Replacement	

This protocol applies to LED lamps with and without motion sensors installed in vertical display refrigerators, coolers, and freezers replacing T8 or T12 linear fluorescent lamps. The LED lamps produce less waste heat than the fluorescent baseline lamps, decreasing the cooling load on the refrigeration system and energy needed by the refrigerator compressor. Additional savings can be achieved from the installation of motion sensors which dim the lights when the space is unoccupied.

ELIGIBILITY

This measure is targeted to non-residential customers who install LED case lighting with or without motion sensors on refrigerators, coolers, and freezers - specifically on vertical displays. The baseline equipment is assumed to be cases with uncontrolled T8 or T12 linear fluorescent lamps.

ALGORITHMS

Savings and assumptions are based on a per door basis.

$$\Delta kWh = \frac{(WATTS_{base} - WATTS_{ee})}{1000} \times N_{doors} \times HOURS \times (1 + IE)$$

$$= \frac{(WATTS_{base} - WATTS_{ee})}{1000} \times N_{doors} \times (1 + IE) \times CF$$

³⁴¹ Theobald, M. A., Emerging Technologies Program: Application Assessment Report #0608, LED Supermarket Case Lighting Grocery Store, Northern California, Pacific Gas and Electric Company, January 2006. http://www.etcc-ca.com/images/stories/pdf/ETCC Report 204.pdf. Assumes 6,205 annual operating hours and 50,000 lifetime hours. Most case lighting runs continuously (24/7) but some can be controlled. 6,205 annual hours of use can be used to represent the mix. Using grocery store hours of use (4,660 hr) is too conservative since case lighting is not tied to store lighting.

State of Pennsylvania – Technical Reference Manual – Rev Date: June 2015

DEFINITION OF TERMS

Table 3-20: LED: Refrigeration Case Lighting - Values and References

Term	Unit	Values	Source
WATTS _{base} , Connected wattage of baseline fixtures	Watts	EDC Data Gathering	EDC Data Gathering
WATTS _{ee} , Connected wattage of efficient fixtures	Watts	EDC Data Gathering	EDC Data Gathering
N_{doors} , Number of doors	None	EDC Data Gathering	EDC Data Gathering
HOURS, Annual operating hours	Hours	EDC Data Gathering Default: 6,205	1
IE, Interactive Effects factor for energy to account for cooling savings from efficient lighting	None	Refrigerator and cooler: 0.41 Freezer: 0.52	2
CF, Coincidence factor	None	0.92	3
1000, Conversion factor from watts to kilowatts	$\frac{W}{kW}$	1000	Conversion Factor

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Theobald, M. A., Emerging Technologies Program: Application Assessment Report #0608, LED Supermarket Case Lighting Grocery Store, Northern California, Pacific Gas and Electric Company, January 2006. http://www.etcc-ca.com/images/stories/pdf/ETCC Report 204.pdf. Assumes 6,205 annual operating hours and 50,000 lifetime hours. Most case lighting runs continuously (24/7) but some can be controlled. 6,205 annual hours of use can be used to represent the mix. Using grocery store hours of use (4,660 hr) is too conservative since case lighting is not tied to store lighting.
- 2. Values adopted from Hall, N. et al, New York Standard Approach for Estimating Energy Savings from Energy Efficiency Measures in Commercial and Industrial Programs,

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TecMarket Works, September 1, 2009. http://www3.dps.ny.gov/W/PSCWeb.nsf/0/06f2fee55575bd8a852576e4006f9af7/\$FILE/TechManualNYRevised10-15-10.pdf

3. Methodology adapted from Kuiken et al, "State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Deemed Savings Parameter Development", KEMA, November 13, 2009, assuming summer coincident peak period is defined as June through August on weekdays between 3:00 p.m. and 6:00 p.m., unless otherwise noted. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf

3.2 HVAC

3.2.1 HVAC SYSTEMS

Measure Name	HVAC Systems
Target Sector	Commercial and Industrial Establishments
Measure Unit	HVAC System
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	Variable
Measure Life	15 years ³⁴²
Measure Vintage	Replace on Burnout, New Construction, or Early Replacement

ELIGIBILITY

The energy and demand savings for Commercial and Industrial HVAC systems is determined from the algorithms listed below. This protocol excludes water source, ground source, and groundwater source heat pumps measures that are covered in Section 3.2.3. All HVAC applications other than comfort cooling and heating, such as process cooling, are defined as non-standard applications and are ineligible for this measure.

ALGORITHMS

Air Conditioning (includes central AC, air-cooled DX, split systems, and packaged terminal AC)

For A/C units < 65,000 $\frac{Btu}{hr}$, use SEER instead of EER-to calculate ΔkWh and convert SEER to EER to calculate ΔkW_{peak} using 11.3/13 as the conversion factor. For units rated in both EER and IEER, use IEER for energy savings calculations.

$$\begin{split} \Delta kWh &= \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{EER_{base}} - \frac{1}{EER_{ee}}\right) \times EFLH_{cool} \\ &= \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{IEER_{base}} - \frac{1}{IEER_{ee}}\right) \times EFLH_{cool} \\ &= \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{SEER_{base}} - \frac{1}{SEER_{ee}}\right) \times EFLH_{cool} \\ &= \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{EER_{base}} - \frac{1}{EER_{ee}}\right) \times CF \end{split}$$

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³⁴² The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007. http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

Air Source and Packaged Terminal Heat Pump

For ASHP units < $65,000 \, \frac{Btu}{hr}$, use SEER instead of EER to calculate ΔkWh_{cool} and HSPF instead of COP to calculate ΔkWh_{heat} . Convert SEER to EER to calculate ΔkW_{peak} using 11.3/13 as the conversion factor. For units rated in both EER and IEER, use IEER for energy savings calculations.

$$\Delta kWh_{cool} + \Delta kWh_{heat} \\ = \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{EER_{base}} - \frac{1}{EER_{ee}}\right) \times EFLH_{cool} \\ = \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{IEER_{base}} - \frac{1}{IEER_{ee}}\right) \times EFLH_{cool} \\ = \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{SEER_{base}} - \frac{1}{SEER_{ee}}\right) \times EFLH_{cool} \\ = \left(\frac{Btu_{heat}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{SEER_{base}} - \frac{1}{SEER_{ee}}\right) \times EFLH_{cool} \\ = \left(\frac{Btu_{heat}}{hr} \times \frac{1}{1000\,W}\right) \times \frac{1}{3.412} \times \left(\frac{1}{COP_{base}} - \frac{1}{COP_{ee}}\right) \times EFLH_{heat} \\ = \left(\frac{Btu_{heat}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{HSPF_{base}} - \frac{1}{HSPF_{ee}}\right) \times EFLH_{heat} \\ = \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000\,W}\right) \times \left(\frac{1}{EER_{base}} - \frac{1}{EER_{ee}}\right) \times CF$$

DEFINITION OF TERMS

Table 3-21: Variables for HVAC Systems

Term_Component	<u>Unit</u> Type	Value <u>s</u>	Source
$\frac{\mathit{Btu}_{\mathit{cool}}}{\mathit{hr}}$, Rated cooling capacity of the energy efficient unit	Btu hr Variabl e	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
$\frac{Btu_{heat}}{hr}$, Rated heating capacity of the energy efficient unit	Hr Variable	Nameplate data (AHRI or AHAM)	EDC Data Gathering
		Early Replacement: Nameplate data	ED <u>C</u> Data Gathering
$\it IEER_{base}$, Integrated energy efficiency ratio of the baseline unit.	$\frac{Btu/hr}{W}$	New Construction or Replace on Burnout: Default values from Table 3-22Table 3-22	See <u>Table 3-22Table</u> 3-22
$\it IEER_{ee}$, Integrated energy efficiency ratio of the energy efficientey unit.	$\frac{Btu/_{hr}}{W}$	Nameplate data (AHRI or AHAM)	EDC Data Gathering
EER _{base} , Energy efficiency ratio rating of the baseline unit. For air-source AC and	$\frac{{}^{Btu}/{}_{hr}}{W}$ Varia	Early Replacement: Nameplate data	EDC's Data Gathering

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Term_Component	<u>Unit</u> Type	Value <u>s</u>	Source
ASHP units < 65,000 $\frac{Btu}{hr}$, SEER should be used for cooling savings	ble -	New Construction or Replace on Burnout: Default values from Table 3-22Table 3-22	See <u>Table 3-22</u> Table 3-22
EER_{ee} , Energy efficiency ratiorating of the energy efficientey unit. For air-source AC and ASHP units < 65,000 $\frac{Btu}{hr}$, SEER should be used for cooling savings.	Btu/ _{hr} W - Variable	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
		Early Replacement: Nameplate data	EDC ¹ Data Gathering
$SEER_{base}$, Seasonal <u>energy</u> efficiency ratioing of the baseline unit. For units > 65,000 $\frac{Btu}{hr}$, EER should be used for cooling savings.	Btu/hr W Varia	New Construction or Replace on Burnout: Default values from	See <u>Table 3-22Table</u> 3-22
		Table 3-22 Table 3-22	
$SEER_{ee}$, Seasonal <u>energy</u> efficiency ratioing of the energy efficientey unit. For units > $65,000 \frac{Btu}{hr}$, EER should be used for cooling savings.	$\frac{Btu}{W}$	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
		Early Replacement: Nameplate data	EDC's Data Gathering
COP_{base} , Efficiency ratingCoefficient of performance of the baseline unit. For ASHP units < $65,000\frac{Btu}{hr}$, HSPF should be used for heating savings.	None- Variable	New Construction or Replace on Burnout: Default values from	See Table 3-22Table 3-22_
		Table 3-22Table 3-22	
COP_{ee} , Efficiency ratingCoefficient of performance of the energy efficientey unit. For ASHP units < 65,000 $\frac{Btu}{hr}$ HSPF should be used for heating savings.	None Varia	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
HSPF _{base} , Heating seasonal performance		Early Replacement: Nameplate data	EDC 's-Data Gathering
factor of the baseline unit. For units > $65,000$ $\frac{Btu}{hr}$, COP should be used for heating savings.	Btu/hr W ble	New Construction or Replace on Burnout: Default values from Table 3-22Table	See <u>Table 3-22Table</u> 3-22
		3-22	

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Term Component		<u>Unit</u> Type	Value <u>s</u>	Source
$HSPF_{ee}$, Heating seasonal factor of the energy efficience $> 65,000 \frac{Btu}{hr}$, COP should heating savings.	ency unit. For units	Btu/ _{hr} Varia	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
CF, Demand Coincidence Section 1.5.)	Factor (See	None Fixed	55%See Table 3-24Table 3-24	<u>1</u> 2
EFLH _{cool} , Equivalent Full the cooling season – The entire operating season d	kWh during the	HoursVari	Based on Logging, BMS data or Modeling ³⁴³	EDC's Data Gathering
at design conditions.	,		Default values from Table 3-23	1
EFLH _{heat} , Equivalent Full the heating season – The		Hours	Based on Logging, BMS data or Modeling ³⁴⁴	EDC Data Gathering
entire operating season d at design conditions.	ivided by the kW	Hours	Default values from - Table 3-25Table 3-25	<u>1</u> -
11.3/13, Conversion facto EER, based on average E unit		None	11.3 13	2
1000, conversion from wa	tts to kilowatts	$\frac{W}{kW}$	1000	Conversion Factor

Note: For water-source and evaporatively-cooled air conditioners, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment withcapacity modulation.

Table 3-22: HVAC Baseline Efficiencies 345

Equipment Type and Capacity	Cooling Baseline	Heating Baseline
Air-Source Air Conditioners		
< 65,000 ^{Btu} / _{hr}	13.0 SEER	N/A
\geq 65,000 $\frac{Btu}{hr}$ and <135,000 $\frac{Btu}{hr}$	11.2 EER 11.4 IEER	N/A
\geq 135,000 $\frac{Btu}{hr}$ and < 240,000 $\frac{Btu}{hr}$	11.0 EER 11.2 IEER	N/A

³⁴³ Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

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 ³⁴ Ibid
 345 Baseline values from IECC 2009 (https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf), after Jan 1, 2010 or Jan 23, 2010 as applicable. Integrated Energy Efficiency Ratio (IEER) requirements have been incorporated from ASHRAE 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings", 2008 Supplement (Addendum S: (Tables 6.8.1A and 6.8.1B). IECC 2009 does not present IEER requirements.

Equipment Type and Capacity	Cooling Baseline	Heating Baseline		
\geq 240,000 $\frac{Btu}{hr}$ and < 760,000 $\frac{Btu}{hr}$	10.0 EER- 10.1 IEER	N/A		
≥ 760,000 ^{Btu} / _{hr}	$\frac{9.7 \text{ EER}}{hr}$ 9.7 EER 9.8 IEER			
Air-Source Heat Pumps				
< 65,000 \(\frac{Btu}{hr}\)	13 SEER	7.7 HSPF		
$\geq 65,000 \frac{Btu}{hr}$ and <135,000 $\frac{Btu}{hr}$	11.0 EER 11.2 IEER	3.3 COP		
$\geq 135,000 \frac{Btu}{hr}$ and $< 240,000 \frac{Btu}{hr}$	10.6 EER 10.7 IEER	3.2 COP		
≥ 240,000 ^{Btu} / _{hr}	$\geq 240,000 \frac{Btu}{hr}$ 9.5 EER 9.6 IEER			
Packaged Terminal Systems (Non	standard Size) 346, 347			
PTAC (cooling)	10.9 - (0.213 x Cap / 1000) EER	N/A		
PTHP	10.8 - (0.213 x Cap / 1000) EER	2.9 - (0.026 x Cap / 1000) COP		
Packaged Terminal Systems (Standard Size) 348, 349				
PTAC (cooling)	12.5 - (0.213 x Cap / 1000) EER	N/A		
PTHP	12.3 - (0.213 x Cap / 1000) EER	3.2 - (0.026 x Cap / 1000) COP		
Water-Cooled Air Conditioners				

HVAC

³⁴⁶Nonstandard size packaged terminal air conditioners and heat pumps with existing sleeves having an external wall opening of less than 16 in. high or less than 42 in. wide and having a cross-sectional area less than 670 in. Shall be factory labeled as follows: Manufactured for nonstandard size applications only: not to be installed in new construction

projects.

347 Cap represents the rated cooling capacity of the product in Btu/hr. If the unit's capacity is less than 7,000 Btu/hr, 7,000 Btu/hr is used in the calculation. If the unit's capacity is greater than 15,000 Btu/hr, 15,000 Btu/hr is used in the

Cap represents the rated cooling capacity of the product in Btu/hr. If the unit's capacity is greater than 15,000 Btu/hr, 15,000 Btu/hr, 7,000 Btu/hr, 7,000 Btu/hr is used in the calculation.

Equipment Type and Capacity	Cooling Baseline	Heating Baseline
< 65,000 Btu/hr	12.1 EER 12.3 IEER	N/A
\geq 65,000 $\frac{Btu}{hr}$ and <135,000 $\frac{Btu}{hr}$	12.1 EER 12.3 IEER	N/A
$\geq 135,000 \frac{Btu}{hr}$ and $\leq 240,000 \frac{Btu}{hr}$	12.5 EER 12.7 IEER	N/A
$\geq 240,000 \frac{Btu}{hr}$ and $< 760,000 \frac{Btu}{hr}$	12.4 EER 12.6 IEER	N/A
> 760,000 Btu/hr	11.0 EER 11.1 IEER	N/A
Evaporatively-Cooled Air Conditions	<u>ers</u>	
< 65,000 Btu/hr	12.1 EER 12.3 IEER	N/A
\geq 65,000 $\frac{Btu}{hr}$ and <135,000 $\frac{Btu}{hr}$	12.1 EER 12.3 IEER	N/A
$\geq 135,000 \frac{Btu}{hr}$ and $\leq 240,000 \frac{Btu}{hr}$	12.0 EER 12.2 IEER	N/A
$\geq 240,000 \frac{Btu}{hr}$ and $\leq 760,000 \frac{Btu}{hr}$	11.9 EER 12.1 IEER	N/A
> 760,000 Btu/hr	11.0 EER 11.1 IEER	N/A

Notes: For air-source air conditioners and air-source heat pumps, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment with capacity modulation.

Table 1-3: Cooling EFLH for Pennsylvania Cities 350,351

	d Ollios		Ф	±	# # P	hia	
Space and/or Building Type	Allentown	Erio	Harrisburg	Pittsburgh	Williamsport	Philadolphia	Scranton
Arena/Auditorium/Convention Center	602	332	640	508	454	711	428
College: Classes/Administrative	690	380	733	582	520	815	490
Convenience Stores	1,216	671	1,293	1,026	917	1,436	864
Dining: Bar Lounge/Leisure	912	503	969	769	688	1,077	648
Dining: Cafeteria / Fast Food	1,227	677	1,304	1,035	925	1,449	872
Dining: Restaurants	912	503	969	769	688	1,077	648
Gymnasium/Performing Arts Theatre	690	380	733	582	520	815	490
Hospitals/Health care	1,396	770	1,483	1,177	1,052	1,648	992
Industrial: 1 Shift/Light Manufacturing	727	401	773	613	548	859	517
Industrial: 2 Shift	988	545	1,050	833	745	1,166	702
Industrial: 3 Shift	1,251	690	1,330	1,055	944	1,478	889
Lodging: Hotels/Motels/Dormitories	756	418	805	638	571	894	538
Lodging: Residential	757	418	805	638	571	894	538
Multi-Family (Common Areas)	1,395	769	1,482	1,176	1,052	1,647	991
Museum/Library	851	469	905	718	642	1,005	605
Nursing Homes	1,141	630	1,213	963	861	1,348	811
Office: General/Retail	851	469	905	718	642	1,005	605
Office: Medical/Banks	851	469	905	718	642	1,005	605
Parking Garages & Lots	938	517	997	791	707	1,107	666
Penitentiary	1,091	602	1,160	920	823	1,289	775
Police/Fire Stations (24 Hr)	1,395	769	1,482	1,176	1,052	1,647	991
Post Office/Town Hall/Court House	851	469	905	718	642	1,005	605

HVAC

³⁶⁰US Department of Energy. ENERGY STAR Calculator and Bin Analysis Models
³⁶¹The Equivalent Full Load Hours (EFLH) for Pennsylvania are calculated based on the degree day scalingmethodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using fullload hours (FLH) from the US-DOE-ENERGY STAR Calculator to account for differences in weather conditions.

Space and/or Building Type	Allentown	Еніо	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Religious Buildings/Church	602	332	640	508	454	711	428
Retail	894	493	950	75 4	674	1,055	635
Schools/University	634	350	674	535	478	749	451
Warehouses (Not Refrigerated)	692	382	735	583	522	817	492
Warehouses (Refrigerated)	692	382	735	583	522	817	492
Waste Water Treatment Plant	1,251	690	1,330	1,055	944	1,478	889

Table 3-23: Air Conditioning EFLHs for Pennsylvania Cities

Space and/or Building Type	Allentown	Erie	Harrisburg	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsport
Assembly	<u>753</u>	<u>607</u>	<u>820</u>	<u>1,087</u>	<u>706</u>	<u>629</u>	<u>685</u>
Education - Community College	603	<u>436</u>	<u>620</u>	<u>695</u>	<u>557</u>	<u>515</u>	<u>594</u>
Education - Primary School	<u>250</u>	<u>154</u>	<u>277</u>	<u>302</u>	<u>255</u>	<u>204</u>	<u>208</u>
Education - Relocatable Classroom	<u>301</u>	<u>198</u>	<u>326</u>	<u>359</u>	<u>303</u>	<u>229</u>	<u>246</u>
Education - Secondary School	249	<u>204</u>	<u>327</u>	<u>375</u>	<u>262</u>	<u>219</u>	<u>264</u>
Education - University	<u>677</u>	<u>520</u>	<u>693</u>	<u>773</u>	<u>630</u>	<u>550</u>	<u>595</u>
Grocery	<u>654</u>	<u>636</u>	<u>453</u>	<u>536</u>	<u>638</u>	434	442
Health/Medical - Hospital	1,030	1,038	892	1,059	<u>788</u>	1,022	<u>1,013</u>
Health/Medical - Nursing Home	<u>477</u>	<u>481</u>	<u>540</u>	<u>684</u>	<u>511</u>	<u>467</u>	<u>476</u>
<u>Lodging - Hotel</u>	<u>1,386</u>	<u>1,392</u>	<u>1,523</u>	<u>1,732</u>	<u>1,478</u>	<u>1,348</u>	<u>1,384</u>
Manufacturing - Bio/Tech	<u>785</u>	<u>548</u>	<u>766</u>	<u>858</u>	<u>710</u>	<u>594</u>	<u>627</u>
Manufacturing - Light Industrial	<u>355</u>	<u>274</u>	<u>465</u>	<u>506</u>	<u>349</u>	<u>296</u>	329
Office - Large	480	433	<u>601</u>	<u>754</u>	<u>749</u>	<u>595</u>	<u>490</u>
Office - Small	<u>435</u>	<u>391</u>	<u>529</u>	<u>653</u>	<u>692</u>	<u>404</u>	442
Restaurant - Fast-Food	<u>545</u>	<u>478</u>	<u>574</u>	<u>790</u>	<u>602</u>	<u>524</u>	<u>569</u>

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<u>Table 3-24: Air Conditioning Demand CFs for Pennsylvania Cities</u>

Space and/or Building Type	Allentown	Erie	Harrisburg	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsport
<u>Assembly</u>	0.53	<u>0.45</u>	0.60	0.72	0.56	0.48	0.52
Education - Community College	0.49	0.37	0.49	0.53	0.49	<u>0.48</u>	<u>0.52</u>
Education - Primary School	0.10	0.07	<u>0.16</u>	<u>0.16</u>	<u>0.17</u>	<u>0.11</u>	<u>0.12</u>
Education - Relocatable Classroom	<u>0.15</u>	<u>0.11</u>	0.18	0.19	0.20	0.14	<u>0.15</u>
Education - Secondary School	<u>0.11</u>	0.10	0.20	0.21	0.18	0.13	<u>0.17</u>
Education - University	0.47	0.38	0.47	0.49	0.47	0.42	<u>0.45</u>
Grocery	0.33	0.27	0.24	0.26	0.27	0.21	0.24
Health/Medical - Hospital	0.43	0.37	0.39	0.44	0.39	0.37	0.42
Health/Medical - Nursing Home	0.26	0.27	0.30	0.34	0.32	0.28	0.29
Lodging - Hotel	0.72	0.77	0.78	0.83	0.83	0.73	0.78
Manufacturing - Bio/Tech	0.62	0.47	0.61	0.67	0.64	0.54	0.55
Manufacturing - Light Industrial	0.39	0.31	0.49	0.52	0.42	0.36	0.40
Office - Large	0.33	0.32	0.42	0.27	0.35	0.39	0.37
Office - Small	<u>0.31</u>	0.30	0.39	0.27	0.34	0.33	0.36
Restaurant - Fast-Food	0.36	0.33	0.39	0.47	0.44	0.38	0.42
Restaurant - Sit-Down	0.39	0.41	0.45	0.53	0.54	0.40	0.48
Retail - Multistory Large	0.52	0.42	0.56	0.53	<u>0.51</u>	0.48	<u>0.51</u>
Retail - Single-Story Large	0.50	0.40	0.53	0.63	<u>0.55</u>	0.47	0.47
Retail - Small	0.53	0.56	<u>0.51</u>	<u>0.55</u>	0.63	0.45	<u>0.50</u>
Storage - Conditioned	<u>0.18</u>	0.13	0.24	0.30	0.23	<u>0.15</u>	0.20
Warehouse - Refrigerated	0.50	0.48	0.52	0.53	<u>0.51</u>	0.48	<u>0.51</u>

<u>Table</u> 3-25: <u>Heat Pump EFLHs for Pennsylvania Cities</u>

Space and/or Building Type	Allentown	Erie	Harrisburg	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsport
Assembly	<u>1,178</u>	1,437	1,098	<u>1,121</u>	<u>1,163</u>	<u>1,401</u>	1,066
Education - Community College	<u>816</u>	<u>966</u>	<u>620</u>	<u>521</u>	<u>734</u>	<u>977</u>	<u>783</u>
Education - Primary School	<u>795</u>	<u>830</u>	<u>651</u>	<u>557</u>	<u>819</u>	<u>879</u>	<u>543</u>
Education - Secondary School	<u>752</u>	1,002	<u>710</u>	<u>654</u>	<u>776</u>	<u>893</u>	<u>677</u>
Education - University	<u>621</u>	<u>748</u>	<u>483</u>	<u>407</u>	<u>567</u>	<u>670</u>	<u>527</u>
Grocery	<u>733</u>	<u>534</u>	<u>1,269</u>	<u>1,217</u>	<u>564</u>	<u>1,737</u>	<u>1,419</u>
Health/Medical - Hospital	<u>147</u>	<u>95</u>	<u>361</u>	<u>345</u>	418	<u>106</u>	<u>154</u>
Health/Medical - Nursing Home	944	<u>1,304</u>	<u>854</u>	<u>805</u>	1,023	<u>1,193</u>	<u>958</u>
Lodging - Hotel	2,371	3,077	<u>2,159</u>	2,017	2,411	<u>2,591</u>	2,403
Manufacturing - Bio/Tech	<u>178</u>	<u>193</u>	<u>138</u>	<u>111</u>	<u>172</u>	<u>176</u>	<u>141</u>
Manufacturing - Light Industrial	633	<u>752</u>	<u>609</u>	<u>567</u>	<u>627</u>	<u>705</u>	<u>550</u>
Office - Large	<u>218</u>	<u>292</u>	<u>230</u>	<u>22</u>	<u>30</u>	<u>176</u>	<u>231</u>
Office - Small	<u>423</u>	<u>551</u>	<u>430</u>	<u>38</u>	<u>62</u>	<u>481</u>	<u>448</u>
Restaurant - Fast-Food	<u>1,227</u>	1,627	<u>1,112</u>	<u>1,078</u>	<u>1,363</u>	<u>1,612</u>	<u>1,295</u>
Restaurant - Sit-Down	1,074	1,747	<u>968</u>	908	<u>1,316</u>	<u>1,390</u>	<u>1,187</u>
Retail - Multistory Large	<u>687</u>	<u>828</u>	<u>582</u>	<u>447</u>	<u>620</u>	<u>736</u>	<u>587</u>
Retail - Single-Story Large	<u>791</u>	<u>979</u>	<u>674</u>	<u>735</u>	849	929	<u>654</u>
Retail - Small	949	<u>1,133</u>	<u>689</u>	<u>109</u>	<u>164</u>	900	<u>785</u>
Storage - Conditioned	<u>847</u>	<u>1,114</u>	843	900	978	<u>1,008</u>	800
Warehouse - Refrigerated	<u>363</u>	<u>534</u>	<u>307</u>	222	<u>409</u>	<u>439</u>	<u>328</u>

Table 1-7: Heating EFLH for Pennsylvania Cities 352, 353

Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Arena/Auditorium/Convention Center	1,719	2,002	1,636	1,642	1,726	1,606	1,747
College: Classes/Administrative	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Convenience Stores	603	701	573	576	605	563	612
Dining: Bar Lounge/Leisure	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Dining: Cafeteria / Fast Food	582	678	554	556	585	544	592
Dining: Restaurants	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Gymnasium/Performing Arts Theatre	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Hospitals/Health care	276	321	263	264	277	258	280
Industrial: 1 Shift/Light Manufacturing	1,491	1,737	1,420	1,425	1,498	1,394	1,516
Industrial: 2 Shift	1,017	1,184	968	972	1,022	951	1,034
Industrial: 3 Shift	538	626	512	513	540	502	546
Lodging: Hotels/Motels/Dormitories	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Lodging: Residential	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Multi-Family (Common Areas)	277	322	263	264	278	259	281
Museum/Library	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Nursing Homes	738	859	702	704	740	689	749
Office: General/Retail	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Office: Medical/Banks	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Parking Garages & Lots	1,110	1,292	1,056	1,060	1,114	1,037	1,128
Penitentiary	829	965	789	792	832	774	842
Police/Fire Stations (24 Hr)	277	322	263	264	278	259	281
Post Office/Town Hall/Court House	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Religious Buildings/Church	1,718	2,001	1,635	1,641	1,725	1,605	1,746
Retail	1,188	1,383	1,130	1,135	1,193	1,110	1,207
Schools/University	1,661	1,933	1,580	1,586	1,667	1,551	1,687
Warehouses (Not Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Warehouses (Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Waste Water Treatment Plant	538	626	512	513	540	502	546

HVAC

³⁶² US Department of Energy. ENERGY STAR Calculator and Bin Analysis Models
363 The Equivalent Full Load Hours (EFLH) for Pennsylvania are calculated based on the degree day scalingmethodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using fullload hours (FLH) from the US DOE ENERGY STAR Calculator to account for differences in weather conditions.

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. The Equivalent Full Load Hours (EFLH) for EFLHs and CFs for Pennsylvania are calculated based on Nexant's eQuest modeling analysis 2014. the degree day scaling methodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using full load hours (FLH) from the US DOE ENERGY STAR Calculator to account for differences in weather conditions. Degree day scaling ratios were calculated using heating degree day and cooling degree day values for seven Pennsylvania cities: Allentown, Erie, Harrisburg, Philadelphia, Pittsburgh, Scranton, and Williamsport.
- a. US Department of Energy. ENERGY STAR Calculator and Bin Analysis Models
- 2.1.UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating Company, September 2011, Pages 219-220.
- C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. The study reports CF values in the range of 44% to 63% depending on the size of the unit for the Mid Atlantic PJM hours. The study, therefore, assumes an average CF value of 55% for the PJM peak demand period that is applied to all units.
- 4-2. Average EER for SEER 13 units as calculated by EER = -0.02 × SEER² + 1.12 × SEER based on U.S. DOE Building America House Simulation Protocol, Revised 2010. http://www.nrel.gov/docs/fy11osti/49246.pdf

3.2.2 **ELECTRIC CHILLERS**

Measure Name	Electric Chillers
Target Sector	Commercial and Industrial Establishments
Measure Unit	Electric Chiller
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	20 years ³⁵⁴
Measure Vintage	Replace on Burnout, New Construction, or Early Replacement

ELIGIBILITY

This protocol estimates savings for installing high efficiency electric chillers as compared to chillers that meet the minimum performance allowed by the current PA Energy Code. The measurement of energy and demand savings for chillers is based on algorithms with key variables (i.e., Efficiency, Coincidence Factor, and Equivalent Full Load Hours (EFLHs). These prescriptive algorithms and stipulated values are valid for standard commercial applications, defined as unitary electric chillers serving a single load at the system or sub-system level. The savings calculated using the prescriptive algorithms need to be supported by a certification that the chiller is appropriately sized for site design load condition.

All other chiller applications, including existing multiple chiller configurations (including redundant or 'stand-by' chillers), existing chillers serving multiple load groups, and chillers in industrial applications are defined as non-standard applications and must follow a site_specific custom protocol. Situations with existing non-VFD chillers upgrading to VFD chillers may use the protocol algorithm. The algorithms, assumptions, and default factors in this Section may be applied to nNew cConstruction applications.

ALGORITHMS

Efficiency ratings in EER

Efficiency ratings in kW/ton

$$\Delta kWh = Tons_{ee} \times \left(\frac{IPLV_{base}}{ton_{base}} - \frac{kW}{ton_{base}}\right) \times EFLH$$

$$= Tons_{ee} \times \left(\frac{kW}{ton_{base}} - \frac{kW}{ton_{ee}}\right) \times CF$$

354 DEER Effective Useful Life. October 10, 2008. For the purpose of calculating the total Resource Cost Test for Act 129, measure cannot claim savings for more than fifteen years.

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DEFINITION OF TERMS

Table 3-26: Electric Chiller Variables

Component <u>Term</u> -	Type Unit	Values	Source
Tons_{ee} , The capacity of the chiller at site design conditions accepted by the program	Variable Tons	Nameplate Data	EDC Data Gathering
$\frac{kW}{ton_{base}}$, Design Rated Efficiency of the baseline chiller.	Variable kw ton	Early Replacement: Nameplate Data	EDC Data Gathering
		New Construction or Replace on Burnout: Default value from <u>Table</u> 3-27 Table 3-27	See <u>Table</u> 3-27 Table 3-27
$\frac{kW}{ton_{ee}}$, Design Rated Efficiency of the energy efficient chiller from the manufacturer data and equipment ratings in accordance with ARI Standards.	Variable $\frac{kW}{ton}$	Nameplate Data (ARI Standards 550/590). At minimum, must satisfy standard listed in <u>Table</u> <u>3-27Table 3-27</u>	EDC Data Gathering
EER _{base} , Energy Efficiency Ratio of the baseline unit. See Table 1 1 for values.	Variable Btu/hr	Early Replacement: Nameplate Data	EDC Data Gathering
	W	New Construction or Replace on Burnout: Default value from <u>Table</u> 3-27 Table 3-27	See <u>Table</u> 3-27 Table 3-27
EER _{ee} , Energy Efficiency Ratio of the efficient unit from the manufacturer data and equipment ratings in accordance with ARI Standards.	Variable Btu/hr W	Nameplate Data (ARI Standards 550/590). At minimum, must satisfy standard listed in <u>Table</u> 3-27 Table 3-27	EDC Data Gathering
IPLV _{base} _Integrated Part Load Value of the baseline unit.	None or $\frac{kW}{ton}$	New Construction or Replace on Burnout: See Table 3-27 Table 3-27	See Table 3-27 Table 3-27
IPLV _{ee} , Integrated Part Load Value of the efficient unit.	None or $\frac{kW}{ton}$	Nameplate Data (ARI_Standards 550/590). At minimum, must satisfy standard listed in Table 3-27 Table 3-27	EDC Data Gathering

Component <u>Term</u> -	Type Unit	Values	Source
CF, Demand Coincidence Factor (See Section1.5)	Fixed None	<u>See</u>	1
		<u>Table 3-29</u>	
		Table 3 2980%	
EFLH, Equivalent Full Load Hours – The kWh during the entire operating season divided by the kW at design	<u>Hours</u>	Based on Logging, BMS data or Modeling ³⁵⁵	EDC Data Gathering
conditions. The most appropriate EFLH from shall be utilized in the calculation.		Default values from Table 3-28	1

Table 3-27: Electric Chiller Baseline Efficiencies (IECC 2009)356

Chiller Type	Size	Path A	Path B	Source
Air Cooled Chillers	< 150 tons	Full load: 9.562 EER IPLV: 12.500 EER	N/A	IECC 2009 Table 503.2.3
	>=150 tons	Full load: 9.562 EER IPLV: 12.750 EER	N/A	(7) Post 1/1/20102
Water Cooled Positive	< 75 tons	Full load: 0.780 kW/ton	Full load: 0.800 kW/ton	
Displacement or Reciprocating Chiller		IPLV: 0.630 kW/ton	IPLV: 0.600 kW/ton	
	>=75 tons and < 150 tons	Full load: 0.775 kW/ton	Full load: 0.790 kW/ton	
		IPLV: 0.615 kW/ton	IPLV: 0.586 kW/ton	
	>=150 tons and < 300 tons	Full load: 0.680 kW/ton	Full load: 0.718 kW/ton	
		IPLV: 0.580 kW/ton	IPLV: 0.540 kW/ton	
	>=300 tons	Full load: 0.620 kW/ton	Full load: 0.639 kW/ton	

³⁵⁵ Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

356 IECC 2009 – Table 503.2.3(7). Chillers must satisfy efficiency requirements for both full load and IPLV efficiencies for either Path A or Path B. The table shows the efficiency ratings to be used for the baseline chiller efficiency in the savings estimation algorithm, which must be consistent with the expected operating conditions of the efficient chiller. For example, if the efficient chiller satisfies Path A and generally performs at part load, the appropriate baseline chiller efficiency is the IPLV value under Path A for energy savings. If the efficient chiller satisfies Path B and generally performs at full load, the appropriate baseline chiller efficiency is the full load value under Path B for energy savings. Generally, chillers operating above 70 percent load for a majority (50% or more) of operating hours should use Path A and chillers below 70% load for a majority of operating hours should use Path B. The "full load" efficiency from the appropriate Path A or B should be used to calculate the Peak Demand Savings as it is expected that the chillers would be under full load during the peak demand periods.

SECTION 3: Commercial and Industrial Measures

Chiller Type	Size	Path A	Path B	Source
		IPLV: 0.540 kW/ton	IPLV: 0.490 kW/ton	
Water Cooled Centrifugal	<300 tons	Full load: 0.634 kW/ton	Full load: 0.639 kW/ton	
Chiller		IPLV: 0.596 kW/ton	IPLV: 0.450 kW/ton	
	>=300 tons and < 600 tons	Full load: 0.576 kW/ton	Full load: 0.600 kW/ton	
		IPLV: 0.549 kW/ton	IPLV: 0.400 kW/ton	
	>=600 tons	Full load: 0.570 kW/ton	Full load: 0.590 kW/ton	
		IPLV: 0.539 kW/ton	IPLV: 0.400 kW/ton	

Table 1-3: Chiller Cooling EFLH by Location 357,358

					tt	ď	
	Allentown		Harrisburg	ittsburgh	Williamsport	Philadelphia	Scranton
Space and/or Building Type	Alle	Erio	Hari	Pitts	Will	Phil	Ser
Arena/Auditorium/Convention- Center	602	332	640	508	454	711	428
College: Classes/Administrative	690	380	733	582	520	815	490
Convenience Stores	1,216	671	1,293	1,026	917	1,436	864
Dining: Bar Lounge/Leisure	912	503	969	769	688	1,077	648
Dining: Cafeteria / Fast Food	1,227	677	1,304	1,035	925	1,449	872
Dining: Restaurants	912	503	969	769	688	1,077	648
Gymnasium/Performing Arts Theatre	690	380	733	582	520	815	490
Hospitals/Health care	1,396	770	1,483	1,177	1,052	1,648	992
Lodging: Hotels/Motels/Dormitories	756	418	805	638	571	894	538
Lodging: Residential	757	418	805	638	571	894	538
Multi-Family (Common Areas)	1,395	769	1,482	1,176	1,052	1,647	991
Museum/Library	851	469	905	718	642	1,005	605
Nursing Homes	1,141	630	1,213	963	861	1,348	811
Office: General/Retail	851	469	905	718	642	1,005	605
Office: Medical/Banks	851	469	905	718	642	1,005	605
Parking Garages & Lots	938	517	997	791	707	1,107	666
Penitentiary	1,091	602	1,160	920	823	1,289	775
Police/Fire Stations (24 Hr)	1,395	769	1,482	1,176	1,052	1,647	991
Post Office/Town Hall/Court House	851	469	905	718	642	1,005	605
Religious Buildings/Church	602	332	640	508	454	711	428
Retail	894	493	950	754	674	1,055	635

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³⁵⁷ US Department of Energy. Energy Star Calculator, and Bin Analysis Models
358 The Equivalent Full Load Hours (ELFH) for Pennsylvania are calculated based on the degree day scaling methodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using full load hours (FLH) from the US-DOE ENERGY STAR Calculator to account for differences in weather conditions.

Space and/or Building Type	Allentown	Erio	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Schools/University	634	350	674	535	478	749	451
Warehouses (Not Refrigerated)	692	382	735	583	522	817	492
Warehouses (Refrigerated)	692	382	735	583	522	817	492
Waste Water Treatment Plant	1,251	690	1,330	1,055	944	1,478	889

Table 3-28: Chiller EFLHs for Pennsylvania Cities

Space and/or Building Type	Allentown	Erie	Harrisburg	Philadelphia	Pittsburgh	Scranton	Williamsport
Education - Community College	<u>634</u>	<u>453</u>	<u>661</u>	<u>734</u>	<u>564</u>	<u>502</u>	<u>608</u>
Education - Secondary School	<u>275</u>	<u>214</u>	<u>344</u>	<u>389</u>	<u>282</u>	244	<u>316</u>
Education - University	<u>695</u>	<u>526</u>	<u>730</u>	<u>805</u>	<u>635</u>	<u>545</u>	<u>629</u>
Health/Medical - Hospital	<u>1,240</u>	<u>1,100</u>	1,362	<u>1,556</u>	<u>1,185</u>	<u>1,134</u>	<u>1,208</u>
Health/Medical - Nursing Home	<u>459</u>	<u>408</u>	<u>520</u>	<u>622</u>	<u>472</u>	<u>418</u>	<u>462</u>
<u>Lodging - Hotel</u>	<u>1.397</u>	<u>1.317</u>	<u>1,511</u>	<u>1,654</u>	<u>1,432</u>	<u>1.352</u>	<u>1,415</u>
Manufacturing - Bio/Tech	<u>708</u>	<u>527</u>	<u>700</u>	<u>780</u>	<u>631</u>	<u>574</u>	<u>614</u>
Office - Large	<u>463</u>	<u>411</u>	<u>546</u>	<u>604</u>	<u>451</u>	<u>427</u>	<u>472</u>
Office - Small	<u>429</u>	<u>374</u>	<u>495</u>	<u>567</u>	<u>434</u>	<u>393</u>	<u>433</u>
Retail - Multistory Large	<u>749</u>	<u>609</u>	<u>836</u>	<u>897</u>	<u>699</u>	<u>659</u>	<u>742</u>

Table 3-29: Chiller Demand CFs for Pennsylvania Cities

Space and/or Building Type	Allentown	Erie	Harrisburg	Philadelphia	Pittsburgh	Scranton	Williamsport
Education - Community College	0.43	0.31	0.44	0.47	0.42	0.36	0.43
Education - Secondary School	<u>0.11</u>	0.09	0.18	<u>0.18</u>	<u>0.17</u>	0.12	0.17
Education - University	0.40	0.30	0.41	0.44	0.39	0.32	0.37
Health/Medical - Hospital	0.50	0.48	0.50	0.54	0.48	0.48	0.50
Health/Medical - Nursing Home	0.24	0.22	0.28	0.30	0.28	0.23	0.26
<u>Lodging - Hotel</u>	0.62	0.61	0.68	0.69	0.71	0.60	0.68
Manufacturing - Bio/Tech	0.53	0.43	0.53	0.58	0.54	0.48	0.50
Office - Large	0.30	0.28	0.36	0.25	0.33	0.30	0.33
Office - Small	0.28	0.26	0.33	0.21	0.30	0.28	0.31
Retail - Multistory Large	0.46	0.38	0.54	0.55	0.48	0.43	0.48

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Nexant's eQuest modeling analysis 2014.
- Average based on coincidence factors from Ohio, New Jersey, Mid-Atlantic, Massachusetts, Connecticut, Illinois, New York, CEE and Minnesota. (74%, 67%, 81%, 94%, 82%, 72%, 100%, 70% and 76% respectively)IECC 2009 Table 503.2.3 (7). https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf

SECTION 3: Commercial and Industrial Measures

3.2.3 WATER SOURCE AND GEOTHERMAL HEAT PUMPS

Measure Name	Water Source and Geothermal Heat Pumps
Target Sector	Commercial and Industrial Establishments
Measure Unit	Geothermal Heat Pump
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ³⁵⁹
Measure Vintage	Replace on Burnout, New Construction, or Early Replacement

This protocol shall apply to ground source, groundwater source, water source heat pumps, and water source and evaporatively cooled air conditioners in commercial applications as further described below. This measure may apply to early replacement of an existing system, replacement on burnout, or installation of a new unit in a new or existing non-residential building for HVAC applications. The base case may employ a different system than the retrofit case.

ELIGIBILITY

In order for this characterization to apply, the efficient equipment is a high-efficiency groundwater source, water source, or ground source heat pump system that meets or exceeds the energy efficiency requirements of the International Energy Conservation Code (IECC) 2009, Table 503.2.3(2). The following retrofit scenarios are considered:

- Ground source heat pumps for existing or new non-residential HVAC applications
- Groundwater source heat pumps for existing or new non-residential HVAC applications
- · Water source heat pumps for existing or new non-residential HVAC applications

These retrofits reduce energy consumption by the improved thermodynamic efficiency of the refrigeration cycle of new equipment, by improving the efficiency of the cooling and heating cycle, and by lowering the condensing temperature when the system is in cooling mode and raising the evaporating temperature when the equipment is in heating mode as compared to the base case heating or cooling system. It is expected that the retrofit system will use a similar conditioned-air distribution system as the base case system.

This protocol does not apply to heat pump systems coupled with non-heat pump systems such as chillers, rooftop AC units, boilers, or cooling towers. Projects that use unique, combined systems such as these should use a site-specific M&V plan (SSMVP) to describe the particulars of the project and how savings are calculated. All HVAC applications other than comfort cooling and heating, such as process cooling, are defined as non-standard applications and are ineligible for this measure.

Definition of Baseline Equipment

In order for this protocol to apply, the baseline equipment could be a standard-efficiency air source, water source, groundwater source, or ground source heat pump system, or an electric

³⁵⁹ DEER values, updated October 10, 2008. Various sources range from 12 to 20 years, DEER represented a reasonable midrange.

chiller and boiler system, or other chilled/hot water loop system. To calculate savings, the baseline system type is assumed to be an air source heat pump of similar size except for cases where the project is replacing a ground source, groundwater source, or water source heat pump; in those cases, the baseline system type is assumed to be a similar system at code.

Table 3-30: Water Source or Geothermal Heat Pump Baseline Assumptions

	Baseline Scenario	Baseline Efficiency Assumptions				
New Construction	n	Standard efficiency air source heat pump system				
Retrofit	Replacing any technology besides a ground source, groundwater source, or water source heat pump	Standard efficiency air source heat pump system				
	Replacing a ground source, groundwater source, or water source heat pump	Efficiency of the replaced geothermal system for early replacement only (if known), else code for a similar system				

ALGORITHMS

There are three primary components that must be accounted for in the energy and demand calculations. The first component is the heat pump unit energy and power, the second is the circulating pump in the ground/water loop system energy and power, and the third is the well pump in the ground/water loop system energy and power. For projects where the retrofit system is similar to the baseline system, such as a standard efficiency ground source system replaced with a high efficiency ground source system, the pump energy is expected to be the same for both conditions and does not need to be calculated. The kWh savings should be calculated using the basic equations below. For baseline units rated in both EER and IEER, use IEER in place of EER where listed in energy savings calculations below.

For air-cooled base case units with cooling capacities less than 65 kBtu/h:

$$\begin{split} \Delta kWh_{heat} & = \left\{ \left(\frac{Btu_{heat}}{hr} \times \frac{1 \ kW}{1000 \ W} \right) \times \left(\frac{1}{HSPF_{base}} \right) \times EFLH_{heat} \right\} \\ & - \left\{ \left(\frac{Btu_{heat}}{hr} \times \frac{1 \ kW}{1000 \ W} \right) \times \left(\frac{1}{COP_{ee}} \right) \times \left(\frac{1}{3.412} \right) \right. \end{split}$$

$$& \times EFLH_{heat} \bigg\}$$

HVAC

For air-cooled base case units with cooling capacities equal to or greater than 65 kBtu/h, and all other units:

$$\Delta kWh_{cool} \\ \Delta kWh_{cool} \\ = \left\{ \left(\frac{Btu_{cool}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{EER_{base}} \right) \times EFLH_{cool} \right\} \\ - \left\{ \left(\frac{Btu_{cool}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{EER_{base}} \right) \times EFLH_{cool} \right\} \\ - \left\{ \left(\frac{Btu_{heat}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{3.412} \right) \times EFLH_{heat} \right\} \\ - \left\{ \left(\frac{Btu_{heat}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{3.412} \right) \times EFLH_{heat} \right\} \\ - \left\{ \left(\frac{Btu_{heat}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{COP_{base}} \right) \times \left(\frac{1}{3.412} \right) \times EFLH_{heat} \right\} \\ - \left\{ HP_{basemotor} \times LF_{base} \times 0.746 \times \left(\frac{1}{\eta_{basemotor}} \right) \times \left(\frac{1}{\eta_{basepump}} \right) \times HOURS_{basepump} \right\} \\ - \left\{ HP_{eemotor} \times LF_{ee} \times 0.746 \times \left(\frac{1}{\eta_{eemotor}} \right) \times \left(\frac{1}{\eta_{eepump}} \right) \times HOURS_{eepump} \right\} \\ - \left\{ \left(\frac{Btu_{cool}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{EER_{base}} \right) \times CF_{cool} \right\} \\ - \left\{ \left(\frac{Btu_{cool}}{hr} \times \frac{1 \, kW}{1000 \, W} \right) \times \left(\frac{1}{EER_{base}} \right) \times CF_{cool} \right\}$$

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 $\Delta kW_{peak\;pump}$

$$= \left\{ HP_{basemotor} \times LF_{base} \times 0.746 \times \left(\frac{1}{\eta_{basemotor}}\right) \times \left(\frac{1}{\eta_{basepump}}\right) \times CF_{pump} \right\} \\ - \left\{ HP_{eemotor} \times LF_{ee} \times 0.746 \times \left(\frac{1}{\eta_{eemotor}}\right) \times \left(\frac{1}{\eta_{eepump}}\right) \times CF_{pump} \right\} \\ \times CF_{pump} \right\}$$

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³⁶⁰ The cooling efficiency ratings of the baseline and efficient units should be used not including pumps where appropriate.

Table 3-31: Geothermal Heat Pump- Values and Assumptions

TermComponent	<u>Unit</u> Type	Values	Source
$\frac{Btu_{cool}}{hr}$, Rated cooling capacity of the energy efficient unit	Btu _{cool} Varia	Nameplate data (ARI or AHAM)	EDC Data Gathering
$\frac{Btu_{heat}}{hr}$, Rated heating capacity of the energy efficient unit	Btuheat hr ble	Nameplate data (ARI or AHAM) Use $\frac{Btu_{cool}}{hr}$ if the heating capacity is not known	EDC Data Gathering
$\it SEER_{base}$, the cooling SEER of the baseline unit	$\frac{Btu/hr}{W}$ _Fixed	Early Replacement: Nameplate data	EDC Data Gathering
		New Construction or Replace on Burnout: Default values from Table 3-34	See Table 3-34
IEER _{base} , Integrated energy efficiency ratio of the baseline unit.	$\frac{Btu}{W}$	Early Replacement: Nameplate data	EDC Data Gathering
		Default: Table 3-34	See Table 3-34
EER _{base} , the cooling EER of the baseline unit	$\frac{Btu/hr}{W}$ _Fixed	Early Replacement: Nameplate data	EDC Data Gathering
		= SEER _{base} X (11.3/13) if EER not available	
		New Construction or Replace on Burnout: Default values from Table 3-34	See Table 3-34
HSPF _{base} . Heating Season Performance Factor of the baseline unit	$\frac{Btu/hr}{W}$ Fixed	Early Replacement: Nameplate data	EDC Data Gathering
Unit		New Construction or Replace on Burnout: Default values from Table 3-34	See Table 3-34
${\it COP_{base}}$, Coefficient of Performance of the baseline unit	None Fixed	Early Replacement: Nameplate data	EDC Data Gathering
		New Construction or Replace on Burnout: Default values from Table 3-34	See Table 3-34
EER _{ee} , the cooling EER of the new ground source, groundwater source, or water source heat pumpground being installed	Htu/ _{hr} W- Variable	Nameplate data (ARI or AHAM) = SEER _{ee} X (11.3/13) if EER not available 362	EDC Data Gathering

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 $^{^{361}}$ 11.3/13 = Conversion factor from SEER to EER, based on average EER of a SEER 13 unit. 362 11.3/13 = Conversion factor from SEER to EER, based on average EER of a SEER 13 unit.

TermComponent	<u>Unit</u> Type	Values	Source
COP _{ee} , Coefficient of Performance of the new ground source, groundwater source, or water source heat pump being installed	NoneVariable	Nameplate data (ARI or AHAM)	EDC Data Gathering
EFLH _{cool} , Cooling annual Equivalent Full Load Hours EFLH for Commercial	Hours Variable	Based on Logging, BMS data or Modeling ³⁶³	EDC Data Gathering
HVAC for different occupancies		Default values from Table 3-23 Table 3-23	2
EFLH _{heat} , Heating annual Equivalent Full Load Hours EFLH for Commercial	Hours Variable	Based on Logging, BMS data or Modeling ³⁶⁴	EDC Data Gathering
HVAC for different occupancies		Default values from Table 3-25 Table (Heat Pump EFLHs- for Pennsylvania Cities)	2_
CF _{cool} , Demand Coincidence Factor (See Section 1.5) for Commercial HVAC	<u>None</u> Fixed	Default= 55% See Table 3-24Table 3-24	<u>2</u> 3
CF _{pump} , Demand Coincidence Factor (See Section 1.5) for ground source	<u>None</u> Fixed	If unit runs 24/7/365, defaultCF- =1.0;	<u>2</u> 4
loop pump		If unit runs only with heat pump unit compressor, default = 0.67_ See Table 3-24Table 3-24	
$\mathit{HP}_{basemotor},$ Horsepower of base case ground loop pump motor	<u>HP</u> Variable	Nameplate	EDC Data Gathering
LF_{base} , Load factor of the base case ground loop pump motor; ratio of the	VariableNon e	Based on spot metering and nameplate	EDC Data Gathering
peak running load to the nameplate rating of the pump motor.		Default <u>:</u> 75%	1
$\eta_{basemotor}$, efficiency of base case ground loop pump motor	None Variable	Nameplate	EDC's Data Gathering
		If unknown, assume the federal minimum efficiency requirements in Table 3-32	See Table 3-32
$\eta_{basepump}$, efficiency of base case ground loop pump at design point	NoneVariable	Nameplate	EDC's Data Gathering
		If unknown, assume program compliance efficiency in Table 3-33	See Table 3-33

³⁶³ Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

364 Ibid

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<u>Term</u> Component	<u>Unit</u> Type	Values	Source
HOURS _{basepump} , Run hours of base case ground loop pump motor	<u>Hours</u> Fixed	Based on Logging, BMS data or Modeling ³⁶⁵	EDC's Data Gathering
		EFLH _{cool} + EFLH _{heat} ³⁶⁶ Default values from Table 3-23 Table 3-23 and Table 3-25Table 3-25	2
HP _{eemotor} , Horsepower of retrofit case ground loop pump motor	<u>HP</u> Variable	Nameplate	EDC's Data Gathering
LF _{ee} , Load factor of the retrofit case ground loop pump motor; Ratio of the peak running load to the nameplate	None Variable	Based on spot metering and nameplate	EDC Data Gathering
rating of the pump motor.		Default <u>.</u> 75%	1
$\eta_{eemotor}$, efficiency of retrofit case ground loop pump motor	None Variable	Nameplate	EDC's Data Gathering
		If unknown, assume the federal minimum efficiency requirements in Table 3-32	Table 3-32
η_{eepump} , efficiency of retrofit case ground loop pump at design point	None Variable	Nameplate	EDC's Data Gathering
		If unknown, assume program compliance efficiency in Table 3-33	See Table 3-33
HOURS _{eepump} , Run hours of retrofit case ground loop pump motor	Hours Variab le	Based on Logging, BMS data or Modeling ³⁶⁷	EDC Data Gathering
		EFLH _{cool} + EFLH _{heat} ³⁶⁸ Default values from Table 3-23 Table 3-23 and Table 3-25Table 3-25	2
3.412, conversion factor from kWh to kBtu	kBtu kWh	3.412	Conversion Factor
0.746, conversion factor from horsepower to kW	$\frac{kW}{hp}$	0.746	Conversion Factor

Notes: For water-source and evaporatively-cooled air conditioners, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

³⁶⁵ Ibid

³⁶⁶ EFLH_{cool} + EFLH_{heat} represent the addition of cooling and heating annual equivalent full load hours for commercial HVAC for different occupancies, respectively.

³⁶⁷ Ibid

³⁶⁸ EFLH_{cool} + EFLH_{heat} represent the addition of cooling and heating annual equivalent full load hours for commercial HVAC for different occupancies, respectively.

Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment with capacity modulation.

Table 3-32: Federal Minimum Efficiency Requirements for Motors³⁶⁹

	Open	Drip Proof # of Poles	(ODP)	Totally	Enclosed Far (TEFC)	n-Cooled	
Size HP	6	4	2	6	4	2	
	;	Speed (RPM)	Speed (RPM)			
	1200	1800	3600	1200	1800	3600	
1	82.50%	85.50%	77.00%	82.50%	85.50%	77.00%	
1.5	86.50%	86.50%	84.00%	87.50%	86.50%	84.00%	
2	87.50%	86.50%	85.50%	88.50%	86.50%	85.50%	
3	88.50%	89.50%	85.50%	89.50%	89.50%	86.50%	
5	89.50%	89.50%	86.50%	89.50%	89.50%	88.50%	
7.5	90.20%	91.00%	88.50%	91.00%	91.70%	89.50%	
10	91.70%	91.70%	89.50%	91.00%	91.70%	90.20%	
15	91.70%	93.00%	90.20%	91.70%	92.40%	91.00%	
20	92.40%	93.00%	91.00%	91.70%	93.00%	91.00%	

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³⁶⁹ Table is based on NEMA premium efficiency motor standards. The Department of Energy published a final rule on May 29, 2014 that applies to electric motors manufactured on or after June 1, 2016. Therefore, baseline efficiencies for electric motors will be updated in the 2016 TRM to comply with federal energy conservation standards. http://www.ecfr.gov/cgi-bin/text-idx?SID=ba6d4f97451f89bcaa13b3f5a91c54c1&node=10:3.0.1.4.19.2.47.11&rgn=div8

Technical Reference Manual

НР	Minimum Pump Efficiency at Design Point (η _{pump})
1.5	65%
2	65%
3	67%
5	70%
7.5	73%
10	75%
15	77%
20	77%

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 $^{^{\}rm 370}$ Based on program requirements submitted during protocol review.

Table 3-34: Default Baseline Equipment Efficiencies

Equipment Type and Capacity	Cooling Baseline	Heating Baseline						
Water-Source and Evaporatively-Cooled Air Condition	Water-Source and Evaporatively-Cooled Air Conditioners							
< 65,000 Print Apr	12.1 EER / 12.3 IEER	N/A						
> 65,000 Arr and <135,000 Arr	11.5 EER / 11.7 IEER	N/A						
$> 135,000 \frac{Btu}{hx}$ and $< 240,000 \frac{Btu}{hx}$	11.0 EER / 11.2 IEER	N/A						
> 240,000 Btu hr	11.0 EER / 11.1 IEER	N/A						
Water-Source Heat Pumps								
< 17,000 \(\frac{Btu}{hr}\)	11.2 EER	4.2 COP						
$> 17,000 \frac{Btu}{hr}$ and $< 65,000 \frac{Btu}{hr}$	12.0 EER	4.2 COP						
Ground Water Source Heat Pumps								
< 135,000 ^{Btu} / _{hr}	16.2 EER	3.6 COP						
Ground Source Heat Pumps								
< 135,000 ^{Btu} / _{hr}	13.4 EER	3.1 COP						

Notes: For water-source and evaporatively-cooled air conditioners, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment with capacity modulation.

MEASURE LIFE

The expected measure life is assumed to be 15 years. 374

DEFAULT SAVINGS

3.2.3.1

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with

SECTION 3: Commercial and Industrial Measures

³⁷¹ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.

verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

SOURCES

- 1. California Public Utility Commission. Database for Energy Efficiency Resources 2005.
- Based on Nexant's eQuest modeling analysis 2014. Provides a conservative estimate in the absence of logging or modeling data.
- C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. The study reports CF values in the range of 44% to 63% depending on the size of the unit for the Mid Atlantic PJM hours. The study, therefore, assumes an average CF value of 55% for the PJM peak demand period that is applied to all units.
- 4. Engineering Estimate See definition in Section 3.3.2 for specific algorithm to be used when performing spot metering analysis to determine alternate load factor.

3.2.4 DUCTLESS MINI-SPLIT HEAT PUMPS - COMMERCIAL < 5.4 TONS

Measure Name	Ductless Mini-Split Heat Pumps – Commercial < 5.4 Ttons Heat-Pumps
Target Sector	Commercial and Industrial Establishments(non-residential)
Measure Unit	Ductless Heat Pump
Unit Energy Savings	Variable based on efficiency of systems
Unit Peak Demand Reduction	Variable based on efficiency of systems
Measure Life	15 years ³⁷²
Measure Vintage	Replace on Burnout

ENERGY STAR ductless "mini-split" heat pumps (DHP) utilize high efficiency SEER/EER and HSPF energy performance factors of 14.5/12 and 8.2, respectively, or greater. This technology typically converts an electric resistance heated space into a space heated/cooled with a single or multi-zonal ductless heat pump system.

ELIGIBILITY

This protocol documents the energy savings attributed to ENERGY STAR ductless mini-split heat pumps with energy-efficiency performance of 14.5/12 SEER/EER and 8.2 HSPF or greater with inverter technology. The baseline heating system could be an existing electric resistance, a lower-efficiency ductless heat pump system, a ducted heat pump, packaged terminal heat pump (PTHP), electric furnace, or a non-electric fuel-based system. The baseline cooling system could be a standard efficiency heat pump system, central air conditioning system, packaged terminal air conditioner (PTAC), or room air conditioner. The DHP could be a new device in an existing space, a new device in a new space, or could replace an existing heating/cooling device. The DHP systems could be installed as a single-zone system (one indoor unit, one outdoor unit) or a multi-zone system (multiple indoor units, one outdoor unit). In addition, the old systems should be de—energized, completely uninstalled and removed in order to ensure that the full savings is realized. All HVAC applications other than comfort cooling and heating, such as process cooling, are defined as non-standard applications and are ineligible for this measure.

ALGORITHMS

The savings depend on three main factors: baseline condition, usage, and the capacity of the indoor unit.

The algorithm is separated into two calculations: single zone and multi-zone ductless heat pumps. The savings algorithm is as follows:

³⁷² DEER values, updated October 10, 2008. Various sources range from 12 to 20 years, DEER represented a reasonable midrange.

range.

373 The measure energy efficiency performance is based on ENERGY STAR minimum specification requirements as specified in ARHI and CEE directory for ductless mini-split heat pumps. Ductless heat pumps fit these criteria and can easily exceed SEER levels of 16 or greater. ENERGY STAR Air Source Heat Pumps and Central Air Conditioners Key Product Criteria. http://www.energystar.gov/index.cfm?c=airsrc heat.pr crit as heat pumps

For heat prump units < 65,000 $\frac{Btu}{hr}$, use SEER to calculate ΔkWh_{cool} and HSPF to calculate ΔkWh_{heat} . Convert SEER to EER to calculate ΔkW_{peak} using 11.3/13 as the conversion factor.

Single Zone:

$$\begin{array}{ll} \Delta kWh & = \Delta kWh_{cool} + \Delta kWh_{heat} \\ \Delta kWh_{heat} & = \frac{CAPY_{heat}}{1000 \frac{W}{kW}} \times \left(\frac{1}{HSPF_b} - \frac{1}{HSPF_e}\right) \times EFLH_{heat} \times LF \\ \Delta kWh_{cool} & = \frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e}\right) \times EFLH_{cool} \times LF \\ \Delta kW_{peak} & = \frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{EER_b} - \frac{1}{EER_e}\right) \times CF \end{array}$$

Multi-Zone:

$$\Delta kWh \\ \Delta kWh_{heat} \\ = \begin{bmatrix} CAPY_{heat} \\ 1000 \frac{W}{kW} \\ \times \\ \end{bmatrix} \times \left(\frac{1}{HSPF_b} - \frac{1}{HSPF_e} \right) \times EFLH_{heat} \times LF \\ + \begin{bmatrix} CAPY_{heat} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{HSPF_b} - \frac{1}{HSPF_e} \right) \times EFLH_{heat} \\ \times LF \\ \end{bmatrix}_{ZONE2} \\ + \begin{bmatrix} CAPY_{heat} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{HSPF_b} - \frac{1}{HSPF_e} \right) \times EFLH_{heat} \\ \times LF \\ \end{bmatrix}_{ZONEn} \\ = \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE1} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE2} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE2} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE2} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3} \\ + \begin{bmatrix} CAPY_{cool} \\ 1000 \frac{W}{kW} \\ \end{bmatrix} \times \left(\frac{1}{SEER_b} - \frac{1}{SEER_e} \right) \times EFLH_{cool} \times LF \\ \end{bmatrix}_{ZONE3}$$

 ΔkW_{peak}

$$\begin{split} = & \left[\frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{EER_b} - \frac{1}{EER_e} \right) \times CF \right]_{ZONE1} \\ & + \left[\frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{EER_b} - \frac{1}{EER_e} \right) \times CF \right]_{ZONE2} \\ & + \left[\frac{CAPY_{cool}}{1000 \frac{W}{kW}} \times \left(\frac{1}{EER_b} - \frac{1}{EER_e} \right) \times CF \right]_{ZONE2} \end{split}$$

DEFINITION OF TERMS

Table 3-35: DHP - Values and References

Component <u>Term</u>	Typ e <u>Unit</u>	Values	Source
$CAPY_{cool.}$ The cooling capacity of the indoor unit, given in $\frac{Btu}{hr}$ as appropriate for the calculation. This protocol is limited to units < 65,000 $\frac{Btu}{hr}$ (5.4 tons) $CAPY_{heat.}$ The heating capacity of the indoor unit, given in $\frac{Btu}{hr}$ as appropriate for the calculation.	Btu hr Variab le	Nameplate	AEPS- Application; EDC Data Gathering
EFLH _{cool} , Equivalent Full Load Hours		Based on Logging, BMS data or Modeling ³⁷⁴	EDC Data Gathering
for cooling $EFLH_{cool}$, Equivalent Full Load Hours for heating	HoursVar iable	Default: See Table 3-23 Table 3-23and Table 3-25Table 3-25	1
HSPF_b , Heating Seasonal Performance Factor, heating efficiency of the installed DHP	Btu/hr W ed	Standard DHP: 7.7 Electric resistance: 3.413 ASHP: 7.7 PTHP ³⁷⁵ (Replacements): 2.9 - (0.026 x Cap / 1000) COP PTHP (New Construction): 3.2 - (0.026 x Cap / 1000) COP Electric furnace: 3.242 For new space, no heat in an existing space, or non-electric heating in an existing space: use standard DHP: 7.7	2, 4, <u>7</u> 9

HVAC

³⁷⁴ Ibid
375 Cap represents the rated cooling capacity of the product in Btu/h. If the unit's capacity is less than 7,000 Btu/h, 7,000 Btu/h is used in the calculation. If the unit's capacity is greater than 15,000 Btu/h, 15,000 Btu/h is used in the calculation. Use HSPF = COP X 3.413.

Component <u>Term</u>	Type Unit	Values	Source
SEER _b , Seasonal Energy Efficiency Ratio cooling efficiency of baseline unit	Btu/hr W ed	DHP, ASHP, or central AC: 13 Room AC: 11.3 PTAC ³⁷⁶ (Replacements): 10.9 - (0.213 x Cap / 1000) EER PTAC (New Construction): 12.5 - (0.213 x Cap / 1000) EER PTHP (Replacements): 10.8 - (0.213 x Cap / 1000) EER PTHP (New Construction): 12.3 - (0.213 x Cap / 1000) EER PTHP (New Construction): 12.3 - (0.213 x Cap / 1000) EER For new space or no cooling in an existing space: use Central AC: 13	3,4,5, <u>6</u> 7, <u>7</u> 9
$\mathit{HSPF_e}$, Heating Seasonal Performance Factor, heating efficiency of the installed DHP	^{Btu/hr} √ari able	Based on nameplate information. Should be at least ENERGY STAR.	AEPS Application; EDC Data Gathering
SEER _e , Seasonal Energy Efficiency Ratio cooling efficiency of the installed DHP	Wari able	Based on nameplate information. Should be at least ENERGY STAR.	AEPS Application; EDC Data Gathering
CF, Demand Coincidence Factor (See Section 1.5)	NoneFixe d	70%See Table <u>Table</u> 3-24Table 3-24	<u>1</u> 6

Table 1-2: Cooling EFLH for Pennsylvania Cities 377, 378

Space and/or Building Type	Allentown	Егіө	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Arena/Auditorium/Convention Center	602	332	640	508	454	711	428
College: Classes/Administrative	690	380	733	582	520	815	490
Convenience Stores	1,216	671	1,293	1,026	917	1,436	864
Dining: Bar Lounge/Leisure	912	503	969	769	688	1,077	648
Dining: Cafeteria / Fast Food	1,227	677	1,304	1,035	925	1,449	872
Dining: Restaurants	912	503	969	769	688	1,077	648
Gymnasium/Performing Arts Theatre	690	380	733	582	520	815	490
Hospitals/Health care	1,396	770	1,483	1,177	1,052	1,648	992
Industrial: 1 Shift/Light Manufacturing	727	401	773	613	548	859	517

 $^{^{376}}$ Cap represents the rated cooling capacity of the product in Btu/h. If the unit's capacity is less than 7,000 Btu/h, 7,000 Btu/h is used in the calculation. If the unit's capacity is greater than 15,000 Btu/h, 15,000 Btu/h is used in the calculation. Use SEER = EER X (13/11.3).

HVAC

³⁷⁷ US Department of Energy. ENERGY STAR Calculator
378 A zip code mapping table is located in Appendix F. This table should be used to identify the reference Pennsylvania city for all zip codes in Pennsylvania

Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Industrial: 2 Shift	988	545	1,050	833	745	1,166	702
Industrial: 3 Shift	1,251	690	1,330	1,055	944	1,478	889
Lodging: Hotels/Motels/Dormitories	756	418	805	638	571	894	538
Lodging: Residential	757	418	805	638	571	894	538
Multi-Family (Common Areas)	1,395	769	1,482	1,176	1,052	1,647	991
Museum/Library	851	469	905	718	642	1,005	605
Nursing Homes	1,141	630	1,213	963	861	1,348	811
Office: General/Retail	851	469	905	718	642	1,005	605
Office: Medical/Banks	851	469	905	718	642	1,005	605
Parking Garages & Lots	938	517	997	791	707	1,107	666
Penitentiary	1,091	602	1,160	920	823	1,289	775
Police/Fire Stations (24 Hr)	1,395	769	1,482	1,176	1,052	1,647	991
Post Office/Town Hall/Court House	851	469	905	718	642	1,005	605
Religious Buildings/Church	602	332	640	508	454	711	428
Retail	894	493	950	754	674	1,055	635
Schools/University	634	350	674	535	478	749	451
Warehouses (Not Refrigerated)	692	382	735	583	522	817	492
Warehouses (Refrigerated)	692	382	735	583	522	817	4 92
Waste Water Treatment Plant	1,251	690	1,330	1,055	944	1,478	889

Rev Date: June 2015

Table 1-3: Heating EFLH for Pennsylvania Cities 379, 380

Tuble 1 3. Heating Et Et 1011 et insylvani							
Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Arena/Auditorium/Convention Center	1,719	2,002	1,636	1,642	1,726	1,606	1,747
College: Classes/Administrative	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Convenience Stores	603	701	573	576	605	563	612
Dining: Bar Lounge/Leisure	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Dining: Cafeteria / Fast Food	582	678	554	556	585	544	592
Dining: Restaurants	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Gymnasium/Performing Arts Theatre	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Hospitals/Health care	276	321	263	264	277	258	280
Industrial: 1-Shift/Light- Manufacturing	1,491	1,737	1,420	1,425	1,498	1,394	1,516
Industrial: 2 Shift	1,017	1,184	968	972	1,022	951	1,034
Industrial: 3 Shift	538	626	512	513	540	502	546
Lodging: Hotels/Motels/Dormitories	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Lodging: Residential	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Multi-Family (Common Areas)	277	322	263	264	278	259	281
Museum/Library	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Nursing Homes	738	859	702	704	740	689	749
Office: General/Retail	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Office: Medical/Banks	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Parking Garages & Lots	1,110	1,292	1,056	1,060	1,114	1,037	1,128
Penitentiary	829	965	789	792	832	774	842
Police/Fire Stations (24 Hr)	277	322	263	26 4	278	259	281

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³⁷⁹ US-Department of Energy. ENERGY STAR Calculator³⁸⁰ A zip code mapping table is located in Appendix F. This table should be used to identify the reference Pennsylvania city for all zip codes in Pennsylvania

Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Post Office/Town Hall/Court House	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Religious Buildings/Church	1,718	2,001	1,635	1,641	1,725	1,605	1,746
Retail	1,188	1,383	1,130	1,135	1,193	1,110	1,207
Schools/University	1,661	1,933	1,580	1,586	1,667	1,551	1,687
Warehouses (Not Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Warehouses (Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Waste Water Treatment Plant	538	626	512	513	540	502	546

DEFAULT SAVINGS

State of Pennsylvania

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering.

SOURCES

- Based on Nexant's eQuest modeling analysis 2014. US Department of Energy. ENERGY STAR Calculator and Bin Analysis Models.
- COP = HSPF/3.413. HSPF = 3.413 for electric resistance heating, HSPF = 7.7 for standard DHP. Electric furnace COP typically varies from 0.95 to 1.00 and thereby assumed a COP 0.95 (HSPF = 3.242).
- Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.
- Air-Conditioning, Heating, and Refrigeration Institute (AHRI); the directory of the available ductless mini-split heat pumps and corresponding efficiencies (lowest efficiency currently available). Accessed 8/16/2010. https://www.ahridirectory.org/ahridirectory/pages/home.aspx
- ENERGY STAR and Federal Appliance Standard minimum EERs for a 10,000 Btuh unit with louvered sides. http://www.energystar.gov/index.cfm?c=roomac.pr_crit_room_ac SEER based on average EER of 9.8 for room AC unit. From Pennsylvania's Technical Reference Manual.

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- 6. Based on an analysis of six different utilities by Proctor Engineering. From Pennsylvania's Technical Reference Manual.
- 7.6. Average EER for SEER 13 units as calculated by EER = -0.02 × SEER² + 1.12 × SEER based on U.S. DOE Building America House Simulation Protocol, Revised 2010. http://www.nrel.gov/docs/fy11osti/49246.pdf _ Average EER for SEER 13 unit. From Pennsylvania's Technical Reference Manual.
- 8. The load factor is used to account for inverter-based DHP units operating at partial loads. The value was chosen to align savings with what is seen in other jurisdictions: based on personal communication with Bruce Manclark, Delta-T, Inc. who is working with Northwest Energy Efficiency Alliance (NEEA) on the Northwest DHP Project http://www.nwductless.com/, and the results found in the "Ductless Mini Pilot Study" by KEMA, Inc., June 2009. The adjustment is required to account for partial load conditions and because the EFLH used are based on central ducted systems which may overestimate actual usage for baseboard systems.
- 9-7. Package terminal air conditioners (PTAC) and package terminal heat pumps (PTHP) COP and EER minimum efficiency requirements is based on CAPY value. If the unit's capacity is less than $7,000 \, \frac{Btu}{hr}$, use $7,000 \, \frac{Btu}{hr}$ in the calculation. If the unit's capacity is greater than $15,000 \, \frac{Btu}{hr}$, use $15,000 \, \frac{Btu}{hr}$ in the calculation.

3.2.5 FUEL SWITCHING: SMALL COMMERCIAL ELECTRIC HEAT TO NATURAL GAS / PROPANE / OIL HEAT

Measure Name	Fuel Switching: Small Commercial Electric Heat to Natural Gas / Propane / Oil Heat
Target Sector	Commercial and Industrial Establishments
Measure Unit	Water Heater
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	<u>Variable</u>
Measure Life	20 years ³⁸¹
Measure Vintage	Replace on Burnout or Early Retirement or New Construction

ELIGIBILITY

The energy and demand savings for small commercial fuel switching for heating systems is determined from the algorithms listed below. This protocol excludes water source, ground source, and groundwater source heat pumps.

The baseline for this measure is an existing commercial facility with an electric primary heating source. The heating source can be electric baseboards, packaged terminal heat pump (PTHP) units, electric furnace, or electric air source heat pump. The retrofit condition for this measure is the installation of a new standard efficiency natural gas, propane, or oil furnace or boiler. This algorithm does not apply to combination space and water heating units. This protocol applies to measures with input rating of less than $225,000 \, \frac{Btu}{hr}$.

To encourage adoption of the highest efficiency units, older units which meet outdated ENERGY STAR standards may be incented up through the given sunset dates (see table below).

Table 3-36: Act 129 Sunset Dates for ENERGY STAR Furnaces

ENERGY STAR Product Criteria Version	ENERGY STAR Effective Manufacture Date	Act 129 Sunset Date ^a
ENERGY STAR Furnaces Version 4.0	February 1, 2013	N/A
ENERGY STAR Furnaces Version 3.0	February 1, 2012	May 31, 2014
ENERGY STAR Furnaces Version 2.0, Tier II units	October 1, 2008	May 31, 2013

^a Date after which Act 129 programs may no longer offer incentives for products meeting the criteria for the listed ENERGY STAR version."

³⁸¹DEER Effective Useful Life. October 10, 2008. For the purpose of calculating the total Resource Cost Test for Act 129, measure cannot claim savings for more than fifteen years.

EDCs may provide incentives for equipment with efficiencies greater than or equal to the applicable ENERGY STAR requirement per the following table.

Table 3-37: ENERGY STAR Requirements for Furnaces and Boilers

Equipment	ENERGY STAR Requirements ³⁸²
Gas Furnace	AFUE rating of 95% or greater Less than or equal to 2.0% furnace fan efficiency Less than or equal to 2.0% air leakage
Oil Furnace	AFUE rating of 85% or greater Less than or equal to 2.0% furnace fan efficiency Less than or equal to 2.0% air leakage
Boiler	AFUE rating of 85% or greater

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The energy savings are the full energy consumption of the electric heating source minus the energy consumption of the fossil fuel furnace blower motor. The energy savings are obtained through the following formulas:

Electric furnace or air source heat pump

For ASHP units < 65,000
$$\frac{Btu}{hr}$$
, use HSPF instead of COP to calculate ΔkWh_{heat} .
$$\Delta kWh_{heat} = \left(\frac{Btu_{heat}}{hr} \times \frac{1}{1000 \, W}\right) \times \frac{1}{3.412} \times \frac{1}{COP_{base}} \times EFLH_{heat}$$
$$= \left(\frac{Btu_{heat}}{hr} \times \frac{1}{1000 \, W}\right) \times \frac{1}{HSPF_{base}} \times EFLH_{heat}$$

Baseboard heating, packaged terminal heat pump

$$\Delta kWh_{heat} = \frac{\frac{Btu_{heat}}{hr} \times EFLH_{heat}}{3413 \frac{Btu}{kWh} \times COP_{base}} - \frac{HP_{motor} \times 746 \frac{W}{HP} \times EFLH_{heat}}{\eta_{motor} \times 1000 \frac{W}{kW}}$$

The motor consumption of a gas furnace is subtracted from the savings for a baseboard or PTHP heating system, as these existing systems do not require a fan motor while the replacement furnace does (the electric furnace and air source heat pumps require fan motors with similar consumption as a gas furnace and thus there is no significant change in motor load). For boilers, the annual pump energy consumption is negligible (<50 kWh per year) and not included in this calculation. 384

³⁸² Residential Furnace and Boiler Energy Star product criteria. http://www.energystar.gov/index.cfm?c=furnaces.pr_crit_furnaces and http://www.energystar.gov/index.cfm?c=furnaces.pr_crit_furnaces and http://www.energystar.gov/index.cfm?c=furnaces.pr_crit_furnaces and http://www.energystar.gov/index.cfm?fuseaction=find a product.showProductGroup&pgw_code=BO

³⁸³ EDC's may use billing analysis using program participant data to claim measure savings, in lieu of using the defaults provided in this measure protocol. Billing analysis should be conducted using at least 12 months of billing data (pre- and post-retrofit).

³⁸⁴ Pump motors are typically 1/25 HP. With 1,000 hour runtime and 80% assumed efficiency, this translates to 37 kWh.

There are no peak demand savings as it is a heating only measure.

Although there isare a significant electric savings, there is also an associated increase in fossil fuel energy consumption. While this fuel consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel energy is obtained through the following formula:

Fuel consumption with fossil fuel furnace or boiler:

Fuel Consumption (MMBTU)
$$= \frac{1}{AFUE_t}$$

$$= \frac{\frac{Btu_{fuel}}{hr} \times EFLH_{heat}}{AFUE_{fuel} \times 1,000,000 \frac{Btu}{MMBtu}}$$

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Table 3-38: Variables for HVAC Systems

<u>Term</u> Component	UnitTy pe	Value <u>s</u>	Source
$rac{Btu_{fuel}}{hr}$, Rated heating capacity of the new fossil fuel unit	Btu hr ble	Nameplate data (AHRI or AHAM)	EDC Data Gathering
$\frac{Btu_{heat}}{hr}$, Rated heating capacity of the existing electric unit	BtuVaria	Nameplate data (AHRI or AHAM) Default: set equal to $\frac{Btu_{fuel}}{hr}$	EDC Data Gathering
COP _{base} , Efficiency rating of the baseline unit. For ASHP units <	Variable	Early Replacement: Nameplate data	EDC Data Gathering
65,000 BtuH, HSPF should be used for heating savings	None	New Construction or Replace on Burnout: Default values from Table 3-39	See Table 3-39
<i>HSPF</i> _{base} , Heating seasonal performance factor of the baseline	Variable	Early Replacement: Nameplate data	EDC Data Gathering
unit. For units >65,000 BtuH, COP should be used for heating savings	Btu/hr W	New Construction or Replace on Burnout: Default values from Table 3-39	See Table 3-39
		Default = >= 95% (natural gas/propane furnace) >= 95% (natural gas/propane	
$AFUE_{fuel}$, Annual Fuel Utilization Efficiency rating of the fossil fuel	Variable None	steam boiler) >= 95% (natural gas/propane hot water boiler)	ENERGY STAR requirement
unit		>= 85% (oil furnace) >= 85% (oil steam boiler)	
		>= 85% (oil hot water boiler)	
		Nameplate data (AHRI or AHAM)	EDC Data Gathering
EFLH _{heat} , Equivalent Full Load Hours for the heating season – The kWh during the entire	Variable	Based on Logging, EMS data or Modeling ³⁸⁵	EDC Data Gathering
operating season divided by the kW at design conditions	<u>Hours</u>	Default values from <u>Table</u> <u>3-25Table</u> 3-25	1
HP_{Motor} , Gas furnace blower motor horsepower (hp)	Variable <u>HP</u>	Default: = ½ hp for furnace	Average blower motor capacity for gas furnace (typical range = 1/4 hp to 3/4 hp)
		Nameplate	EDC Data Gathering
η_{motor} , Efficiency of furnace	Variable None	From nameplate	EDC Data Gathering
blower motor	<u>inulle</u>	Default: 0.50 for furnace	Typical efficiency of ½ hp blower motor for ga furnace

³⁸⁵ Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

Table 3-39: HVAC Baseline Efficiencies386

Equipment Type and Capacity	Heating Baseline					
Air-Source Heat Pumps						
< 65,000 BtuH	7.7 HSPF					
> 65,000 BtuH and <135,000 BtuH	3.3 COP					
> 135,000 BtuH and < 240,000 BtuH	3.2 COP					
> 240,000 BtuH (IPLV for units with capacity-modulation only)	3.2 COP					
Electric Resistance Heat (Electric Furnace or Baseboa	ard)					
All sizes	1.0 COP					
Packaged Terminal Systems (Replacements)387						
РТНР	2.9 - (0.026 x Cap / 1000) COP					

Table 1-5: Heating EFLH for Pennsylvania Cities388, 389Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Arena/Auditorium/Convention- Center	1,719	2,002	1,636	1,642	1,726	1,606	1,747
College: Classes/Administrative	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Convenience Stores	603	701	573	576	605	563	612
Dining: Bar Lounge/Leisure	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Dining: Cafeteria / Fast Food	582	678	554	556	585	544	592
Dining: Restaurants	1,156	1,346	1,100	1,104	1,161	1,080	1,175
Gymnasium/Performing Arts- Theatre	1,559	1,815	1,484	1,489	1,565	1,457	1,584
Hospitals/Health care	276	321	263	264	277	258	280
Industrial: 1 Shift/Light Manufacturing	1,491	1,737	1,420	1,425	1,498	1,394	1,516
Industrial: 2 Shift	1,017	1,184	968	972	1,022	951	1,034
Industrial: 3 Shift	538	626	512	513	540	502	546

 $^{^{386}}$ Baseline values from IECC 2009, after Jan 1, 2010 or Jan 23, 2010 as applicable.

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³⁸⁷ Cap represents the rated cooling capacity of the product in Btu/h. If the unit's capacity is less than 7,000 Btu/h, 7,000 Btu/h is used in the calculation. If the unit's capacity is greater than 15,000 Btu/h, 15,000 Btu/h is used in the calculation.

³⁸⁸ US Department of Energy. Energy Star Calculator389 The Equivalent Full Load Hours (EFLH) for Pennsylvania are calculated based on the degree day scaling methodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using full load hours (FLH) from the US-DOE ENERGY STAR Calculator to account for differences in weather conditions.

Table 1-5: Heating EFLH for- Pennsylvania Cities388,- 389Space and/or Building Type	Allentown	Ērie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Lodging: Hotels/Motels/Dormitories	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Lodging: Residential	1,438	1,675	1,369	1,374	1,444	1,344	1,462
Multi-Family (Common Areas)	277	322	263	264	278	259	281
Museum/Library	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Nursing Homes	738	859	702	704	740	689	749
Office: General/Retail	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Office: Medical/Banks	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Parking Garages & Lots	1,110	1,292	1,056	1,060	1,114	1,037	1,128
Penitentiary	829	965	789	792	832	774	842
Police/Fire Stations (24 Hr)	277	322	263	264	278	259	281
Post Office/Town Hall/Court House	1,266	1,474	1,205	1,209	1,271	1,183	1,286
Religious Buildings/Church	1,718	2,001	1,635	1,641	1,725	1,605	1,746
Retail	1,188	1,383	1,130	1,135	1,193	1,110	1,207
Schools/University	1,661	1,933	1,580	1,586	1,667	1,551	1,687
Warehouses (Not Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Warehouses (Refrigerated)	1,555	1,810	1,480	1,485	1,561	1,453	1,580
Waste Water Treatment Plant	538	626	512	513	540	502	546

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

1. The Equivalent Full Load Hours (EFLH) for Pennsylvania are calculated based on Nexant's eQuest modeling analysis.

the degree day scaling methodology. The EFLH values reported in the Connecticut Program Savings Documentation were adjusted using full load hours (FLH) from the US DOE ENERGY STAR Calculator to account for differences in weather conditions. Degree day scaling ratios were calculated using heating degree day and cooling degree day values for seven Pennsylvania cities: Allentown, Erie, Harrisburg, Philadelphia, Pittsburgh, Scranton, and Williamsport. US Department of Energy. ENERGY STAR Calculator

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UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating-Company, September 2011, Pages 219-220.

http://www.energizect.com/sites/default/files/2012%20CT%20Program%20Savings%20Document

ation%20FINAL.pdf Measure Life

Measure life = 20 years390

390 DEER 2008 for nonresidential high efficiency furnace-

(http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls) PA 2013 TRM Appendix A: Measure-Lives. Note that PA Act 129 savings can be claimed for no more than 15 years.

3.2.6 SMALL C/I HVAC REFRIGERANT CHARGE CORRECTION

Measure Name	Refrigerant Charge Correction
Target Sector	Commercial and Industrial EstablishmentsSmall C/I HVAC-
Measure Unit	Tons of Refrigeration Capacity
Unit Energy Savings	<u>Variable</u> <u>Varies</u>
Unit Peak Demand Reduction	<u>Variable</u> Varies-
Measure Life	10 years ³⁹¹
Measure Vintage	Retrofit

This protocol describes the assumptions and algorithms used to quantify energy savings for refrigerant charging on packaged AC units and heat pumps operating in small commercial applications. The protocol herein describes a partially deemed energy savings and demand reduction estimation.

ELIGIBILITY

This protocol is applicable for small commercial and industrial customers, and applies to documented tune-ups for package or split systems up to 20 tons. All HVAC applications other than comfort cooling and heating, such as process cooling, are defined as non-standard applications and are ineligible for this measure.

ALGORITHMS

This section describes the process of creating energy savings and demand reduction calculations.

Air Conditioning:

For A/C units < 65,000 $\frac{Btu}{hr}$, use SEER instead of EER to calculate ΔkWh and convert SEER to EER to calculate ΔkW_{peak} using 11.3/13 as the conversion factor. For A/C units > 65,000 $\frac{Btu}{hr}$, if rated in both EER and IEER, use IEER for energy savings calculations.

391 DEER 2008. Commercial Results Review: Non-Updated Measures.

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Heat Pumps

For Heat Pump units < 65,000 $\frac{Btu}{hr}$, use SEER instead of EER to calculate ΔkWh_{cool} and HSPF instead of COP to calculate ΔkWh_{heat} . Convert SEER to calculate ΔkW_{peak} using 11.3/13 as the conversion factor. For Heat Pump units > 65,000 $\frac{Btu}{hr}$, if rated in both EER and IEER, use IEER to calculate:

<u>e</u>

$$\begin{array}{ll} \Delta kWh_{cool} \\ \Delta kWh \\ \Delta kWh_{cool} \\ \end{array} &= \left(EFLH_c \times \frac{CAPY_c}{1000 \frac{W}{kW}} \right) \times \left(\frac{1}{[IEER \times RCF]} - \frac{1}{IEER} \right) \\ \Delta kWh_{cool} \\ &= \left(EFLH_c \times \frac{CAPY_c}{1000 \frac{W}{kW}} \right) \times \left(\frac{1}{[SEER \times RCF]} - \frac{1}{SEER} \right) \\ \Delta kWh_{heat} \\ &= \left(EFLH_{mh} \times \frac{CAPY_h}{1000 \frac{W}{kW}} \right) \times \frac{1}{3.412} \times \left(\frac{1}{[COP \times RCF]} - \frac{1}{COP} \right) \\ \Delta kWh_{heat} \\ &= \left(EFLH_{mh} \times \frac{CAPY_h}{1000 \frac{W}{kW}} \right) \times \left(\frac{1}{[HSPF \times RCF]} - \frac{1}{HSPF} \right) \\ \Delta kW_{peak} \\ &= \left(\frac{Btu_{cool}}{hr} \times \frac{1}{1000 \frac{W}{kW}} \right) \times \left(\frac{1}{EER_{base}} - \frac{1}{EER_{ee}} \right) \times CF \\ \end{array}$$

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Table 3-40: Refrigerant Charge Correction Calculations Assumptions

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
CAPY_c , Unit capacity for cooling	BtuVariable e	From nNameplate	EDC Data Gathering
CAPY_h , Unit capacity for heating	Btu Variable e	From nNameplate	EDC Data Gathering
EER, Energy Efficiency Ratio. For A/C and heat pump units < 65,000	Variable Btu/hr W	From nNameplate	EDC Data Gathering
$\frac{Btu}{hr}$, SEER should be used for cooling savings.	W	Default: See Table 3-22Table 3-22in 2014 PA TRM	See Table 3-22 Table 3-222014 PA TRM
IEED leterante de grante efficience article f	Day /ha	From Nnameplate	EDC Data Gathering
IEER, Integrated energy efficiency ratio of the baseline unit.	$\frac{Btu/hr}{W}$	Default: See Table 3-22Table 3-22	See Table 3-22Table 3-22
SEER, Seasonal Energy Efficiency Ratio. For A/C and heat pump units > 65,000	Rtu/hr	From nNameplate	EDC Data Gathering
Btu hr, EER should be used for cooling savings.	Btu/hr W	Default: See Table 3-22Table 3-22	See Table 3-22Table 3-22
HSPF, Heating Seasonal Performance Factor. For heat pump units > 65,000	Variable Btu/hr	From nNameplate	EDC Data Gathering
$\frac{Btu}{hr}$, COP should be used for heating savings.	W	Default: See <u>Table 3-22Table</u> 3-22in 2014 PA TRM	See Table 3-22Table 3-222014 PA TRM
COP, Coefficient of Performance. For		From nNameplate	EDC Data Gathering
heat pump units < 65,000 Btu hr, HSPF should be used for heating savings.	None	Default: See Table 3-22Table 3-22	See Table 3-22Table 3-22
$\it EFLH_c$, Equivalent Full-Load Hours for mechanical cooling	HoursVari able	Default: See Table 3-23	2014 PA TRM Air- Conditioning 1
		Table 3-23in 2014 PA TRM	
		Based on Logging, BMS data or Modeling ³⁹²	EDC's Data Gathering

³⁹² Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

TermComponent	<u>Unit</u> Type	Value <u>s</u>	Source
EFLH _{mth} , Equivalent Full-Load Hours for Heating Mechanical Heating 393	HoursVari able	Take EFLH _{mm} as- 70% of the listed- EFLH _R in See Table Table 3-25in- 2014 PA TRM	2) 1
RCF, COP Degradation Factor for Cooling	NoneVaria ble	See Table 3-41	24
CF, Demand Coincidence Factor (See Section 1.5)	NoneFixe d	55% <u>See Table</u> 3-24Table 3-24	1
1000, convert from watts to killowatts	$\frac{W}{kW}$	$1000 \frac{W}{kW}$	Conversion Factor
11.3/13, Conversion factor from SEER to EER. based on average EER of a SEER 13 unit	None	11.3 13	3

Note: For air-source air conditioners and air-source heat pumps, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment withcapacity modulation.

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³⁹³ Here it is assumed that the compressor provides 70% of the heat, while the fan and supplemental heat strips provide the remaining 30% of the heating. The efficiency gains from refrigerant charging do not apply to the fan or supplemental heat strips.

Table 3-41: Refrigerant charge correction COP degradation factor (RCF) for various relative charge adjustments for both TXV metered and non-TXV units. 394

% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)	% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)	% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)
60%	68%	13%	28%	95%	83%	(4%)	100%	100%
59%	70%	16%	27%	96%	84%	(5%)	100%	99%
58%	71%	19%	26%	96%	85%	(6%)	100%	99%
57%	72%	22%	25%	97%	87%	(7%)	99%	99%
56%	73%	25%	24%	97%	88%	(8%)	99%	99%
55%	74%	28%	23%	97%	89%	(9%)	99%	98%
54%	76%	31%	22%	98%	90%	(10%)	99%	98%
53%	77%	33%	21%	98%	91%	(11%)	99%	97%
52%	78%	36%	20%	98%	92%	(12%)	99%	97%
51%	79%	39%	19%	98%	92%	(13%)	99%	96%
50%	80%	41%	18%	99%	93%	(14%)	98%	96%
49%	81%	44%	17%	99%	94%	(15%)	98%	95%
48%	82%	46%	16%	99%	95%	(16%)	98%	95%
47%	83%	48%	15%	99%	95%	(17%)	98%	94%
46%	84%	51%	14%	99%	96%	(18%)	98%	93%
45%	85%	53%	13%	100%	97%	(19%)	98%	93%
44%	86%	55%	12%	100%	97%	(20%)	97%	92%
43%	86%	57%	11%	100%	98%	(21%)	97%	91%
42%	87%	60%	10%	100%	98%	(22%)	97%	90%
41%	88%	62%	9%	100%	98%	(23%)	97%	90%
40%	89%	64%	8%	100%	99%	(24%)	97%	89%
39%	89%	65%	7%	100%	99%	(25%)	96%	88%

³⁹⁴ Small HVAC Problems and PotentailPotential Savings Report, CalifnormiaCalifornia Energy Commission, 2003. http://www.energy.ca.gov/2003publications/CEC-500-2003-082/CEC-500-2003-082-A-25.PDF_CA 2003 RTU Survey

% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)	% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)	% of nameplate charge added (removed)	RCF (TXV)	RCF (Orifice)
38%	90%	67%	6%	100%	99%	(26%)	96%	87%
37%	91%	69%	5%	100%	100%	(27%)	96%	86%
36%	91%	71%	4%	100%	100%	(28%)	96%	85%
35%	92%	73%	3%	100%	100%	(29%)	95%	84%
34%	92%	74%	2%	100%	100%	(30%)	95%	83%
33%	93%	76%	1%	100%	100%	(31%)	95%	82%
32%	94%	77%	(0%)	100%	100%	(32%)	95%	81%
31%	94%	79%	(1%)	100%	100%	(33%)	95%	80%
30%	95%	80%	(2%)	100%	100%	(34%)	94%	78%
29%	95%	82%	(3%)	100%	100%	(35%)	94%	77%

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

According to the 2008 Database for Energy Efficiency Resources (DEER) EUL listing, the measure life for refrigerant charging is 10 years395...

SOURCES

- 1. Nexant's eQuest modeling analysis.
- 4-2. Small HVAC Problems and PotentailPotential Savings Report, CalifnormiaCalifornia Energy Commission, 2003. http://www.energy.ca.gov/2003publications/CEC-500-2003-082/CEC-500-2003-082-A-25.PDF CA 2003 RTU Survey-

³⁹⁵ http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

- 2. Assumes 70% of heating is done by compressor, 30% by fan and supplemental resistive heat
- 3. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. The study reports CF values in the range of 44% to 63% depending on the size of the unit for the Mid Atlantic PJM hours. The study, therefore, assumes an average CF value of 55% for the PJM peak demand period that is applied to all units.
- 4-3. Average EER for SEER 13 units as calculated by EER = -0.02 × SEER² + 1.12 × SEER based on U.S. DOE Building America House Simulation Protocol, Revised 2010. http://www.nrel.gov/docs/fy11osti/49246.pdf

3.2.7 ENERGY STAR ROOM AIR CONDITIONER

Measure Name	ENERGY STAR Room Air Conditioner
Target Sector	Commercial and Industrial Establishments
Measure Unit	Room Air Conditioner
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	12 years ³⁹⁶
Measure Vintage	Replace on Burnout, Early Retirement, or New Construction

ELIGIBILITY

This protocol is for ENERGY STAR room air conditioner units installed in small commercial spaces. All HVAC applications other than comfort cooling and heating, such as process cooling, are defined as non-standard applications and are ineligible for this measure. Only ENERGY STAR units qualify for this protocol.

ALGORITHMS

If CEER is not available, use EER.

DEFINITION OF TERMS

³⁹⁶ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007. http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

Table 3-42: Variables for HVAC Systems

Component <u>Term</u>	<u>Unit</u> Type	Value <u>s</u>	Source
$\frac{\mathit{Btu}_{\mathit{cool}}}{\mathit{hr}}$, Rated cooling capacity of the energy efficient unit	A Btu Variable	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
$\begin{cal}{lll} \it{CEER}_{base}, \it{EER}_{base} \end{array}$, Efficiency rating of the baseline unit	None Variable	New Construction or Replace on Burnout: Default values from Table 3-43 to Table 3-45	See Table 3-43 to Table 3-45
		Early Replacement: Nameplate data	EDC's Data Gathering
CEER _{ee} , EER _{ee} , Efficiency rating of the energy efficiency unit. heater	None Variable	Nameplate data (AHRI or AHAM)	EDC's Data Gathering
CF, Demand Coincidence Factor (Section 1.5)	FixedNone	55% <u>Table 3-24Table</u> 3-24	1
EFLH _{cool} , Equivalent Full Load Hours for the cooling season – The kWh during the entire	.	Based on Logging, BMS data or Modeling ³⁹⁷	EDC's Data Gathering
operating season divided by the kW at design conditions.		Default values from Table 3-23 Table 3-23	SeeSee 1

Table 3-43_Table_below lists the minimum federal efficiency standards for room air conditioners (effective as of June 1, 2014) and minimum ENERGY STAR efficiency standards for RAC units of various capacity ranges, and with and without louvered sides. Units without louvered sides are also referred to as "through the wall" units or "built-in" units. Note that the new federal standards are based on the Combined Energy Efficiency Ratio Metric (CEER), which is a metric that incorporates energy use in all modes, including standby and off modes.

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³⁹⁷ Modeling is an acceptable substitute to metering and BMS data if modeing is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit). 398 Federal standards: U.S. Department of Energy. *Federal Register*. 164th ed. Vol. 76, August 24, 2011.

http://www.gpo.gov/fdsys/pkg/FR-2013-07-16/pdf/FR-2013-07-16.pdf

Table 3-43: RAC Federal Minimum Efficiency and ENERGY STAR Version 3.0 Standards³⁹⁹

Capacity (Btu/h)	Federal Standard CEER, with louvered sides	ENERGY STAR EER, with louvered sides	Federal Standard EER, without louvered sides	ENERGY STAR CEER, without louvered sides	
< 6,000	- ≥11.0	11.2	10.0	10.4	
6,000 to 7,999	211.0	11.2	10.0	10.4	
8,000 to 10,999	- ≥10.9	11.3	9.6		
11,000 to 13,999	210.9	11.5	9.5		
14,000 to 19,999	≥10.7	11.2	9.3	9.8	
20,000 to 24,999	≥9.4	9.8	9.4		
≥25,000	≥9.0				

Table 3-44 lists the minimum federal efficiency standards and minimum ENERGY STAR efficiency standards for casement-only and casement-slider RAC units. Casement-only refers to a RAC designed for mounting in a casement window with an encased assembly with a width of \leq 14.8 inches or less and a height of \leq 11.2 inches or less. Casement-slider refers to a RAC with an encased assembly designed for mounting in a sliding or casement window with a width of \leq 15.5 inches or less.

Table 3-44: Casement-Only and Casement-Slider RAC Federal Minimum Efficiency and ENERGY STAR Version 3.0 Standards (effective 2014 TRM)

Casement	Federal Standard CEER	ENERGY STAR EER
Casement-only	≥ 9.5	≥ 10.0
Casement-slider	≥ 10.4	≥ 10.9

Table 3-45 lists the minimum federal efficiency standards and minimum ENERGY STAR efficiency standards for reverse-cycle RAC units.

^{399 &}lt;u>Ibid.</u>Federal standards: U.S. Department of Energy. Federal Register. 164th ed. Vol. 76, August 24, 2011.

Table 3-45: Reverse-Cycle RAC Federal Minimum Efficiency Standards and ENERGY STAR Version 3.0 Standards (effective 2014 TRM) 400

Capacity (Btu/h)	Federal Standard CEER, with louvered sides	ENERGY STAR EER, with louvered sides	Federal Standard CEER, without louvered sides	ENERGY STAR EER, without louvered sides
< 14,000	2/0	n/a	≥ 9.3	≥ 9.8
≥ 14,000	n/a	II/a	≥ 8.7	≥ 9.2
< 20,000	≥ 9.8	≥ 10.4	nlo	n/a
≥ 20,000	≥ 9.3	≥ 9.8	n/a	I I / a

Table 1-5: Cooling EFLH for Pennsylvania Cities 401

Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
College: Classes/Administrative	690	380	733	582	520	815	490
Convenience Stores	1,216	671	1,293	1,026	917	1,436	864
Dining: Bar Lounge/Leisure	912	503	969	769	688	1,077	648
Dining: Cafeteria / Fast Food	1,227	677	1,304	1,035	925	1,449	872
Dining: Restaurants	912	503	969	769	688	1,077	648
Lodging: Hotels/Motels/Dormitories	756	418	805	638	571	894	538
Lodging: Residential	757	418	805	638	571	894	538
Multi-Family (Common Areas)	1,395	769	1,482	1,176	1,052	1,647	991
Nursing Homes	1,141	630	1,213	963	861	1,348	811
Office: General/Retail	851	469	905	718	642	1,005	605
Office: Medical/Banks	851	4 69	905	718	642	1,005	605
Penitentiary	1,091	602	1,160	920	823	1,289	775
Police/Fire Stations (24 Hr)	1,395	769	1,482	1,176	1,052	1,647	991
Post Office/Town Hall/Court House	851	469	905	718	642	1,005	605
Religious Buildings/Church	602	332	640	508	454	711	428
Retail	894	493	950	754	674	1,055	635
Schools/University	634	350	674	535	478	749	451
Warehouses (Not Refrigerated)	692	382	735	583	522	817	492

 $^{^{400}}$ ENERGY STAR standards: ENERGY STAR Program Requirements Product Specification for Room Air Conditioners, Eligibility Criteria Version 3.0. June 22, 2012.

Space and/or Building Type	Allentown	Erie	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Warehouses (Refrigerated)	692	382	735	583	522	817	492

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

1. Based on Nexant's eQuest Modeling Analysis 2014.C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. The study reports CF values in the range of 44% to 63% depending on the size of the unit for the Mid Atlantic PJM hours. The study, therefore, assumes an average CF value of 55% for the PJM peak demand period that is applied to all units. http://energizect.com/sites/default/files/NEEP%20CI%20Lighting%20LS%20FINAL%20Report_ver%205_7-19-11.pdf

3.2.8 CONTROLS: GUEST ROOM OCCUPANCY SENSOR

Measure Name	Controls: Guest Room Occupancy Sensor
Target Sector	Commercial and Industrial Establishments
Measure Unit	Occupancy Sensor
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁴⁰²

This protocol applies to the installation of a control system in hotel guest rooms to automatically adjust the temperature setback during unoccupied periods. Savings are based on the management of the guest room's set temperatures and controlling the HVAC unit for various occupancy modes. The savings are per guestroom controlled, rather than per sensor, for multiroom suites.

ELIGIBILITY

This measure is targeted to hotel customers whose guest rooms are equipped with energy management thermostats replacing manual heating/cooling temperature set-point and fan On/Off/Auto thermostat controls.

Acceptable baseline conditions are hotel guest rooms with manual heating/cooling temperature set-point and fan On/Off/Auto thermostat controls.

Efficient conditions are hotel/motel guest rooms with energy management controls of the heating/cooling temperature set-points and operation state based on occupancy modes.

ALGORITHMS

Energy savings estimates are deemed using the tables below. Estimates were derived using an EnergyPlus model of a motel 403. Model outputs were normalized to the installed capacity and reported here as kWh/Ton and coincident peak kW/Ton. Motels and hotels show differences in shell performance, number of external walls per room and typical heating and cooling efficiencies, thus savings values are presented for hotels and motels separately. Savings also depend on the size and type of HVAC unit, and whether housekeeping staff are directed to set-back/turn-off the thermostats when rooms are unrented.

 ΔkWh = CAPY * ESF ΔkW_{peak} = CAPY * DSF

⁴⁰² DEER 2008 value for energy management systems.

⁴⁰³ S. Keates, ADM Associates Workpaper: "Suggested Revisions to Guest Room Energy Management (PTAC & PTHP)", 11/14/2013 and spreadsheet summarizing the results: 'GREM Savings Summary_IL TRM_1_22_14.xlsx'.

DEFINITION OF TERMS

Table 3-46: Guest Room Occupancy Sensor – Values and References

Term	Unit	Values	Source
CAPY, Cooling capacity of controlled unit in Tons	tons	EDC Data Gathering	
ESF, Energy savings factor	kWh tons	See <u>Table 3-47Table</u> 3-47 and <u>Table</u> 3-48 Table 3-48	1
DSF, Demand savings factor	$\frac{kW}{tons}$	See <u>Table 3-49Table</u> 3-49 and <u>Table</u> 3-50 Table 3-50	1

Table 3-2: Energy Savings for Guest Room Occupancy Sensors - Motels

Table 3-47: Energy Savings for Guest Room Occupancy Sensors – Motels

HVAC Type	Baseline	ESF; Energy Savings Factor (kWh/Ton)
PTAC with Electric Resistance	Housekeeping Setback	559
Heating	No Housekeeping Setback	1,877
PTAC with Gas Heating	Housekeeping Setback	85
The man odd risating	No Housekeeping Setback	287
PTHP	Housekeeping Setback	260
	No Housekeeping Setback	1,023

Table 3-48: Energy Savings for Guest Room Occupancy Sensors – Hotels

HVAC Type	Baseline	ESF; Energy Savings Factor (kWh/Ton)
PTAC with Electric Resistance	Housekeeping Setback	322
Heating	No Housekeeping Setback	1,083
DTAC with Coo Heating	Housekeeping Setback	259
PTAC with Gas Heating	No Housekeeping Setback	876
РТНР	Housekeeping Setback	283
	No Housekeeping Setback	1,113
Central Hot Water Fan Coil with	Housekeeping Setback	245
Electric Resistance Heating	No Housekeeping Setback	822
Central Hot Water Fan Coil with Gas	Housekeeping Setback	182
Heating	No Housekeeping Setback	615

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Table 3-49: Peak Demand Savings for Guest Room Occupancy Sensors – Motels

HVAC Type	Baseline	DSF; Demand Savings Factor (kW/Ton)
PTAC with Electric Resistance	Housekeeping Setback	0.10
Heating	No Housekeeping Setback	0.28
PTAC with Gas Heating	Housekeeping Setback	0.10
	No Housekeeping Setback	0.28
РТНР	Housekeeping Setback	0.10
	No Housekeeping Setback	0.28

Table 3-50: Peak Demand Savings for Guest Room Occupancy Sensors – Hotels

HVAC Type	Baseline	DSF; Demand Savings Factor (kW/Ton)
PTAC with Electric Resistance	Housekeeping Setback	0.04
Heating	No Housekeeping Setback	0.10
DTAC with Con Heating	Housekeeping Setback	0.03
PTAC with Gas Heating	No Housekeeping Setback	0.08
РТНР	Housekeeping Setback	0.03
	No Housekeeping Setback	0.09
Central Hot Water Fan Coil with	Housekeeping Setback	0.03
Electric Resistance Heating	No Housekeeping Setback	0.08
Central Hot Water Fan Coil with Gas	Housekeeping Setback	0.02
Heating	No Housekeeping Setback	0.06

DEFAULT SAVINGS

Energy savings estimates may be claimed using the algorithms, deemed values above, and required customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

1. S. Keates, ADM Associates Workpaper: "Suggested Revisions to Guest Room Energy Management (PTAC & PTHP)", 11/14/2013 and spreadsheet summarizing the results: 'GREM Savings Summary_IL TRM_1_22_14.xlsx.' Five cities in IL were part of this study. Values in this protocol are based on the model for the city of Belleville, IL due to the similarity in the weather heating and cooling degree days with the city of Philadelphia, PA

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3.2.9 CONTROLS: ECONOMIZER

State of Pennsylvania

Measure Name	Controls: Economizer
Target Sector	Commercial and Industrial Establishments
Measure Unit	Economizer
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	10 years ⁴⁰⁴
Measure Vintage	Retrofit

Dual enthalpy economizers regulate the amount of outside air introduced into the ventilation system based on the relative temperature and humidity of the outside and return air. If the enthalpy (latent and sensible heat) of the outside air is less than that of the return air when space cooling is required, then outside air is allowed in to reduce or eliminate the cooling requirement of the air conditioning equipment. Since the economizers will not be saving energy during peak hours, the demand savings are zero.

ELIGIBILITY

This measure is targeted to non-residential establishments whose HVAC equipment is not equipped with a functional economizer.

Baseline condition is an HVAC unit with no economizer installed or with a non-functional/disabled economizer.

Efficient condition is an HVAC unit with an economizer and dual enthalpy (differential) control.

ALGORITHMS

$$\Delta kWh = \frac{SF \times AREA \times FCH_r \times 12 \frac{MBh}{ton}}{Eff}$$

$$= 0$$

DEFINITION OF TERMS

⁴⁰⁴ Based on ERS measure life study for Massachusetts Joint Utilities which looked at economizer life on HVAC systems in large commercial and industrial applications.

http://rtf.nwcouncil.org/subcommittees/nonreslighting/Measure%20Life%20Study_MA%20Joint%20Utilities_2005_ERS-1.pdf

Table 3-51: Economizer – Values and References

Term	Unit	Values	Source
SF, Savings factor; Annual cooling load savings per unit area of conditioned space in the building when compared with a baseline HVAC system with no economizer.	$\frac{tons}{ft^2}$	0.002	1
AREA, Area of conditioned space served by controlled unit	ft ²	EDC Data Gathering	
FCH_r , Free cooling hours with outdoor temperature between 60 F and 70 F. Typical operating hour conditions are defined below with standard climate zones for PA.	Hours	See Table 3-52	2
Eff, Efficiency of existing HVAC equipment. Depending on the size and age, this will either be the SEER, IEER, or EER (use EER only if SEER or IEER are not available)	$\frac{MBh}{kW}$	EDC Data Gathering Default: See Table 3-53	3

Table 3-52: FCH_r for PA Climate Zones and Various Operating Conditions

Location	FCH _r by Operating Schedule					
Operating Schedule	1 Shift, 5 days per week	2 Shift, 5 days per week	3 Shift, 5 days per week	24/7		
Allentown	419	653	1057	1688		
Erie	384	606	977	1563		
Harrisburg	377	605	1000	1746		
Philadelphia	413	634	1050	1694		
Pittsburgh	401	603	947	1622		
Scranton	465	705	1117	1787		
Williamsport	383	605	1004	1682		

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Table 3-53: Default HVAC Efficiencies for Non-Residential Buildings⁴⁰⁵

Equipment Type and Capacity	Cooling Efficiency	Heating Efficiency
Air-Source Air Conditioners		
< 65,000 BtuH	13.0 SEER	N/A
≥ 65,000 BtuH and <135,000 BtuH	11.2 EER / 11.4 IEER	N/A
≥ 135,000 BtuH and < 240,000 BtuH	11.0 EER / 11.2 IEER	N/A
≥ 240,000 BtuH and < 760,000 BtuH (IPLV for units with capacity-modulation only)	10.0 EER / 10.1 IEER	N/A
≥ 760,000 BtuH (IPLV for units with capacity-modulation only)	9.7 EER / 9.8 IEER	N/A
Water-Source and Evaporatively-Cooled Air	Conditioners	
< 65,000 BtuH	12.1 EER / 12.3 IEER	N/A
≥ 65,000 BtuH and <135,000 BtuH	11.5 EER / 11.7 IEER	N/A
≥ 135,000 BtuH and < 240,000 BtuH	11.0 EER / 11.2 IEER	N/A
≥ 240,000 BtuH	11.0 EER / 11.1 IEER	N/A

Note: For air-source air conditioners, water-source and evaporatively-cooled air conditioners, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults along with required EDC data gathering of customer data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

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⁴⁰⁵ Values from IECC 2009, Tables 503.2.3(1), 503.2.3(2), and 503.2.3(3). After Jan 1, 2010 or Jan 23, 2010 as applicable. Integrated Energy Efficiency Ratio (IEER) requirements have been incorporated from ASHRAE 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings", 2008 Supplement (Addendum S: (Tables 6.8.1A and 6.8.1B). IECC 2009 does not present IEER requirements.

SOURCES

- Bell Jr., Arthur A., 2007. HVAC Equations, Data, and Rules of Thumb, second edition, pages 51-52. Assuming 500 CFM/ton (total heat of 300-500 cfm/ton @20F delta) and interior supply flow of 1 CFM/Sq Ft as rule of thumb for all spaces, divide 1 by 500 to get 0.002 ton/Sq Ft savings factor used. This is the assumed cooling load per sq ft of a typical space and what the economizer will fully compensate for during free cooling temperatures.
- Hours calculated based on local TMY weather data with outdoor temperature between 60°F and 70°F.
- 3. Baseline values from IECC 2009, Tables 503.2.3(1), 503.2.3(2), and 503.2.3(3). After Jan 1, 2010 or Jan 23, 2010 as applicable. Integrated Energy Efficiency Ratio (IEER) requirements have been incorporated from ASHRAE 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings", 2008 Supplement (Addendum S: (Tables 6.8.1A and 6.8.1B). IECC 2009 does not present IEER requirements. https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf

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3.3 MOTORS AND VFDs

3.3.1 Premium Efficiency Motors

Measure Name	Premium Efficiency Motors
Target Sector	Commercial and Industrial Establishments
Measure Unit	Motor
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	Variable
Measure Life	<u>15 years⁴⁰⁶</u>
Measure Vintage	Replace on Burnout, New Construction, or Early Replacement-

ELIGIBILITY

For constant speed and uniformly loaded motors, the prescriptive measurement and verification protocols described below apply for to the replacement of old motors with new energy efficient motors of the same rated horsepower and for New Construction. Replacements where the old motor and new motor have different horsepower ratings are considered custom measures. Motors with variable speeds, variable loading, or industrial-specific applications are also considered custom measures.

Note that the Coincidence Factor (CF) and Run Hours of Use (RHRS) for motors specified below do not take into account systems with multiple motors serving the same load, such as duplex motor sets with a lead-lag setup. Under these circumstances, a custom measure protocol is required. Duplex motor sets in which the second motor serves as a standby motor can utilize this protocol with an adjustment made such that savings are correctly attributed to a single motor.

ALGORITHMS

From AEPS application form or EDC data gathering calculate ΔkW where:

$$\begin{array}{lll} \Delta kWh & = kWh_{base} - kWh_{ee} \\ kWh_{base} & = 0.746 \times HP \times \frac{LF}{\eta_{base}} \times RHRS \\ kWh_{ee} & = 0.746 \times HP \times \frac{LF}{\eta_{ee}} \times RHRS \\ \Delta kW_{peak} & = kW_{base} - kW_{ee} \\ kW_{base} & = 0.746 \times HP \times \frac{LF}{\eta_{base}} \times CF \\ kW_{ee} & = 0.746 \times HP \times \frac{LF}{\eta_{ee}} \times CF \end{array}$$

⁴⁰⁶ DEER Effective Useful Life. October 10, 2008.

DEFINITION OF TERMS

Relative to the algorithms in section (3.3.1), $\triangle kW$ values will be calculated for each motor improvement in any project (account number). For the efficiency of the baseline motor, if a new motor was purchased as an alternative to rewinding an old motor, the nameplate efficiency of the old motor may be used as the baseline.

Motors and VFDs Page 34

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Table 3-54: Building Mechanical System Variables for Premium Efficiency Motor Calculations

	<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
	HP, Rated horsepower of the baseline and energy efficient motor	<u>HP</u> Variable	Nameplate	EDC Data Gathering
Ī	DUDG07 Arrand are borne of the contact	<u>Hours</u> Variabl	Based on logging, panel data or modeling ⁴⁰⁸	EDC Data Gathering
	RHRS ⁴⁰⁷ , Annual run hours of the motor	е	Default <u>: Table</u> 3-57Table 3-57 to Table 3-61	1
-	LF ⁴⁰⁹ , Load Factor. Ratio between the actual load and the rated load. Motor		Based on spot metering and nameplate	EDC Data Gathering
	efficiency curves typically result in motors being most efficient at approximately 75% of the rated load. The default value is 0.75. Variable loaded motors should use custom measure protocols.	None Variable	Default: 75%	<u>2</u> 4
			Early Replacement: Nameplate	EDC Data Gathering
	η_{base} , Efficiency of the baseline motor	NoneVariable	New Construction or Replace on Burnout: Default comparable standard motor. NEMA Premium (See Table 3-55 and Table 3-56)	Table 3-55 and Table 3-56
	η_{ee} , Efficiency of the energy-efficient motor	None Variable	Nameplate	EDC Data Gathering
	CF ^{+±±} , Demand Coincidence Factor	<u>None</u> Variable	Table 3-57Table 3-57Table 1-5-to Table 3-61_Single Motor- Configuration: 74% Duplex Motor- Configuration: 37%	1

Note: that tThe Energy Independence and Security Act of 2007⁴¹² restates the definition of General Purpose Electric Motors and classifies them as Subtype I or Subtype II.

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⁴⁰⁷ Default <u>v</u>Value can be used by EDC but is subject to metering _and adjustment by evaluators or SWE

⁴⁰⁸ Modeling is an acceptable substitute to metering and panel data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

⁴⁰⁹ Default <u>v</u>¥alue can be used by EDC but is subject to metering and adjustment by evaluators or SWE

⁴¹⁰ See definition in section <u>3.3.11.1.2</u>3.3.2 for specific algorithm to be used when performing spot metering analysis to determine alternate load factor.

⁴¹¹ Need to confirm source through TWG

⁴¹² US Congress, Energy Independence and Security Act of 2007 (EISA), January 4, 2007. http://www.gpo.gov/fdsys/pkg/BILLS-110hr6enr/pdf/BILLS-110hr6enr.pdf

The term 'General Purpose electric motor (Subtype I)' means any motor that meets the definition of 'General Purpose' as established in the final rule issued by the Department of Energy titled "Energy Efficiency Program for Certain Commercial and Industrial Equipment: Test Procedures, Labeling, and Certification Requirements for Electric Motors" (10 CFR 431), as in effect on the date of enactment of the Energy Independence and Security Act of 2007.

The term 'General Purpose electric motor (Subtype II)' means motors incorporating the design elements of a general purpose electric motor (Subtype I) that are configured as one of the following:

- A U-Frame Motor
- A Design C Motor
- A close-coupled pump motor
- A Footless motor
- A vertical solid shaft normal thrust motor (as tested in a horizontal configuration)
- An 8-pole motor (900 rpm)
- A poly-phase motor with voltage of not more than 600 volts (other than 230 or 460 volts)

Motors and VFDs

Table 3-55: Baseline Motor Nominal Efficiencies for General Purpose Electric Motors (Subtype I)⁴¹³

	Open Drip Proof (ODP) # of Poles			Totally End	closed Fan-Cooled (TEFC) # of Poles		
	6	4	2	6	4	2	
		Speed (RPM)			Speed (RPM)		
Size HP	1200	1800	3600	1200	1800	3600	
1	82.50%	85.50%	77.00%	82.50%	85.50%	77.00%	
1.5	86.50%	86.50%	84.00%	87.50%	86.50%	84.00%	
2	87.50%	86.50%	85.50%	88.50%	86.50%	85.50%	
3	88.50%	89.50%	85.50%	89.50%	89.50%	86.50%	
5	89.50%	89.50%	86.50%	89.50%	89.50%	88.50%	
7.5	90.20%	91.00%	88.50%	91.00%	91.70%	89.50%	
10	91.70%	91.70%	89.50%	91.00%	91.70%	90.20%	
15	91.70%	93.00%	90.20%	91.70%	92.40%	91.00%	
20	92.40%	93.00%	91.00%	91.70%	93.00%	91.00%	
25	93.00%	93.60%	91.70%	93.00%	93.60%	91.70%	
30	93.60%	94.10%	91.70%	93.00%	93.60%	91.70%	
40	94.10%	94.10%	92.40%	94.10%	94.10%	92.40%	
50	94.10%	94.50%	93.00%	94.10%	94.50%	93.00%	
60	94.50%	95.00%	93.60%	94.50%	95.00%	93.60%	
75	94.50%	95.00%	93.60%	94.50%	95.40%	93.60%	
100	95.00%	95.40%	93.60%	95.00%	95.40%	94.10%	
125	95.00%	95.40%	94.10%	95.00%	95.40%	95.00%	
150	95.40%	95.80%	94.10%	95.80%	95.80%	95.00%	
200	95.40%	95.80%	95.00%	95.80%	96.20%	95.40%	
250	95.40%	95. <u>4</u> 80%	9 <u>4.5</u> 5.00%	95. <mark>80</mark> 0%	9 <u>5.0</u> 6.20%	95. <mark>84</mark> 0%	
300	95.40%	95. <u>4</u> 80%	95.4 <u>0</u> 0%	95. <u>0</u> 80%	9 6.2 5.40%	95. <u>4</u> 80%	
350	95.40%	95. <mark>84</mark> 0%	95.4 <u>0</u> 0%	95. <u>0</u> 80%	9 <u>5.4</u> 6.20%	95. <u>4</u> 80%	
400	<u>N/A</u> 95.80%	95. <u>4</u> 80%	95. <u>4</u> 80%	<u>N/A</u> 95.80%	9 <u>5.4</u> 6.20%	95. <u>4</u> 80%	
450	N/A96.20%	9 <u>5</u> 6.2 <u>8</u> 0%	95.80%	<u>N/A</u> 95.80%	9 <u>5.4</u> 6.20%	95. <u>4</u> 80%	
500	<u>N/A</u> 96.20%	9 <u>5</u> 6.2 <u>8</u> 0%	95.80%	<u>N/A</u> 95.80%	9 6.2 5.80%	95. <u>4</u> 80%	

⁴¹³ Table is based on NEMA premium efficiency motor standards. The Department of Energy published a final rule on May 29, 2014 that applies to electric motors manufactured on or after June 1, 2016. Therefore, baseline efficiencies for electric motors will be updated in the 2016 TRM to comply with federal energy conservation standards. http://www.ecfr.gov/cgi-bin/text-idx?SID=ba6d4f97451f89bcaa13b3f5a91c54c1&node=10:3.0.1.4.19.2.47.11&rgn=div8

Table 3-56: Baseline Motor Nominal Efficiencies for General Purpose Electric Motors (Subtype II)414

		Open Drip F # of F			Totally Enclosed Fan-Cooled (TEFC # of Poles			
	<u>8</u>	6	4	2	<u>8</u>	6	4	2
Size		Speed	(RPM)			Speed	(RPM)	
HP	900	1200	1800	3600	900	1200	1800	3600
1	74.0%	80.0%	82.5%	N/A	74.0%	80.0%	82.5%	75.5%
1.5	<u>75.5%</u>	84.0%	84.0%	82.5%	<u>77.0%</u>	85.5%	84.0%	82.5%
2	<u>85.5%</u>	85.5%	84.0%	84.0%	82.5%	86.5%	84.0%	84.0%
3	86.5%	86.5%	86.5%	84.0%	84.0%	87.5%	87.5%	85.5%
5	87.5%	87.5%	87.5%	85.5%	<u>85.5%</u>	87.5%	87.5%	87.5%
7.5	<u>88.5%</u>	88.5%	88.5%	87.5%	<u>85.5%</u>	89.5%	89.5%	88.5%
10	89.5%	90.2%	89.5%	88.5%	88.5%	89.5%	89.5%	89.5%
15	<u>89.5%</u>	90.2%	91.0%	89.5%	<u>88.5%</u>	90.2%	91.0%	90.2%
20	90.2%	91.0%	91.0%	90.2%	<u>89.5%</u>	90.2%	91.0%	90.2%
25	90.2%	91.7%	91.7%	91.0%	<u>89.5%</u>	91.7%	92.4%	91.0%
30	91.0%	92.4%	92.4%	91.0%	91.0%	91.7%	92.4%	91.0%
40	91.0%	93.0%	93.0%	91.7%	91.0%	93.0%	93.0%	91.7%
50	91.7%	93.0%	93.0%	92.4%	91.7%	93.0%	93.0%	92.4%
60	92.4%	93.6%	93.6%	93.0%	91.7%	93.6%	93.6%	93.0%
75	93.6%	93.6%	94.1%	93.0%	93.0%	93.6%	94.1%	93.0%
100	93.6%	94.1%	94.1%	93.0%	93.0%	94.1%	94.5%	93.6%
125	93.6%	94.1%	94.5%	93.6%	93.6%	94.1%	94.5%	94.5%
150	93.6%	94.5%	95.0%	93.6%	93.6%	95.0%	95.0%	94.5%
200	93.6%	94.5%	95.0%	94.5%	94.1%	95.0%	95.0%	95.0%
250	94.5%	9 <u>5</u> 4. <u>54</u> %	95.4%	94.5%	94.5%	95.0%	95.0%	95.4%
300	N/A	9 <u>5</u> 4. <u>54</u> %	95.4%	95.0%	N/A	95.0%	95.4%	95.4%
350	N/A	9 <u>5</u> 4. <u>54</u> %	95.4%	95.0%	N/A	95.0%	95.4%	95.4%
400	N/A	N/A	95.4%	95.4%	N/A	N/A	95.4%	95.4%
450	N/A	N/A	95.8%	95.8%	N/A	N/A	95.4%	95.4%
500	N/A	N/A	95.8%	95.8%	N/A	N/A	95.8%	95.4%

Table 1-4: Stipulated Hours of Use for Motors in Commercial Buildings⁴¹⁵

Facility Type	Fan Motor	Chilled Water- Pumps/Cooling Tower Fan	Heating Pumps
Auto Related	4,056	1,878	6,000

Motors and VFDs

⁴¹⁴Table is based on NEMA premium efficiency motor standards.- <u>Ibid.</u>-445-Operating hours subject to adjustment with data provided by EDCs and accepted by SWE

Bakery	2,85 4	1,445	6,000
Banks, Financial Centers-	3,748	1,767	6,000
Church-	1,955	1,121	6,000
College – Cafeteria	6,376	2,713	6,000
College - Classes/Administrative	2,586	1,348	6,000
College - Dormitory	3,066	1,521	6,000
Commercial Condos-	4,055	1,877	6,000
Convenience Stores	6,376	2,713	6,000
Convention Center	1,954	1,121	6,000
Court House	3,748	1,767	6,000
Dining: Bar Lounge/Leisure	4 ,182	1,923	6,000
Dining: Cafeteria / Fast Food	6,456	2,742	6,000
Dining: Family	4,182	1,923	6,000
Entertainment	1,952	1,120	6,000
Exercise Center	5,836	2,518	6,000
Fast Food Restaurants	6,376	2,713	6,000
Fire Station (Unmanned)	1,953	1,121	6,000
Food Stores	4,055	1,877	6,000
Gymnasium	2,586	1,348	6,000
Hospitals	7,674	3,180	6,000
Hospitals / Health Care	7,666	3,177	6,000
Industrial - 1 Shift	2,857	1,446	6,000
Industrial - 2 Shift	4,730	2,120	6,000
Industrial - 3 Shift	6,631	2,805	6,000
Laundromats-	4,056	1,878	6,000
Library	3,748		6,000
Light Manufacturers	2,857	1,446	6,000
Lodging (Hotels/Motels)	3,064	1,521	6,000
Mall Concourse	4,833	2,157	6,000
Manufacturing Facility	2,857	1,446	6,000
Medical Offices	3,748	1,767	6,000
Motion Picture Theatre	1,954	1,121	6,000
Multi-Family (Common Areas)	7,665	3,177	6,000
Museum	3,748	1,767	6,000
Nursing Homes	5,840	2,520	6,000
Office (General Office Types)	3,748	1,767	6,000
Office/Retail	3,748	1,767	6,000
Parking Garages & Lots	4,368	1,990	6,000
Penitentiary	5,477	2,389	6,000

Performing Arts Theatre	2,586	1,348	6,000
Police / Fire Stations (24 Hr)	7,665	3,177	6,000
Post Office-	3,748	1,767	6,000
Pump Stations	1,949	1,119	6,000
Refrigerated Warehouse	2,602	1,354	6,000
Religious Building	1,955	1,121	6,000
Residential (Except Nursing Homes)	3,066	1,521	6,000
Restaurants	4,182	1,923	6,000
Retail	4,057	1,878	6,000
School / University	2,187	1,205	6,000
Schools (Jr./Sr. High)	2,187	1,205	6,000
Schools (Preschool/Elementary)	2,187	1,205	6,000
Schools (Technical/Vocational)	2,187	1,205	6,000
Small Services	3,750	1,768	6,000
Sports Arena	1,95 4	1,121	6,000
Town Hall	3,748	1,767	6,000
Transportation	6,456	2,742	6,000
Warehouse (Not Refrigerated)	2,602	1,354	6,000
Waste Water Treatment Plant	6,631	2,805	6,000
Workshop-	3,750	1,768	6,000
Other ⁴¹⁶	3,985	1,852	6,000

⁴¹⁶ To be used only when no other category is applicable.

Table 3-57: <u>Default RHRS and CFs for Supply Fan Motors in Commercial Buildings417</u>

Facility Type	<u>Parameter</u>	Allentown	Erie	Harrisburg	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsburg
Accombly	Run Hours	<u>5,188</u>	<u>5,217</u>	<u>5,172</u>	<u>5,186</u>	<u>5,201</u>	<u>5,207</u>	<u>5,184</u>
<u>Assembly</u>	<u>CF</u>	0.53	<u>0.45</u>	0.60	0.72	<u>0.56</u>	0.47	0.52
Education -	Run Hours	<u>5,972</u>	6,081	<u>5,772</u>	<u>5,878</u>	<u>5,911</u>	<u>5,795</u>	<u>5,824</u>
Community College	<u>CF</u>	0.44	0.32	0.45	0.48	0.43	0.40	0.47
Education -	Run Hours	<u>3,753</u>	<u>3,961</u>	3,699	3,894	3,790	3,881	<u>3,763</u>
Primary School	<u>CF</u>	<u>0.10</u>	0.07	<u>0.16</u>	<u>0.16</u>	<u>0.17</u>	0.11	<u>0.12</u>
Education -	Run Hours	<u>5,467</u>	<u>5,649</u>	<u>5,375</u>	<u>5,321</u>	<u>5,556</u>	<u>5,607</u>	<u>5,439</u>
Relocatable Classroom	<u>CF</u>	<u>0.15</u>	<u>0.11</u>	<u>0.18</u>	0.19	0.20	0.14	<u>0.15</u>
Education -	Run Hours	3,920	<u>4,106</u>	<u>3,866</u>	3,937	3,900	3,983	3,928
Secondary School	<u>CF</u>	<u>0.11</u>	0.09	0.18	0.19	0.17	0.12	<u>0.17</u>
Education -	Run Hours	<u>6,111</u>	<u>6,196</u>	<u>5,948</u>	6,053	6,053	<u>5,957</u>	<u>5,985</u>
University	<u>CF</u>	0.41	<u>0.31</u>	0.43	0.45	0.40	0.36	<u>0.40</u>
Cracomi	Run Hours	6,708	6,738	6,692	6,669	<u>6,718</u>	6.725	<u>6,710</u>
<u>Grocery</u>	<u>CF</u>	0.24	0.22	0.24	0.26	0.29	0.21	0.24
Health/Medical -	Run Hours	<u>8,760</u>	8,760	<u>8,760</u>	8,760	<u>8,760</u>	8.760	<u>8,760</u>
<u>Hospital</u>	<u>CF</u>	0.43	0.39	0.45	<u>0.51</u>	<u>0.45</u>	0.40	<u>0.41</u>
Health/Medical -	Run Hours	<u>8,760</u>	8,760	8,760	8,760	8,760	8.760	<u>8,760</u>
Nursing Home	<u>CF</u>	0.24	0.23	0.29	0.31	0.29	0.25	0.28
Lodging Hotel	Run Hours	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>
<u>Lodging - Hotel</u>	<u>CF</u>	0.64	0.65	0.71	0.71	0.73	0.65	<u>0.71</u>
Manufacturing -	Run Hours	3,570	3,616	3,539	3,565	<u>3,571</u>	3,552	<u>3,573</u>
Bio/Tech	<u>CF</u>	0.56	0.44	0.57	0.61	0.57	0.50	0.52

⁴¹⁷ Operating hours subject to adjustment with data provided by EDCs and accepted by SWE.

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Facility Type	<u>Parameter</u>	Allentown	Erie	<u>Harrisburg</u>	<u>Philadelphia</u>	<u>Pittsburgh</u>	Scranton	Williamsburg
Manufacturing -	Run Hours	4,092	4,338	3,998	<u>4,111</u>	<u>4,167</u>	<u>4,251</u>	4,084
Light Industrial	<u>CF</u>	0.39	0.31	0.49	0.52	0.42	0.36	0.40
Office - Large	Run Hours	<u>4,400</u>	<u>4,696</u>	<u>4,298</u>	<u>4,342</u>	<u>4,503</u>	<u>4,441</u>	<u>4,353</u>
Office - Large	<u>CF</u>	0.30	0.29	0.39	0.39	<u>0.34</u>	0.34	0.35
Office Small	Run Hours	3,990	<u>4,185</u>	3,876	3,784	3,976	4,014	3,924
Office - Small	<u>CF</u>	0.29	0.27	0.35	0.38	0.35	0.30	0.33
Restaurant -	Run Hours	7,328	7,398	<u>7,300</u>	7,238	<u>7,313</u>	7,342	7,332
Fast-Food	<u>CF</u>	0.36	0.33	0.39	0.47	0.44	0.38	0.42
Restaurant - Sit-	Run Hours	<u>5,236</u>	<u>5,332</u>	<u>5,203</u>	<u>5,213</u>	<u>5,286</u>	<u>5,288</u>	<u>5,239</u>
<u>Down</u>	<u>CF</u>	0.39	0.41	0.45	0.53	0.54	0.40	0.48
Retail -	Run Hours	<u>4,893</u>	4,897	4,885	4,885	4,907	4,890	4,896
Multistory Large	<u>CF</u>	0.48	0.39	0.54	0.53	0.48	0.44	0.49
Retail - Single-	Run Hours	<u>5,486</u>	<u>5,494</u>	<u>5,481</u>	<u>5,497</u>	<u>5,502</u>	<u>5,493</u>	<u>5,487</u>
Story Large	<u>CF</u>	0.50	0.40	0.53	0.63	<u>0.55</u>	0.47	0.47
Datail Consil	Run Hours	<u>5,031</u>	5,083	4,959	<u>4,895</u>	<u>5,030</u>	<u>5,063</u>	<u>5,018</u>
Retail - Small	<u>CF</u>	0.53	0.52	0.51	0.53	0.59	0.45	0.50
Storage -	Run Hours	5.037	5,222	4.980	<u>5,168</u>	<u>5,110</u>	<u>5,188</u>	5.028
Conditioned	<u>CF</u>	<u>0.18</u>	0.13	0.24	0.30	0.23	0.15	0.20
Warehouse -	Run Hours	4.041	4,041	4.041	4.041	4.041	4,041	4,041
Refrigerated	<u>CF</u>	<u>0.50</u>	0.48	0.52	<u>0.53</u>	<u>0.51</u>	0.48	<u>0.51</u>

Motors and VFDs

Table 3-58: Default RHRS and CFs for Chilled Water Pump (CHWP) Motors in Commercial Buildings⁴¹⁸

Facility Type	<u>Parameter</u>	Allentown	<u>Erie</u>	<u>Harrisburg</u>	<u>Philadelphia</u>	<u>Pittsburgh</u>	Scranton	Williamsburg
Education - Community	Run Hours	2,868	<u>2,561</u>	2,937	3,307	<u>2,775</u>	2,660	<u>2,727</u>
College	<u>CF</u>	0.42	0.30	0.43	0.46	<u>0.41</u>	<u>0.35</u>	0.43
Education - Secondary	Run Hours	<u>2,721</u>	<u>2,175</u>	<u>2,730</u>	<u>3,505</u>	<u>2,676</u>	<u>2,310</u>	<u>2,573</u>
School	<u>CF</u>	<u>0.10</u>	0.09	<u>0.18</u>	<u>0.18</u>	<u>0.17</u>	0.12	<u>0.16</u>
Education -	Run Hours	<u>5,145</u>	<u>4,721</u>	<u>5,177</u>	<u>5,314</u>	<u>5,056</u>	<u>4,995</u>	<u>5,016</u>
<u>University</u>	<u>CF</u>	<u>0.39</u>	0.29	<u>0.40</u>	0.43	<u>0.38</u>	<u>0.31</u>	<u>0.36</u>
Health/Medical -	Run Hours	<u>5,588</u>	<u>5,109</u>	<u>5,717</u>	<u>6,086</u>	<u>5,593</u>	<u>5,266</u>	<u>5,628</u>
<u>Hospital</u>	<u>CF</u>	0.46	0.42	<u>0.50</u>	<u>0.54</u>	0.48	0.44	<u>0.47</u>
Health/Medical -	Run Hours	<u>3,892</u>	<u>3,456</u>	<u>4,104</u>	<u>4.535</u>	<u>3,900</u>	<u>3.710</u>	<u>3,818</u>
Nursing Home	<u>CF</u>	0.24	0.22	0.28	0.30	0.28	0.23	<u>0.26</u>
Lodging - Hotel	Run Hours	<u>5,845</u>	<u>5,198</u>	<u>6,045</u>	<u>6,161</u>	<u>5,686</u>	<u>5,655</u>	<u>5,776</u>
<u>Loughing - Floter</u>	<u>CF</u>	<u>0.61</u>	<u>0.60</u>	<u>0.66</u>	<u>0.67</u>	<u>0.69</u>	<u>0.59</u>	<u>0.66</u>
Manufacturing -	Run Hours	<u>1,735</u>	<u>1,448</u>	<u>1,742</u>	<u>1,891</u>	<u>1,606</u>	<u>1,558</u>	<u>1,633</u>
Bio/Tech	<u>CF</u>	0.53	0.43	0.53	0.58	<u>0.54</u>	0.48	<u>0.50</u>
Office - Large	Run Hours	<u>1.873</u>	<u>1,713</u>	<u>1,912</u>	<u>2,173</u>	<u>1,876</u>	<u>1,741</u>	<u>1,815</u>
Office - Large	<u>CF</u>	0.30	0.28	<u>0.36</u>	0.37	0.33	0.30	<u>0.33</u>
Office - Small	Run Hours	<u>1,705</u>	<u>1,456</u>	<u>1,696</u>	<u>1,899</u>	<u>1,602</u>	<u>1,534</u>	<u>1,606</u>
Office - Striati	<u>CF</u>	0.28	<u>0.26</u>	0.33	<u>0.35</u>	<u>0.32</u>	0.28	<u>0.31</u>
Retail -	Run Hours	2,957	2,653	3,085	3,225	<u>2,795</u>	<u>2,735</u>	<u>2,898</u>
Multistory Large	<u>CF</u>	<u>0.46</u>	0.38	0.53	0.54	0.47	0.42	0.47

418 Ibid.

SECTION 3: Commercial and Industrial Measures

Table 3-59: Default RHRS and CFs for Cooling Tower Fan (CTF) Motors in Commercial Buildings⁴¹⁹

Rev Date: June 2015

Facility Type	<u>Parameter</u>	Allentown	<u>Erie</u>	<u>Harrisburg</u>	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsburg
Education - Community	Run Hours	2,868	<u>2,560</u>	<u>2,937</u>	<u>3,306</u>	<u>2,774</u>	<u>2,660</u>	<u>2,727</u>
College	<u>CF</u>	0.42	0.30	0.43	<u>0.46</u>	<u>0.41</u>	<u>0.35</u>	0.42
Education - Secondary	Run Hours	<u>2,742</u>	<u>2,178</u>	<u>2,744</u>	<u>3,517</u>	<u>2,685</u>	<u>2,313</u>	<u>2,604</u>
School	<u>CF</u>	<u>0.11</u>	0.09	<u>0.18</u>	<u>0.18</u>	<u>0.17</u>	0.12	<u>0.17</u>
Education -	Run Hours	<u>5,143</u>	<u>4,721</u>	<u>5,176</u>	<u>5,312</u>	<u>5,053</u>	<u>4,993</u>	<u>5,015</u>
<u>University</u>	<u>CF</u>	<u>0.39</u>	0.29	<u>0.40</u>	0.43	<u>0.38</u>	<u>0.31</u>	<u>0.36</u>
Health/Medical -	Run Hours	<u>5,587</u>	<u>5,107</u>	<u>5,714</u>	<u>6,084</u>	<u>5,591</u>	<u>5,263</u>	<u>5,626</u>
<u>Hospital</u>	<u>CF</u>	<u>0.45</u>	<u>0.41</u>	<u>0.49</u>	0.54	<u>0.47</u>	0.44	<u>0.46</u>
Health/Medical -	Run Hours	<u>3,894</u>	<u>3,457</u>	<u>4,106</u>	<u>4.537</u>	<u>3.902</u>	<u>3,711</u>	<u>3,819</u>
Nursing Home	<u>CF</u>	0.24	0.22	0.28	0.30	0.28	0.23	<u>0.26</u>
Lodging - Hotel	Run Hours	<u>5,844</u>	<u>5,197</u>	<u>6,043</u>	<u>6,159</u>	<u>5,683</u>	<u>5,652</u>	<u>5,773</u>
<u>Louging - Hotel</u>	<u>CF</u>	<u>0.61</u>	<u>0.61</u>	<u>0.67</u>	0.68	<u>0.70</u>	<u>0.59</u>	<u>0.66</u>
Manufacturing -	Run Hours	<u>1,735</u>	<u>1,448</u>	<u>1,742</u>	<u>1,891</u>	<u>1,606</u>	<u>1,558</u>	<u>1,633</u>
Bio/Tech	<u>CF</u>	0.53	0.43	<u>0.54</u>	<u>0.59</u>	0.54	0.48	<u>0.50</u>
Office - Large	Run Hours	<u>1.873</u>	<u>1,713</u>	<u>1,912</u>	<u>2.173</u>	<u>1,876</u>	<u>1,741</u>	<u>1,815</u>
Office - Large	<u>CF</u>	0.30	0.28	<u>0.36</u>	0.37	0.33	0.30	<u>0.33</u>
Office - Small	Run Hours	<u>1,705</u>	<u>1,456</u>	<u>1,696</u>	<u>1,899</u>	<u>1,602</u>	<u>1,534</u>	<u>1,606</u>
Office - Small	<u>CF</u>	0.28	<u>0.26</u>	0.33	<u>0.35</u>	0.32	0.28	<u>0.31</u>
Retail -	Run Hours	2,957	2,653	3,085	3,226	2,795	<u>2,736</u>	2,898
Multistory Large	<u>CF</u>	<u>0.46</u>	0.38	0.53	<u>0.54</u>	0.47	0.42	0.47

419 Ibid.

SECTION 3: Commercial and Industrial Measures

Table 3-60: Default RHRS and CFs for Heating Hot Water Pump (HHWP) Motors in Commercial Buildings⁴²⁰

Facility Type	<u>Parameter</u>	Allentown	<u>Erie</u>	<u>Harrisburg</u>	<u>Philadelphia</u>	<u>Pittsburgh</u>	Scranton	Williamsburg
Education - Community	Run Hours	<u>4,454</u>	<u>4,941</u>	<u>4,150</u>	<u>3,838</u>	<u>4,447</u>	<u>4,562</u>	<u>4,408</u>
College	<u>CF</u>	<u>0.01</u>	<u>0.01</u>	0.00	0.00	<u>0.01</u>	<u>0.01</u>	<u>0.01</u>
Education - Secondary	Run Hours	<u>3,651</u>	<u>4,080</u>	<u>3,492</u>	<u>3,341</u>	<u>3,705</u>	<u>3,830</u>	<u>3,658</u>
School	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Education -	Run Hours	<u>4,642</u>	<u>5,131</u>	<u>4,350</u>	<u>4,190</u>	<u>4,697</u>	<u>4,714</u>	<u>4,566</u>
<u>University</u>	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Health/Medical -	Run Hours	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>	<u>8,760</u>
<u>Hospital</u>	<u>CF</u>	0.09	<u>0.09</u>	0.09	0.09	0.09	<u>0.09</u>	<u>0.09</u>
Health/Medical -	Run Hours	<u>5,934</u>	<u>6,280</u>	<u>5.823</u>	<u>5,477</u>	<u>5,991</u>	<u>6,223</u>	<u>6,045</u>
Nursing Home	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Lodging - Hotel	Run Hours	<u>6,469</u>	<u>6,829</u>	<u>6,155</u>	<u>6,077</u>	<u>6,574</u>	<u>6,628</u>	<u>6,387</u>
<u>Loughig - Hotel</u>	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Manufacturing -	Run Hours	<u>1,258</u>	<u>1,555</u>	<u>1,184</u>	<u>1,028</u>	<u>1,287</u>	<u>1,393</u>	<u>1,277</u>
Bio/Tech	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Office - Large	Run Hours	<u>3,705</u>	<u>4,097</u>	<u>3,503</u>	<u>3,112</u>	<u>3,703</u>	<u>3,809</u>	<u>3,652</u>
Office - Large	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Office - Small	Run Hours	<u>2,723</u>	<u>3,124</u>	<u>2,525</u>	<u>2,267</u>	<u>2,788</u>	<u>2,863</u>	<u>2,685</u>
Office - Small	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Retail -	Run Hours	<u>2,676</u>	<u>2,960</u>	<u>2,561</u>	<u>2,398</u>	<u>2,908</u>	<u>2,841</u>	<u>2,660</u>
Multistory Large	<u>CF</u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00

420 Ibid.

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Table 3-61: Default RHRS and CFs for Condenser Water Pump Motors in Commercial Buildings 421

Facility Type	<u>Parameter</u>	Allentown	Erie	Harrisburg	<u>Philadelphia</u>	Pittsburgh	Scranton	Williamsburg
Education - Community	Run Hours	<u>2,611</u>	2,344	<u>2,733</u>	3,000	2,668	<u>2,429</u>	<u>2,530</u>
College	<u>CF</u>	0.42	0.30	0.43	0.46	<u>0.41</u>	<u>0.35</u>	<u>0.42</u>
Education - Secondary	Run Hours	<u>2,448</u>	2,039	2,539	<u>3,346</u>	2,409	<u>2,164</u>	<u>2,423</u>
School	<u>CF</u>	<u>0.11</u>	0.09	<u>0.18</u>	<u>0.18</u>	<u>0.17</u>	0.12	<u>0.17</u>
Education -	Run Hours	<u>4,443</u>	3,782	<u>4,471</u>	<u>5,059</u>	<u>4,830</u>	<u>4,571</u>	<u>4,448</u>
<u>University</u>	<u>CF</u>	0.39	0.29	0.40	0.43	0.38	<u>0.31</u>	<u>0.36</u>
Health/Medical -	Run Hours	<u>3,950</u>	<u>3,698</u>	<u>3,687</u>	<u>4,168</u>	4,093	<u>3,713</u>	<u>3,670</u>
<u>Hospital</u>	<u>CF</u>	0.45	<u>0.41</u>	0.49	0.54	0.47	0.44	<u>0.46</u>
Health/Medical -	Run Hours	<u>3,675</u>	3,394	3,725	4.304	<u>3,571</u>	3,687	3,722
Nursing Home	<u>CF</u>	0.24	0.22	0.28	0.30	0.28	0.23	<u>0.26</u>
Lodging - Hotel	Run Hours	<u>5,544</u>	<u>4,766</u>	<u>5,569</u>	<u>5,886</u>	<u>5,239</u>	<u>5,353</u>	<u>5,328</u>
<u>Louging - Hotel</u>	<u>CF</u>	<u>0.61</u>	<u>0.61</u>	<u>0.67</u>	<u>0.68</u>	<u>0.70</u>	<u>0.59</u>	<u>0.66</u>
Manufacturing -	Run Hours	<u>1,735</u>	<u>1,445</u>	<u>1,737</u>	<u>1,889</u>	<u>1,602</u>	<u>1,558</u>	<u>1,632</u>
Bio/Tech	<u>CF</u>	0.53	0.43	<u>0.54</u>	<u>0.59</u>	<u>0.54</u>	0.48	<u>0.50</u>
Office - Large	Run Hours	<u>1,857</u>	<u>1,685</u>	<u>1,891</u>	<u>2,156</u>	<u>1,862</u>	<u>1.728</u>	<u>1,798</u>
Office - Large	<u>CF</u>	<u>0.30</u>	0.28	<u>0.36</u>	0.37	0.33	0.30	0.33
Office - Small	Run Hours	<u>1,705</u>	<u>1,453</u>	<u>1,693</u>	<u>1,898</u>	<u>1,597</u>	<u>1,533</u>	<u>1,606</u>
Office - Small	<u>CF</u>	0.28	0.26	0.33	<u>0.35</u>	0.32	0.28	<u>0.31</u>
Retail -	Run Hours	2,889	<u>2,616</u>	3,025	<u>3,185</u>	<u>2,757</u>	<u>2,702</u>	<u>2,847</u>
Multistory Large	<u>CF</u>	0.46	0.38	0.53	0.54	0.47	0.42	0.47

421 Ibid.

SECTION 3: Commercial and Industrial Measures

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

Motor projects achieving expected kWh savings of 250,000 kWh or higher must⁴²² be metered to calculate *ex ante* and/or *ex post* savings. In addition, if any motor within a sampled project uses the "Other" category to stipulate hours, the threshold is decreased to 25,000 kWh. Metering is not mandatory where the motors in question are constant speed and hours can be easily verified through a building automation system schedule that clearly shows motor run time.

SOURCES

- 1. Results are based on Nexant eQuest modeling analysis 2014.
- California Public Utility Commission. Database for Energy Efficiency Resources 2005.
 The coincident peak demand period is 12:00 p.m. to 6:00 p.m., weekdays, May October.

UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating Company, September 2011

Other category calculated based on simple averages.

Motors and VFDs

⁴²² The Commission allows the EDCs to use alternative methods for obtaining customer-specific data where customer processes do not support metering. The EDCs are required to provide supporting documentation to the SWE for review if there are any such exceptions.

SECTION 3: Commercial and Industrial Measures

3.3.2 VARIABLE FREQUENCY DRIVE (VFD) IMPROVEMENTS

Measure Name	Variable Frequency Drive (VFD) Improvements
Target Sector	Commercial and Industrial Establishments
Measure Unit	Variable Frequency Drive
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	13 years ⁴²³
Measure Vintage	Retrofit

ELIGIBILITY

The following protocol for the measurement of energy and demand savings applies to the installation of Variable Frequency Drives (VFDs) in standard commercial building applications shown in Table 3-63. The baseline condition is a motor without a VFD control. The efficient condition is a motor with a VFD control.

ALGORITHMS

$$\Delta kWh = HP \times \frac{LF}{\eta_{motor}} \times RHRS_{base} \times ESF$$

$$\Delta kW_{peak} = HP \times \frac{LF}{\eta_{motor}} \times CF \times DSF$$

DEFINITIONS OF TERMS

⁴²³ The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007. http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

Rev Date: June 2015

	TermComponent-	<u>Unit</u> Type	Value <u>s</u>	Source
	Motor HP, Rated horsepower of the motor	<u>HP</u> Variable	Nameplate	EDC Data Gathering
1	$\it RHRS_{\it base}^{\it 424}$, Annual run hours of the baseline	Hours Variable	Based on logging, panel data or modeling	EDC Data Gathering
	motor	<u>riours v ariable</u>	Default: <u>See Table</u> 3-57Table 3-57_to Table 3-61	1
	<i>LF</i> ⁴²⁵ , Load Factor. Ratio between the actual load and the rated load. Motor efficiency curves typically result in motors being most efficient at	None Variable	Based on spot metering and nameplate	EDC Data Gathering
	approximately 75% of the rated load. The default value is 0.75.		Default <u>:</u> 75%	1
	ESF, Energy Savings Factor. Percent of baseline energy consumption saved by installing VFD.	<u>None</u> Variable	Default See Table 3-63 Table 3-63	See Table 3-63 <u>Table 3-63</u> Table 3-63
			Based on logging and panel data	EDC Data Gathering
	DSF, Demand Savings Factor. Percent of baseline demand saved by installing VFD	None Variable	Default: See Table 3-63 Table 3-63	See <u>Table 3-63</u> Table 3-63
	, ,		Based on logging and panel data	EDC Data Gathering
	Efficiency – η_{base} η_{motor} , Motor efficiency at the full-rated load. For VFD installations, this can be either an energy efficient motor or standard efficiency motor. Motor efficiency varies with load and decreases dramatically below 50% load; this is reflected in the ESF term of the algorithm.	<u>None</u> Variable	Nameplate	EDC Data Gathering
	CF ⁴²⁶ , Demand Coincidence Factor	<u>None</u> Fixed	74%See Table 3-57Table 3-57 to Table 3-61	2

SECTION 3: Commercial and Industrial Measures

Motors and VFDs

⁴²⁴ Default Value can be used by EDC but is subject to metering and adjustment by evaluators or SWE
425 Default Value can be used by EDC but is subject to metering and adjustment by evaluators or SWE<u>lbid.</u>
426.Need to confirm source through TWG

Table 3-63: ESF and DSF for Typical Commercial VFD Installations^{427,428}

HVAC Fan VFD Savings Factors								
Baseline	ESF	DSF						
Constant Volume ⁴²⁹	0.534	0.347						
Air Foil/Backward Incline	0.354	0.26						
Air Foil/Backward Incline with Inlet Guide Vanes	0.227	0.13						
Forward Curved	0.179	0.136						
Forward Curved with Inlet Guide Vanes	0.092	0.029						
HVAC Pump VFD Sav	ings Factors							
System	ESF	DSF						
Chilled Water Pump	0.411	0.299						
Hot Water Pump	0.424	0						

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOL

VFD projects achieving expected kWh savings of 250,000 kWh or higher must⁴³⁰ be metered to calculate *ex ante* and/or *ex post* savings. In addition, if any VFD within a sampled project uses the "Other" category to stipulate hours, the threshold is decreased to 25,000 kWh. Metering is not mandatory where hours can be easily verified through a building automation system schedule that clearly shows motor run time.

SOURCES

- California Public Utility Commission. Database for Energy Efficiency Resources 2005.
 The coincident peak demand period is 12:00 p.m. to 6:00 p.m., weekdays, May October.
- 4.2. Results are based on Nexant's eQuest modeling analysis 2014

SECTION 3: Commercial and Industrial Measures

⁴²⁷ UI and CL&P Program Savings Documentation for 2012 Program Year, United Illuminating Company, September 2011. Pages
<u>44-45. http://www.energizect.com/sites/default/files/2012%20CT%20Program%20Savings%20Documentation%20FINAL.pdf</u>
428 UI and CL&P Program Savings Documentation for 2011 Program Year, United Illuminating Company, September 2010. Page 44.

⁴²⁹ The ESF and DSF values for the constant volume baseline condition are taken from the 2011 Connecticut TRM whereas 2012 Connecticut TRM was used to report values for all other baseline conditions. This is because the 2012 Connecticut TRM does not report values for constant volume condition. Note that the values for all baseline conditions for HVAC fans are same in both versions of the Connecticut TRM. The values were only updated for the HVAC pump baseline conditions.

⁴³⁰ The Commission allows the EDCs to use alternative methods for obtaining customer-specific data where customer processes do not support metering. The EDCs are required to provide supporting documentation to the SWE for review if there are any such exceptions.

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3.3.3 ECM CIRCULATING FAN

Measure Name	ECM Circulating Fan
Target Sector	Commercial and Industrial Establishments
Measure Unit	ECM Circulating Fan
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	18 years ⁴³¹
Measure Vintage	Early Replacement

This protocol covers energy and demand savings associated with retrofit of existing shaded-pole (SP) or permanent-split capacitor (PSC) evaporator fan motors in an air handling unit with an electronically commutated motor (ECM).

ELIGIBILITY

This measure is targeted to non-residential customers whose air handling equipment currently uses a SP or PSC fan motor rather than an ECM. This measure applies only to circulating fan motors of 1 hp or less. Above 1 hp motors are governed by NEMA standards and would see little to no efficiency benefit by adding an ECM.

The targeted fan can supply heating or cooling only, or both heating and cooling. A default savings option is offered if motor input wattage is not known. However, these parameters should be collected by EDCs for greatest accuracy.

Acceptable baseline conditions are an existing circulating fan with a SP or PSC fan motor 1 hp or less

Efficient conditions are a circulating fan with an ECM.

ALGORITHMS

 $\begin{array}{ll} \varDelta kWh & = \Delta kWh_{neat} + \Delta kWh_{cool} \\ \Delta kW_{peak} & = \Delta kW_{cool} \end{array}$

SECTION 3: Commercial and Industrial Measures

⁴³¹ "Energy Savings from Efficient Furnace Fan Air Handlers in Massachusetts," ACEEE, Sachs and Smith, 2003. For the purpose of calculating the total Resource Cost Test for Act 129, measure cannot claim savings for more than fifteen years.

Heating

$$\Delta kWh_{heat} = \frac{(WATTS_{base} - WATTS_{ee})}{1000} \times LF \times EFLH_{heat} \times (1 + IF_{kWh})$$

$$= 0$$

Cooling

$$\Delta kWh_{cool} = \frac{(WATTS_{base} - WATTS_{ee})}{1000} \times LF \times EFLH_{cool} \times (1 + IF_{kWh})$$

$$\Delta kW_{cool} = \frac{(WATTS_{base} - WATTS_{ee})}{1000} \times LF \times CF \times (1 + IF_{kW})$$

DEFINITION OF TERMS

Table 3-64: ECM Circulating Fan – Values and References

	Term	Unit	Values	Source
	vivienna B. III. III.		Nameplate data	EDC Data Gathering
	WATTS _{base} , Baseline watts	Watts	Default : See <u>Table 3-65</u> Table 3-65	1, 2, 3
.			Nameplate data	EDC Data Gathering
	WATTS _{ee} , Energy efficient watts	Watts	Default : See <u>Table 3-65</u> Table 3-65	1, 2, 3
	I.C. Lond factor	Nama	EDC Data Gathering	EDC Data Gathering
	LF , Load factor	Load factor None	Default: 0.9	4
	EFLH _{heat} , Equivalent Full-Load	hours	EDC Data Gathering	EDC Data Gathering
	Hours for heating only	year	Default : See <u>Table 3-25Table</u> 3-25	7
			EDC Data Gathering	EDC Data Gathering
	EFLH _{cool} , Equivalent Full-Load Hours for cooling only	hours year	Default : See Table 3-23 Table 3-23	7
			EDC Data Gathering	EDC Data Gathering
	CF, Coincidence Factor	None	Default : See <u>Table 3-24Table</u> 3-24	7
			EDC Data Gathering	EDC Data Gathering
	IF_{kWh} , Energy Interactive Factor	None	$ \begin{array}{l} IF_{kW} \\ \times \left(1 - \frac{EFLH_{heat}}{EFLH_{heat} + EFLH_{cool}}\right) \\ \times \frac{13}{11.3} \end{array} $	5
	IF Demand Interactive Factor	None	EDC Data Gathering	EDC Data Gathering
I	IF_{kW} , Demand Interactive Factor	NOILE	Default : 30%	6

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Table 3-65: Default Motor Wattage (WATTSbase and WATTSee) for Circulating Fan

	Motor Category					
Motor Type	1/40 HP (16-23 watts) (Using 19.5 watt as industry average)	1/20 HP (~37 watts)	1/15 HP (~49 watts)			
Motor Output Watts	19.5	37	49			
SP	93	142	191			
PSC	48	90	120			
ECM	30	56	75			

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDC's may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery Display Case ECM, FY2010, V2. http://rtf.nwcouncil.org/measures/measure.asp?id=107&decisionid=230
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Deemed MeasuresV26 _walkinevapfan.
- AO Smith New Product Notification. I-motor 9 & 16 Watt. Stock Numbers 9207F2 and 9208F2.
 Web address: http://www.electricmotorwarehouse.com/PDF/Bulletin%206029B.pdf
- PSC of Wisconsin, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, p. 4-103 to 4-106. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.odf
- Assuming that the waste heat is within the conditioned air stream, then the energy associated with removing the waste heat during peak times is approximated as the inverse of the COP, or 3.413/EER = 0.30 if one uses 11.3 as a default value for cooling system EER.
- This is an approximation that accounts for the coincidence between cooling and fan operation and corrects with a factor of 11.3/13 to account for seasonal cooling efficiency rather then peak cooling efficiency.
- 7. Nexant eQuest modeling analysis 2014.

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3.3.4 VSD on KITCHEN EXHAUST FAN

Measure Name	VSD on Kitchen Exhaust Fan
Target Sector	Commercial and Industrial Establishments
Measure Unit	VSD on Kitchen Exhaust Fan
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁴³²
Measures Vintage	Retrofit

Installation of variable speed drives (VSD) on commercial kitchen exhaust fans allows the variation of ventilation based on cooking load and/or time of day.

ELIGIBILITY

This measure is targeted to non-residential customers whose kitchen exhaust fans are equipped with a VSD that varies the exhaust rate of kitchen ventilation based on the energy and effluent output from the cooking appliances (i.e., the more heat and smoke/vapors generated, the more ventilation needed). This involves installing a temperature sensor in the hood exhaust collar and/or an optic sensor on the end of the hood that sense cooking conditions which allows the system to automatically vary the rate of exhaust to what is needed by adjusting the fan speed. The baseline equipment is kitchen ventilation that has a constant speed ventilation motor.

The energy efficient condition is a kitchen ventilation system equipped with a VSD and demand ventilation controls and sensors.

ALGORITHMS

Annual energy and demand savings values are based on monitoring results from five different types of sites, as summarized in the PG&E work paper⁴³³. The sites included an institutional cafeteria, a casual dining restaurant, a hotel kitchen, a supermarket kitchen, and a university dining facility. Units are based on savings per total exhaust fan rated horsepower. Savings values are applicable to new and retrofit units.

 $^{^{\}rm 432}$ PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116. June 1, 2009 $^{\rm 433}$ lbid.

DEFINITION OF TERMS

Table 3-66: VSD on Kitchen Exhaust Fan - Variables and References

Term	Unit	Values	Source
4,486, Annual energy savings per total exhaust fan horsepower	$\frac{kWh}{HP}$	4,486	1, 2
0.76, Coincident peak demand savings per total exhaust fan horsepower	$\frac{kW}{HP}$	0.76	1, 2
HP, Horsepower rating of the exhaust fan	HP	Nameplate data	EDC Data Gathering

DEFAULT SAVINGS

Savings for this measure are partially deemed based on motor horsepower.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116. June 1, 2009
- SDGE Workpaper, Work Paper WPSDGENRCC0019, Commercial Kitchen Demand Ventilation Controls-Electric, Revision 0. June 15, 2012. https://www.google.com/url?sa=t&rct=j&q=&esrc=s&source=web&cd=1&ved=0CCsQFjA A&url=https%3A%2F%2Fwww.sdge.com%2Fsites%2Fdefault%2Ffiles%2Fregulatory%2 FWPSDGENRCC0019%2520Rev%25200%2520Demand%2520Ventilation%2520Controls.doc&ei=RZuMU5vCNK-lsATEqoCoAg&usg=AFQjCNFltI0wjiCmylhK06gWIEYcX7b3lw&sig2=mwHSNhg EnQF7 eSVUyvgpA&bvm=bv.67720277,d.cWc&cad=rja

SECTION 3: Commercial and Industrial Measures

3.4 DOMESTIC HOT WATER

3.4.1 ELECTRIC RESISTANCE WATER HEATERS

Measure Name	Efficient Electric Resistance Water Heaters	
Target Sector	Small Commercial and Industrial Establishments	
Measure Unit	Electric Resistance Water Heater	
Unit Energy Savings	<u>Variable</u> Varies	
Unit Peak Demand Reduction	<u>Variable</u> Varies	
Measure Life	15 years ⁴³⁴	
Measure Vintage	Replace on Burnout	

Efficient electric resistance water heaters use resistive heating coils to heat the water. Premium efficiency models primarily generally use increased tank insulation to achieve energy factors of 0.93 to 0.96.

ELIGIBILITY

This protocol documents the energy savings attributed to efficient electric resistance water heaters with a minimum energy factor of 0.93 compared to a baseline electric resistance water heater with an energy factor of 0.904. However, other energy factors are accommodated with the partially deemed scheme. The target sector includes domestic hot water applications in small commercial settings such as small retail establishments, small offices, small clinics, and small lodging establishments such as small motels.

ALGORITHMS

The energy savings calculation utilizes average performance data for available premium and standard electric resistance water heaters and typical hot water usages. The energy savings are obtained through the following formula:

For efficient resistive water heaters, demand savings result primarily from reduction in standby losses. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage.

$$\Delta kW_{peak}$$
 = ETDF × Energy Savings × RDF

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⁴³⁴ DEER Effective Useful Life. October 10, 2008

The Energy to Demand Factor is defined below:

ETDF =
$$\frac{Average\ Usage}{Annual\ Energy\ Usage}$$

Loads

The annual loads are taken from data from the DEER database data for gas energy usage for the domestic hot water end use for various small commercial buildings. The loads are averaged over all 16 climate zones and all six vintage types in the DEER database. Finally, the loads are converted to average annual gallons of use using the algorithm below. The loads are summarized in Table 3-67 below.

$$HW (Gallons) = \frac{Load \times EF_{ng,base} \times 1,000 \frac{Btu}{kBtu} \times Typical SF}{8.3 \frac{lb}{gal} \times \frac{1}{kBtu} \times Typical SF}$$

Table 3-67: Typical water heating loads

Building Type	Typical Square Footage	Average Annual Load In kBTU	Average Annual Use, Gallons
Motel	30,000	2,963	97,870
Small Office	10,000	2,214	24,377
Small Retail	7,000	1,451	11,183

Energy to Demand Factor

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA⁴³⁶. The usage profiles are shown in Figure 3-1 Figure 3-1. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Figure 3-2 Figure 3-2, indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for all building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania⁴³⁷.

⁴³⁵ DEER 2008. Commercial Results Review Non-Updated Measures.

⁴³⁶ ibid

⁴³⁷ One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).

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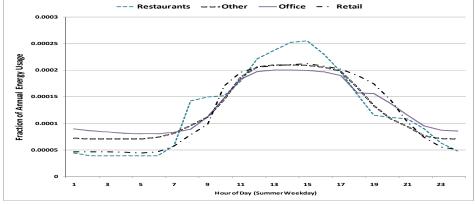


Figure 3-1: Load shapes for hot water in four commercial building types

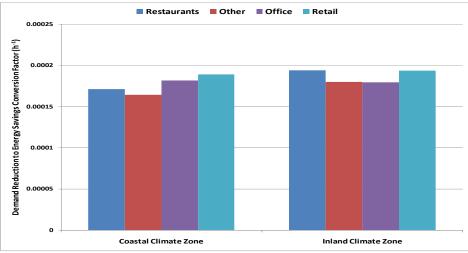


Figure 3-2: Energy to demand factors for four commercial building types

DEFINITION OF TERMS

The parameters in the above equation are listed in Table 3-68.

Domestic Hot Water

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Table 3-68: Electric Resistance Water Heater Calculation Assumptions

Component-Term	Type <u>Unit</u>	Values	Source
$\it EF_{base}$, Energy Factor of baseline water heater	None Variable	See	1
		<u>Table 3-69</u> Table 3-69	
$\mathit{EF}_{proposed}$, Energy Factor of proposed efficient water heater	<u>None</u> Variable	Default: 0.93	Program Design
		Nameplate	EDC Data Gathering
Load, Average annual Load	<u>kBTU</u> Fixed	Varies	DEER Database
T_{hot} , Temperature of hot water	<u>°FDegrees</u> <u>Fahrenheit</u> Fixed	1 <u>19</u> 23 °F	2
T_{cold} , Temperature of cold water supply	<u>°FDegrees</u> <u>Fahrenheit</u> Fixed	55 °F	3
ETDF, Energy_To_Demand_Factor	<u>None</u> Fixed	0.000178	4
HW, Average annual gallons of use	Gal lons	Default: See	Calculation
	Variable	<u>Table 3-67</u>	
		Table 3-67	
		EDC Data Gathering	EDC Data Gathering
$\it EF_{ng,base}$, Energy Factor of baseline gas water heater	<u>None</u> Fixed	0.594	5
RDF, Resistive Discount Factor	<u>None</u> Fixed	1.0	6

Energy Factors based on Tank Size

Federal Standards for Energy Factors are equal to $0.97 - 0.00132 \times Rated\ Storage\ (gallons)0.97 - 0.00132 \times Rated\ Storage\ in\ Gallons$. The following table shows the Energy Factors for various tank sizes.

Table 3-69: Minimum Baseline Energy Factors based on Tank Size

Tank Size (gallons)	Minimum Energy Factors (E _{base})
40	0.9172
50	0.9040
65	0.8842
80	0.8644
120	0.8116

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DEFAULT SAVINGS

Savings for the installation of efficient electric resitanceresistance water heaters in different building types are calculated using the formulae formulas below:

Table 3-70: Energy Savings Algorithms

Building Type	Default Algorithms
Motel	$\Delta kWh = 16,185 \left(\frac{1}{EF_{base}} - \frac{1}{EF_{proposed}} \right)$
Small Office	$\Delta kWh = 4,031 \left(\frac{1}{EF_{base}} - \frac{1}{EF_{proposed}} \right)$
Small Retail	$\Delta kWh = 1,849 \left(\frac{1}{EF_{base}} - \frac{1}{EF_{proposed}}\right)$

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, an electric water heater's lifespan is 15 years 438.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US <u>DeptDept.</u> of Energy Docket Number: EE–2006–BT-STD–0129, p. 30 http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- 2. 20142 SWE Residential Baseline Study Mid-Atlantic TRM. http://www.puc.pa.gov/Electric/pdf/Act129/SWE-2014 PA Statewide Act129 Residential Baseline Study.pdf

http://www.puc.pa.gov/electric/pdf/Act129/PA_Residential_Baseline_Report2012.pdf

2-3. Mid-Atlantic TRM, footnote #24. http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM_March2013Version.pdf

http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

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⁴³⁸ DEER values, updated October 10, 2008

- 3.4. The ETDF is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 4.5_Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons. For a 40-gallon tank this is 0.594. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept_Dept_ of Energy Docket Number: EE-2006-BT-STD-0129, p. 30 http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- 5.6. Engineering Estimate. No discount factor is needed because this measure is already an electric resisitanceresistance water heater system.

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3.4.2 HEAT PUMP WATER HEATERS

Measure Name	Heat Pump Water Heaters
Target Sector	Commercial and Industrial Establishments
Measure Unit	Heat Pump Water Heater
Unit Energy Savings	<u>Variable</u> Varies
Unit Peak Demand Reduction	<u>Variable</u> Varies
Measure Life	10 years ⁴³⁹
Measure Vintage	Replace on Burnout

Heat pump water heaters take heat from the surrounding air and transfer it to the water in the tank, unlike conventional electrical water heaters which use resistive heating coils to heat the water.

ELIGIBILITY

This protocol documents the energy savings attributed to heat pump water heaters with an energy factors of 2.2. However, other energy factors are accommodated with the partially deemed scheme. The target sector includes domestic hot water applications in small commercial settings such as small retail establishments, small offices, small clinics, and small lodging establishments such as small motels. The measure described here involves a direct retrofit of a resistive electric water heater with a heat pump water heater. It does not cover systems where the heat pump is a pre-heater or is combined with other water heating sources. More complicated installations can be treated as custom projects.

ALGORITHMS

The energy savings calculation utilizes average performance data for available heat pump and standard electric resistance water heaters and typical hot water usages. The energy savings are obtained through the following formula:

$$\Delta kWh = \frac{\left\{ \left(\frac{1}{EF_{base}} - \left(\frac{1}{EF_{proposed}} \times \frac{1}{F_{adjust}} \right) \right) \times HW \times \frac{X}{4} 8.3 \frac{lb}{gal} \times 1.0 \frac{Btu}{lb \cdot {}^{\circ}F} \times (T_{hot} - T_{cold}) \right\} }{3413 \times \frac{Btu}{kWh}}$$

For heat pump water heaters, demand savings result primarily from a reduced connected load. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage.

$$\Delta kW_{peak} \hspace{1.5cm} = ETDF \hspace{0.1cm} \times Energy \hspace{0.1cm} Savings \hspace{0.1cm} \times RDF$$

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⁴³⁹ DEER Effective Useful Life. October 10, 2008.

The Energy to Demand Factor is defined below:

ETDF =
$$\frac{Average\ Usage}{Annual\ Energy\ Usage}$$

Loads

The annual loads are taken from data from the DEER database data for gas energy usage for the domestic hot water end use for various small commercial buildings. The loads are averaged over all 16 climate zones and all six vintage types in the DEER database. Finally, the loads are converted to average annual gallons of use using the algorithm below. The loads are summarized in Table 3-71 below.

$$HW (Gallons) = \frac{Load \times EF_{ng,base} \times 1,000 \frac{Btu}{kBtu} \times Typical SF}{8.3 \frac{lb}{gal} \times \frac{1}{2} (T_{hot} - T_{cold}) \times 1,000 SF}$$

Table 3-71: Typical water heating loads

Building Type	Typical Square Footage	Average Annual Load In kBTU	Average Annual Use, Gallons
Motel	30,000	2,963	97,870
Small Office	10,000	2,214	24,377
Small Retail	7,000	1,451	11,183

Energy to Demand Factor

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA⁴⁴¹. The usage profiles are shown in Figure 3-3. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Figure 3-4, indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for al building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania⁴⁴².

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⁴⁴⁰ DEER 2008 Commercial Results Review Non Updated Measures

⁴⁴¹ ibid

⁴⁴² One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).

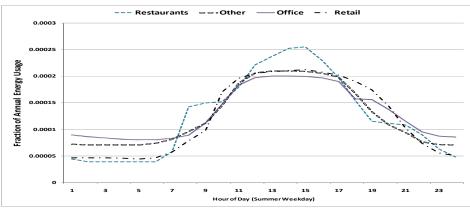


Figure 3-3: Load shapes for hot water in four commercial building types

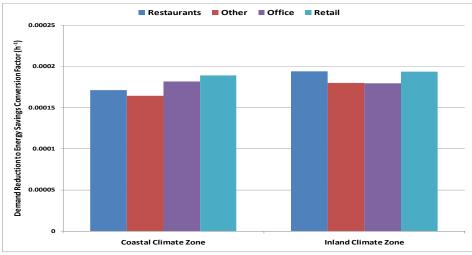


Figure 3-4: Energy to demand factors for four commercial building types

Resistive Heating Discount Factor

The resistive heating discount factor is an attempt to account for possible increased reliance on back-up resistive heating elements during peak usage conditions. Although a brief literature review failed to find data that may lead to a quantitative adjustment, two elements of the demand reduction calculation are worth considering.

- The hot water temperature in this calculation is somewhat conservative at 123 °F.
- · The peak usage window is eight hours long.
- In conditioned space, heat pump capacity is somewhat higher in the peak summer window.
- In unconditioned space, heat pump capacity is dramatically higher in the peak summer window.

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Under these operating conditions, one would expect a properly sized heat pump water heater with adequate storage capacity to require minimal reliance on resistive heating elements. A resistive heating discount factor of 0.9, corresponding to a 10% reduction in COP during peak times, is therefore taken as a conservative estimation for this adjustment.

Heat Pump COP Adjustment Factor

The energy factors are determined from a DOE testing procedure that is carried out at 56 °F wetbulb temperature. However, the average wetbulb temperature in PA is closer to 45 °F ⁴⁴³, while the average wetbulb temperature in conditioned typically ranges from 50 °F to 80 °F. The heat pump performance is temperature dependent. Figure 3-5 Figure 3-5 below shows relative coefficient of performance (COP) compared to the COP at rated conditions ⁴⁴⁴. According to the plotted profile, the following adjustments are recommended.

Table 3-72: COP Adjustment Factors

Heat Pump Placement	Typical WB Temperature °F	COP Adjustment Factor
Unconditioned Space	44	0.80
Conditioned Space	63	1.09
Kitchen	80	1.30

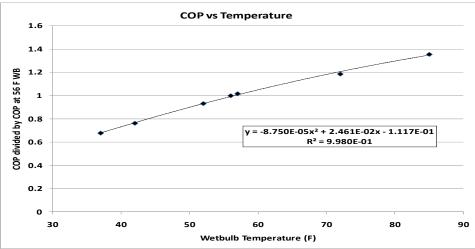


Figure 3-5: Dependence of COP on outdoor wetbulb temperature-

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 $^{^{443}}$ Based on TMY2 weather files from DOE2.com for Erie, Harrisburg, Pittsburgh, Wilkes-Barre, And Williamsport, the average annual wetbulb temperature is 45 ± 1.3 °F. The wetbulb temperature in garages or attics, where the heat pumps are likely to be installed, are likely to be two or three degrees higher, but for simplicity, 45 °F is assumed to be the annual average wetbulb temperature.

⁴⁴⁴ The performance curve is adapted from Table 1 in http://wescorhvac.com/HPWH%20design%20details.htm#Single-stage%20HPWHs. The performance curve depends on other factors, such as hot water set point. Our adjustment factor of 0.84 is a first order approximation based on the information available in literature.

DEFINITION OF TERMS

The parameters in the above equation are listed in <u>Table 3-73</u>Table 3-73.

Table 3-73: Heat Pump Water Heater Calculation Assumptions

Component <u>Term</u>	Type Unit	Values	Source
$\it EF_{\it base}$, Energy Factor of baseline water heater	None Variable	See <u>Table</u> 3-74 Table 3-74	1
EF _{proposed} , Energy Factor of proposed	None Variable	Default: 2.2	Program Design
efficient water heater		Nameplate	EDC Data Gathering
Load, Average annual Load	<u>kBTU</u> Fixed	Varies	5
T_{hot} , Temperature of hot water	Degrees FahrenheitFixed	1 <u>19</u> 23 °F	2
T_{cold} , Temperature of cold water supply	Degrees FahrenheitFixed	55 °F	3
ETDF, Energy to Demand Factor	None Fixed	0.000178	4
F_{adjust} ,COP Adjustment factor	None Fixed	0.80 if outdoor 1.09 if indoor 1.30 if in kitchen	4
RDF, Resistive Discount Factor	NoneFixed-	0.90	6
HW, Average annual gallons of use	<u>Gallons</u> Variable	Default: See <u>Table</u> 3-71 Table 3-71	Calculation
		EDC Data Gathering	EDC Data Gathering
$\it EF_{ng,base}$, Energy Factor of baseline gas water heater	Fixed None	0.594	7

Energy Factors based on Tank Size

Federal Standards for Energy Factors are equal to $0.97-0.00132 \times Rated\ Storage\ (gallons)0.97-0.00132 \times Rated\ Storage\ (gallons)0.97-0.00132 \times Rated\ Storage\ in\ Gallons.$ The following table shows the Energy Factors for various tank sizes.

Table 3-74: Minimum Baseline Energy Factor Based on Tank Size

Tank Size (gallons)	Minimum Energy Factors (Ebase)
40	0.9172
50	0.9040
65	0.8842
80	0.8644
120	0.8116

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DEFAULT SAVINGS

As an example, the default savings for the installation of heat pump electric water heaters with <a>an energy factor of 2.2 in various applications are calculated using the algorithms below:

Building Type	Location Installed	Algorithm
Motel	Outdoor	$\Delta kWh = (\frac{16,184.54}{EF_{base}} - \frac{20,230.67}{EF_{proposed}})$
Motel	Indoor	$\Delta kWh = (\frac{16,184.54}{EF_{base}} - \frac{14,848.20}{EF_{proposed}})$
Motel	Kitchen	$\Delta kWh = (\frac{16,184.54}{EF_{base}} - \frac{12,449.64}{EF_{proposed}})$
Small Office	Outdoor	$\Delta kWh = (\frac{4,031.17}{EF_{base}} - \frac{5,038.96}{EF_{proposed}})$
Small Office	Indoor	$\Delta kWh = (\frac{4,031.17}{EF_{base}} - \frac{3,698.32}{EF_{proposed}})$
Small Office	Kitchen	$\Delta kWh = (\frac{4,031.17}{EF_{base}} - \frac{3,100.90}{EF_{proposed}})$
Small Retail	Outdoor	$\Delta kWh = (\frac{1,849.31}{EF_{base}} - \frac{2,311.63}{EF_{proposed}})$
Small Retail	Indoor	$\Delta kWh = (\frac{1,849.31}{EF_{base}} - \frac{1,696.6}{EF_{proposed}})$
Small Retail	Kitchen	$\Delta kWh = (\frac{1,849.31}{EF_{base}} - \frac{1,422.54}{EF_{proposed}})$

MEASURE LIFE

According to an October 2008 report for the CA Database for Energy Efficiency Resources, an electric water heater's lifespan is 10 years445.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

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⁴⁴⁵ DEER values, updated October 10, 2008. http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

SOURCES

- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30
- 20124 SWE Residential Baseline Study Mid-Atlantic TRM. http://www.puc.pa.gov/Electric/pdf/Act129/SWE-2014 PA Statewide Act129 Residential Baseline Study.pdf http://www.puc.pa.gov/electric/pdf/Act129/PA Residential Baseline Report2012.pdf
- Mid-Atlantic TRM, footnote #24. http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM March2013Version.pdf
- The EnergyToDemandFactor_ETDF_is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 5. <u>DEER 2008. Commercial Results Review Non-Updated Measures. DEER Database</u>
- 6. Engineering Estimate
- Federal Standards are 0.67 -0.0019 x Rated Storage in Gallons. For a 40-gallon tank this is 0.594. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30 http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf

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3.4.3 Low Flow Pre-Rinse Sprayers for Retrofit Programs

Measure Name	Low Flow Pre-Rinse Sprayers for Retrofit Programs
Target Sector	Commercial Kitchens and Industrial Establishments
Measure Unit	Pre Rinse Sprayer
Unit Energy Savings	Groceries: 151 kWh; Non-Groceries: 1,222 kWh
Unit Peak Demand Reduction	Groceries: 0.03kW; Non-Groceries: 0.22 kW
Measure Life	5 years ⁴⁴⁶
Measure Vintage	Early Replacement

ELIGIBILITY

This protocol documents the energy savings and demand reductions attributed to efficient low flow pre-rinse sprayers in grocery and non-grocery (primarily food service) applications. The most likely areas of application are kitchens in restaurants and hotels. Only premises with electric water heating may qualify for this incentive. In addition, the replacement pre-rinse spray nozzle must use less than 1.6 gallons per minute with a cleanability performance of 26 seconds per plate or less. Low flow pre-rinse sprayers reduce hot water usage and save energy associated with water heating.

This protocol is applicable to Retrofit programs only. The baseline for Retrofit Program is assumed to be ana 2.25 GPM⁴⁴⁷ and 2.15 GPM⁴⁴⁸ for non-grocery and grocery applications respectively.

ALGORITHMS

The energy savings and demand reduction are calculated through the protocols documented below.

The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage.

$$\Delta kW_{peak}$$
 = ETDF × Energy Savings

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⁴⁴⁶ Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve Installation Program (Phase 2), SBW Consulting, 2007, p. 30

¹⁴⁷ ibid

⁴⁴⁸ ibid

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The Energy to Demand Factor is defined below:

$$ETDF = \frac{Average\ Usage_{Summer\ WD\ 2-6\ PM}}{Annual\ Energy\ Usage}$$

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA. The usage profiles are shown in Figure 3-6. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Figure 3-7, indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for al building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania⁴⁴⁹.

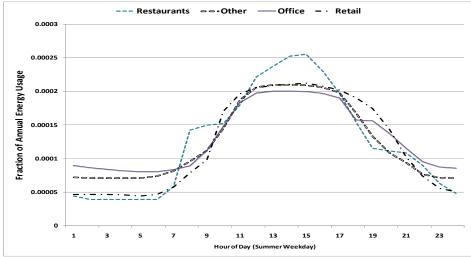


Figure 3-6: Load shapes for hot water in four commercial building types

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⁴⁴⁹ One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).

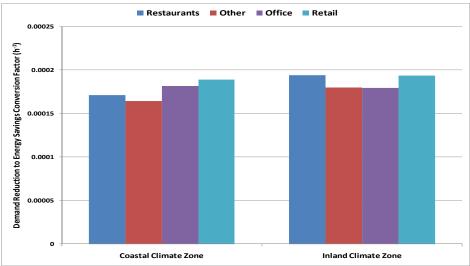


Figure 3-7: Energy to demand factors for four commercial building types.

DEFINITION OF TERMS

The parameters in the above equation are listed in Table 3-75 below. The values for all parameters except incoming water temperature are taken from impact evaluation of the 2004-2005 California Urban Water council Pre-Rinse Spray Valve Installation Program.

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Table 3-75: Low Flow Pre-Rinse Sprayer Calculations Assumptions

Description <u>Term</u>	<u>Unit</u> Type	Value <u>s</u>	Source
${\it F}_{bng}$, Baseline flow rate of sprayer for nongrocery applications	Gallons per minuteFix edGPM-	Retrofit: 2.25-gpm	1, 7
${\it F}_{png},$ Post measure flow rate of sprayer for nongrocery applications	Gallons per minute VariableG	EDC Data Gathering	EDC Data Gathering
	PM	Default: 1.12	1
${\it U}_{bng},$ Baseline water usage duration for nongrocery applications	$\frac{min}{day}$ Fixed	$32.4 \frac{min}{day}$	2
${\it U}_{png},$ Post measure water usage duration for non-grocery applications	$\frac{min}{day}$ Fixed	$43.8 \frac{min}{day}$	2
${\it F}_{bg},$ Baseline flow rate of sprayer for grocery applications	GPMGallo ns per minuteFix ed	Retrofit: 2.15 gpm	1, 7
${\cal F}_{pg},$ Post measure flow rate of sprayer for grocery applications	GPMGallo ns per minuteVari	EDC Data Gathering	EDC Data Gathering
	able	Default: 1.12	1
U_{bg} , Baseline water usage duration for grocery applications	min day Fixed	4.8 min day 4.8	2
U_{pg} , Post measure water usage duration for grocery applications	$\frac{min}{day}$ Fixed	6 min day 6	2
T_{hng} , Temperature of hot water coming from the spray nozzle for non-grocery application	Degrees Fahrenheit Fixed°F	107°F	3
T_c , Incoming cold water temperature for grocery and non-grocery application	°FDegrees Fahrenheit Fixed	55°F	6
$T_{hg},$ Temperature of hot water coming from the spray nozzle for grocery application	°FDegrees Fahrenheit Fixed	97.6°F	3
EF _{electric} , Energy factor of existing electric water heater system	None Variable	EDC Data Gathering	EDC Data Gathering
		0.904	4
ETDF, Energy to demand factor	NoneFixe	0.000178	5

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DEFAULT SAVINGS

The default savings for the installation of a low flow pre-rinse sprayer compared to a standard efficiency sprayer is 151 kWh/year for pre-rinse sprayers installed in grocery stores and 1,222 kWh/year for pre-rinse sprayers installed in non-groceries—grocery building types such as restaurants. The deemed demand reductions for the installation of a low flow pre-rinse sprayer compared to a standard efficiency sprayer is 0.03 kW for pre-rinse sprayers installed in grocery stores and 0.22 kW for pre-rinse sprayers installed in non-groceries-grocery building types such as restaurants.

MEASURE LIFE

The effective life for this measure is 5 years 450.

EVALUATION PROTOCOL

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve Installation Program (Phase 2), SBW Consulting, 2007, Table 3-4, p. 23_ http://www.allianceforwaterefficiency.org/WorkArea/DownloadAsset.aspx?id=976
- Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve Installation Program (Phase 2), SBW Consulting, 2007, Table 3-6, p. 24_ http://www.allianceforwaterefficiency.org/WorkArea/DownloadAsset.aspx?id=976
- Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve Installation Program (Phase 2), SBW Consulting, 2007, Table 3-5, p. 23_ http://www.allianceforwaterefficiency.org/WorkArea/DownloadAsset.aspx?id=976_
- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept. of Energy Docket Number: EE–2006–BT-STD–0129, p. 30 http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- The EnergyToDemandFactor is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 6. Mid-Atlantic TRM, footnote #24.

⁴⁵⁰ Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve-Installation Program (Phase 2), SBW Consulting, 2007, p. 30

http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM March2013Version.pdf

 The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 GPM at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006.

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3.4.4 Low Flow Pre-Rinse Sprayers for Time of Sale / Retail Programs

I	
Measure Name	Low Flow Pre-Rinse Sprayers for Time of Sale / Retail Programs
Target Sector	Commercial-Kitchens and Industrial Establishments
Measure Unit	Pre Rinse Sprayer
Unit Energy Savings	See <u>Table 3-77</u> Table 3-77
Unit Peak Demand Reduction	See <u>Table 3-77</u> Table 3-77
Measure Life	5 years ⁴⁵¹
Measure Vintage	Replace on Burnout

ELIGIBILITY

This protocol documents the energy savings and demand reductions attributed to efficient low flow pre-rinse sprayers in small quick service restaurants, medium-sized casual dining restaurants, and large institutional establishments with cafeterias. Low flow pre-rinse sprayers reduce hot water usage and save energy associated with water heating. Only premises with electric water heating may qualify for this incentive. In addition, the new pre-rinse spray nozzle must have a cleanability performance of 26 seconds per plate or less.

This protocol is applicable to Time of Sale/Retail programs only. The baseline for Time of Sale-/Retail programs is assumed to be 1.52 GPM⁴⁵².

ALGORITHMS

The energy savings and demand reduction are calculated through the protocols documented below.

$$\Delta kWh = \frac{\left(F_b - F_p\right) \times U \times 60 \times 365\frac{12}{2} \times 8.33 \times 1 \times (T_h - T_c)}{EF \times 3413 \frac{Btu}{kWh}}$$

The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage <u>during-between</u> noon and 8PM on summer weekdays to the total annual energy usage.

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⁴⁵¹ Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse Spray Valve Installation Program (Phase 2), SBW Consulting, 2007, p. 30

⁴⁵² The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 GPM at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. The federal baseline is adjusted using a baseline adjustment factor of 0.95 to arrive at 1.52 GPM i.e. 1.6 GPM X 0.95 = 1.52 GPM. This value is derived based on the performance rating results of 29 models listed on the Food Service Technology Center Website which showed that the highest rated flow was 1.51 GPM.

Web address: http://www.fishnick.com/equipment/sprayvalves/, Accessed September 21, 2012. Sprayer by T&S Brass Model JetSpray B-0108 was rated at 1.48 GPM, and tested at 1.51 GPM.

$$\Delta kW_{peak}$$
 = ETDF × Energy Savings

The ETDF is defined below:

$$ETDF = \frac{Average\ Usage_{Summer\ WD\ 2-6\ PM}}{Annual\ Energy\ Usage}$$

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA. The usage profiles are shown in Figure 3-8Figure 3-8. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Table 3-77Table 3-77 indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for all building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania.

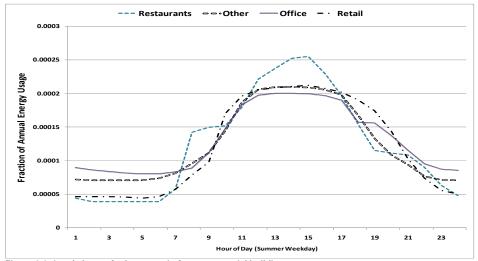


Figure 3-8: Load shapes for hot water in four commercial building types

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⁴⁵³ One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).

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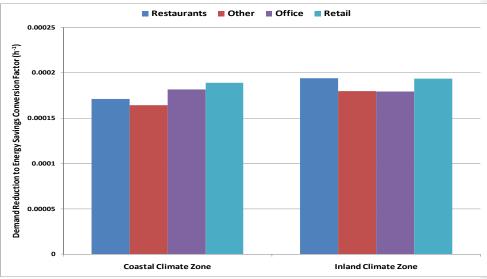


Figure 3-9: Energy to demand factors for four commercial building types.

DEFINITION OF TERMS

SECTION 3: Commercial and Industrial Measures

Table 3-76: Low Flow Pre-Rinse Sprayer Calculations Assumptions

Description Term	Type <u>Unit</u>	Value <u>s</u>	Source
${\it F_b}$, Baseline flow rate of sprayer	Gallons- per- MinuteFi xedGPM	Default: Time of Sale/Retail: 1.52 GPM	1, 2
	Gallons per	EDC Data Gathering	EDC Data Gathering
F_{p} , Post measure flow rate of sprayer	minuteFi xed Variable GPM	Default: Time of Sale/Retail: 1.06 GPM	3
U, Baseline and post measure water usage duration based on application	hours day - Fixed	Default: Small, quick- service restaurants: 0.5 Medium-sized casual dining restaurants: 1.5 Large institutional establishments with cafeteria: 3	4
T_h , Temperature of hot water coming from the spray nozzle	Degrees *Fahrenh eit_Fixed	120 <u>5.6</u> °F	<u>51</u>
T_c , Incoming cold water temperature	<u>°F</u> <u>Degrees</u> <u>Fahrenhe</u> <u>it</u> Fixed	55º₽	<u>5</u> 6
$\it EF_{electric}$, Energy factor of existing electric water heater system	Fixed Variable	EDC Data Gathering	EDC Data Gathering
neater system	None	Default: 0.904	6
ETDF, EnergyToDemandFactor	Fixed None	0.000178	<u>7</u> 8
Specific mass in pounds of one gallon of water	$\frac{lb}{gal}$	8.33 th / _{gat} 8.33	8
Specific heat of water	$\frac{Btu}{lb\cdot °F}$	1 ### 1.0	8
Days per year pre-rinse spray valve is used at the	<u>Days</u>	312 365 ⁴⁵⁴	1

⁴⁵⁴ Days per year pre-rinse spray valve is used at the site is assumed to be 365days/y.ryr.

SECTION 3: Commercial and Industrial Measures

n Term	Type <u>Unit</u>	Value

<u>Description</u> Term	TypeUnit	Value <u>s</u>	Source
site			
Minutes per hour pre-rinse spray valve	<u>Minutes</u>	<u>60</u>	Conversion Factor
3,413	Btul kWh	3.413	Conversion Factor

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DEFAULT SAVINGS

The <u>deemed_default</u> savings for the installation of a low flow pre-rinse sprayer compared to a standard efficiency sprayer for retail programs are listed in <u>Table 3-77Table 3-77</u> below.

Table 3-77: Low Flow Pre-Rinse Sprayer Default Savings

Application	Retail		
Application	kWh	kW	
Small quick service restaurants	<u>961</u> 814	0.1 <u>71</u> 4 5	
Medium-sized casual dining restaurants	2, <u>882</u> 441	0. <u>513</u> 4 3 4	
Large institutional establishments with cafeteria	<u>5,765</u> 4 <u>,882</u>	<u>1.026</u> 0	

MEASURE LIFE

The effective life for this measure is 5 years 455.

EVALUATION PROTOCOL

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

 Verification measurements taken at 195 installations showed average pre and post flowrates of 2.23 and 1.12 gallon per minute, respectively." from Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-

⁴⁵⁵ Impact and Process Evaluation Final Report for California Urban Water Conservation Council 2004-5 Pre-Rinse-Spray Valve-Installation Program (Phase 2), SBW Consulting, 2007, p. 30

Rinse Spray Valve Installation Program (Phase 2) (PG&E Program # 1198-04; SoCalGas Program 1200-04) ("CUWCC Report", Feb 2007). http://www.allianceforwaterefficiency.org/WorkArea/DownloadAsset.aspx?id=976

Rev Date: June 2015

- 2. The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 GPM at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. The federal baseline is adjusted using a baseline adjustment factor of 0.95. This value is derived based on the performance rating results of 29 models listed on the Food Service Technology Center Website showed that the highest rated flow was 1.51 GPM. Web address: http://www.fishnick.com/equipment/sprayvalves/, Accessed September 21, 2012. Sprayer by T&S Brass Model JetSpray B-0108 was rated at 1.48 GPM, and tested at 1.51 GPM.
- 3. 1.6 gallons per minute used to be the high efficiency flow, but more efficient spray valves are available ranging down to 0.64 gallons per minute per Federal Energy Management Program which references the Food Services Technology Center web site with the added note that even more efficient models may be available since publishing the data. The average of the nozzles listed on the FSTC website is 1.06. http://www.fishnick.com/equipment/sprayvalves/
- Hours primarily based on PG&E savings estimates, algorithms, sources (2005), Food Service Pre-Rinse Spray Valves with review of 2010 Ohio Technical Reference Manual and Act on Energy Business Program Technical Resource Manual Rev05.
- 3. Food Service Technology Center calculator assumptions to account for variations in mixing and water heater efficiencies. In the algorithm, $T_{\rm ft} = T_e + 70^{\circ}F$ temperature rise from $T_{\rm cr}$
- Mid-Atlantic TRM, footnote #24. http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM March2013Version.pdf
- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept. of Energy Docket Number: EE-2006-BT-STD-0129, p. 30 http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- The ETDF is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 8. The Engineering ToolBox. "Water-Thermal Properties." http://www.engineeringtoolbox.com/water-thermal-properties-d_162.html

3.4.6 FUEL SWITCHING: DOMESTIC HOT WATER ELECTRIC ELECTRIC RESISTANCE WATER HEATERS TO GAS / OIL / PROPANE

Measure Name	Fuel Switching: DHW-Electric Resistance Water Heaters to Gas/Oil/Propane
Target Sector	Small-Commercial and Industrial Establishments
Measure Unit	Water Heater
Unit Energy Savings	Varies <u>Variable</u>
Unit Peak Demand Reduction	<u>Variable</u> Varies
Measure Life	13 years ⁴⁵⁶ for natural gas or propane 8 years ⁴⁵⁷ for oil
Measure Description	Replace on Burnout

ELIGIBILITY

Natural gas, propane, and oil water heaters generally offer the customer lower costs compared to standard electric water heaters. Additionally, they typically see an overall energy savings when looking at the source energy of the electric unit versus the fossil fuel unit. Federal standard electric water heaters have energy factors of ≥>=0.904 and energy STAR gas and propane-fired water heaters have energy factors of 0.67 for a 50 gal unit and 0.495 for an oil-fired 50 gal unit. This protocol does not apply for units >55 gal.

This protocol documents the energy savings attributed to converting from a standard electric water heater with Energy Factor of ≥0.904 or greater to an ENERGY STAR natural gas/propane-fired water heater with Energy Factor of ≥0.67 or greater and ≥0.495 or greater for a standard oil-fired water heater. The target sector primarily consists of motels, small offices, and small retail establishments. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted.

ALGORITHMS

The energy savings calculation utilizes average performance data for available small commercial standard electric and natural gas water heaters and typical water usage. Because there is little electric energy associated with a natural gas or propane water heater, the energy savings are the full energy utilization of the electric water heater. The energy savings are obtained through the following formula:

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⁴⁵⁶ From ENERGY STAR: http://www.energystar.gov/index.cfm?c=gas_storage.pr_savings_benefits_

⁴⁵⁷ http://www.aceee.org/consumer/water-heating

$$= \frac{\left\{ \left(\frac{1}{EF_{elec,bl}} \right) \times \left(HW \times 1 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold}) \right) \right\}}{3413 \frac{Btu}{kWh}}$$

Although there is a significant electric savings, there is an associated increase in fossil fuel energy consumption. While this fossil fuel consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel usage is obtained through the following formula:

$$\frac{Fuel\ Consumption}{(MMBtu)} = \frac{\left\{ \left(\frac{1}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}} \right) \times \left(HW \times 1 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{co}) \right\} \right\}}{1,000,000 \frac{Btu}{MMBtu}}$$

Where EF_{fuel} changes depending on the fossil fuel used by the water heater.

For resistive water heaters, the demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage.

$$\begin{array}{ll} & = EnergytoDemandFactorTDF \times Energy\ Savings \\ & \times ResistiveDiscountFactorDF \end{array}$$

The Energy to Demand Factor is defined below:

$$\frac{EnergyToDemandFactor}{ETDF} = \frac{Average\ Usage_{summer\ WD\ 2-6PM}}{Annual\ Energy\ Usage}$$

Loads

The annual loads are taken from data from the DEER database 458. The DEER database has data for gas energy usage for the domestic hot water end use for various small commercial buildings. The loads are averaged over all 16 climate zones and all six vintage types in the DEER database. Finally, the loads are converted to average annual gallons of use using the algorithm below. The loads are summarized in Table 3-78 below, assuming a 40 gal natural gas water heater with a standard efficiency of 0.594.

$$HW \left(Gallons \right) \\ = \frac{Load \times EF_{ng,base} \times 1000 \frac{Btu}{kBtu} \times Typical \, SF}{1 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold}) \times 1000 \, SF}$$

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⁴⁵⁸ DEER 2008. Commercial Results Review Non-Updated Measures.

Table 3-78: Typical Water Heating Loads

Building Type	Typical Square Footage	Average Annual Load in $\frac{kBtu}{1000ft^2}$	Average Annual Use, Gallons
Motel	30,000	2,963	97,870
Small Office	10,000	2,214	24,377
Small Retail	7,000	1,451	11,183

Energy to Demand Factor

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA⁴⁵⁹. The usage profiles are shown in Figure 3-10Figure 3-10. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Figure 3-11Figure 3-14, indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for all building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania⁴⁶⁰.



Figure 3-10: Load shapes for hot water in four commercial building types

⁴⁵⁹ ibid

⁴⁶⁰ One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).

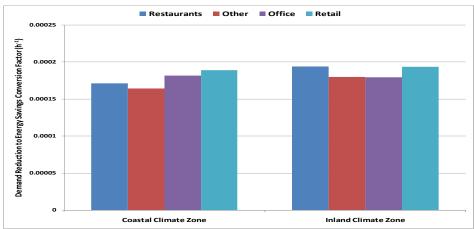


Figure 3-11: Energy to demand factors for four commercial building types

DEFINITION OF TERMS

The parameters in the above equation are listed in <u>Table 3-79</u>Table 3-79.

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Table 3-79: Commercial Water Heater Fuel Switch Calculation Assumptions

Component-Term	Type Unit	Values	Source
<i>EF</i> _{base} , Energy Factor of baseline water heater	None Variable	Default: 0.904	1
water ricater		Nameplate	EDC Data Gathering
$EF_{fuel}^{ 461}$, Energy Factor of installed fossil fuel water heater*	None Variable	>=0.67 for Natural Gas and Propane >=0.495 for Oil	5, EDC Data Gathering
EF _{tankless water heater} . Energy Factor of installed tankless water heater	None Variable	>=0.82	5
$DF_{fuel,adjust}$, Fossil fuel water heaters derating adjustment factor	None Fixed	Storage Water Heaters: 1.0 Tankless Water Heaters: 0.91	7
Load, Average annual load	kBtu_Fixed	Varies	DEER Database
T _{hot} , Temperature of hot water	Degrees Fahrenheit Fixed	1 <u>19</u> 23-°F	2
T_{cold} , Temperature of cold water supply	Degrees FahrenheitFixed	55 °F	3
HW, Average annual gallons of use	<u>Gallons</u> Variable	Default: See <u>Table 3-78</u> Table 3-78	Calculation
		EDC Data Gathering	EDC Data Gathering
EnergyToDemandFactorETDF, Energy To Demand Factor	NoneFixed	0.000178	4
⁴⁶² EF _{NG,base} , Energy Factor of baseline gas water heater	None Fixed	0.594	5
Resistive Discount Factor RDF. Resistive Discount Factor	None Fixed	1.0	6

Energy Factors based on Tank Size

Federal Standards for Factors Energy are equal to $0.97-0.00132 \times Rated\ Storage\ (gallons) 0.97-0.00132 \times Rated\ Storage\ in\ Gallons.$ following table shows the Energy Factors for various tank sizes. The

Domestic Hot Water

⁴⁶¹ Note that the federal minimum energy efficiency standards for electric and fossil fuel water heaters will increase starting April 16,

^{2015.} These new standards will be included in the 2015 TRM.

462 The protocol assumes a 40 gal natural gas water heater with a standard efficiency of 0.594 to calculate the loads summarized in Table 3-78Table 3-80Table 3-78: Typical Water Heating Loads.

Table 3-80: Minimum Baseline Energy Factors based on Tank Size

Tank Size (gallons)	Minimum Energy Factors $(E_{\it base})$	
40	0.9172	
50	0.9040	
65	0.8842	
80	0.8644	
120	0.8116	

DEFAULT SAVINGS

The default savings for the replacement of 50 gal electric water heater with a 50 gal fossil fuel units in various applications are listed below.

Table 3-81: Water Heating Fuel Switch Energy Savings Algorithms

Building Type	∆kWh	Fuel Consumption (MMBtu)
Motel	$\frac{16,184.54}{EF_{elec,bl}}$	$\frac{55.24}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}}$
Small Office	$\frac{4,031.17}{EF_{elec,bl}}$	$\frac{13.76}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}}$
Small Retail	$\frac{1,849.31}{EF_{elec,bl}}$	$\frac{6.31}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}}$

MEASURE LIFE

A natural gas or propane water heater's lifespan is 13 years463 and an oil-fired water heater has an 8 year lifetime. 464

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

⁴⁶³ From ENERGY STAR https://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=WGS 464.http://www.aceee.org/consumer/water-heating

SOURCES

- Federal Standards are 0.97 -0.00132 x Rated Storage in Gallons. For a 50-gallon tank this is approximately 0.90. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30. http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- 20124 SWE Residential Baseline Study Mid-Atlantic TRM. http://www.puc.pa.gov/Electric/pdf/Act129/SWE-2014 PA Statewide Act129 Residential Baseline Study.pdf http://www.puc.pa.gov/electric/pdf/Act129/PA Residential Baseline Report2012.pdf
- Mid-Atlantic TRM, footnote #24. http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM_March2013Version.pdf
- The EnergyToDemandFactor_ETDF is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 5. Commission Order⁴⁶⁵ requires fuel switching to ENERGY STAR measures, not standard efficiency measures. The Energy Factor has therefore been updated to reflect the Energy Star standard for natural gas or propane storage water heaters beginning September 1, Residential Water Heaters Kev Product Criteria. http://www.energystar.gov/index.cfm?c=water_heat.pr_crit_water_heaters Accessed June 2013. Federal Standards are 0.59 – 0.0019 x Rated Storage in Gallons for oil. For a 50-gallon tank this is 0.495 for oil. For a 40-gallon tank, this is 0.594 for natural gas. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 30.
- Engineering Estimate. No discount factor is needed because the baseline is already an electric resistance water heater system.
- 7. The disconnect between rated energy factor and in-situ energy consumption is markedly different for tankless units due to significantly higher contributions to overall household hot water usage from short draws. In tankless units the large burner and unit heat exchanger must fire and heat up for each draw. The additional energy losses incurred when the mass of the unit cools to the surrounding space in-between shorter draws was found to be 9% in a study prepared for Lawrence Berkeley National Laboratory by Davis Energy Group, 2006. "Field and Laboratory Testing of Tankless Gas Water Heater Performance" Due to the similarity (storage) between the other categories and the baseline, this derating factor is applied only to the tankless category. http://eetd.lbl.gov/sites/all/files/water heaters and hot water distribution systems.pdf

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⁴⁶⁵ See page 42 of the 2013 TRC Test Final Order

3.4.7 FUEL SWITCHING: HEAT PUMP WATER HEATERS TO GAS / OIL / PROPANE

Measure Name	Heat Pump Water Heaters
Target Sector	Commercial and Industrial Establishments
Measure Unit	Water Heater
Unit Energy Savings	Varies <u>Variable</u>
Unit Peak Demand Reduction	Varies-Variable
Measure Life	13 years ⁴⁶⁶ for natural gas/propane 8 years ⁴⁶⁷ for oil
Measure Vintage	Replace on Burnout

ELIGIBILITY

Natural gas, propane, and oil water heaters generally offer the customer lower costs compared to heat pump water heaters. Additionally, they typically see an overall energy savings when looking at the source energy of the electric unit versus the gas unit. Heat pump water heaters have energy factors of 2 or greater and an ENERGY STAR gas and propane water heater have an energy factor of 0.67 for a 50 gal unit and 0.495 for an oil-fired 50 gal unit. This protocol does not apply for units >55 gal.

This protocol documents the energy savings attributed to converting heat pump water heaters with Energy Factors of 2 or greater to fossil fuel water heaters. The target sector includes domestic hot water applications in small commercial settings such as small retail establishments, small offices, small clinics, and small lodging establishments such as small motels. The measure described here involves a direct retrofit of a heat pump water heater with a fossil fuel water heater. It does not cover systems where the heat pump is a pre-heater or is combined with other water heating sources. If a customer submits a rebate for a product that has applied for ENERGY STAR Certification but has not yet been certified, the savings will be counted for that product contingent upon its eventual certification as an ENERGY STAR measure. If at any point the product is rejected by ENERGY STAR, the product is then ineligible for the program and savings will not be counted. More complicated installations can be treated as custom projects.

ALGORITHMS

The energy savings calculation utilizes average performance data for available heat pump water heaters and typical hot water usages. The energy savings are obtained through the following formula:

$$\Delta kWh = \frac{\left\{ \left((\frac{1}{EF_{base}} \times \frac{1}{F_{adjust}}) \right) \times HW \times \frac{1}{2} 8.3 \frac{lb}{gal} \times 1.0 \frac{Btu}{lb \cdot {}^{\circ}F} \times (T_{hot} - T_{cold}) \right\}}{3413 \frac{Btu}{kWh}}$$

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⁴⁶⁶ From ENERGY STAR: http://www.energystar.gov/index.cfm?c=gas_storage.pr_savings_benefits

⁴⁶⁷ http://www.aceee.org/consumer/water-heating

Although there is a significant electric savings, there is an associated increase in fossil fuel energy consumption. While this fossil fuel consumption does not count against PA Act 129 energy savings, it is expected to be used in the program TRC test. The increased fossil fuel usage is obtained through the following formula:

Fuel Consumption (MMBtu)
$$= \frac{\left\{ \left(\frac{1}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}}\right) \times \left(HW \times 1.0 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times (T_{hot} - T_{cold})\right) \right\}}{1,000,000 \frac{Btu}{MMBtu}}$$

Where EF_{fuel} changes depending on the fossil fuel used by the water heater.

For replacement of heat pump water heaters with fossil fuel units, demand savings result primarily from a reduced connected load. The demand reduction is taken as the annual energy savings multiplied by the ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage.

$$\Delta kW_{peak}$$
 = ETDF × Energy Savings × RDF

The ETDF is defined below:

ETDF =
$$\frac{Average\ Usage}{Annual\ Energy\ Usage}$$

Loads

The annual loads are taken from data from the DEER database 468. The DEER database has data for gas energy usage for the domestic hot water end use for various small commercial buildings. The loads are averaged over all 16 climate zones and all six vintage types in the DEER database. Finally, the loads are converted to average annual gallons of use using the algorithm below. The loads are summarized in

Table 3-82

Table 3-82, assuming a 40 gal natural gas water heater with a standard efficiency of 0.594.

$$HW (Gallons) = \frac{Load \times EF_{ng,base} \times 1,000 \frac{Btu}{kBtu} \times Typical SF}{1 \frac{Btu}{lb \cdot {}^{\circ}F} \times 8.3 \frac{lb}{gal} \times \frac{\varkappa}{\ell} (T_{hot} - T_{cold}) \times 1,000 SF}$$

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⁴⁶⁸ DEER 2008. Commercial Building Results Review of Non-Updated Measures.

Table 3-82: Typical Water Heating Loads

Building Type	Typical Square Footage	Average Annual Load In $\frac{kBTU}{1000 ft^2}$	Average Annual Use, Gallons
Motel	30,000	2,963	97,870
Small Office	10,000	2,214	24,377
Small Retail	7,000	1,451	11,183

Energy to Demand Factor (ETDF)

The ratio of the average energy usage between 2 PM to 6 PM on summer weekdays to the total annual energy usage is taken from usage profile data collected for commercial water heaters in CA⁴⁶⁹. The usage profiles are shown in Figure 3-12Figure 3-12. To ensure that the load shape data derived from observations in CA can be applied to PA, we compared the annual energy usage to peak demand factors for two disparate climate zones in CA. The results, shown in Figure 3-13Figure 3-13, indicate that the ratio of peak demand to annual energy usage is not strongly influenced by climate. Also, though the actual usage profiles may be different, the average usage between 2 PM to 6 PM on summer weekdays is quite similar for all building types. The close level of agreement between disparate climate zones and building types suggest that the results will carry over to Pennsylvania.

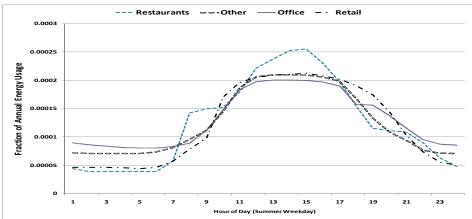
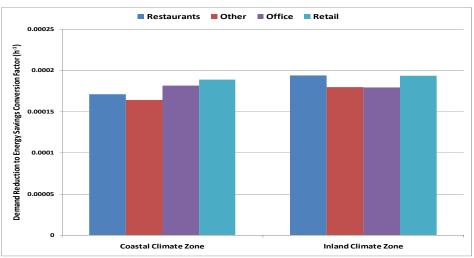


Figure 3-12: Load shapes for hot water in four commercial building types

⁴⁶⁹ Ibid

⁴⁷⁰ One reason for the close agreement is that the factor is a ratio of the energy usage to peak demand for the same location. Even though the energy usages may vary significantly in different climate zones, the hot water usage patterns may be driven by underlying practices that carry over well from state to state (e.g. dishwashing after lunch or dinner in restaurants).



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Figure 3-13: Energy to demand factors for four commercial building types

Resistive Heating Discount Factor

The resistive heating discount factor is an attempt to account for possible increased reliance on back-up resistive heating elements during peak usage conditions. Although a brief literature review failed to find data that may lead to a quantitative adjustment, two elements of the demand reduction calculation are worth considering.

- The hot water temperature in this calculation is somewhat conservative at 120 °F.
- The peak usage window is eight hours long.
- In conditioned space, heat pump capacity is somewhat higher in the peak summer window.
- In unconditioned space, heat pump capacity is dramatically higher in the peak summer window.

Under these operating conditions, one would expect a properly sized heat pump water heater with adequate storage capacity to require minimal reliance on resistive heating elements. A resistive heating discount factor of 0.9, corresponding to a 10% reduction in COP during peak times, is therefore taken as a conservative estimation for this adjustment.

Heat Pump COP Adjustment Factor

The Energy Factors are determined from a DOE testing procedure that is carried out at 56 °F wetbulb temperature. However, the average wetbulb temperature in PA is closer to 45 °F⁴⁷¹, while the average wetbulb temperature in conditioned typically ranges from 50 °F to 80 °F. The heat pump performance is temperature dependent. Figure 3-14 below shows relative

 $^{^{471}}$ Based on TMY2 weather files from DOE2.com for Erie, Harrisburg, Pittsburgh, Wilkes-Barre, And Williamsport, the average annual wetbulb temperature is 45 ± 1.3 °F. The wetbulb temperature in garages or attics, where the heat pumps are likely to be installed, are likely to be two or three degrees higher, but for simplicity, 45 °F is assumed to be the annual average wetbulb temperature.

coefficient of performance (COP) compared to the COP at rated conditions⁴⁷². According to the plotted profile, the following adjustments are recommended.

Table 3-83: COP Adjustment Factors

Heat Pump Placement	Typical WB Temperature °F	COP Adjustment Factor
Unconditioned Space	44	0.80
Conditioned Space	63	1.09
Kitchen	80	1.30

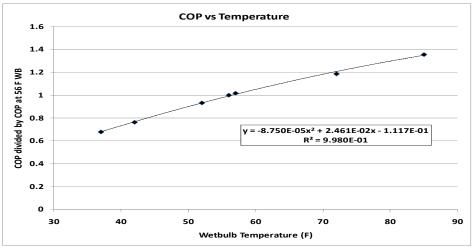


Figure 3-14: Dependence of COP on outdoor wetbulb temperature.

DEFINITION OF TERMS

The parameters in the above equation are listed in <u>Table 3-84 Table 3-84</u>.

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⁴⁷² The performance curve is adapted from Table 1 in http://wescorhvac.com/HPWH%20design%20details.htm#Single-stage%20HPWHs. The performance curve depends on other factors, such as hot water set point. Our adjustment factor of 0.84 is a first order approximation based on the information available in literature.

Table 3-84: Heat Pump Water Heater Fuel Switch Calculation Assumptions

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
EF _{base} , Energy Factor of baseline water heater	None Variable	Default: >= 2	1
water neater		Nameplate	EDC Data Gathering
EF_{fuel}^{473} , Energy Factor of installed fossil fuel water heater*	None Variable	>=0.67 for Natural Gas and Propane >=0.495 for Oil	7, EDC Data Gathering
<i>EF</i> _{tankless water heater} , Energy Factor of installed tankless water heater	None Variable	>=0.82	7
DF _{fuel,adjust} , Fossil Fuel Water Heaters Derating Adjustment factor	None Fixed	Storage Water Heaters: 1.0 Tankless Water Heaters: 0.91	8
Load, Average annual load	<u>kBtu</u> Fixed	Varies	5
T _{hot} , Temperature of hot water	Degrees FahrenheitFixed	1 <u>1923</u> °F	2
T_{cold} , Temperature of cold water supply	Degrees FahrenheitFixed	55 °F	3
ETDF, Energy_To_Demand_Factor	None Fixed	0.000178	4
F_{adjust} ,COP Adjustment factor	None Fixed	0.80 if outdoor 1.09 if indoor 1.30 if in kitchen	4
HW, Average annual gallons of use	<u>Gallons</u> Variable	Default: See	Calculation
		Table 3-82	
		Table 3-82	
		EDC Data Gathering	EDC Data Gathering
RDF, Resistive_Discount_Factor	None Fixed	0.90	6
$^{474}EF_{NG,base}$, Energy Factor of baseline gas water heater, see 3.4.2	None Fixed	0.594	7

 ⁴⁷³ Note that the federal minimum energy efficiency standards for electric and fossil fuel water heaters will increase starting April 16, 2015. These new standards will be included in the 2015 TRM.
 474 The protocol assumes a 40 gal natural gas water heater with a standard efficiency of 0.594 to calculate the loads summarized in

Domestic Hot Water

Energy Factors based on Tank Size

Standards for Federal Energy Factors equal to are $0.97 - 0.00132 \times Rated\ Storage\ (gallons) \frac{0.97}{-} - 0.00132 \times Rated\ Storage\ in\ Gallons.$ The

following table shows the Energy Factors for various tank sizes.

Table 3-85: Minimum Baseline Energy Factors based on Tank Size

Tank Size (gallons)	Minimum Energy Factors (Ebase)	
40	0.9172	
50	0.9040	
65	0.8842	
80	0.8644	
120	0.8116	

DEFAULTEMED SAVINGS

The default deemed savings for the replacement of heat pump electric water heaters with fossil fuel units in various applications are listed below.

<u>Table 3-71: Typical water heating loads</u> <u>Table 3-73: Typical water heating loads</u> <u>Table 3-89: Typical water heating loads</u>.

Domestic Hot Water

Table 3-86: Energy Savings Algorithms

Building Type	Location Installed	∆kWh	Fuel Consumption (MMBtu)
Motel	Outdoor	$\frac{20,230.67}{EF_{base}}$	$\frac{55.24}{EF_{fuel,inst}} imes \frac{1}{DF_{fuel,adjust}}$
Motel	Indoor	$\frac{14,848.20}{EF_{base}}$	EF _{fuel,inst} ^ DF _{fuel,adjust}
Motel	Kitchen	$\frac{12,449.64}{EF_{base}}$	
Small Office	Outdoor	$\frac{5,038.96}{EF_{base}}$	
Small Office	Indoor	$\frac{3,698.32}{EF_{base}}$	$\frac{13.76}{EF_{fuel,inst}} imes \frac{1}{DF_{fuel,adjust}}$
Small Office	Kitchen	$\frac{3,100.90}{EF_{base}}$	
Small Retail	Outdoor	$\frac{2,311.63}{EF_{base}}$	
Small Retail	Indoor	$\frac{1,696.61}{EF_{base}}$	$\frac{6.31}{EF_{fuel,inst}} \times \frac{1}{DF_{fuel,adjust}}$
Small Retail	Kitchen	$\frac{1,422.54}{EF_{base}}$	

MEASURE LIFE

A natural gas or propane water heater's lifespan is 13 years 475 and an oil-fired water heater has an 8 year lifetime. 476

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

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⁴⁷⁵_From ENERGY_STAR https://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=WGS 476_http://www.aceee.org/consumer/water-heating

- Heat pump water heater efficiencies have not been set in a Federal Standard. However, the Federal Standard for water heaters does refer to a baseline efficiency for heat pump water heaters as EF = 2.0. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE-2006-BT-STD-0129, p. 34. http://www.gpo.gov/fdsys/pkg/FR-2010-04-16/pdf/2010-7611.pdf
- 20124 SWE Residential Baseline Study, pg 49. http://www.puc.pa.gov/Electric/pdf/Act129/SWE-2014 PA Statewide Act129 Residential Baseline Study.pdfhttp://www.puc.pa.gov/electric/pdf/Act129/PA Residential Baseline Report2012.pdf
- Mid-Atlantic TRM Version 3.0, March 2013, footnote #314. http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM March2013Version.pdf
- 4. The ETDF is estimated using the California load shapes and reflects PJM's peak demand period. The load shapes can be accessed online: http://www.ethree.com/CPUC/PG&ENonResViewer.zip
- 5. DEER 2008. Commercial Results Review Non-Updated Measures. Database-
- 6. Engineering Estimate.
- 7. Commission Order⁴⁷⁷ requires fuel switching to ENERGY STAR measures, not standard efficiency measures. The Energy Factor has therefore been updated to reflect the ENERGY STAR standard for natural gas or propane storage water heaters beginning September 1, 2010. From Residential Water Heaters Key Product Criteria. http://www.energystar.gov/index.cfm?c=water-heat.pr-crit-water-heaters Accessed June 2013.Federal Standards are 0.59 0.0019 x Rated Storage in Gallons for oil. For a 50-gallon tank this is 0.495 for oil. For a 40-gallon tank this is 0.594 for natural gas. "Energy Conservation Program: Energy Conservation Standards for Residential Water Heaters, Direct Heating Equipment, and Pool Heaters" US Dept of Energy Docket Number: EE–2006–BT-STD–0129, p. 33.
- 8. The disconnect between rated energy factor and in-situ energy consumption is markedly different for tankless units due to significantly higher contributions to overall household hot water usage from short draws. In tankless units the large burner and unit heat exchanger must fire and heat up for each draw. The additional energy losses incurred when the mass of the unit cools to the surrounding space in-between shorter draws was found to be 9% in a study prepared for Lawrence Berkeley National Laboratory by Davis Energy Group, 2006. "Field and Laboratory Testing of Tankless Gas Water Heater Performance" Due to the similarity (storage) between the other categories and the baseline, this derating factor is applied only to the tankless category. http://eetd.lbl.gov/sites/all/files/water heaters and hot water distribution systems.pdf

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⁴⁷⁷ See page 42 of the 2013 TRC Test Final Order

3.5 REFRIGERATION

3.5.1 HIGH-EFFICIENCY REFRIGERATION/FREEZER CASES

Measure Name	High-Efficiency Refrigeration/Freezer Cases
Target Sector	Commercial and Industrial Establishments
Measure Unit	Refrigeration/Freezer Case
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	<u>Variable</u>
Measure Life	12 years ⁴⁷⁸
Measure Vintage	Replace on Burnout

ELIGIBILITY

This protocol estimates savings for installing high efficiency refrigeration and freezer cases that qualify under the ENERGY STAR rating compared to refrigeration and freezer cases allowed by federal standards. The measurement of energy and demand savings is based on algorithms with volume as the key variable.

ALGORITHMS

Products that can be ENERGY STAR 2.0 qQualified:

Examples of product types that may be eligible for qualification include: reach-in, roll-in, or pass-through units; merchandisers; under counter units; milk coolers; back bar coolers; bottle coolers; glass frosters; deep well units; beer-dispensing or direct draw units; and bunker freezers.

$$\Delta kWh = (kWh_{base} - kWh_{ee}) \times \frac{days}{year}$$

$$\Delta kW_{peak} = \frac{(kWh_{base} - kWh_{ee}) \times CF}{24}$$

Products that cannot be ENERGY STAR qualified

Drawer cabinets, prep tables, deli cases, and open air units are not eligible for ENERGY STAR under the Version 2.0 specification.

For these products, savings should be treated under a high-efficiency case fan, Electronically Commutated Motor (ECM) option.

478 DEER Effective Useful Life. October 10, 2008.

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DEFINITION OF TERMS

Table 3-87: Refrigeration Cases - References

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
kWh_{base} , The unit energy consumption of a standard unit	kwh day lated	See Table 3-88 and Table 3-89	1
kWh_{ee} , The unit energy consumption of the ENERGY STAR-qualified unit	kwh day lated	See Table 3-88 and Table 3-89	<u>2</u> 4
V, Internal Volume	ft³_ Variable	EDC data gathering	EDC data gathering
$rac{days}{year}$, days per year	days year Fixed	365	Conversion Factor 4
CF, Demand Coincidence Factor (See Section 1.5)	NoneFixe	0.772	<u>3</u> 2

Table 3-88: Refrigeration Case Efficiencies

Volume (ft^3)	Glass Door		Solid Door	
	kWh _{ee} day	kWh _{base} day	$\frac{kWh_{ee}}{day}$	kWh _{base} day
V < 15	0.118*V + 1.382	0.12*V + 3.34	0.089*V + 1.411	0.10*V + 2.04
15 ≤ V < 30	0.140*V + 1.050		0.037*V + 2.200	
30 ≤ V < 50	0.088*V + 2.625		0.056*V + 1.635	
50 ≤ V	0.110*V + 1.50		0.060*V + 1.416	

Table 3-89: Freezer Case Efficiencies

Volume (ft^3)	Glass Door		Solid Door	
	kWh _{ee} day	kWh _{base} day	$rac{kWh_{ee}}{day}$	kWh _{base} day
V < 15	0.607*V+0.893	0.75*V + 4.10	0.250*V + 1.25	0.4*V + 1.38
15 ≤ V < 30	0.733*V - 1.00		0.40*V - 1.00	
30 ≤ V < 50	0.250*V + 13.50		0.163*V + 6.125	
50 ≤ V	0.450*V + 3.50		0.158*V + 6.333	

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MEASURE LIFE

12 years

DEFAULT SAVINGS

If precise case volume is unknown, default savings given in tables below can be used.

Table 3-90: Refrigeration Case Savings

Volume (ft^3)	Annual Energy Savings (kWh)		Demand Impacts (kW)	
	Glass Door	Solid Door	Glass Door	Solid Door
V < 15	722	268	0.0824	0.0306
15 ≤ V < 30	683	424	0.0779	0.0484
30 ≤ V < 50	763	838	0.0871	0.0957
50 ≤ V	927	1,205	0.1058	0.1427

Table 3-91: Freezer Case Savings

Volume (ft3)	Annual Energy Savings (kWh)		Demand Impacts (kW)	
	Glass Door	Solid Door	Glass Door	Solid Door
V < 15	1,901	814	0.2170	0.0929
15 ≤ V < 30	1,992	869	0.2274	0.0992
30 ≤ V < 50	4,417	1,988	0.5042	0.2269
50 ≤ V	6,680	3,405	0.7625	0.3887

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Energy Conservation Program: Energy Conservation Standards for Commercial Refrigerators, Freezers, and Refrigerator-Freezers. Pg. 538 http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec431-66.pdf
- 21. ENERGY STAR calculator, March, 2010 update.

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- ENERGY STAR Program Requirements for Commercial Refrigerators and Freezers.
 Version 2.1
 http://www.energystar.gov/products/specs/system/files/Commercial Refrigerator and Freezer Program Requirements%20v2 1.pdf
- 22-3. Northeast Energy Efficiency Partnerships, Mid Atlantic TRM Version 3.0. March 2013. Calculated from Itron eShapes, which is 8760 hourly data by end use for Update New York.
 - http://www.neep.org/Assets/uploads/files/emv/emv-products/TRM_March2013Version.pdf
- 23.4. Food Service Technology Center (as stated in ENERGY STAR calculator).

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SECTION 3: Commercial and Industrial Measures

3.5.2 HIGH-EFFICIENCY EVAPORATOR FAN MOTORS FOR REACH-IN REFRIGERATED CASES

Rev Date: June 2015

Measure Name	High-Efficiency Evaporator Fan Motors for Reach-In Refrigerated Cases
Target Sector	Commercial and Industrial Establishments
Measure Unit	Evaporator Fan Motor
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁴⁷⁹
Measure Vintage	Early Replacement

ELIGIBILITY

This protocol covers energy and demand savings associated with the replacement retrofit- of existing shaded-pole evaporator fan motors in reach-in refrigerated display cases with either an Electronically Commutated (ECM) or Permanent Split Capacitor (PSC) motor. PSC motors must replace shaded pole (SP) motors, and ECM motors can replace either SP or PSC motors. A default savings option is offered if case temperature and/or motor size are not known. However, these parameters should be collected by EDCs for greatest accuracy.

There are two sources of energy and demand savings through this measure: There are

- tThe direct savings associated with replacement of an inefficient motor with a more efficient one; and there are
- 4.2. the The indirect savings of a reduced cooling load on the refrigeration unit due to less heat gain from the more efficient evaporator fan motor in the air-stream.

ALGORITHMS

Cooler

$$\begin{array}{ll} \Delta kW_{peak\;per\;unit} & = \frac{W_{base} - W_{ee}}{1,000} \times LF \times DC_{evapcool} \times \left(1 + \frac{1}{DG \times COP_{cooler}}\right) \\ \Delta kWh_{per\;unit} & = \Delta kW_{peak\;per\;unit} \times 8,760 \\ \Delta kW_{peak} & = N \times \Delta kW_{peak\;per\;unit} \\ \Delta kWh & = N \times \Delta kWh_{per\;unit} \end{array}$$

Freezer

$$\begin{array}{ll} \Delta kW_{peak\;per\;unit} & = \frac{W_{base} - W_{ee}}{1,000} \times LF \times DC_{evapfreeze} \times \left(1 + \frac{1}{DG \times COP_{freezer}}\right) \\ \Delta kWh_{per\;unit} & = \Delta kW_{peak\;per\;unit} \times 8,760 \\ \Delta kW_{peak} & = N \times \Delta kW_{peak\;per\;unit} \end{array}$$

⁴⁷⁹ DEER Effective Useful Life. October 10, 2008.

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 $= N \times \Delta kW h_{per\,unit}$ ΔkWh

Default (case service temperature not known)

 $= \frac{(1 - PctCooler) \times kW_{freezer}}{motor} + \frac{PctCooler \times kW_{cooler}}{motor}$ $= \Delta kW_{peak\ per\ unit} \times 8,760$ $= N \times \Delta kW_{peak\ per\ unit}$ $= \frac{N \times kWh_{def\ ault}}{motor}$ $\Delta kW_{peak\ per\ unit}$ $\Delta kWh_{per\;unit}$ ΔkW_{peak} ΔkWh

DEFINITION OF TERMS

Refrigeration

Table 3-92: Variables for High-Efficiency Evaporator Fan Motor⁴⁸⁰

<u>Term</u> Variable	<u>Unit</u> Type	Value <u>s</u>	Source
N, Number of motors replaced	None	EDC Data Gathering	EDC Data Gathering
W. Janut watters of wisting/hosping	10/	Nameplate Input Wattage	EDC Data Gathering
W_{base} , Input wattage of existing/baseline evaporator fan motor	W Variable	Default values from Table 3-93Table 3-93	Table 3-93 Table 3-93
W. land and a second se		Nameplate Input Wattage	EDC Data Gathering
W_{ee} , Input wattage of new energy efficient evaporator fan motor	<u>W</u> Variable	Default values from Table 3-93 Table 3-93	Table 3-93 Table 3-93
LF, Load factor of evaporator fan motor	None Fixed	0.9	1
DC _{evapcool} , Duty cycle of evaporator fan motor for cooler	None Fixed	100%	2
DC _{evapfreeze} , Duty cycle of evaporator fan motor for freezer	None Fixed	94.4%	2
DG, Degradation factor of compressor COP	None Fixed	0.98	3
COP _{cooler} , Coefficient of performance of compressor in the cooler	None Fixed	2.5	1
COP _{freezer} , Coefficient of performance of compressor in the freezer	None Fixed	1.3	1
PctCooler, Percentage of coolers in stores vs. total of freezers and coolers	None Fixed	68%	<u>3</u> 4
8,760, Hours per year	Hours Year	8,760	Conversion Factor

⁴⁸⁰ PSC of Wisconsin, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, p. 4-103 to 4-106. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf

Table 3-93: Variables for HE Evaporator Fan Motor

Motor Category	Weighting Percentage (population	Motor Output Watts	SP Efficiency	SP Input Watt s	PSC Efficiency ²	PSC Input Watt s	ECM Efficiency ¹	ECM Input Watt s
1-14 watts (Using 9 watt as industry average)	91%	9	18%	50	41%	22	66%	14
16-23 watts (Using 19.5 watt as industry average)	3%	19.5	21%	93	41%	48	66%	30
1/20 HP (~37 watts)	6%	37	26%	142	41%	90	66%	56

DEFAULT SAVINGS

Table 3-94: Shaded Pole to PSC Deemed Savings

Measure	W _{base} (Shaded Pole)	W _{ee} (PSC)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
Cooler: Shaded Pole to PSC: 1-14 Watt	50	22	0.9	100%	0.98	2.5	0.0355	311
Cooler: Shaded Pole to PSC: 16-23 Watt	93	48	0.9	100%	0.98	2.5	0.0574	503
Cooler: Shaded Pole to PSC: 1/20 HP (37 Watt)	142	90	0.9	100%	0.98	2.5	0.0660	578
Freezer: Shaded Pole to PSC: 1-14 Watt	50	22	0.9	94.4%	0.98	1.3	0.0425	373
Freezer: Shaded Pole to PSC: 16-23 Watt	93	48	0.9	94.4%	0.98	1.3	0.0687	602
Freezer: Shaded Pole to PSC: 1/20 HP (37 Watt)	142	90	0.9	94.4%	0.98	1.3	0.0790	692

SECTION 3: Commercial and Industrial Measures

Measure	W _{base} (PSC)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
Cooler: PSC to ECM: 1-14 Watt	22	14	0.9	100%	0.98	2.5	0.0105	92
Cooler: PSC to ECM: 16-23 Watt	48	30	0.9	100%	0.98	2.5	0.0228	200
Cooler: PSC to ECM: 1/20 HP (37 Watt)	90	56	0.9	100%	0.98	2.5	0.0433	380
Freezer: PSC to ECM: 1-14 Watt	22	14	0.9	94.4%	0.98	1.3	0.0126	110
Freezer: PSC to ECM: 16-23 Watt	48	30	0.9	94.4%	0.98	1.3	0.0273	239
Freezer: PSC to ECM: 1/20 HP (37 Watt)	90	56	0.9	94.4%	0.98	1.3	0.0518	454

Rev Date: June 2015

Table 3-96: Shaded Pole to ECM Deemed Savings

Measure	W _{base} (Shaded Pole)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
Cooler: Shaded Pole to ECM: 1-14 Watt	50	14	0.9	100%	0.98	2.5	0.0461	404
Cooler: Shaded Pole to ECM: 16-23 Watt	93	30	0.9	100%	0.98	2.5	0.0802	703
Cooler: Shaded Pole to ECM: 1/20 HP (37 Watt)	142	56	0.9	100%	0.98	2.5	0.1093	958
Freezer: Shaded Pole to ECM: 1-14 Watt	50	14	0.9	94.4%	0.98	1.3	0.0551	483
Freezer: Shaded Pole to ECM: 16-23 Watt	93	30	0.9	94.4%	0.98	1.3	0.0960	841
Freezer: Shaded Pole to ECM:	142	56	0.9	94.4%	0.98	1.3	0.1308	1146

SECTION 3: Commercial and Industrial Measures

Measure	W _{base} (Shaded Pole)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
1/20 HP (37 Watt)								

Table 3-97: Default High-Efficiency Evaporator Fan Motor Deemed Savings

Measure	Cooler Weighted Demand Impact (kW)	Cooler Weighted Energy Impact (kWh)	Freezer Weighted Demand Impact (kW)	Freezer Weighted Energy Impact (kWh)	Default Demand Impact (kW)	Default Energy Impact (kWh)
Shaded Pole to PSC	0.0380	333	0.0455	399	0.0404	354
PSC to ECM	0.0129	113	0.0154	135	0.0137	120
Shaded Pole to ECM	0.0509	446	0.0609	534	0.0541	474

MEASURE LIFE

15 years

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- "ActOnEnergy; Business Program-Program Year 2, June, 2009 through May, 2010. Technical Reference Manual, No. 2009-01." Published 12/15/2009.
- "Efficiency Maine; Commercial Technical Reference User Manual No. 2007-1." Published 3/5/07.
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery Display Case ECM, FY2010, V2. Accessed from RTF website http://www.nwcouncil.org/rtf/measures/Default.asp on July 30, 2010.
- 4. AO Smith New Product Notification. I-motor 9 & 16 Watt. Stock Numbers 9207F2 and 9208F2. Web address: http://www.aosmithmotors.com/uploadedFiles/Bulletin%206029B_6-09_web.pdf.-Accessed July 30, 2010.

SECTION 3: Commercial and Industrial Measures

3.5.3 HIGH-EFFICIENCY EVAPORATOR FAN MOTORS FOR WALK-IN REFRIGERATED CASES

Measure Name	High-Efficiency Evaporator Fan Motors for Walk-in Refrigerated Cases
Target Sector	Commercial and Industrial Establishments
Measure Unit	Fan Motor
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁴⁸¹
Measure Vintage	Early Replacement

ELIGIBILITY

This protocol covers energy and demand savings associated with the replacement of existing shaded-pole (SP) or permanent-split capacitor (PSC) evaporator fan motors in walk-in refrigerated display cases with an electronically commutated motor (ECM). A default savings option is offered if case temperature and/or motor size are not known. However, these parameters should be collected by EDCs for greatest accuracy.

There are two sources of energy and demand savings through this measure. There are:

- tThe direct savings associated with replacement of an inefficient motor with a more efficient one; and there are
- 4.2.*The indirect savings of a reduced cooling load on the refrigeration unit due to less heat gain from the more efficient evaporator fan motor in the air-stream.

ALGORITHMS

Cooler

$$\begin{array}{ll} \Delta kW_{peak\;per\;unit} & = \frac{(W_{base} - W_{ee})}{1,000} \times LF \times DC_{evapcool} \times \left[1 + \left(\frac{1}{DG \times COP_{cooler}}\right)\right] \\ \Delta kWh_{per\;unit} & = \Delta kW_{peak\;per\;unit} \times HR \\ \Delta kWh_{peak} & = N \times \Delta kW_{peak\;per\;unit} \\ \Delta kWh & = N \times \Delta kWh_{per\;unit} \end{array}$$

Freezer

$$\begin{array}{ll} \Delta kW_{peak\;per\;unit} & = \frac{(W_{base} - W_{ee})}{1,000} \times LF \times DC_{evapfreeze} \times \left[1 + \left(\frac{1}{DG \times COP_{freezer}}\right)\right] \\ \Delta kWh_{per\;unit} & = \Delta kW_{peak\;per\;unit} \times HR \\ \Delta kWh_{peak} & = N \times \Delta kW_{peak\;per\;unit} \\ \Delta kWh & = N \times \Delta kWh_{per\;unit} \end{array}$$

481 Appliance Magazine. "Evaporator Fan Motor Energy Monitoring." http://www.appliancemagazine.com/editorial.php?article=1570

SECTION 3: Commercial and Industrial Measures

Default (case service temperature not known)

 $= \frac{(1 - PctCooler) \times kW_{freezer}}{motor} + \frac{PctCooler \times kW_{cooler}}{motor}$ $= \Delta kW_{peak \ per \ unit} \times HR$ $= N \times \Delta kW_{peak \ per \ unit}$ $\Delta kW_{peak\;per\;unit}$ $\Delta kWh_{per\;unit}$ ΔkWh_{peak}

 ΔkWh^{r} $= N \times \Delta kW h_{per\,unit}$

DEFINITION OF TERMS

Table 3-98: Variables for High-Efficiency Evaporator Fan Motor

Term	Unit	Values	Source
N. Number of motors replaced	None	EDC Data Gathering	EDC Data Gathering
W_{base} , Input wattage of existing/baseline evaporator fan	Variable -W	Nameplate Input Wattage	EDC Data Gathering
motor	variable <u>vv</u>	Default Table 3-99	Table 3-99
W_{ee} , Input wattage of new energy efficient evaporator fan	Variable W	Nameplate Input Wattage	EDC Data Gathering
motor	Variable <u>vv</u>	Default Table 3-99	Table 3-99
LF, Load factor of evaporator fan motor	<u>None</u> Fixed	0.9	1
$\mathcal{DC}_{evapcool}$, Duty cycle of evaporator fan motor for cooler	<u>None</u> Fixed	100%	2
$\mathit{DC_{evapfreeze}}$, Duty cycle of evaporator fan motor for freezer	None Fixed	94.4%	2
DG, Degradation factor of compressor COP	<u>None</u> Fixed	0.98	3
${\it COP_{cooler}}$, Coefficient of performance of compressor in the cooler	None _{Fixed}	2.5	1
${\it COP_{freezer}}$, Coefficient of performance of compressor in the freezer	<u>None</u> Fixed	1.3	1
PctCooler, Percentage of walk-in coolers in stores vs. total of freezers and coolers	<u>None</u> Fixed	69%	3
Hr, Operating hours per year	<u>Hours</u> Fixed	8,273	2

Refrigeration

Table 3-99: Variables for HE Evaporator Fan Motor

Motor Category	Weighting Number (populatio n)2	Motor Output Watts	SP Efficiency 482,483	SP Input Watts	PSC Efficiency	PSC Input Watts	ECM Efficiency 1	ECM Input Watts
1/40 HP (16-23 watts) (Using 19.5 watt as industry average)	25%	19.5	21%	93	41%	48	66%	30
1/20 HP (~37 watts)	11.5%	37	26%	142	41%	90	66%	56
1/15 HP (~49 watts)	63.5%	49	26%	191	41%	120	66%	75

DEFAULT SAVINGS

⁴⁸² Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery
Display Case ECM, FY2010, V2. Accessed from RTF website: http://www.nwcouncil.org/rtf/measures/Default.asp on July 30, 2010
483 Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Deemed
MeasuresV26_walkinevapfan. Provided by Adam Hadley (adam@hadleyenergy.com). Should be made available on RTF website
http://www.nwcouncil.org/rtf/measures/Default.asp

⁴⁸⁴ AO Smith New Product Notification. I-motor 9 & 16 Watt. Stock Numbers 9207F2 and 9208F2. Web address: http://www.aosmithmotors.com/uploadedFiles/Bulletin%206029B_6-09_web.pdf. Accessed July 30, 2010.

Table 3-100: PSC to ECM Deemed Savings

Measure	W _{base} (PSC)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
Cooler: PSC to ECM: 1/40 HP (16-23 Watt)	48	30	0.9	100%	0.98	2.5	0.0228	189
Cooler: PSC to ECM: 1/20 HP (37 Watt)	90	56	0.9	100%	0.98	2.5	0.0431	356
Cooler: PSC to ECM: 1/15 HP (49 Watt)	120	75	0.9	100%	0.98	2.5	0.0570	472
Freezer: PSC to ECM: 1/40 HP (16-23 Watt)	48	30	0.9	94.4%	0.98	1.3	0.0273	226
Freezer: PSC to ECM: 1/20 HP (37 Watt)	90	56	0.9	94.4%	0.98	1.3	0.0516	427
Freezer: PSC to ECM: 1/15 HP (49 Watt)	120	75	0.9	94.4%	0.98	1.3	0.0682	565

Table 3-101: Shaded Pole to ECM Deemed Savings

Measure	W _{base} (Shaded Pole)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
Cooler: Shaded Pole to ECM: 1/40 HP (16-23 Watt)	93	30	0.9	100%	0.98	2.5	0.0798	661
Cooler: Shaded Pole to ECM: 1/20 HP (37 Watt)	142	56	0.9	100%	0.98	2.5	0.1090	902
Cooler: Shaded Pole to ECM: 1/15 HP (49 Watt)	191	75	0.9	100%	0.98	2.5	0.1470	1,216
Freezer: Shaded Pole to ECM: 1/40 HP (16-23 Watt)	93	30	0.9	94.4%	0.98	1.3	0.0955	790
Freezer: Shaded Pole to ECM: 1/20 HP (37 Watt)	142	56	0.9	94.4%	0.98	1.3	0.1304	1,079
Freezer: Shaded Pole to ECM:	191	75	0.9	94.4%	0.98	1.3	0.1759	1,455

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Measure	W _{base} (Shaded Pole)	W _{ee} (ECM)	LF	DC _{Evap}	DG	COP per case Temp	Demand Impact (kW)	Energy Impact (kWh)
1/15 HP (49 Watt)								

Table 3-102: Default High-Efficiency Evaporator Fan Motor Deemed Savings

Measure	Cooler Weighted Demand Impact (kW)	Cooler Weighted Energy Impact (kWh)	Freezer Weighted Demand Impact (kW)	Freezer Weighted Energy Impact (kWh)	Default Demand Impact (kW)	Default Energy Impact (kWh)
PSC to ECM	0.0469	388	0.0561	464	0.0499	413
Shaded Pole to ECM	0.1258	1,041	0.1506	1,246	0.1335	1,105

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- PSC of Wisconsin, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, p. 4-103 to 4-106. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf
- Efficiency Vermont, Technical Reference Manual 2009-54, 12/08. Hours of operation accounts for defrosting periods where motor is not operating. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- PECI presentation to Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Energy Smart March 2009 SP to ECM 090223.ppt. Accessed from RTF website http://rtf.nwcouncil.org/meetings/2009/03/http://rtf.nwcouncil.org/energy/rtf/meetings/2009/03/default.htmon September 7, 2010.
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery Display Case ECM, FY2010, V2. Accessed from RTF website: http://www.nwcouncil.org/rtf/measures/Default.asp on July 30, 2010
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council,
 Deemed Measures List. Deemed MeasuresV26 _walkinevapfan. Provided by Adam Hadley (adam@hadleyenergy.com). Should be made available on RTF website http://www.nwcouncil.org/rtf/measures/Default.asp
- AO Smith New Product Notification. I motor 9 & 16 Watt. Stock Numbers 9207F2 and 9208F2.
 Web address:

 $\label{lem:http://www.aosmithmotors.com/uploadedFiles/Bulletin\%206029B_6-09_web.pdf.-Accessed July 30, 2010.$

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3.5.4 REFRIGERATION—CONTROLS: EVAPORATOR FAN CONTROLLERS

Measure Name	Controls: Evaporator Fan Controllers
Target Sector	Commercial and Industrial Establishments
Measure Unit	Evaporator Fan Controller
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	Variable
Measure Life	<u>10 years⁴⁸⁵</u>
Measure Vintage	Retrofit

This measure is for the installation of evaporator fan controls⁴⁸⁶ in medium-temperature walk-in coolers with no pre-existing controls. Evaporator fans run constantly to provide cooling when the compressor is running, and to provide air circulation when the compressor is not running. The equations specified in the Algorithms section are for fans that are turned off and/or cycled. A fan controller saves energy by reducing fan usage, by reducing the refrigeration load resulting from the heat given off by the fan and by reducing compressor energy resulting from the electronic temperature control. This protocol documents the energy savings attributed to evaporator fan controls.

ELIGIBILITY

This protocol documents the energy savings attributed to installation of evaporator fan controls in medium-temperature walk-in coolers and low temperature walk-in freezers.

ALGORITHMS 487

$$\begin{array}{lll} \Delta kWh & = \Delta kWh_{fan} + \Delta kWh_{heat} + \Delta kWh_{control} \\ \Delta kWh_{fan} & = kW_{fan} \times 8,760 \times \%0ff \\ \Delta kWh_{heat} & = \Delta kWh_{fan} \times 0.28 \times Eff_{rs} \\ \Delta kW_{control} & = \left[kW_{cp} \times Hours_{cp} + kW_{fan} \times 8,760 \times (1 - \%0ff)\right] \times 5\% \\ \Delta kW & = \frac{\Delta kWh}{8,760} \end{array}$$

Determine kW_{fan} , and kW_{cp} variables using any of the following methods:

1. Calculate using the nameplate horsepower and load factor.

$$\Delta kW_{fan} \ or \ kW_{cp} \qquad \qquad = \frac{HP \times LF \times 0.746}{\eta_{motor}}$$

Refrigeration

⁴⁸⁵ Energy & Resource Solutions (2005). Measure Life Study. Prepared for Thethe Massachusetts Joint Utilities; Table 1-1.

486 An evaporator fan controller is a device or system that lowers airflow across an evaporator in medium-temperature walk-in coolers when there is no refrigerant flow through the evaporator (i.e., when the compressor is in an off-cycle).

487 The assumptions and algorithms used in this section are specific to NRM products and are taken from the Massachusetts Statewide Technical Reference Manual for Estimating Savings from Energy Efficiency Measures, 2012. Version 1.0 http://www.ma-eeac.org/docs/M4%20TRM -2011%20PLAN%20VERSION.PDF

2. Calculate using the nameplate amperage and voltage and a power factor.

$$\Delta kW_{fan} \ or \ kW_{cp} = V \times A \times PF_{motor} \times LF$$

3. Measure the input kW fan using a power meter reading true RMS power.

DEFINITION OF TERMS

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Table 3-103: Evaporator Fan Controller Calculations Assumptions

Component <u>Term</u>	TypeUni t	Value <u>s</u>	Source
ΔkWh _{fan} , Energy savings due to evaporator being shut off	<u>kWh</u>	Calculated	Calculated
ΔkWh _{heat} , Heat energy savings due to reduced heat from evaporator fans	<u>kWh</u>	Calculated	Calculated
$\frac{\Delta kWh_{control}, \text{Control energy savings due}}{\text{to electronic controls on compressor and}} \\ \frac{\text{evaporator}}{\text{evaporator}}$	<u>kWh</u>	Calculated	<u>Calculated</u>
kW _{fan} . Power demand of evaporator fan calculated from any of the methods described above	<u>kW</u>	Calculated	Calculated
kW _{cp} . Power demand of compressor_motor and condenser fan calculated from any of the methods described above	kW	Calculated	Calculated
0.28, Conversion from kW to tons	kW tons	0.28	Conversion Factor
5%, Reduced run-time of compressor and evaporator due to electronic controls	<u>None</u>	<u>5%</u>	7
0.746, Conversion factor from kW to horsepower	$\frac{kW}{hp}$	0.746 kw/hp	Conversion Factor
PF, Power Factor of the motor	NoneFix ed	Fan motor: 0.75 Compressor motor: 0.9	1, 5, 6
%Off, Percent of annual hours that the evaporator is turned off	None Fixed	46%_	2
Eff_{rs} , Efficiency of typical refrigeration system	$\frac{kW}{ton}$ Fixed	1.6 \(\frac{kW}{ton} \)	3
Hours _{cp} , Equivalent annual full load hours of	Hours Va	EDC Data Gathering	EDC Data Gathering
compressor operation	nabic	4,072	1, 4
HP, Rated horsepower of the motor	Horsepo werHPV ariable	EDC Data Gathering	EDC Data Gathering
η_{motor} , Efficiency of the motor	None Vari	EDC Data Gathering	EDC Data Gathering
LF, Load factor of motor	NoneFix ed	0.9	Section 3.5.2

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Voltage, Voltage of the motor	VoltsVari	EDC Data Gathering	EDC Data Gathering
Amperage, Rated amperage of the motor	Amperes Variable	EDC Data Gathering	EDC Data Gathering

DEFAULT SAVINGS

State of Pennsylvania

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- 1. Conservative value based on 15 years of NRM field observations and experience
- Select Energy (2004). Analysis of Cooler Control Energy Conservation Measures. Prepared for NSTAR.
- Estimated average refrigeration efficiency for small business customers, Massachusetts Technical Reference Manual for Estimating Savings from Energy Efficiency Measures. October 2012. Pg. 191
- 2012 Program Year Rhode Island Technical Reference Manual for Estimating Savings from Energy Efficiency Measures
- 5. ESource Customer Direct to Touchstone Energy for Evaporator Fan Controllers, 2005
- LBNL 57651 Energy Savings in Refrigerated Walk-in Boxes, 1998 http://gaia.lbl.gov/btech/papers/57651.pdf
- Conservative estimate supported by less conservative values given by several utilitysponsored 3rd party studies including: Select Energy (2004). Analysis of Cooler Control Energy Conservation Measures. Prepared for NSTAR.

3.5.5 REFRIGERATION CONTROLS: FLOATING HEAD PRESSURE CONTROLS

Measure Name	Controls: Floating Head Pressure Control
Target Sector	Commercial and Industrial EstablishmentsRefrigeration
Measure Unit	Floating Head Pressure Control
Unit Energy Savings	Deemed by location, kWh
Unit Peak Demand Reduction	0 kW
Measure Life	15 years ⁴⁸⁸
Measure Vintage	Retrofit

Installers conventionally design a refrigeration system to condense at a set pressure-temperature setpoint, typically 90° °F. By installing a floating head pressure control (FHPCs) condenser system, the refrigeration system can change condensing temperatures in response to different outdoor temperatures. This means that the minimum condensing head pressure from a fixed setting (180 psig for R-22) is lowered to a saturated pressure equivalent at 70°-F or less. Either a balanced-port or electronic expansion valve that is sized to meet the load requirement at a 70°-F condensing temperature must be installed. Alternatively, a device may be installed to supplement the refrigeration feed to each evaporator attached to a condenser that is reducing head pressure.

ELIGIBILITY

This protocol documents the energy savings attributed to FHPCs applied to a single-compressor refrigeration system in commercial applications. The baseline case is a refrigeration system without FHPC whereas the efficient case is a refrigeration system with FHPC. FHPCs must have a minimum Saturated Condensing Temperature (SCT) programmed for the floating head pressure control of $\leq 70^{\circ}$ F. The use of FHPC would require balanced-port expansion valves, allowing satisfactory refrigerant flow over a range of head pressures. The Gc ompressor must be 1 HP or larger.

ALGORITHMS

The savings are primarily dependent on the following factors:

- Load factor of compressor motor horsepower (HP)
- · Climate zone
- Refrigeration system temperature application

The savings algorithm is as follows:

$$\Delta kWh$$
 = $HP_{compressor} \times \frac{kWh}{HP}$

If the refrigeration system is rated in tonnage:

⁴⁸⁸ Grocery Floating Head Pressure Controls for Single Compressor Systems, FY2010, V1. Accessed from RTF website http://www.nwcouncil.org/rtf/measures/Default.asp on September 06, 2011.

⁴⁸⁹ Also called as flood back control

$$\Delta kWh \\ \Delta kW_{peak} = \frac{4.715}{COP} \times Tons \times \frac{kWh}{HP}$$

DEFINITION OF TERMS

Table 3-104: Floating Head Pressure Controls – Values and References

<u>TermComponent</u>	<u>Unit</u> Type	Values	Source
<i>HP_{compressor}</i> , Rated horsepower (HP) per compressor	Horsepower Variable <u>HP</u> -	EDC Data Gathering	EDC Data Gathering
$\frac{kWh}{HP}$, Annual savings per HP	^{kWh} _{HP} Fixed	<u>Table 3-105</u> Table 3-105	1
		Based on design conditions	EDC Data Gathering
		Default:	
	None Variable	Condensing Unit;	
COP, Coefficient of		Refrigerator (Medium Temp: 28 °F – 40 °F): 2.55 COP	
Performance		Freezer (Low Temp: -20 °F – 0 °F): 1.32 COP	2
		Remote Condenser;	
		Refrigerator (Medium Temp: 28 °F – 40 °F): 2.49 COP	
		Freezer (Low Temp: -20 °F – 0 °F): 1.45 COP	
Tons, Refrigeration tonnage of the system	Tons Variable	EDC Data Gathering	EDC Data Gathering
4.715, Conversion factor to convert from tons to HP	NoneFixed	Engineering Estimate	3

Table 3-105: Annual Savings kWh/HP by Location

	Condensing U	nit (kWh/HP)	Remote Condenser (kWh/HP)			
Climate Zone	Refrigerator (Medium Temp)	(Medium (Low (Temp (M		Refrigerator (Medium Temp)	Freezer (Low Temp)	Default ⁴⁹¹ (Temp Unknown)	
Allentown	630	767	674	380	639	463	
Erie	681	802	720	438	657	508	
Harrisburg	585	737	634	330	623	424	
Philadelphia	546	710	598	286	609	390	
Pittsburgh	617	759	662	366	634	452	
Scranton	686	806	724	443	659	512	
Williamsport	663	790	703	417	651	492	

Rev Date: June 2015

Table 3-106: Default Condenser Type Annual Savings kWh/HP by Location

	Unknown Condenser Type Default492 (kWh/HP)					
Climate Zone	Refrigerator (Medium Temp)	Freezer (Low Temp)	Temp Unknown			
Allentown	505	703	568			
Erie	559	730	614			
Harrisburg	458	680	529			
Philadelphia	416	660	494			
Pittsburgh	491	697	557			
Scranton	564	732	618			
Williamsport	540	720	598			

DEFAULT SAVINGS

There are no default savings for this measure.

⁴⁹⁰ Default based on: 2010 ASHRAE Refrigeration Handbook, page 15.1 "Medium- and low-temperature display refrigerator line-ups account for roughly 68% and 32%, respectively, of a typical supermarket's total display refrigerators."
⁴⁹¹ Ibid.

⁴⁹² No data available to predict if condensing units or remote condensers will be more prevalent, assumed 50/50 split, based on discussion with Portland Energy Conservation, Inc. (PECI) GrocerySmart staff.

MEASURE LIFE

The measure life is 15 years 493 as per the Regional Technical Forum (RTF) of the Northwest Power & Conservation Council.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installed refrigeration capacity coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- The deemed savings values were derived from the Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery Floating Head Pressure Controls for Single Compressor Systems, FY2010, V1. Using RTF Deemed saving estimates for the NW climate zone, data was extrapolated to Pennsylvania climate zones by using cooling degree days comparison based on the locale.
- The given COP values are averaged based on the data from: Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Grocery Floating Head Pressure Controls for Single Compressor Systems, FY2010, V1.
- Conversion factor for compressor horsepower per ton: http://www.engineeringtoolbox.com/refrigeration-formulas-d 1695.html

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⁴⁹³ Grocery Floating Head Pressure Controls for Single Compressor Systems, FY2010, V1. Accessed from RTF websitehttp://www.nwcouncil.org/rtf/measures/Default.asp.on.September 06, 2011.

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3.5.6 ANTI-SWEAT HEATER CONTROLS: ANTI-SWEAT HEATER **CONTROLS**

Measure Name	Anti-Sweat Heater Controls
Target Sector	Commercial and Industrial Establishments
Measure Unit	Case door
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	12 years ⁴⁹⁴
Measure Vintage	Retrofit

ELIGIBILITY

Anti-sweat heater (ASH) controls sense the humidity in the store outside of reach-in, glass door refrigerated cases and turn off anti-sweat heaters during periods of low humidity. Without controls, anti-sweat heaters run continuously whether they are necessary or not. Savings are realized from the reduction in energy used by not having the heaters running at all times. In addition, secondary savings result from reduced cooling load on the refrigeration unit when the heaters are off. The ASH control is applicable to glass doors with heaters, and the savings given below are based on adding controls to doors with uncontrolled heaters. The savings calculated from these algorithms is on a per door basis for two temperatures: Refrigerator/Coolers and Freezers. A default value to be used when the case service temperature is unknown is also calculated. Furthermore, impacts are calculated for both a per-door and a per-linear-feet of case unit basis, because both are used for Pennsylvania energy efficiency programs.

ALGORITHMS

Refrigerator/Cooler

$$\Delta kWh_{per\,unit} = \frac{kW_{coolerbase}}{DoorFt} \times \left(8,760 \times CHA_{off}\right) \times \left(1 + \frac{R_h}{COP_{cool}}\right)$$

$$\Delta kW_{peak\,per\,unit} = \frac{kW_{coolerbase}}{DoorFt} \times CHP_{off} \times \left(1 + \frac{R_h}{COP_{cool}}\right) \times DF$$

$$\Delta kWh = N \times \Delta kWh_{per\,unit}$$

$$\Delta kW_{peak} = N \times \Delta kW_{peak\,per\,unit}$$

Freezer

$$\Delta kWh_{per unit} = \frac{kW_{freezerbase}}{DoorFt} \times \left(8,760 \times FHA_{off}\right) \times \left(1 + \frac{R_h}{COP_{freeze}}\right)$$

$$\Delta kW_{peak \ per unit} = \frac{kW_{freezerbase}}{DoorFt} \times FHP_{off} \times \left(\frac{1 + R_h}{COP_{freeze}}\right) \times DF$$

$$\Delta kWh = N \times \Delta kWh_{per \ unit}$$

$$\Delta kW_{peak} = N \times \Delta kW_{peak \ per \ unit}$$

⁴⁹⁴ DEER Effective Useful Life. October 10, 2008.

Refrigeration

Default (case service temperature is unknown)

This algorithm should only be used when the refrigerated case type or service temperature is unknown or this information is not tracked as part of the EDC data collection.

$$\begin{array}{ll} \Delta kWh_{per\,unit} & = (1-PctCooler) \times \frac{kWh_{freezer}}{DoorFt} + \frac{PctCooler \times kWh_{cooler}}{DoorFt} \\ \Delta kW_{peak\,per\,unit} & = (1-PctCooler) \times \frac{kW_{freezer}}{DoorFt} + \frac{PctCooler \times kW_{cooler}}{DoorFt} \\ \Delta kWh & = N \times \Delta kWh_{per\,unit} \\ \Delta kW_{peak} & = N \times \Delta kW_{peak\,per\,unit} \end{array}$$

DEFINITION OF TERMS

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Table 3-107 Anti-Sweat Heater Controls – Values and References

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
N, Number of doors or case length in linear feet having ASH controls installed	<u>None</u> Variable	# of doors or case length in linear feet	EDC Data Gathering
$R_{\rm h}$, Residual heat fraction; estimated percentage of the heat produced by the heaters that remains in the freezer or cooler case and must be removed by the refrigeration unit	None Fixed	0.65	1
Unit, Refrigeration unit	None Fixed or ft	Door = 1 Linear Feet_= 2.5	2
8,760 <u>. Hours in a year</u>	hours year	8,760	Conversion Factor
Refrigerator/Cooler			
$kW_{cooler\ base}$, Per door power consumption of cooler case ASHs without controls	<u>kW</u> Fixed	0.109	1
$\it CHP_{off}$, Percent of time cooler case ASH with controls will be off during the peak period	None Fixed	20%	1
CHA_{off} , Percent of time cooler case ASH with controls will be off annually	None Fixed	85%	1
DF_{cool} , Demand diversity factor of cooler, accounting for the fact that not all anti-sweat heaters in all buildings in the population are operating at the same time.	None Fixed	1	3
COP _{cool} , Coefficient of performance of cooler	None Fixed	2.5	1
Freezer			
$kW_{freezerbase}$. Per door power consumption of freezer case ASHs without controls	<u>kW</u> Fixed	0.191	1
$\it FHP_{off}$, Percent of time freezer case ASH with controls will be off during the peak period	None Fixed	10%	1
FHA_{off} , Percent of time freezer case ASH with controls will be off annually	None Fixed	75%	1
DF_{freeze} , Demand diversity factor of freezer, accounting for the fact that not all anti-sweat heaters in all buildings in the population are operating at the same time.	<u>None</u> Fixed	1	3
COP _{freeze} , Coefficient of performance of freezer	NoneFixed	1.3	1

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
PctCooler, Typical percent of cases that are medium-temperature refrigerator/cooler cases	NoneFixed	68%	4

MEASURE LIFE

12 Years (DEER 2008, Regional Technical Forum)

DEFAULT SAVINGS

Table 3-108: Recommended Fully Deemed Impact Estimates

Description	Per Door Impact	Per Linear Ft of Case Impact
Refrigerator/Cooler		
Energy Impact	1,023 kWh per door	409 kWh per linear ft.
Peak Demand Impact	0.0275 kW per door	0.0110 kW per linear ft.
Freezer		
Energy Impact	1,882 kWh per door	753 kWh per linear ft.
Peak Demand Impact	0.0287 kW per door	0.0115 kW per linear ft.
Default (case service temperature unknown)		
Energy Impact	1,298 kWh per door	519 kWh per linear ft.
Peak Demand Impact	0.0279 kW per door	0.0112 kW per linear ft.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

 State of Wisconsin, Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs Deemed Savings Manual, March 22, 2010. https://focusonenergy.com/sites/default/files/bpdeemedsavingsmanuav10_evaluationreport.pdf

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- a. Three door heating configurations are presented in this reference: Standard, low-heat, and no-heat. The standard configuration was chosen on the assumption that low-heat and no-heat door cases will be screened from participation.
- 2. Review of various manufacturers' web sites yields 2.5' average door length. Sites include:
 - a. http://www.bushrefrigeration.com/bakery glass door coolers.php
 - b. http://www.brrr.cc/home.php?cat=427
 - c. http://refrigeration-equipment.com/gdm s c series swing door reac.html
- New York Standard Approach for Estimating Energy Savings from Energy Efficiency Measures in Commercial and Industrial Programs, Sept 1, 2009. http://www3.dps.ny.gov/W/PSCWeb.nsf/0/06f2fee55575bd8a852576e4006f9af7/\$FILE/TechManualNYRevised10-15-10.pdf
- 2010 ASHRAE Refrigeration Handbook, page 15.1 "Medium- and low-temperature display refrigerator line-ups account for roughly 68 and 32%, respectively, of a typical supermarket's total display refrigerators."

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3.5.7 CONTROLS: EVAPORATOR COIL DEFROST CONTROL

Measure Name	Controls: Evaporator Coil Defrost Control
Target Sector	Commercial and Industrial Establishments
Measure Unit	Evaporator Coil Defrost Control
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	10 years ⁴⁹⁵
Measure Vintage	Retrofit

Rev Date: June 2015

This protocol applies to electric defrost control on small commercial walk-in cooler and freezer systems. A freezer refrigeration system with electric defrost is set to run the defrost cycle periodically throughout the day. A defrost control uses temperature and pressure sensors to monitor system processes and statistical modeling to learn the operation and requirements of the system. When the system calls for a defrost cycle, the controller determines if it is necessary and skips the cycle if it is not.

ELIGIBILITY

This measure is targeted to non-residential customers whose equipment uses electric defrost controls on small commercial walk-in freezer systems.

Acceptable baseline conditions are existing small commercial walk-in coolers or freezers without defrost controls.

Efficient conditions are small commercial walk-in coolers or freezers with defrost controls installed.

ALGORITHMS

 $\begin{array}{ll} \Delta kW_{peak} & = FANS \times kW_{DE} \times SVG \times BF \\ \Delta kWh & = \Delta kW_{peak} \times HOURS \end{array}$

DEFINITION OF TERMS

⁴⁹⁵ Estimate from Heatcraft based on expected component expected life. The only moving part is a relay which has a cycle life that is well over 15 years based on the frequency of the relay operation.

Table 3-109: Evaporator Coil Defrost Control - Values and References

Term	Unit	Values	Source
FANS , Number of evaporator fans	Fans	EDC Data Gathering	EDC Data Gathering
kW_{DE} , kW of defrost element	kW	EDC Data Gathering Default: 0.9	EDC Data Gathering
SVG, Savings percentage for reduced defrost cycles	None	30%	2
BF , Savings factor for reduced cooling load from eliminating heat generated by the defrost element	None	See <u>Table 3-110</u> Table 3-110	3
HOURS , Average annual full load defrost hours	hours year	EDC Data Gathering Default: 487	EDC Data Gathering

Table 3-110: Savings Factor for Reduced Cooling Load

Equipment Type	Savings Factor for Reduced Cooling Load (BF)		
Cooler	1.3		
Freezer	1.67		

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Efficiency Vermont Technical Reference Manual, 2013. The total Defrost Element kW is
 proportional to the number of evaporator fans blowing over the coil. The typical wattage
 of the defrost element is 900W per fan. See Bohn <Bohn Evap 306-0D.pdf> and Larkin
 <LC-03A.pdf>specifications.
- Smart defrost kits claim 30-40% savings (with 43.6% savings by third party testing by Intertek Testing Service). MasterBilt Demand defrost claims 21% savings for northeast. Smart Defrost Kits are more common so the assumption of 30% is a conservative estimate.

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- 3. ASHRAE Handbook 2006 Refrigeration, Section 46.15 Figure 24.
- 4. Efficiency Vermont Technical Reference Manual, 2013. The refrigeration system is assumed to be in operation every day of the year, while savings from the evaporator coil defrost control will only occur during set defrost cycles. This is assumed to be (4) 20-minute cycles per day, for a total of 487 hours.

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3.5.8 VARIABLE SPEED REFRIGERATION COMPRESSOR

Measure Name	VSD Refrigeration Compressor
Target Sector	Commercial and IndustrialRefrigeration Establishments
Measure Unit	VSD Refrigeration Compressor
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	1 <u>5</u> 0 years ⁴⁹⁶
Measure Vintage	Retrofit

Variable speed drive (VSD) compressors are used to control and reduce the speed of the compressor during times when the refrigeration system does not require the motor to run at full capacity. VSD control is an economical and efficient retrofit option for existing compressor installations. The performance of variable speed compressors can more closely match the variable refrigeration load requirements thus minimizing energy consumption.

ELIGIBILITY

This measure, VSD control for refrigeration systems and its eligibility targets applies to retrofit construction in the commercial and industrial building sectors; it is most applicable to grocery stores or food processing applications with refrigeration systems. This protocol is for a VSD control system replacing a slide valve control system.

ALGORITHMS

The savings algorithms are is as follows:

If the refrigeration system is rated in tonnage:

 $\begin{array}{ll} \Delta kWh & = Tons \times ES_{value} \\ \Delta kW_{peak} & = Tons \times DS_{value} \end{array}$

If the refrigeration system is rated in horsepower:

 $\begin{array}{ll} \Delta kWh & = 0.445 \times HP_{compressor} \times ES_{value} \\ \Delta kW_{peak} & = 0.445 \times HP_{compressor} \times DS_{value} \end{array}$

DEFINITION OF TERMS

⁴⁹⁶ DEER Effective Useful Life. October 10, 2008.

Table 3-111: VSD Compressor – Values and References

	Component-Term	Type-Unit	Values	Sources
I Tons. Retrigeration tonnage of the system T		Variable <u>To</u> ns	EDC Data Gathering	EDC Data Gathering
	$\mathit{HP}_{compressor}$, Rated horsepower per compressor	horsepower Variable	EDC Data Gathering	EDC Data Gathering
	ES_{value} , Energy savings value in kWh per compressor HP	kwh ton Fixed	1,696	1
	DS_{value} , Demand savings value in kW per compressor HP	Fixed kW ton	0.22	1
	0.445, Conversion factor to convert from tons to HP	Fixed None	Engineering Estimate 0.445	2,3

DEFAULT SAVINGS

There are no default savings for this measure.

MEASURE LIFE

According to the 2005 DEER, a VFD compressor has a measure life of 10 years.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installed refrigeration capacity affected by the retrofit coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- Deemed savings values of 1696 kWh ton and 0.22 kW were obtained from the 2005 DEER (Database for Energy Efficiency Resources). This measure considered the associated savings by vintage and by climate zone for compressors. The deemed value was an average across all climate zones and all vintages (excluding new construction). http://www.deeresources.com/index.php/deer2005
- PSC of Wisconsin, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, p. 4-103 to 4-106. Where refrigerator (medium temp: 28 °F – 40 °F) COP equals 2.5 and freezer COP (low temp: -20 °F – 0 °F) equals 1.3. The weighted average COP equals 2.1, based on 2010 ASHRAE Refrigeration Handbook, page 15.1 "Medium-

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⁴⁹⁷ The deemed savings was averaged across all climate zones since the variance between all cases was less than 5%.

and low-temperature display refrigerator line-ups account for roughly 68% and 32%, respectively, of a typical supermarket's total display refrigerators."

3. Conversion factor for compressor horsepower per ton is HP/ton = 4.715/COP, using weighted average COP of 2.1. From http://www.engineeringtoolbox.com/refrigeration-formulas-d 1695.html

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3.5.9 STRIP CURTAINS FOR WALK-IN FREEZERS AND COOLERS

Measure Name	Strip Curtains for Walk-In Coolers and Freezers
Target Sector	Commercial and Industrial EstablishmentsRefrigeration
Measure Unit	Walk-in unit door
Unit Energy Savings	Fixed
Unit Peak Demand Reduction	Fixed
Measure Life	4 years ⁴⁹⁸
Measure Vintage	Retrofit

Strip curtains are used to reduce the refrigeration load associated with the infiltration of non-refrigerated air into the refrigerated spaces of walk-in coolers or freezers.

The primary cause of air infiltration into walk-in coolers and freezers is the air density difference between two adjacent spaces of different temperatures. The total refrigeration load due to infiltration through the main door into the unit depends on the temperature differential between the refrigerated and non-refrigerated airs, the door area and height, and the duration and frequency of door openings. The avoided infiltration depends on the efficacy of the newly installed strip curtains as infiltration barriers. ⁴⁹⁹, and on the efficacy of the supplanted infiltration barriers, if applicable. The calculation of the refrigeration load due to air infiltration and the energy required to meet that load is rather straightforward, but relies on critical assumptions regarding the aforementioned operating parameters. All the assumptions in this protocol are based on values that were determined by direct measurement and monitoring of over 100 walk-in units in the 2006-2008 evaluation for the CA Public Utility Commission. ⁵⁰⁰

ELIGIBILITY

This protocol documents the energy savings attributed to strip curtains applied on walk-in cooler and freezer doors in commercial applications. The most likely areas of application are large and small grocery stores, supermarkets, restaurants, and refrigerated warehouses. The baseline case is a walk-in cooler or freezer that previously had either no strip curtain installed or an old, ineffective strip curtain installed. The efficient equipment is a strip curtain added to a walk-in cooler or freezer. Strip curtains must be at least 0.06 inches thick. Low temperature strip curtains must be used on low temperature applications.

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⁴⁹⁸ DEER Effective Useful Life. October 10, 2008.

⁴⁹⁹ We define *curtain efficacy* as the fraction of the potential airflow that is blocked by an infiltration barrier. For example, a brick wall would have an efficacy of 1.0, while the lack of any infiltration barrier corresponds to an efficacy of 0.

⁵⁰⁰ See source 1 for Table 3-54 Table 3-56 Table 3-14.

⁵⁰¹ http://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

ALGORITHMS

$$\begin{array}{ll} \varDelta kWh & = \frac{\Delta kWh}{sqft} \times A \\ & = \frac{\Delta kW}{sqft} \times A \end{array}$$

The annual energy savings due to infiltration barriers is quantified by multiplying savings per square foot by area using assumptions for independent variables described in the protocol introduction. The source algorithm from which the savings per square foot values are determined is based on Tamm's equation 502 (an application of Bernoulli's equation) and the ASHRAE handbook $_{\scriptscriptstyle 2}^{503}$. To the extent that evaluation findings are able to provide more reliable site specific inputs assumptions, they may be used in place of the default per square foot savings using the following equation.

$$\Delta kWh = \frac{365 \times t_{open} \times (\eta_{new} - \eta_{old}) \times 20 \times CD \times A \times \left\{ \left[\frac{(T_i - T_r)}{T_i} \right] \times g \times H \right\}^{0.5} \times [\rho_i \times h_i - \rho_r \times h_r]}{3,413 \frac{Btu}{kWh} \times COP_{adj} \times A}$$

The peak demand reduction is quantified by multiplying savings per square foot by area. The source algorithm is the annual energy savings divided by 8.760. This assumption is based on general observation that refrigeration is constant for food storage, even outside of normal operating conditions. This is the most conservative approach in lieu of a more sophisticated model.

$$\frac{\Delta k W_{peak}}{8,760} = \frac{\Delta k W h}{8,760}$$

The ratio of the average energy usage during Peak hours to the total annual energy usage is taken from the load shape data collected by ADM for a recent evaluation for the CA Public Utility Commission⁵⁰⁴ in the study of strip curtains in supermarkets, convenience stores, and restaurants.

DEFINITION OF TERMS

 $^{^{502}\ \}textit{Kalterveluste durch kuhlraumoffnungen}.\ Tamm\ W., Kaltetechnik-Klimatisierung\ 1966; 18; 142-144$

⁵⁰³ American Society of Heating, Refrigeration, and Air-Conditioning Engineers (ASHRAE). 2010. ASHRAE Handbook, Refrigeration: 13.4, 13.6

⁵⁰⁴ http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02-18-2010.pdf

Table 3-112: Strip Curtain Calculation Assumptions

Term	<u>Unit</u>	<u>Values</u>	Source
$\frac{\Delta kWh}{ft^2}$, Average annual kWh savings per square foot of infiltration barrier	$\frac{\Delta kWh}{ft^2}$	Calculated	<u>Calculated</u>
$\frac{\Delta kW}{ft^2}$, Average kW savings per square foot of infiltration barrier	$\frac{\Delta kW}{ft^2}$	Calculated	<u>Calculated</u>
20, Product of 60 minutes per hour and an integration factor of 1/3	None	<u>20</u>	4
g, Gravitational constant	$\frac{ft}{s^2}$	32.174	Constant
3.413. Conversion factor: number of Btus in one kWh	Btu kWh	3.413	Conversion factor

DEFAULT SAVINGS

The default savings values are listed in <u>Table 3-113Table 3-113</u>. Default parameters used in the source equations are listed in <u>Table 3-114Table 3-114</u>, Table 3-115, Table 3-116, and Table 3-117. The source equations and the values for the input parameters are adapted from the 2006-2008 California Public Utility Commission's evaluation of strip curtains. The original work included 8.760-hourly bin calculations. The values used herein represent annual average values. For example, the differences in the temperature between the refrigerated and infiltrating airs are averaged over all times that the door to the walk-in unit is open. Recommendations made by the evaluation team have been adopted to correct for errors observed in the *ex ante* savings calculation.

As for the verified savings for all strip curtains installed in the refrigerated warehouses, the study found several issues that resulted in low realization rates despite the relatively high savings if the curtains are found to be installed in an actual warehouse. The main factor was the misclassification of buildings with different end-use descriptions as refrigerated warehouses. For example, the EM&C contractor found that sometimes the facilities where the curtains were installed were not warehouses at all, and sometimes the strip curtain installations were not verified. The Commission, therefore, believes that the savings for strip curtains installed at an actual refrigerated warehouse should be much higher. To accurately estimate savings for this measure, the Commission encourages the EDCs to use billing analysis for refrigerated warehouses for projects selected in the evaluation sample.

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⁵⁰⁵ http://www.cpuc.ca.gov/PUC/energy/Energy+Efficiency/EM+and+V/2006-2008+Energy+Efficiency+Evaluation+Report.htm. The scale factors have been determined with tracer gas measurements on over 100 walk-in refrigeration units during the California Public Utility Commission's evaluation of the 2006-2008 CA investor owned utility energy efficiency programs. The door-open and close times, and temperatures of the infiltrating and refrigerated airs are taken from short-term monitoring of over 100 walk-in units. The temperature and humidity of the infiltrating air and the COP of the units have been modified to reflect the PA climate.

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Table 3-113: Default Energy Savings and Demand Reductions for Strip Curtains

Туре	Pre-existing Curtains	Energy Savings $\frac{\Delta kWh}{ft^2}$	Demand Savings $\frac{\Delta kWh}{ft^2}$
Supermarket - Cooler	Yes	37	0.0042
Supermarket - Cooler	No	108	0.0123
Supermarket - Cooler	Unknown	108	0.0123
Supermarket - Freezer	Yes	119	0.0136
Supermarket - Freezer	No	349	0.0398
Supermarket - Freezer	Unknown	349	0.0398
Convenience Store - Cooler	Yes	5	0.0006
Convenience Store - Cooler	No	20	0.0023
Convenience Store - Cooler	Unknown	11	0.0013
Convenience Store - Freezer	Yes	8	0.0009
Convenience Store - Freezer	No	27	0.0031
Convenience Store - Freezer	Unknown	17	0.0020
Restaurant - Cooler	Yes	8	0.0009
Restaurant - Cooler	No	30	0.0034
Restaurant - Cooler	Unknown	18	0.0020
Restaurant - Freezer	Yes	34	0.0039
Restaurant - Freezer	No	119	0.0136
Restaurant - Freezer	Unknown	81	0.0092
Refrigerated Warehouse	Yes	254	0.0290
Refrigerated Warehouse	No	729	0.0832
Refrigerated Warehouse	Unknown	287	0.0327

Table 3-114: Strip Curtain Calculation Assumptions for Supermarkets

Teri	<u>m Component</u>	<u>Unit</u> Type	Value <u>s</u>		Source
			Cooler	Freezer	
effic thwa	, Efficacy of the new strip curtain – an acy of 1 corresponds to the strip curtain arting all infiltration, while an efficacy of zero esponds to the absence of strip curtains.	None Fixed	0.88	0.88	1
η_{old}	, Efficacy of the old strip curtain	None Fixed			1
with	Pre-existing curtain		0.58	0.58	
with	no Pre-existing curtain		0.00	0.00	
unk	nown		0.00	0.00	
dete diffe pred	Discharge Coefficient: empirically ermined scale factors that account for erences between infiltration as rates dicted by application Bernoulli's law and lal observed infiltration rates	<u>None</u> Fixed	0.366	0.415	1
t_{ope}	_n , Minutes walk-in door is open per day	minutes Fixed	132	102	1
A , [Doorway area_	ft ² Fixed	35	35	1
Н, С	Doorway height	ft_Fixed-	7	7	1
Ran	Ory-bulb temperature of infiltrating air, ikine ahrenheit + 459.67	°F_ Fixed	71	67	1 and 2
	Dry-bulb temperature of refrigerated air, kine	°F_ Fixed	37	5	1
= Fa	ahrenheit + 459.67				
$ ho_i$, RH	Density of the infiltration air, based on 55%	$\frac{lb}{ft^3}$ Fixed	0.074	0.074	3
h _i , E RH.	Enthalpy of the infiltrating air, based on 55%	$\frac{Btu}{lb}$ Fixed	26.935	24.678	3
$ ho_r$, I	Density of the refrigerated air, based on 80%	$\frac{lb}{ft^3}$ Fixed	0.079	0.085	3
	Enthalpy of the refrigerated air, based on 6 RH.	Btu lb Fixed	12.933	2.081	3
coet	ρ_{adj} , Time-dependent (weather dependent) fficient of performance of the refrigeration tem. Based on nominal COP of 1.5 for zers and 2.5 for coolers	None Fixed	3.07	1.95	1 and 2

Table 3-115: Strip Curtain Calculation Assumptions for Convenience Stores

Term Component	Unit Type	Value <u>s</u>		Source
		Cooler	Freezer	
η_{new} , Efficacy of the new strip curtain – an efficacy of 1 corresponds to the strip curtain thwarting all infiltration, while an efficacy of zero corresponds to the absence of strip curtains.	<u>None</u> Fixed	0.79	0.83	1
η_{old} , Efficacy of the old strip curtain	None Fixed			1
with Pre-existing curtain		0.58	0.58	
with no Pre-existing curtain		0.00	0.00	
unknown		0.34	0.30	
\mathcal{C}_d , Discharge Coefficient: empirically determined scale factors that account for differences between infiltration as rates predicted by application Bernoulli's law and actual observed infiltration rates	None Fixed	0.348	0.421	1
t_{open} , Minutes walk-in door is open per day	minutes Fixed	38	9	1
$\it A$, Doorway area	ft ² Fixed	21	21	1
H, Doorway height	ftFixed-	7	7	1
T_i , Dry-bulb temperature of infiltrating air, Rankine = Fahrenheit + 459.67	°F_ Fixed	68	64	1 and 2
T_r , Dry-bulb temperature of refrigerated air, Rankine	°F_ Fixed	39	5	1
= Fahrenheit + 459.67				
ρ_i , Density of the infiltration air, based on 55% RH	Fixed $\frac{lb}{ft^3}$	0.074	0.075	3
h_i , Enthalpy of the infiltrating air, based on 55% RH.	Btu Fixed	25.227	23.087	3
$\rho_r,$ Density of the refrigerated air, based on 80% RH.	$\frac{lb}{ft^3}$ Fixed	0.079	0.085	3
h_r , Enthalpy of the refrigerated air, based on 80% RH.	$\frac{Btu}{lb}$ Fixed	13.750 <u></u>	2.081	3
COP _{adj} , Time-dependent (weather dependent) coefficient of performance of the refrigeration system. Based on nominal COP of 1.5 for freezers and 2.5 for coolers.	None Fixed	3.07	1.95	1 and 2

Table 3-116: Strip Curtain Calculation Assumptions for Restaurants

	<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>		Source
			Cooler	Freezer	
	η_{new} , Efficacy of the new strip curtain – an efficacy of 1 corresponds to the strip curtain thwarting all infiltration, while an efficacy of zero corresponds to the absence of strip curtains.	None Fixed	0.80	0.81	1
	η_{old} , Efficacy of the old strip curtain with Pre-existing curtain with no Pre-existing curtain unknown	None Fixed	0.58 0.00 0.33	0.58 0.00 0.26	1
	\mathcal{C}_d , Discharge Coefficient: empirically determined scale factors that account for differences between infiltration as rates predicted by application Bernoulli's law and actual observed infiltration rates	None Fixed	0.383	0.442	1
	$t_{\it open}$, Minutes walk-in door is open per day	minutes day Fixed	45	38	1
	A , Doorway area	ft^2 Fixed	21	21	1
	H, Doorway height	ft_Fixed	7	7	1
	T_i , Dry-bulb temperature of infiltrating air, Rankine = Fahrenheit + 459.67	°F_ Fixed	70	67	1 and 2
	T_r , Dry-bulb temperature of refrigerated air, Rankine = Fahrenheit + 459.67	°F_ Fixed	39	8	1
1	ρ_i , Density of the infiltration air, based on 55% RH	$\frac{lb}{ft^3}$ Fixed	0.074	0.074	3
	h_i , Enthalpy of the infiltrating air, based on 55% RH.	Btu Fixed	26.356	24.678	3
	$ ho_r$, Density of the refrigerated air, based on 80% RH.	$\frac{lb}{ft^3}$ Fixed	0.079	0.085	3
	h_r , Enthalpy of the refrigerated air, based on 80% RH.	Btu Fixed	13.750	2.948	3
	COP_{adj} , Time-dependent (weather dependent) coefficient of performance of the refrigeration system. Based on nominal COP of 1.5 for freezers and 2.5 for coolers.	NoneFixed	3.07	1.95	1 and 2

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Table 3-117: Strip Curtain Calculation Assumptions for Refrigerated Warehouses

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
η_{new} , Efficacy of the new strip curtain – an efficacy of 1 corresponds to the strip curtain thwarting all infiltration, while an efficacy of zero corresponds to the absence of strip curtains.	None Fixed	0.89	1
η_{old} , Efficacy of the old strip curtain	None Fixed		1
with Pre-existing curtain		0.58	
with no Pre-existing curtain		0.00	
unknown		0.54	
\mathcal{C}_d , Discharge Coefficient: empirically determined scale factors that account for differences between infiltration as rates predicted by application Bernoulli's law and actual observed infiltration rates	None Fixed	0.425	1
t_{open} , Minutes walk-in door is open per day	minutes Fixed	494	1
A , Doorway area	ft ² Fixed	80	1
H, Doorway height	ft_Fixed	10	1
T_i , Dry-bulb temperature of infiltrating air, Rankine = Fahrenheit + 459.67	°F_ Fixed	59	1 and 2
T_r , Dry-bulb temperature of refrigerated air, Rankine = Fahrenheit + 459.67	°F_ Fixed	28	1
$ ho_i$, Density of the infiltration air, based on 55% RH	$\frac{lb}{ft^3}$ Fixed	0.076	3
h_i , Enthalpy of the infiltrating air, based on 55% RH.	Btu Fixed	20.609	3
$ ho_r$, Density of the refrigerated air, based on 80% RH.	$\frac{lb}{ft^3}$ Fixed	0.081	3
$\it h_r$, Enthalpy of the refrigerated air, based on 80% RH.	Btu Fixed	9.462	3
${\it COP}_{adj}$, Time-dependent (weather dependent) coefficient of performance of the refrigeration system. Based on nominal COP of 1.5 for freezers and 2.5 for coolers.	None Fixed	1.91	1 and 2

MEASURE LIFE

The measure life is estimated to be 4 years. Sources:

24. Commercial Facilities Contract Group 2006-2008 Direct Impact Evaluation, http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02-18-2010.pdf

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 The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007

EVALUATION PROTOCOLS

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings according to store type. The strip curtains are not expected to be installed directly. As such, the program tracking / evaluation effort must capture the following key information:

- Fraction of strip curtains installed in each of the categories (e.g. freezer / cooler and store type)
- · Fraction of customers that had pre-existing strip curtains

The rebate forms should track the above information. During the M&V process, interviews with site contacts should track this fraction, and savings should be adjusted accordingly.

SOURCES

- http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02_18-2010.pdf.
 The scale factors have been determined with tracer gas measurements on over 100 walk-in refrigeration units during the California Public Utility Commission's evaluation of the 2006-2008 CA investor owned utility energy efficiency programs. The door-open and close times, and temperatures of the infiltrating and refrigerated airs are taken from short-term monitoring of over 100 walk-in units. http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02-18-2010.pdf.
- 2. For refrigerated warehouses, we used a bin calculation method to weight the outdoor temperature by the infiltration that occurs at that outdoor temperature. This tends to shift the average outdoor temperature during times of infiltration higher (e.g. from 54 °F year-round average to 64 °F). We also performed the same exercise to find out effective outdoor temperatures to use for adjustment of nominal refrigeration system COPs.
- 3. Density and enthalpy of infiltrating and refrigerated air are based on psychometric equations based on the dry bulb temperature and relative humidity. Relative humidity is estimated to be 55% for infiltrating air and 80% for refrigerated air. Dry bulb temperatures were determined through the evaluation cited in Source 1.
- 4. In the original equation (Tamm's equation) the height is taken to be the difference between the midpoint of the opening and the 'neutral pressure level' of the cold space. In the case that there is just one dominant doorway through which infiltration occurs, the neutral pressure level is half the height of the doorway to the walk-in refrigeration unit. The refrigerated air leaks out through the lower half of the door, and the warm, infiltrating air enters through the top half of the door. We deconstruct the lower half of the door into infinitesimal horizontal strips of width W and height dh. Each strip is treated as a separate window, and the air flow through each infinitesimal strip is given by 60 x CD x A x {[(Ti Tr) / Ti] x g x ΔHNPL }^0.5 where ΔHNPL represents the distance to the vertical midpoint of the door. In effect, this replaces the implicit wh1.5 (one power from the area, and the other from ΔHNPL) with the integral from 0 to h/2 of wh'0.5 dh' which results in wh1.5/(3×20.5¬). For more information see: Are They Cool(ing)?:Quantifying the Energy Savings from Installing / Repairing Strip Curtains, Alereza, Baroiant, Dohrmann, Mort, Proceedings of the 2008 IEPEC Conference.

3.5.10 REFRIGERATION - NIGHT COVERS FOR DISPLAY CASES

Measure Name	Night Covers for Display Cases
Target Sector	Commercial Refrigeration and Industrial Establishments
Measure Unit	Display Cases
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	5 years ^{506,507}
Measure Vintage	Retrofit

ELIGIBILITY

This measure is the documents the energy savings associated with the installation of night covers on existing open-type refrigerated display cases, where covers are deployed during the facility's unoccupied hours in order to reduce refrigeration energy consumption. These types of display cases can be found in small and medium to large size grocery stores. The air temperature inside low-temperature display cases is below 0_°F for low-temperature display cases, and between 0_°F to 30_°F for medium-temperature display cases, and between 35_°F to 55_°F for high-temperature display cases. The main benefit of using night covers on open display cases is a reduction of infiltration and radiation cooling loads. It is recommended that these covers have small, perforated holes to decrease moisture buildup.

ALGORITHMS

The energy savings and demand reduction are obtained through the following calculations. 509

 $\Delta kWh = W \times SF \times HOU$

There are no demand savings for this measure because the covers will not be in use during the peak $period^{510}$.

DEFINITION OF TERMS

⁵⁰⁶ DEER Effective Useful Life. October 10, 2008.

⁵⁰⁷ The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007. http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf

⁵⁰⁸ Massachusetts 2012 Technical Reference Manual, pgpg. 229

^{509 &}quot;Effects of the Low Emissivity Shields on Performance and Power Use of a Refrigerated Display Case" Southern California Edison Refrigeration Technology and Test Center Energy Efficiency Division August 8, 1997.

⁵¹⁰ Assumed that the continuous covers are deployed at night (usually 1:00 a.m. – 5:00 a.m.); therefore no demand savings is usually reported for this measure.

· ·	•		
Component Term	Type Unit	Value <u>s</u>	Source
W, Width of the opening that the night covers protect	Variable <u>ft</u>	EDC's Data Gathering	EDC's Data Gathering
SF, Savings factor based on the temperature of the case	$\frac{kW}{ft}$ Fixed	Default values in <u>Table</u> 3-119Table 3-119	1
HOU, Annual hours that the night covers are in use	<u>Hours</u> Variable	EDC's Data Gathering Default: 2,190 ⁵¹¹	EDC's Data Gathering

Table 3-119: Savings Factors

Cooler Case Temperature	Savings Factor
Low Temperature (-35 F to -5 F)	0.03 kW/ft
Medium Temperature (0 F to 30 F)	0.02 kW/ft
High Temperature (35 F to 55 F)	0.01 kW/ft

The demand and energy savings assumptions are based on analysis performed by Southern California Edison (SCE). SCE conducted this test at its Refrigeration Technology and Test Center (RTTC). The RTTC's sophisticated instrumentation and data acquisition system provided detailed tracking of the refrigeration system's critical temperature and pressure points during the test period. These readings were then utilized to quantify various heat transfer and power related parameters within the refrigeration cycle. The results of SCE's test focused on three typical scenarios found mostly in supermarkets.

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.—

Measure Life

The expected measure life is 5 years 512,513.

⁵¹¹ Hours should be determined on a case-by-case basis. Default value of 2,190 hours is estimated assuming that the annual operating hours of the refrigerated case is 8,760 hours as per Ohio 2010 Technical Reference Manual and night covers must be applied for a period of at least six hours in a 24-hour period. https://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

$\textcolor{red}{\textbf{S}} \, \texttt{OURCES}$

 CL&P Program Savings Documentation for 2011 Program Year (2010). Factors based on Southern California Edison (1997). Effects of the Low Emissive Shields on Performance and Power Use of a Refrigerated Display Case. http://www.energizect.com/sites/default/files/2012%20CT%20Program%20Savings%20
 Documentation%20FINAL.pdf

542 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.

⁵¹³ The Measure Life Report for Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June-2007

3.5.11 AUTO CLOSERS

Measure Name	Auto Closers
Target Sector	Commercial Refrigerationand Industrial Establishments
Measure Unit	Walk-in Cooler and Freezer Door
Unit Energy Savings	Fixed
Unit Peak Demand Reduction	Fixed
Measure Life	8 years ⁵¹⁴
Measure Vintage	Retrofit

The auto-closer should be applied to the main insulated opaque door(s) of a walk-in cooler or freezer. Auto-closers on freezers and coolers can reduce the amount of time that doors are open, thereby reducing infiltration and refrigeration loads. These measures are for retrofit of doors not previously equipped with auto-closers, and assume the doors have strip curtains.

ELIGIBILITY 515

This protocol documents the energy savings attributed to installation of auto closers in walk-in coolers and freezers. The auto-closer must be able to firmly close the door when it is within one inch of full closure. The walk-in door perimeter must be \geq 16 ft.

ALGORITHMS

Auto-cClosers are treated in the Database for Energy Efficient Resources (DEER) as weather-sensitive; therefore the recommended default deemed savings values indicated below are derived from the DEER-runs in California climate zones most closely associated to the climate zones of the main seven Pennsylvania cities Climate zone 4 has been chosen as the most similar zone to to the climate zones of the main seven Pennsylvania cities. Thise association between California climate zones and the Pennsylvania cities is based on cooling degree hours (CDHs) and wet bulb temperatures. Savings estimates for each measure are averaged across six building vintages for each climate-zone for and building type 9, Grocery Stores. The peak demand savings provided by DEER was calculated using the following peak definition:

"Tthe demand savings due to an energy efficiency measure is calculated as the average reduction in energy use over a defined nine-hour demand period." 3

The nine hours correspond to 2 PM through 5 PM during 3-day heat waves.⁴

⁵¹⁴ http://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

³ http://www.deeresources.com/files/DEER2011/download/2011_DEER_Documentation_Appendices.pdf
4 Zarnikau, J. and Zhu, S.S. (2014) The Identification of Peak Period Impacts When a TMY Weather
File Is Used in Building Energy Use Simulation. Open Journal of Energy Efficiency, 3, 25-33.
http://dx.doi.org/10.4236/ojee.2014.31003

Main Cooler Doors

 $\begin{array}{lll} \Delta kWh & = \; \Delta kWh_{cooler} \\ \Delta kW_{peak} & = \; \Delta kW_{cooler} \end{array}$

Main Freezer Doors

 $\begin{array}{lll} \Delta kWh & = \; \Delta kWh_{freezer} \\ \Delta kW_{peak} & = \; \Delta kW_{freezer} \end{array}$

DEFINITION OF TERMS

Table 3-120: <u>Auto Closers Calculation Assumptions</u>

<u>Term</u>	<u>Unit</u>	<u>Values</u>	Source
ΔkWh _{cooler} , Annual kWh savings for main cooler doors	kWh	<u>Table</u> 3-121 Table 3-121	1
ΔkW _{cooler} , Summer peak kW savings for main cooler doors	kW	<u>Table</u> <u>3-121</u> Table <u>3-121</u>	1
ΔkWh _{freezer} , Annual kWh savings for main freezer doors	kWh	<u>Table</u> <u>3-121</u> Table <u>3-121</u>	1
ΔkW _{freezer} , Summer peak kW savings for main freezer doors	kW	<u>Table</u> 3-121 3-121	1

DEEMEDFAULT SAVINGS

Table 3-121: Refrigeration Auto Closers Deemedfault Savings

	Associated		<u>Val</u>	<u>ue</u>	
Reference City	California Climate Zone	<u>Cooler</u>		<u>Freezer</u>	
		<u>kWh</u> cooler	<u>kW_{cooler}</u>	<u>kWh</u> freezer	<u>kW_{freezer}</u>
All PA cities	<u>4</u>	<u>961</u>	<u>0.135</u>	<u>2319</u>	0.327

Table 3-75: Refrigeration Auto Closers Calculations Assumptions

- 4	Associated-					
Reference City	California Climate Zone ⁵¹⁶	Coc	ler	Fr	eezer	Source
		kWh _{cooler}	kW _{cooler}	kWh _{freezer}	kW_{freezer}	

⁶¹⁶ The deemed savings values were adopted from the California region and are adjusted to account for differences in weather conditions in Pennsylvania based on cooling degree hours and wet bulb temperatures. The Refrigeration – Auto Closers measure-protocol follows the weather mapping table shown in Section 1.17.

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Reference City	Associated- California Climate Zone 516		Val	ue		Source
Allentown	15	882	0.061	2351	0.142	4
Williamstown	15	882	0.061	2351	0.142	4
Pittsburgh	4	961	0.135	2319	0.327	4
Philadelphia	15	882	0.061	2351	0.142	4
Erie	9	989	0.175	2369	0.395	4
Harrisburg	15	882	0.061	2351	0.142	4
Scranton	4	961	0.135	2319	0.327	4

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The expected measure life is 8 years 517.

SOURCES

2.—2005 DEER weather sensitive commercial data; DEER Database, http://www.deeresources.com/ _

⁵¹⁷ http://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

3.5.12 REFRIGERATION — DOOR GASKETS FOR WALK-IN AND REACH-IN COOLERS AND FREEZERS

Measure Name	Door Gaskets for Walk-in and Reach-in Coolers and Freezers
Target Sector	Commercial and Industrial Establishments
Measure Unit	Walk-in Coolers and Freezers
Unit Energy Savings	Fixed
Unit Peak Demand Reduction	<u>Fixed</u>
Measure Life	4 years ⁵¹⁸
Measure Vintage	Replace on Burnout

The following protocol for the measurement of energy and demand savings is applicable to commercial refrigeration and applies to the replacement of worn-out gaskets with new better-fitting gaskets. Applicable gaskets include those located on the doors of walk-in and/or reach-in coolers and freezers.

Tight fitting gaskets inhibit infiltration of warm, moist air into the cold refrigerated space, thereby reducing the cooling load. Aside from the direct reduction in cooling load, the associated decrease in moisture entering the refrigerated space also helps prevent frost on the cooling coils. Frost build-up adversely impacts the coil's, heat transfer effectiveness, reduces air passage (lowering heat transfer efficiency), and increases energy use during the defrost cycle. Therefore, replacing defective door gaskets reduces compressor run time and improves the overall effectiveness of heat removal from a refrigerated cabinet.

ELIGIBILITY

This protocol applies to the main doors of both low temperature ("freezer" – below 32_°F) and medium temperature ("cooler" – above 32_°F) walk-ins.

ALGORITHMS

The demand and energy savings assumptions are based on analysis performed by Southern California Edison.

The energy savings and demand reduction are obtained through the following calculations:

$$\Delta kWh = \frac{\Delta kWh}{ft} \times L$$

$$\Delta kW_{peak} = \frac{\Delta kW}{ft} \times L$$

⁵¹⁸ Commercial Facilities Contract Group 2006-2008 Direct Impact Evaluation, California Public Utility Commission, February 2010.

DEFINITION OF TERMS

Table 3-122: Door Gasket Assumptions

Term	Unit	Values	Source
$\frac{\Delta kWh}{ft}$, Annual energy savings per linear foot of gasket	$\frac{\Delta kWh}{ft}$	Table 3-123	1
$\frac{\Delta kW}{ft}$, Demand savings per linear foot of gasket	$\frac{\Delta kW}{ft}$	Table 3-123	1
L, Total gasket length	Feet	As Measured	EDC Data Gathering

DEFAULT SAVINGS

The– default savings values below are weather sensitive, therefore the values reference CA climate zone 4, which is the zone chosen as the most similar to the seven major Pennsylvania cities. The demand and energy savings assumptions are based on DEER 2005 and analysis performed by Southern California Edison (SCE)⁵¹⁹.

Table 3-123: Door Gasket Savings Per Linear Foot for Walk-in and Reach-in Coolers and Freezers

	Coolers		Freezers	
Building Type	$\frac{\Delta kW}{ft}$	$\frac{\Delta kWh}{ft}$	$\frac{\Delta kW}{ft}$	$\frac{\Delta kWh}{ft}$
Restaurant	0.000886	<u>18</u>	0.001871	<u>63</u>
Small Grocery Store/ Convenience Store	0.000658	<u>15</u>	0.00162	<u>64</u>
Medium/Large Grocery Store/ Supermarkets	0.0006425	<u>15</u>	0.001593	<u>91</u>

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The expected measure life is 4 years 520.

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⁵¹⁹ Work papers developed by SCE filed with the CA PUC in support of its 2006 – 2008 energy efficiency program plans ⁵²⁰ http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02-18-2010.pdf

SOURCES

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3.5.13 REFRIGERATION: SPECIAL DOORS WITH LOW OR NO ANTI-SWEAT HEAT FOR LOW TEMP CASE

Measure Name	Special Doors with Low or No Anti-Sweat Heat for Low Temp <u>erature</u> Case <u>s</u>
Target Sector	Commercial and Industrial EstablishmentsRefrigeration
Measure Unit	Display Cases
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁵²¹
Measure Vintage	Retrofit

Traditional clear glass display case doors consist of two-pane glass (three-pane in low and medium temperature cases), and aluminum doorframes and door rails. Glass heaters may be included to eliminate condensation on the door or glass. The door heaters are traditionally designed to overcome the highest humidity conditions as cases are built for nation-wide applications. New low heat/no heat door designs incorporate heat reflective coatings on the glass, gas inserted between the panes, non-metallic spacers to separate the glass panes, and/or non-metallic frames (such as fiberglass).

This protocol documents the energy savings attributed to the installation of special glass doors w/low/no anti-sweat heaters for low temp cases. The primary focus of this rebate measure is on new cases to incent customers to specify advanced doors when they are purchasing refrigeration cases.

ELIGIBILITY

For this measure, a no-heat/low-heat clear glass door must be installed on an upright display case. It is limited to door heights of 57 inches or more. Doors must have either heat reflective treated glass, be gas filled, or both. This measure applies to low temperature cases only—those with a case temperature below 0°F. Doors must have 3 or more panes. Total door rail, glass, and frame heater amperage (@ 120 volt) cannot exceed 0.3925⁵²² amps per door for low temperature display cases. Rebate is based on the door width (not including case frame).

ALGORITHMS

The energy savings and demand reduction are obtained through the following calculations adopted from <u>San Diego Gas & Electric Statewide Express Efficiency Program California's Southern California Edison</u> 523.

⁵²¹ http://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

⁵²² Pacific Gas & Electric Company. Refrigeration Rebate Catalog. Pg. 5.

http://www.fishnick.com/saveenergy/rebates/2014 PG&E refrigeration catalog final.pdfSouthern California Edison. Non-Residential Express 2003 Refrigeration Work Paper. Pg. 27.

⁵²³ San Diego Gas & Electric Statewide Express Efficiency Program

https://www.sdge.com/sites/default/files/regulatory/Express%20and%20SBS%20Workpapers.pdf

Assumptions: Indoor Dry-Bulb Temperature of 75 $^{\circ}$ F and Relative Humidity of 55%, (4-minute opening intervals for 16-second), neglect heat conduction through doorframe / assembly.

Compressor Savings (excluding condenser):

$$\begin{array}{ll} \Delta kW_{compressor} & = \frac{1}{1000} \times \frac{Q - cooling_{cooling_svg}}{EER} \\ \Delta kWh_{compressor} & = \Delta kW \times EFLH \\ Q_{cooling_svg} & = Q_{cooling} \times K_{ASH} \end{array}$$

Anti-Sweat Heater Savings:

$$\Delta kW_{ASH} \qquad \qquad = \frac{\Delta \, ASH}{1000} \\ \Delta kWh_{ASH} \qquad \qquad = \Delta kW_{ASH} \times t$$

DEFINITION OF TERMS

Table 3-124: Special Doors with Low or No Anti-Sweat Heat for Low Temp Case Calculations Assumptions

<u>Term</u> Parameter	<u>Unit</u> Type	Value <u>s</u>	Source
$Q_{cooling} Q - cooling$, Case rating by manufacturer	$\frac{Btu}{hr} \times \frac{1}{door} \text{Variable}$	From Nnameplate	EDC Data Gathering
$Q_{cooling_svg}Q - coolingQ - cooling_{svg_L}$. Cooling savings	$\frac{Btu}{hr} \times \frac{1}{door}$	Calculated Value	Calculated Value
$\Delta kW_{compressor}$, Compressor power savings	kW door	Calculated Value	Calculated Value
ΔkW _{ASH} . Reduction due to ASH	$\frac{kW}{door}$	<u>Calculated Value</u>	Calculated Value
K — ASH , Kash. % of cooling load reduction due to low anti-sweat heater	Fixed None	1.5%	1
Δ ASH, Reduction in ASH power per door	watts door Fixed	83 ⁵²⁴	1
$\Delta kWh_{compressor}$, Annual compressor energy savings (excluding condenser energy)	kWh door	Calculated Value	Calculated Value
ΔkWh _{ASH} , Annual reduction in energy	$\frac{kWh}{door}$	Calculated Value	Calculated Value
EER, Compressor rating from manufacturer	<u>None</u> Variable	Nameplate	EDC Data Gathering
EFLH, Equivalent full load annual operating hours	<u>Hours</u> Variable	Based on Logging, BMS data or Modeling	EDC Data Gathering
		Default: 5,700 ⁵²⁵	1
t, Annual operating hours of Anti-sweat heater	Fixed <u>Hours</u>	8,760	1

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

⁵²⁴ From Actual Test: 0.250 kW per 3 doors

⁵²⁵ EFLH was determined by multiplying annual available operation hours of 8,760 by overall duty cycle factors. Duty cycle is a function of compressor capacity, defrost and weather factor. The units are assumed to be operating 24/7, 8760 hrs/yr.

MEASURE LIFE

The expected measure life is 15 years 526.

SOURCES

 San Diego Gas & Electric Statewide Express Efficiency Program https://www.sdge.com/sites/default/files/regulatory/Express%20and%20SBS%20Workpa pers.pdf Southern California Edison. Non-Residential Express 2003 Refrigeration Work Paper. Pg. 27

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⁵²⁶ http://energysmartonline.org/documents/EnergySmart_BPA_T&Cs.pdf

3.5.14 Suction Pipe Insulation for Walk-In Coolers and Freezers

Measure Name	Refrigeration-Suction Pipe Insulation for Walk-In Coolers and Freezers
Target Sector	Commercial and Industrial EstablishmentsRefrigeration
Measure Unit	Walk-In Coolers and FreezersRefrigreation
Unit Energy Savings	Fixed
Unit Peak Demand Reduction	Fixed
Measure Life	11 years ^{527,528}
Measure Vintage	Retrofit

Rev Date: June 2015

ELIGIBILITY

This protocol documents the energy savings attributed to insulation of bare refrigeration suction pipes. The following are the eligibility requirements⁵²⁹:

- Must insulate bare refrigeration suction lines of 1-5/8 inches in diameter or less on existing equipment only
- Medium temperature lines require 3/4 inch of flexible, closed-cell, nitrite rubber or an equivalent insulation
- Low temperature lines require 1-inch of insulation that is in compliance with the specifications above
- Insulation exposed to the outdoors must be protected from the weather (i.e. jacketed with a medium-gauge aluminum jacket)

ALGORITHMS

The demand and energy savings assumptions are based on <u>DEER 2005 and</u> analysis performed by Southern California Edison (SCE)⁵³⁰. Measure savings per linear foot of insulation installed on bare suction lines in Restaurants and Grocery Stores is provided in <u>Table 3-125</u>

SECTION 3: Commercial and Industrial Measures

 ⁵²⁷ Southern California Edison Company, "Insulation of Bare Refrigeration Suction Lines", Work Paper WPSCNRRN0003, California-Measurement Advisory Committee Public Workpapers on PY 2001 Energy Efficiency Programs. September 2000. Appendix F, P.14
 528 DEER database, EUL/RUL for insulation bare suction pipes

⁵²⁹ Commonwealth Edison Refrigeration Incentives Worksheet 2014. https://www.comed.com/documents/business-savings/sifyb_py6_refrigeration.pdf_

http://www.energysmartgrocer.org/pdfs/PGE/2010_2012%20External%20Equipment%20SpecificationTandCs%20v3.pdf

⁵³⁰ Work papers developed by SCE filed with the CA PUC in support of its 2006 – 2008 energy efficiency program plans

Table 3-125 and

Table 3-126

Table 3-126 below lists the "default" savings for the associated with California Climate Z₂one 4s and their respective which has been chosen as the representative zone for all seven major Pennsylvania citiesy.

$$\Delta kWh = \frac{\Delta kWh}{ft} \times L$$

$$\Delta kW_{peak} = \frac{\Delta kW}{ft} \times L$$

DEFINITION OF TERMS

Table 3-125: Insulate Bare Refrigeration Suction Pipes Calculations Assumptions

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
$\frac{\Delta kWh}{ft}$, Annual energy savings per linear foot of insulation	$\frac{\Delta kWh}{ft}$ Variable	Table 3-126 Table 3-126Error! Reference source not found.	1
$\frac{\Delta kW}{ft}$, Demand savings per linear foot of insulation	$\frac{\Delta kW}{ft}$ Variable	<u>Table 3-126</u> Table 3-126	1
L, Total insulation length	Variableftft.	As Measured	EDC Data Gathering

DEFAULT SAVINGS

Table 3-126: Insulate Bare Refrigeration Suction Pipes Savings per Linear Foot for Walk-in Coolers and Freezers of Restaurants and Grocery Stores⁵³¹

City	Associated California Climate Zone ⁵³²	Medium-Temperature Walk- in Coolers		Low-Temperature Walk-in Freezers	
	Zone	ΔkW/ <mark>ft<u>ft.</u></mark>	ΔkWh/ <mark>ft</mark> ft.	ΔkW/ <mark>ft</mark> ft.	ΔkWh/ft <u>ft.</u>
All PA cities Allentown	<u>4</u> 15	0.002190.002488	<u>11.3</u> 14.5	0.0027260.002861	<u>14.8</u> 17.6
Williamstown	15	0.002488	14.5	0.002861	17.6
Pittsburgh	4	0.001507	8	0.0023	13

⁵³¹ A zip code mapping table is located in Appendix F. This table should be used to identify the <u>reference</u> Pennsylvania city for all zip codes in Pennsylvania

Refrigeration

⁵³² The deemed_default savings values were adopted from the California region and are adjusted to account for differences in weather conditions in Pennsylvania based on cooling degree hours and wet bulb temperatures. Thetemperatures. The Refrigeration – Suction Pipes Insulation measure protocol follows the weather mapping table shown in Section 1.17Section 1.17.

City	Associated California Climate Zone ⁵³²	Medium-Temperature Walk- in Coolers		Low-Temperatur Freezers	
Philadelphia Philadelphia	15	0.002488	14.5	0.002861	17.6
Erie	9	0.001757	9.5	0.00246	14
Harrisburg	15	0.002488	14.5	0.002861	17.6
Scranton	4	0.001507	8	0.0023	13

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

$\textcolor{red}{\textbf{S}} \, \texttt{OURCES}$

 Southern California Edison Company, "Insulation of Bare Refrigeration Suction Lines", Work Paper WPSCNRRN0003.

3.6 APPLIANCES

3.6.1 ENERGY STAR CLOTHES WASHER

Measure Name	ENERGY STAR Clothes Washer
Target Sector	Commercial and Industrial EstablishmentsMultifamily Common-Area Laundry and Laundromats
Measure Unit	Clothes WasherPer Washing Machine
Unit Energy Savings	See Table 3-128 to Table 3-131
Unit Peak Demand Reduction	See Table 3-128 to Table 3-131
Measure Life	11.3 years for Multifamily and 7.1 years for Laundromats ⁵³³
Measure Vintage	Replace on Burnout

This protocol discusses the calculation methodology and the assumptions regarding baseline equipment, efficient equipment, and usage patterns used to estimate annual energy savings expected from the replacement of a standard clothes washer with an ENERGY STAR clothes washer with a minimum Modified Energy Factor (MEF) of $\geq 2.2 \frac{(ft^3 \times cycle)}{kWh}$. The Federal efficiency standard is $\geq 1.60 \frac{(ft^3 \times cycle)}{kWh}$ for Top Loading washers and $\geq 2.0 \frac{(ft^3 \times cycle)}{kWh}$ for Front Loading washers⁵³⁴.

ELIGIBILITY

This protocol documents the energy savings attributed to efficient clothes washers meeting ENERGY STAR or better in small commercial applications. This protocol is limited to clothes washers in laundry rooms of multifamily complexes and commercial Laundromats.

ALGORITHMS

The general form of the equation for the ENERGY STAR Clothes Washer measure savings algorithm is:

Total Savings = Number of Clothes Washers \times Savings per Clothes Washer

To determine resource savings, the per-unit estimates in the algorithms will be multiplied by the number of clothes washers.

Per unit energy and demand savings are obtained through the following calculations:

$$\Delta kWh = \left[\left(HE_{t,base} + ME_{t,base} + D_{e,base} \right) - \left(HE_{t,new} + ME_{t,new} + D_{e,new} \right) \right] \times N$$

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^{533 &}quot;Energy Conservation Program: Energy Conservation Standards for Certain Consumer Products (Dishwashers, Dehumidifiers, Microwave Ovens, and Electric and Gas Kitchen Ranges and Ovens) and for Certain Commercial and Industrial Equipment (Commercial Clothes Washers); Final Rule," 75 Federal Register 5 (January 8, 2010), pp.1140.
534 ihid

```
Where:
                                        = LAF \times WGHT_{max} \times DEF \times DUF \times (RMC^3 - 4\%)
D_e
                                        = (-0.156 \times MEF) + 0.734
RMC
                                        = \left(\frac{Cap}{MEF}\right) - ME_t - D_e
= \Delta kWh_kWh_savings_{} \times UF
HE_t
\Delta kW_{peak}
```

The algorithms used to calculate energy savings are taken from the U.S. Department of Energy's <u>Supplemental Notice of Proposed Rulemaking (SNOPR)Life Cycle Cost and Payback Period</u> tool. 535 DOE adopted the algorithms for commercial clothes washers in a final rule published on October 18, 2005. Commercial clothes washer per-cycle energy consumption is composed of three components: water-heating energy, machine energy, and drying energy. DOE established the annual energy consumption of commercial clothes washers by multiplying the per-cycle energy and water use by the number of cycles per year.

In the above equations, MEF is the Modified Energy Factor, which is the energy performance metric for clothes washers. MEF is defined as:

MEF is the quotient of the capacity of the clothes container, C, divided by the total clothes washer energy consumption per cycle, with such energy consumption expressed as the sum of the machine electrical energy consumption, M, the hot water energy consumption, E, and the energy required for removal of the remaining moisture in the wash load, D. The higher the value, the more efficient the clothes washer is. $MEF = \frac{c}{_{M+E+D}}$

$$MEF = \frac{c}{M + E + D}$$

The following steps should be taken to determine per-cycle energy consumption for top-loading and front-loading commercial clothes washers for both old and new clothes washers. Per-cycle energy use is disaggregated into water heating, machine, and clothes drying.

- Calculate the remaining moisture content (RMC) based on the relationship between RMC and MEF.
- 2. Calculate the per-cycle clothes-drying energy use using the equation that determines the per-cycle energy consumption for the removal of moisture.
- 3. Use the per-cycle machine energy use value of 0.133 $\frac{kWh}{cycle}$ for MEFs up to 1.40 and 0.114 $\frac{kWh}{cycle}$ for MEFs greater than 1.40. These values are estimated from 2000 TSD for residential clothes washers' database.
- 4. With the per-cycle clothes dryer and machine energy known, determine the per-cycle water-heating energy use by first determining the total per-cycle energy use (the clothes container volume divided by the MEF) and then subtracting from it the per-cycle clothesdrying and machine energy.

The utilization factor, (UF) is equal to the average energy usage between noon and 8PM on summer weekdays to the annual energy usage. The utilization rate is derived as follows:

1. Obtain normalized, hourly load shape data for residential clothes washing.

http://www.energystar.gov/index.cfm?c=clotheswash.pr crit clothes washers

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⁵³⁵ Commercial Clothes Washer Supplemental Notice of Proposed Rulemaking, Chapter 6. http://www1.eere.energy.gov/buildings/appliance_standards/commercial/pdfs/ccw_snopr_chap6.pdf

⁵³⁶ Definition provided on ENERGY STAR Clothes Washers Key Product Criteria website:

- Smooth the load shape by replacing each hourly value with a 5-hour average centered about that hour. This step is necessary because the best available load shape data exhibits erratic behavior commonly associated with metering of small samples. The smoothing out effectively simulates diversification.
- 3. Take the UF to be the average of all load shape elements corresponding to the hours between noon and 8PM on weekdays from June to September.

The value is the June-September, weekday noon to 8 PM average of the normalized load shape values associated with residential clothes washers in PG&E service territory (northern CA). Although Northern CA is far from PA, the load shape data is the best available at the time and the temporal dependence washer usage is not expected to have a strong geographical dependency. Figure 3-15 shows the utilization factor for each hour of a sample week in July. Because the load shape data derived from monitoring of in-house clothes washers is being imputed to multifamily laundry room washers (which have higher utilization rates), it is important to check that the resulting minutes of usage per hour is significantly smaller than 60. If the minutes of usage per hour approaches 60, then it should be assumed that the load shape for multi-family laundry room clothes washers must be different than the load shape for in-house clothes washers. The maximum utilization per hour is 36.2 minutes.

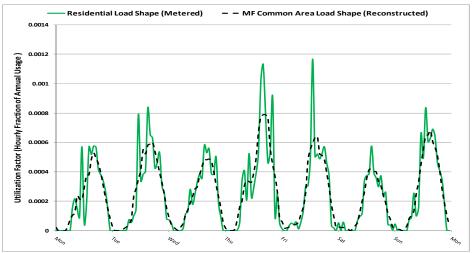


Figure 3-15: Utilization factor for a sample week in July⁵³⁷

DEFINITION OF TERMS

The parameters in the above equation are listed in

Table 3-127

Table 3-127 below.

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⁵³⁷ The solid green profile is derived from a normalized load shape based on metering of residential in-unit dryers. The dashed black profile is a smoothed version of the green profile and represents the utilization factors for common laundry facilities in multifamily establishments.

Table 3-127: Commercial Clothes Washer Calculation Assumptions

Component <u>Term</u>	Type Unit	Values	Source
MEF_b , Base Federal Standard Modified Energy Factor	None Fixed	Top loading: 1.6 Front loading: 2.0	<u>3</u> 4
MEF _p , Modified Energy Factor of ENERGY STAR Qualified Washing	None Variable	Nameplate	EDC Data Gathering
Machine	None Default	Default: 2.2	<u>3</u> 4
HE_t , Per-cycle water heating consumption	kWh Variable	Calculation	Calculation
D_{e} , Per-cycle energy consumption for removal of moisture i.e. dryer energy consumption	kWh Variable	Calculation	Calculation
ME_t , Per-cycle machine electrical energy consumption	kWh cycle Fixed	0.114 ⁵³⁸	1
	ft³Variable Default	Nameplate	EDC Data Gathering
Cap _{base} , Capacity of baseline clothes washer		Default: Front Loading: 2.84 Top Loading: 2.95	5
Capee, Capacity of efficient clothes	ft³Varible Default	Nameplate	EDC Data Gathering
washer		Front Loading: 2.84 Top Loading: 2.84	5
LAF, Load adjustment factor	<u>None</u> Fixed	0.52	1 ,2
DEF, Nominal energy required for clothes dryer to remove moisture from clothes	$\frac{kWh}{lb}$ Fixed	0.5	1 ,2
DUF, Dryer usage factor, percentage of washer loads dried in a clothes dryer	NoneFixed	0.84	1 ,2
WGHT _{max} , Maximum test-load weight	lbs cycle Fixed	11.7	1 ,2
RMC, Remaining moisture content	lbs Variable	Calculation ⁵³⁹	Calculation
N, Number of cycles per year	<u>Cycle</u> Fixed	Multifamily: 1,241 Laundromats: 2,190	1
UF, Utilization Factor	NoneFixed	0.0002382	<u>2</u> 3

DEFAULTEMED SAVINGS

The defaultemed savings for the installation of a washing machine with a MEF of 2.2 or higher, is dependent on the energy source for washer. Table 3-128 The table Tables 1-2 thru Table 3-131 1-5 below shows savings for ENERGY STAR washing machines for with different combinations of water heater and dryer types in multifamily buildings and landromats. The values are based on the difference between the baseline clothes washer with MEF Federal efficiency standard of ≥

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⁵³⁸ Based on residential clothes washer data from DOE 2000 Technical Support Document (TSD)

 $^{^{539}}$ Based on the relationship: RMC = -0.156* MEF + 0.734

1.60 $\frac{(ft^3 \times cycle)}{kWh}$ for $\text{$\stackrel{\bot}{=}$}$ for $\text{$\stackrel{\bot}{=}$}$ coading washers and $\text{$\stackrel{\succeq}{=}$}$ 2.0 $\frac{(ft^3 \times cycle)}{kWh}$ for $\text{$\stackrel{\bot}{=}$}$ for $\text{$\stackrel{\bot}{=}$}$ coading washers and minimum ENERGY STAR qualified front loading⁵⁴⁰ clothes washer of \geq 2.2 $\frac{(ft^3 \times cycle)}{trut}$.

For clothes washers where fuel mix is unknown, calculate default savings using the algorithms below and EDC specific saturation values. For EDCs where saturation information is not accessible, use "Default values" described in Table 3-128 thru Table 3-131 below.

$$ESav_{cw} = kWh_{gwh-gd} \times \%GWH - GD_{cw} + kWh_{gwh-ed} \times \%GWH - ED_{cw} + kWh_{ewh-gd} \times \%EWH - GD_{cw} + kWh_{ewh-ed} \times \%EWH - ED_{cw}$$
Where:
$$| kWh_{gwh-gd} = Energy \text{ savings for clothes washers with gas water heater and non-electric dryer fuel from tables below} = Energy \text{ savings for clothes washers with gas water heater and electric dryer fuel from tables below} = Energy \text{ savings for clothes washers with electric water heater and non-electric dryer fuel from tables below} = Energy \text{ savings for clothes washers with electric water heater and non-electric dryer fuel from tables below} = Energy \text{ savings for clothes washers with electric water heater and electric dryer fuel from tables below} = Percent of clothes washers with gas water heater and non-electric dryer fuel} = Percent of clothes washers with gas water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and non-electric dryer fuel} = Percent of clothes washers with electric water heater and non-electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers with electric water heater and electric dryer fuel} = Percent of clothes washers wit$$

Table 3-128: Defaultemed Savings for Top Loading ENERGY STAR Clothes Washer for Laundry in Multifamily Buildings⁵⁴¹

Fuel Source	Cycles/ Year	Energy Savings (kWh)	Demand Reduction (kW)
Electric Hot Water Heater, Electric Dryer	1,241	686	0.163
Electric Hot Water Heater, Gas Dryer	1,241	341	0.081
Gas Hot Water Heater, Electric Dryer	1,241	345	0.082
Gas Hot Water Heater, Gas Dryer	1,241	0	0
Default (20% Electric DHW 40% Electric Dryer) ⁵⁴²	1,241	206	0.049

⁵⁴⁰ ENERGY STAR-qualified commercial clothes washers in 2013 are likely to be front-loading units because there are no toploading commercial clothes washers at this time which have been certified by DOE as meeting the 2013 standards ⁵⁴¹Based on a container volume of 2.8 cu. ft., maximum test-load weight of 11.7 lb./cycle and electric water heater at 100%

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⁵⁴² Commercial Clothes Washer SNOPR Life Cycle Calculator Excel Spread Sheet http://www1.eere.energy.gov/buildings/appliance_standards/commercial/clothes_washers_snopr_spreadsheets.html

Table 3-129: Defaultemed Savings for Front Loading ENERGY STAR Clothes Washer for Laundry in Multifamily Buildings 543

Fuel Source	Cycles Year	Energy Savings (kWh)	Demand Reduction (kW)
Electric Hot Water Heater, Electric Dryer	1,241	160	0.038
Electric Hot Water Heater, Gas Dryer	1,241	61	0.015
Gas Hot Water Heater, Electric Dryer	1,241	99	0.024
Gas Hot Water Heater, Gas Dryer	1,241	0	0
Default (20% Electric DHW 40% Electric Dryer) ⁵⁴⁴	1,241	52	0.012

Table 3-130: De<u>fault</u>emed Savings for Top Loading ENERGY STAR Clothes Washer for Laundromats 545

Fuel Source	Cycles Year	Energy Savings (kWh)	Demand Reduction (kW)
Electric Hot Water Heater, Electric Dryer	2,190	1,211	0.288
Electric Hot Water Heater, Gas Dryer	2,190	602	0.143
Gas Hot Water Heater, Electric Dryer	2,190	609	0.145
Gas Hot Water Heater, Gas Dryer	2,190	0	0
Default (0% Electric DHW 0% Electric Dryer) ⁵⁴⁶	2,190	0	0

⁵⁴³ibid ⁵⁴⁴ ibid

⁵⁴⁵ibid

⁵⁴⁶ ibid

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Table 3-131: Defaultemed Savings Front Loading ENERGY STAR Clothes Washer for Laundromats 547

Fuel Source	Cycles Year	Energy Savings (kWh)	Demand Reduction (kW)
Electric Hot Water Heater, Electric Dryer	2,190	283	0.067
Electric Hot Water Heater, Gas Dryer	2,190	108	0.026
Gas Hot Water Heater, Electric Dryer	2,190	175	0.042
Gas Hot Water Heater, Gas Dryer	2,190	0	0
Default (0% Electric DHW 0% Electric Dryer) ⁵⁴⁸	2,190	0	0

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- U.S. Department of Energy. Commercial Clothes Washer Supplemental Notice of Proposed Rulemaking, Chapter 6. http://www1.eere.energy.gov/buildings/appliance-standards/commercial/pdfs/ccw_snopr-chap6.pdf
- U.S. Department of Energy's Life-Cycle Cost and Payback Period tool, available at: http://www1.eere.energy.gov/buildings/appliance_standards/commercial/clothes_washers _snepr_spreadsheets.html-
- 3-2. Annual hourly load shapes taken from Energy Environment and Economics (E3), Reviewer2: http://www.ethree.com/cpuc_cee_tools.html. The average normalized usage for the hours noon to 8 PM, Monday through Friday, June 1 to September 30 is 0.000243
- 4-3. "Energy Conservation Program: Energy Conservation Standards for Certain Consumer Products (Dishwashers, Dehumidifiers, Microwave Ovens, and Electric and Gas Kitchen Ranges and Ovens) and for Certain Commercial and Industrial Equipment (Commercial Clothes Washers); Final Rule," 75 Federal Register 5 (8 January 2010), pp. 1123http://www1.eere.energy.gov/buildings/appliance_standards/commercial/clothes_washers.html
- 5.4. ENERGY STAR. U.S. Environmental Protection Agency and U.S. Department of Energy. "ENERGY STAR Program Requirements Product Specification for Clothes Washers." ENERGY STAR Version 6.1 Clothes Washers Specification (Jan. 2013): 5. http://www.energystar.gov/products/specs/system/files/Clothes Washers Program Requirements Version 6 1.pdf

548 ibid

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⁵⁴⁷lbid

http://www1.eere.energy.gov/buildings/appliance_standards/commercial/clothes_washers.html

6.5. California Energy Commission ("CEC") Appliance Efficiency database, http://www.appliances.energy.ca.gov/QuickSearch.aspx

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3.7 FOOD SERVICE EQUIPMENT

3.7.1 HIGH-EFFICIENCY ICE MACHINES

Measure Name	High-Efficiency Ice Machines
Target Sector	Commercial and Industrial Establishments
Measure Unit	Ice Machine
Unit Energy Savings	<u>Variable</u>
Unit Peak Demand Reduction	Variable
Measure Life	<u>10 Years⁵⁴⁹</u>
Measure Vintage	Replace on Burnout

ELIGIBILITY

This measure applies to the installation of a high-efficiency ice machine as either a new item or replacement for an existing unit. The machine must be air-cooled to qualify, which can include self-contained, ice-making heads, or remote-condensing units. The machine must conform with-to the minimum ENERGY STAR efficiency requirements, which are equivalent to the CEE Tier 2 specifications for high-efficiency commercial ice machines. A qualifying machine must also meet the ENERGY STAR requirements for water usage given under the same criteria.

The baseline equipment is taken to be a unit with efficiency specifications less than or equal to CEE Tier 1 equipment.

ALGORITHMS

The energy savings are dependent on machine type and capacity of ice produced on a daily basis. A machine's capacity is generally reported as an ice harvest rate, or amount of ice produced each day.

$$\Delta kWh = \frac{(kWh_{base} - kWh_{he})}{100} \times H \times 365 \times D$$

$$= \frac{\Delta kWh}{8760 \times D} \times CF$$

DEFINITION OF TERMS

The reference values for each component of the energy impact algorithm are shown in Table 3-132._-A default duty cycle (D) is provided as based on referenced values from several studies, however, EDC data gathering may be used to adjust the duty cycle for custom applications.

⁵⁴⁹ DEER Effective Useful Life. October 10, 2008.

⁵⁵⁰ Commercial Ice Machines Key Product Criteria, ENERGY STAR, accessed 8/2010, http://www.energystar.gov/index.cfm?c=comm_ice_machines.pr_crit_comm_ice_machines

Table 3-132: Ice Machine Reference Values for Algorithm Components

	Term	<u>Unit Type</u>	Value <u>s</u>	Source
	kWh_{base} , Baseline ice machine energy usage per 100 lbs. of ice	wh Variable	<u>Table 3-133</u> Table 3-133	1
	kWh_{he} , High-efficiency ice machine energy usage per 100 lbs. of ice	^{kWh} / _{100 lbs} Variable	Table 3-133Table 3-133	2
	H, Ice harvest rate per 24 hrs.	lbs Variable	Manufacturer Specs	EDC Data Gathering
	D, Duty cycle of ice machine expressed as a percentage of time machine produces ice	None Variable	Custom	EDC Data Gathering
l			Default: 0.4 ⁵⁵¹	3
	365. Days per year	days year	<u>365</u>	Conversion Factor
	100, Conversion to obtain energy per pound of ice	<u>lbs</u> 100 lbs	100	Conversion Factor
	8760, Hours per year	hours year	<u>8,760</u>	Conversion Factor
	Ice maker type	None Variable	Manufacturer Specs	EDC Data Gathering
	CF, Demand Coincidence Factor (SeeSection 1.5).	<u>None</u> Fixed	0.77	4

ENERGY SAVINGS CALCULATIONS

DEFAULT SAVINGS

Ice machine energy usage levels are dependent on the ice harvest rate (H), and are calculated using CEE specifications as shown in <u>Table 3-133-Table 3-133</u>. The default energy consumption for the baseline ice machine (kWh_{base}) is calculated using the formula for CEE Tier 1 specifications, and the default energy consumption for the high-efficiency ice machine (kWh_{he}) is calculated using the formula for CEE Tier 2 specifications⁵⁵². The two energy consumption values are then applied to the energy savings algorithm above.

⁵⁵¹ State of Ohio Energy Efficiency Technical Reference Manual, Including Predetermined Savings Values and Protocols for Determining Energy and Demand Savings, August 6, 2010. Prepared for the Public Utilities Commission of Ohio by Vermont Energy Investment Corporation.

⁵⁵² High Efficiency Specifications for Commercial Ice Machines, Consortium for Energy Efficiency, accessed 8/2010, http://www.cee1.org/com/com-kit/files/IceSpecification.pdf

⁰¹ July 2011, http://library.cee1.org/sites/default/files/library/4280/CEE_lce_Machines_Spec_Final_Effective_01Jul2011 - updated_equivalencies.pdf_

Table 3-133: Ice Machine Energy Usage

Ice machine type	Ice harvest rate (H) $\left(\frac{lbs}{day}\right)$	Baseline energy use per 100 lbs. of ice $(kWh_{base})^{553}$	High-efficiency energy use per 100 lbs. of ice $(kWh_{he})^{554}$
Ice-Making Head	<450	10.26 – 0.0086*H	9.23 – 0.0077*H
ice-iviaking nead	≥450	6.89 – 0.0011*H	6.20 - 0.0010*H
Remote-Condensing w/out remote compressor	<1000	8.85 – 0.0038*H	8.05 – 0.0035*H
	≥1000	5.1	4.64
Remote-Condensing with remote compressor	<934	8.85 – 0.0038*H	8.05 – 0.0035*H
	≥934	5.3	4.82
Self-Contained	<175	18 – 0.0469*H	16.7 – 0.0436*H
Seir-Contained	≥175	9.8	9.11

MEASURE LIFE

Measure life = 10 years 555.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Specifications for CEE Tier 1 ice machines.
- 2. Specifications for CEE Tier 2 ice machines.
- 3. State of Ohio Energy Efficiency Technical Reference Manual cites a default duty cycle of 40% as a conservative value. Other studies range as high as 75%.
- 4. State of Ohio Energy Efficiency Technical Reference Manual cites a CF = 0.772 as adopted from the Efficiency Vermont TRM. Assumes CF for ice machines is similar to that for general commercial refrigeration equipment.

⁵⁵³ "Energy Conservation Standards for Certain Consumer Products and Commercial and Industrial Equipment; Final Rule," 70 Federal Register 200 (18 October 2005), pp. 60416.

^{554 &}quot;High Efficiency Specifications for Commercial Ice Machines", Consortium for Energy Efficiency, 01 July 2011,

http://library.cee1.org/sites/default/files/library/4280/CEE_lce_Machines_Spec_Final_Effective_01Jul2011 - updated_equivalencies.pdf

⁵⁵⁵_DEER EUL Summary, Database for Energy Efficient Resources, accessed 8/2010, http://www.deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls

- 5. Karas, A., Fisher, D. (2007), A Field Study to Characterize Water and Energy Use of Commercial Ice-Cube Machines and Quantify Saving Potential, Food Service Technology Center, December 2007, http://www.fishnick.com/publications/appliancereports/special/Ice
 - cube_machine_field_study.pdf
- 6. Energy-Efficient Products, How to Buy an Energy-Efficient Commercial Ice Machine, U.S. Department of Energy, Energy Efficiency and Renewable Energy, accessed August 2010 at http://www1.eere.energy.gov/femp/procurement/eep_ice_makers.html

State of Pennsylvania – Technical Reference Manual – Rev Date: June 2015

3.7.2 CONTROLS: BEVERAGE MACHINE CONTROLS

Measure Name	Controls: Bevearage Machine Controls
Target Sector	Commercial and Industrial Establishments
Measure Unit	Machine Control
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	<u>5 years 556,557</u>
Measure Vintage	Retrofit

ELIGIBILITY

This measure is intended for the addition of control systems to existing, non-ENERGY STAR, beverage vending machines. The applicable machines contain refrigerated, non-perishable beverages that are kept at an appropriate temperature. The control systems are intended to reduce energy consumption due to lighting and refrigeration during times of lower customer sales. Typical control systems contain a passive infrared occupancy sensor to shut down the machine after a period of inactivity in the area. The compressor will power on_for one to three hour intervals, sufficient to maintain beverage temperature, and when powered on at any time will be allowed to complete at least one cycle to prevent excessive wear and tear.

The baseline equipment is taken to be an existing standard refrigerated beverage vending machine that does not contain control systems to shut down the refrigeration components and lighting during times of low customer use.

ALGORITHMS

Energy savings are dependent on decreased machine lighting and cooling loads during times of lower customer sales. The savings will be dependent on the machine environment, noting that machines placed in locations such as a day-use office will result in greater savings than those placed in high-traffic areas such as hospitals that operate around the clock. The algorithm below takes into account varying scenarios and can be taken as representative of a typical application.

$$\Delta kWh = kWh_{base} \times E$$

$$\Delta kW_{peak} = 0$$

There are no peak demand savings because this measure is aimed to reduce demand during times of low beverage machine use, which will typically occur during off-peak hours.

⁵⁵⁶ DEER Effective Useful Life. October 10, 2008.

⁵⁵⁷ Deru et al. suggest that beverage machine life will be extended from this measure due to fewer lifetime compressor cycles.

DEFINITION OF TERMS

Table 3-134: Beverage Machine Control Calculation Assumptions

<u>Term</u>	<u>Unit</u>	<u>Values</u>	Source
kWh _{base} . Baseline annual beverage machine energy consumption	kWh year	EDC Data Gathering Default: Table 3-135Table 3-135	EDC Data Gathering
E, Efficiency factor due to control system, which represents percentage of energy reduction from baseline	None	EDC Data Gathering	EDC Data Gathering

DEFAULT SAVINGS ENERGY SAVINGS CALCULATIONS

The decrease in energy consumption due to the addition of a control system will depend on the number or hours per year during which lighting and refrigeration components of the beverage machine are powered down. The average decrease in energy use from refrigerated beverage vending machines with control systems installed is 46%. 558,559,560,561. _ It should be noted that various studies found savings values ranging between 30-65%, most likely due to differences in customer occupation.

The default baseline energy consumption and default energy savings are shown in Table 3-135. The default energy savings were derived by applying a default efficiency factor of $E_{default}=46\%$ to the energy savings algorithm above. Where it is determined that the default efficiency factor (E) or default baseline energy consumption (kWh_{base}) is not representative of specific applications, EDC data gathering can be used to determine an application-specific energy savings factor (E), and/or baseline energy consumption (kWh_{base}) , for use in the Energy Savings algorithm.

⁵⁵⁸ Deru, M., et al., (2003), Analysis of NREL Cold-Drink Vending Machines for Energy Savings, National Renewable Energy Laboratory, NREL/TP-550-34008, http://www.nrel.gov/docs/fy03osti/34008.pdf

⁵⁵⁹ Ritter, J., Hugghins, J., (2000), *Vending Machine Energy Consumption and VendingMiser Evaluation*, Energy Systems Laboratory, Texas A&M University System, http://repository.tamu.edu/bitstream/handle/1969.1/2006/ESL-TR-00-11-01.pdf;jsessionid=6E215C09FB80BC5D2593AC81E627DA97?sequence=1

⁵⁶⁰ State of Ohio Energy Efficiency Technical Reference Manual, Including Predetermined Savings Values and Protocols for Determining Energy and Demand Savings, August 6, 2010. Prepared for the Public Utilities Commission of Ohio by Vermont Energy Investment Corporation

⁵⁶¹ Vending Machine Energy Savings, Michigan Energy Office Case Study 05-0042, http://www.michigan.gov/documents/CIS_EO_Vending_Machine_05-0042_155715_7.pdf

Table 3-135: Beverage Machine Controls Energy Savings

Machine Can Capacity	Default Baseline Energy Consumption (kWh_{base}) $\left(\frac{kWh}{year}\right)$	Default Energy Savings $(\Delta kWh); {kWh \choose year}$	Source
< 500	3,113	1,432	1
500	3,916	1,801	1
600	3,551	1,633	1
700	4,198	1,931	1
800+	3,318	1,526	1

MEASURE LIFE

Measure life = 5 years

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

SOURCES

 ENERGY STAR Calculator, Assumptions for Vending Machines, accessed 8/2010 http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_Vend_MachBulk.xls

3.7.3 CONTROLS: SNACK MACHINE CONTROLS

Measure Name	Controls: Snack Machine Controls
Target Sector	Commercial and Industrial Establishments
Measure Unit	Machine Control
Unit Energy Savings	Variable
Unit Peak Demand Reduction	0 kW
Measure Life	5 years ⁵⁶²
Measure Vintage	Retrofit

A snack machine controller is an energy control device for non-refrigerated snack vending machines. The controller turns off the machine's lights based on times of inactivity. This protocol is applicable for conditioned indoor installations.

ELIGIBILITY

This measure is targeted to non-residential customers who install controls to non-refrigerated snack vending machines.

Acceptable baseline conditions are non-refrigerated snack vending machines.

Efficient conditions are non-refrigerated snack vending machines with controls.

ALGORITHMS

The energy savings for this measure result from reduced lighting operation.

$$\Delta kWh = \frac{Watts_{base}}{1000} * HOURS * ESF$$

$$\Delta kW_{peak} = 0$$

DEFINITION OF TERMS

⁵⁶² Measure Life Study, prepared for the Massachusetts Joint Utilities, Energy & Resource Solutions, November 2005. http://rtf.nwcouncil.org/subcommittees/nonreslighting/Measure%20Life%20Study_MA%20Joint%20Utilities_2005_ERS-1.pdf

Table 3-136: Snack Machine Controls - Values and References

Term	Unit	Values	Source
${\it Watts_{base}}$, Wattage of vending machine	Watts	EDC Data Gathering Default: 85	EDC Data Gathering
HOURS , Annual hours of operation	hours	EDC Data Gathering Default: 8,760	EDC Data Gathering
ESF, Energy savings factor	None	46	1

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

 Illinois Statewide TRM, 2014. Machine wattages assume that the peak period is coincident with periods of high traffic diminishing the demand reduction potential of occupancy based controls. Hours of operation assume operation 24 hrs/day, 365 days/yr. http://www.efi.org/specs/snackmiser.pdf.

3.7.4 ENERGY STAR ELECTRIC STEAM COOKER

Measure Name	ENERGY STAR Electric Steam Cooker
Target Sector	Commercial and Industrial Establishments
Measure Unit	Electric Steam Cooker
Unit Energy Savings	See Table 3-138 Table 3-138
Unit Peak Demand Reduction	See Table 3-138 Table 3-138
Measure Life	12 years ⁵⁶³
Measure Vintage	Replace on Burnout

ELIGIBILITY

This measure applies to the installation of electric ENERGY STAR steam cookers as either a new item or replacement for an existing unit. Gas steam cookers are not eligible. The steam cookers must meet minimum ENERGY STAR efficiency requirements. A qualifying steam cooker must meet a minimum cooking efficiency of 50 percent and meet idle energy rates specified by pan capacity.

The baseline equipment is a unit with efficiency specifications that do not meet the minimum ENERGY STAR efficiency requirements.

ALGORITHMS

The savings depend on three main factors: pounds of food steam cooked per day, pan capacity, and cooking efficiency.

$$\Delta kWh \\ \Delta kWh_{cooking} = \Delta kWh_{cooking} + \Delta kWh_{idle} \\ = lbsfood \times EnergytoFood \times \left(\frac{1}{Eff_b} - \frac{1}{Eff_{ee}}\right) \\ = \left[\left(Power_{idle-b} \times (1 - \%HOURS_{consteam}) + \%HOURS_{consteam} \times CAPY_b \right. \\ \left. \times Qty_{pans} \times \left(\frac{EnergytoFood}{Eff_b}\right) \times HOURS_{op} \\ - \left(\frac{lbsfood}{CAPY_b \times Qty_{pans}}\right) - HOURS_{pre}\right)\right] \\ - \left[\left(Power_{idle-ee} \times (1 - \%HOURS_{consteam}) + \%HOURS_{consteam} \times CAPY_{ee} \times Qty_{pans} \times \left(\frac{EnergytoFood}{Eff_{ee}}\right) \right. \\ \left. \times HOURS_{op} - \left(\frac{lbsfood}{CAPY_{ee} \times Qty_{pans}}\right) - HOURS_{pre}\right)\right] \\ \Delta kW_{peak} = \frac{\Delta kWh}{EFLH} \times CF$$

563 http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC

DEFINITION OF TERMS

Table 3-137: Steam Cooker - Values and References

Component <u>Term</u>	Type <u>Unit</u>	Values	Source
lbsfood, Pounds of food cooked per day in the steam cooker	<i>lbs</i> Variable	Nameplate	EDC Data Gathering
day in the steam cooker		Default values in <u>Table</u> 3-138 Table 3-138	<u>Table 3-138</u> Table 3-138
EnergyToFood, ASTM energy to food ratio; energy (kilowatt-hours) required per pound of food during cooking	kWh pound Fixed	$0.0308 \frac{kWh}{pound}$	1
Eff_{ee} , Cooking energy efficiency of the new unit	None Variable	Nameplate	EDC Data Gathering
the new unit		Default values in <u>Table</u> 3-138Table 3-138	<u>Table 3-138</u> Table 3-138
${\it Eff_b}$, Cooking energy efficiency of the baseline unit	<u>None</u> Fixed	See <u>Table 3-138</u> Table 3-138	<u>Table 3-138</u> Table 3-138
$Power_{idle-b}$, Idle power of the baseline unit	kW Variable	See <u>Table 3-138</u> Table 3-138	<u>Table 3-138</u> Table 3-138
Power _{idle-ee} , Idle power of the new unit	kW Variable	Nameplate	EDC Data Gathering
unit		Default values in <u>Table</u> 3-138 Table 3-138	<u>Table 3-138</u> Table 3-138
${\it HOURS_{op}}$, Total operating hours per day	<u>Hours</u> Variable	Nameplate	EDC Data Gathering
		12 hours	1
${\it HOURS_{pre}}$, Daily hours spent preheating the steam cooker	<u>Hours</u> Fixed	0.25	1
%HOURS _{consteam} , Percentage of idle time per day the steamer is in continuous steam mode instead of timed cooking. The power used in this mode is the same as the power in cooking mode.	<u>None</u> Fixed	40%	1
CAPY_b , Production capacity per pan of the baseline unit	$\frac{lb}{hr}$ Fixed	See <u>Table 3-138</u> Table 3-138	<u>Table 3-138</u> Table 3-138
CAPY_{ee} , Production capacity per pan of the new unit	$\frac{lb}{hr}$ Fixed	See <u>Table 3-138</u> Table 3-138	<u>Table 3-138</u> Table 3-138
$\mathit{Qty}_{\mathit{pans}}$, Quantity of pans in the unit	None Variable	Nameplate	EDC Data Gathering
EFLH, Equivalent full load hours per year	<u>Hours</u> Fixed	4,380	2
CF, Demand Coincidence Factor (See Section 1.5)	<u>None</u> Fixed	0.84	3,4
1000, Conversion from watts to kilowatts	$\frac{W}{kW}$	1000 <u>w</u>	Conversion factor

DEFAULT SAVINGS

Table 3-138: Default Values for Electric Steam Cookers by Number of Pans⁵⁶⁴

# of Pans	Parameter	Baseline Model	Efficient Model	Savings
	Power _{idle} (kW) ⁵⁶⁵	1.000	0.27	
	$CAPY\left(\frac{lb}{hr}\right)$	23.3	16.7	
3	lbsfood	100	100	
	Eff ⁵⁶⁶	30%	59%	
	ΔkWh			2,813
	ΔkW_{peak}			0.54
	Power _{idle} (kW)	1.325	0.30	
	$CAPY\left(\frac{lb}{hr}\right)$	21.8	16.8	
4	lbsfood	128	128	
	Eff	30%	57%	
	ΔkWh			3,902
	ΔkW_{peak}			0.75
	Power _{idle} (kW)	1.675	0.31	
	$CAPY\left(\frac{lb}{hr}\right)$	20.6	16.6	
5	lbsfood	160	160	
	Eff	30%	70%	
	ΔkWh			5,134
	ΔkW_{peak}			0.98
	Power _{idle} (kW)	2.000	0.31	
	$CAPY\left(\frac{lb}{hr}\right)$	20.0	16.7	
6	lbsfood	192	192	
	Eff	30%	65%	
	ΔkWh			6,311
	ΔkW_{peak}			1.21

⁵⁶⁴ Values for ASTM parameters for baseline and efficient conditions (unless otherwise noted) were determined by FSTC according to ASTM F1484, the Standard Test Method for Performance of Steam Cookers. Pounds of Food Cooked per Day based on the default value for a 3 pan steam cooker (100 lbs from FSTC) and scaled up based on the assumption that steam cookers with a greater number of pans cook larger quantities of food per day.

⁵⁶⁵ Efficient values calculated from a list of ENERGY STAR qualified products.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC data gathering or assignment of stipulated energy savings.

SOURCES

- ENERGY STAR. US Environmental Protection Agency and US Department of Energy. <u>ENERGY STAR Commercial Kitchen Equipment Calculator US Department of Energy.</u> <u>ENERGY STAR Calculator.</u>
- Food Service Technology Center (FSTC) 2012, Commercial Cooking Appliance Technology Assessment, pg 8-14. based on an assumption that the restaurant is open 12 hours a day, 365 days a year.
- 3. State of Ohio Energy Efficiency Technical Reference Manual cites a CF = 0.84 as adopted from the Efficiency Vermont TRM. Assumes CF is similar to that for general commercial industrial lighting equipment.
- RLW Analytics. Coincidence Factor Study Residential and Commercial Industrial Lighting Measures. Spring 2007. The peak demand period used to estimate the CF value is 1PM-5PM, weekday, non-holiday, June-August. http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/116 RLW CF%20Res%20C&I%20Itg.pdf

3.7.5 ENERGY STAR REFRIGERATED BEVERAGE MACHINE

Measure Name	ENERGY STAR Refrigerated Beverage Vending Machine
Target Sector	Commercial and Industrial Establishments
Measure Unit	Refrigerated Beverage Vending Machine
Unit Energy Savings	Variable
Unit Peak Demand Reduction	0 kW
Measure Life	14 years ⁵⁶⁷
Measure Vintage	Replace on Burnout

ENERGY STAR vending machines are equipped with more efficient compressors, fan motors and lighting systems. In addition to more efficient components, ENERGY STAR qualified machines are programmed with software that reduces lighting and refrigeration loads during times of inactivity.

ELIGIBILITY

This measure is targeted to non-residential customers who purchase and install a beverage vending machine that meets ENERGY STAR specifications rather than a non-ENERGY STAR unit. The energy efficient refrigerated vending machine can be new or rebuilt.

ALGORITHMS

Energy savings are dependent on decreased machine lighting and cooling loads during times of lower customer sales. The savings are dependent on the machine environment, noting that machines placed in locations such as a day-use office will result in greater savings than those placed in high-traffic areas such as hospitals that operate around the clock. The algorithm below takes into account varying scenarios and can be taken as representative of a typical application. There are no peak demand savings because this measure is aimed to reduce demand during times of low beverage machine use, which will typically occur during off-peak hours.

Class A Vending Machine

A Class A machine is defined as a refrigerated bottled or canned beverage vending machine that is fully cooled, and is not a combination vending machine.

```
\begin{array}{lll} \varDelta kWh & = kWh_{base} - kWh_{ee} \\ \varDelta kWh_{base} & = (0.055V + 2.56) \times 365 \\ \varDelta kWh_{ee} & = (0.0523V + 2.432) \times 365 \\ \varDelta kW_{peak} & = 0 \end{array}
```

Class B Vending Machine

A Class B machine is defined as any refrigerated bottled or canned beverage vending machine not considered to be Class A, and is not a combination vending machine.

⁵⁶⁷ "Energy Savings Potential and R&D Opportunities for Commercial Refrigeration," Navigant Consulting, Final Report submitted to US DOE, September 2009. http://apps1.eere.energy.gov/buildings/publications/pdfs/corporate/commercial_refrig_report_10-09.pdf

```
      \Delta kWh
      = kWh_{base} - kWh_{ee}

      \Delta kWh_{base}
      = (0.073V + 3.16) \times 365

      \Delta kWh_{ee}
      = (0.0657V + 2.844) \times 365

      \Delta kW_{peak}
      = 0
```

DEFINITION OF TERMS

Table 3-139: ENERGY STAR Refrigerated Beverage Vending Machine - Values and Resources

Term	Unit	Values	Source
kWh_{base} ,energy usage of baseline vending machine	kWh	EDC Data Gathering ⁵⁶⁸	EDC Data Gathering
kWh_{ee} , energy usage of ENERGY STAR vending machine	kWh	EDC Data Gathering ⁵⁶⁹	EDC Data Gathering
V, refrigerated volume of the vending machine	ft³	EDC Data Gathering Default: 24.3	3
365, days per year	$\frac{Days}{yr}$	365	Conversion Factor

DEFAULT SAVINGS

Table 3-140: Default Beverage Vending Machine Energy Consumption

Equipment Class	Default kWh Savings
Class A	1,422
Class B	180

Energy savings for this measure are fully deemed and may be claimed using the algorithm above and the variable defaults.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

⁵⁶⁸ Data gathering of vending machine volume

⁵⁶⁹ Data gathering of annual energy consumption of vending machine

SOURCES

- ENERGY STAR. US Environmental Protection Agency and US Department of Energy. "Program Requirements; Product Specification for Refrigerated Beverage Vending Machines."
 - $\frac{\text{http://www.energystar.gov/products/specs/system/files/Vending\%20Machines\%20Program\%20Requirements\%20Version\%203\%201.pdf}{}$
- 2. US Department of Energy. "Refrigerated Beverage Vending Machines." http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/24
- 3. ENERGY STAR. US Environmental Protection Agency and US Department of Energy. "ENERGY STAR Certified Vending Machines Spread Sheet" http://www.energystar.gov/productfinder/product/certified-vending-machines/results

3.8 BUILDING SHELL

3.8.1 WALL AND CEILING INSULATION

Measure Name	Wall and Ceiling Insulation
Target Sector	Commercial and Industrial Establishments
Measure Unit	Wall and Ceiling Insulation
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	<u>15 years⁵⁷⁰</u>
Measure Vintage	New Construction or Retrofit

Wall and ceiling insulation is one of the most important aspects of the energy system of a building. Insulation dramatically minimizes energy expenditure on heating and cooling. Increasing the R-value of wall insulation above building code requirements generally lowers heating and cooling costs. Incentives are offered with regard to increases in R-value rather than type, method, or amount of insulation.

An R-value indicates the insulation's resistance to heat flow – the higher the R-value, the greater the insulating effectiveness. The R-value depends on the type of insulation and its material, thickness, and density. When calculating the R-value of a multilayered installation, add the R-values of the individual layers.

ELIGIBILITY

This measure applies to non-residential buildings or common areas in multifamily complexes heated and/or cooled using electricity. Existing construction buildings are required to meet or exceed the code requirement. New construction buildings must exceed the code requirement. Eligibility may vary by PA EDC; savings from chiller-cooled buildings are not included.

ALGORITHMS

The savings depend on four main factors: baseline condition, heating system type and size, cooling system type and size, and location. The algorithm for Central AC and Air Source Heat Pumps (ASHP) is as follows:

Ceiling/Wall Insulation

$$\begin{array}{ll} \Delta kWh & = \Delta kWh_{cool} + \Delta kWh_{heat} \\ \Delta kWh_{cool} & = \left(\frac{A \times CDD \times 24}{Eff EER} \times 1,000\right) \times \left(\frac{1}{R_i} - \frac{1}{R_f}\right) \\ \Delta kWh_{heat} & = \left(\frac{A \times HDD \times 24}{COP \times 3,413}\right) \times \left(\frac{1}{R_i} - \frac{1}{R_f}\right) \end{array}$$

⁵⁷⁰DEER 2008 uses 20 years; Northwest Regional Technical Forum uses 45 years. Capped based on the requirements of the Pennsylvania Technical Reference Manual (June 2010). This value is less than that used by other jurisdictions for insulation.

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 $\frac{\Delta kW_{peak}}{\textit{EFLH}_{cool}} \times \textit{CF}$

DEFINITION OF TERMS

Table 3-141: Non-Residential Insulation – Values and References

Term Component	<u>Unit Type</u>	Value <u>s</u>	Source
A, Area of the insulation that was installed	ft ² Variable	EDC Data Gathering Application	AEPS- Application; EDC Data Gathering
HDD, Heating degree days with 65 degree base	°D_Fixed	Allentown = 5318 Erie = 6353 Harrisburg = 4997 Philadelphia = 4709 Pittsburgh = 5429 Scranton = 6176 Williamsport = 5651	1
CDD, Cooling degree days with a 65 degree base	°D_Fixed	Allentown = 787 Erie = 620 Harrisburg = 955 Philadelphia = 1235 Pittsburgh = 726 Scranton = 611 Williamsport = 709	1
24, Hours per day	Hours Fixed	24	Conversion Factor n/a
1000, Watts per kilowatt	$\frac{W}{kW}$ Fixed	1000	Conversion Factor n/a
3,413, Btu per kWh	$\frac{Btu}{kWh}$	3,413	Conversion Factor
Ceilling R_i , the R-value of the ceiling insulation and support structure before the additional insulation is installed	NoneExistin g: Variable New- Construction : Fixed	For new construction buildings and when variable is unknown for existing buildings: See Table 3-142 and Table 3-143 Table 3-143 for values by building type	AEPS- Application; EDC Data Gathering; 2, 4
$Wall\ R_i$, the R-value of the wall insulation and support structure before the additional insulation is installed	NoneExistin g: Variable New- Construction : Fixed	For new construction buildings and when variable is unknown for existing buildings: See Table 3-142 and Table 3-143 for values by building type	AEPS- Application; EDC Data Gathering; 3, 4
R_f , Total R-value of all insulation after the additional insulation is installed	None Variable	EDC Data Gathering	AEPS Application; EDC Data Gathering;
EFLH_{cool} , Equivalent full load cooling hours	None Variable	Based on Logging, BMS data or Modeling ⁵⁷¹ Default: See Table 3-23 Table 3-23	EDC Data Gathering 5
CF, Demand Coincidence Factor (See	None Fixed	55%See Table	5

⁵⁷¹ Modeling is an acceptable substitute to metering and BMS data if modeling is conducted using building- and equipment-specific information at the site and the facility consumption is calibrated using 12 months of billing data (pre-retrofit).

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Term Component	Unit Type	Value <u>s</u>	Source
Section 1.5)		3-24Table_3-24	
EER, Efficiency of the cooling system Eff. Efficiency of existing HVAC equipment.	Btu/hr Wariabl	Nameplate	EDC Data Gathering
Depending on the size and age, this will either be the SEER, IEER, or EER (use EER only if SEER or IEER are not available)	e	Default: See <u>Table</u> 3-22Table 3-22	See <u>Table</u> 3-22 Table 3-22
COP, Efficiency of the heating system	None Variabl	Nameplate	EDC Data Gathering
cor, Emolency of the heating system	Ф	Default: See <u>Table</u> 3-22 <u>Table</u> 3-22	See <u>Table</u> 3-22 Table 3-22

Table 3-142: Ceiling R-Values by Building Type

Building Type	Ceiling R _i -Value (New Construction)	Ceiling R _i -Value (Existing)
Large Office	20	9
Large Retail		
Lodging		
Health		
Education		
Grocery		
Small Office	24.4	13.4
Warehouse		
Small Retail	20	9
Restaurant		
Convenience Store		

Field Code Changed

Table 3-143: Wall R-Values by Building Type

Building Type	Wall R _i -Value (New Construction)	Wall R _i -Value (Existing)
Large Office	14	1.6
Small Office Large Retail Small Retail Convenience Store	14	3.0
Lodging Health Education Grocery	13	2.0
Restaurant	14	3.2
Warehouse	14	2.5

Table 1-4: HVAC Baseline Efficiencies for Non-Residential Buildings 572

Equipment Type and Capacity	Cooling Efficiency	Heating Efficiency				
Air-Source Air Conditioners						
<-65,000 BtuH	13.0 SEER	N/A				
≥ 65,000 BtuH and <135,000 BtuH	11.2 EER / 11.4 IEER	N/A				
≥ 135,000 BtuH and < 240,000 BtuH	11.0 EER / 11.2 IEER	N/A				
≥ 240,000 BtuH and < 760,000 BtuH (IPLV for units with capacity-modulation- only)	10.0 EER / 10.1 IEER	N/A				
≥ 760,000 BtuH (IPLV for units-with capacity-modulation-only)	9.7 EER / 9.8 IEER	N/A				
Water-Source and Evaporatively-Cooled Air Conditioners						
<65,000 BtuH	12.1 EER / 12.3 IEER	N/A				
≥ 65,000 BtuH and <135,000 BtuH	11.5 EER / 11.7 IEER	N/A				
≥ 135,000 BtuH and < 240,000 BtuH	11.0 EER / 11.2 IEER	N/A				
≥ 240,000 BtuH	11.0 EER / 11.1 IEER	N/A				
Air-Source Heat Pumps						
<65,000 BtuH	13 SEER	7.7 HSPF				
≥65,000 BtuH and <135,000 BtuH	11.0 EER / 11.2 IEER	3.3 COP				
≥ 135,000 BtuH and < 240,000 BtuH	10.6 EER / 10.7 IEER	3.2 COP				
≥ 240,000 BtuH (IPLV for units with capacity modulation only)	9.5 EER / 9.6 IEER	3.2 COP				
Water-Source Heat Pumps						
<17,000 BtuH	11.2 EER	4.2 COP				
≥ 17,000 BtuH and ≤ 65,000 BtuH	12.0 EER	4.2 COP				
Ground Water Source Heat Pumps						

⁵⁷² Baseline values from IECC 2009, Tables 503.2.3(1), 503.2.3(2), and 503.2.3(3). After Jan 1, 2010 or Jan 23, 2010 as applicable. Integrated Energy Efficiency Ratio (IEER) requirements have been incorporated from ASHRAE 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings", 2008 Supplement (Addendum S: (Tables 6.8.1A and 6.8.1B). IECC 2009 does not present IEER requirements.

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Equipment Type and Capacity	Cooling Efficiency	Heating Efficiency
< 135,000 BtuH	16.2 EER	3.6 COP
Ground Source Heat Pumps		
< 135,000 BtuH	13.4 EER	3.1 COP
Packaged Terminal Systems		
PTAC (cooling)	12.5 - (0.213 x Cap / 1000) EER	N/A
PTHP-	12.3 - (0.213 x Cap / 1000) EER	3.2 - (0.026 x Cap / 1000) COP

Note:

- For air-source air conditioners, water-source and evaporatively-cooled air conditionersand air-source heat pumps, subtract 0.2 from the required baseline efficiency rating value if unit has heating section other than electric resistance.
- Integrated Energy Efficiency Ratios (IEERs) are only applicable to equipment with capacity modulation.

Table 1-5: Cooling EFLH for Key PA Cities 573

Space and/or Building Type	Allentown	Erio	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Arena/Auditorium/Convention- Center	602	332	640	508	454	711	428
College: Classes/Administrative	690	380	733	582	520	815	490
Convenience Stores	1,216	671	1,293	1,026	917	1,436	864
Dining: Bar Lounge/Leisure	912	503	969	769	688	1,077	648
Dining: Cafeteria / Fast Food	1,227	677	1,304	1,035	925	1,449	872
Dining: Restaurants	912	503	969	769	688	1,077	648
Gymnasium/Performing Arts- Theatre	690	380	733	582	520	815	490
Hospitals/Health care	1,396	770	1,483	1,177	1,052	1,648	992
Lodging: Hotels/Motels/Dormitories	756	418	805	638	571	894	538
Lodging: Residential	757	418	805	638	571	894	538
Multi-Family (Common Areas)	1,395	769	1,482	1,176	1,052	1,647	991
Museum/Library	851	469	905	718	642	1,005	605
Nursing Homes	1,141	630	1,213	963	861	1,348	811
Office: General/Retail	851	469	905	718	642	1,005	605
Office: Medical/Banks	851	469	905	718	642	1,005	605
Parking Garages & Lots	938	517	997	791	707	1,107	666
Penitentiary	1,091	602	1,160	920	823	1,289	775
Police/Fire Stations (24 Hr)	1,395	769	1,482	1,176	1,052	1,647	991
Post Office/Town Hall/Court House	851	469	905	718	642	1,005	605
Religious Buildings/Church	602	332	640	508	454	711	428
Retail	894	493	950	754	674	1,055	635
Schools/University	634	350	674	535	478	749	4 51

⁵⁷³ US Department of Energy. ENERGY STAR Calculator, and Bin Analysis Models

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Space and/or Building Type	Allentown	Егіе	Harrisburg	Pittsburgh	Williamsport	Philadelphia	Scranton
Warehouses (Not Refrigerated)	692	382	735	583	522	817	492
Warehouses (Refrigerated)	692	382	735	583	522	817	492
Waste Water Treatment Plant	1,251	690	1,330	1,055	944	1,478	889

MEASURE LIFE

15 years

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific quidelines and requirements for evaluation procedures.

SOURCES

- U.S. Department of Commerce. Climatography of the United States No. 81 Supplement No. 2. Annual Degree Days to Selected Bases 1971 – 2000. Scranton uses the values for Wilkes-Barre. HDD were adjusted downward to account for business hours. CDD were not adjusted for business hours, as the adjustment resulted in an increase in CDD and so not including the adjustment provides a conservative estimate of energy savings.
- The initial R-value for a ceiling for existing buildings is based on the EDC eligibility requirement that at least R-11 be installed and that the insulation must meet at least IECC 2009 code. The initial R-value for new construction buildings is based on IECC 2009 code for climate zone 5. https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf
- 3. The initial R-value for a wall assumes that there was no existing insulation, or that it has fallen down resulting in an R-value equivalent to that of the building materials. Building simulation modeling using DOE-2.2 model (eQuest) was performed for a building with no wall insulation. The R-value is dependent upon the construction materials and their thickness. Assumptions were made about the building materials used in each sector.
- 4. 2009 International Energy Conservation Code. Used climate zone 5 which covers the majority of Pennsylvania. The R-values required by code were used as inputs in the eQuest building simulation model to calculate the total R-value for the wall including the building materials. https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf
- 5. <u>Based on results from Nexant's eQuest modeling analysis 2014.</u>

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6. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. The study reports CF values in the range of 44% to 63% depending on the size of the unit for the Mid Atlantic PJM hours. The study, therefore, assumes an average CF value of 55% for the PJM peak demand period that is applied to all units. Baseline values from IECC 2009, Tables 503.2.3(1), 503.2.3(2), and 503.2.3(3)., after Jan 1, 2010 or Jan 23, 2010 as applicable. Integrated Energy Efficiency Ratio (IEER) requirements have been incorporated from ASHRAE 90.1-2007, "Energy Standard for Buildings Except Low Rise Residential Buildings", 2008 Supplement (Addendum S: (Tables 6.8.1A and 6.8.1B). IECC 2009 does not present IEER requirements.

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3.9 CONSUMER ELECTRONICS

3.9.1 ENERGY STAR OFFICE EQUIPMENT

Measure Name	ENERGY STAR Office Equipment
Target Sector	Commercial and Industrial Establishments
Measure Unit	Office Equipment
Unit Energy Savings	<u>Fixed</u>
Unit Peak Demand Reduction	<u>Fixed</u>
Measure Life	<u>Table 3-145</u> <u>Table 3-145</u>
Measure Vintage	Replace on Burnout

ELIGIBILITY

This protocol estimates savings for installing ENERGY STAR office equipment compared to standard efficiency equipment. The measurement of energy and demand savings is based on a deemed savings value multiplied by the quantity of the measure.

ALGORITHMS

The general form of the equation for the ENERGY STAR Office Equipment measure savings' algorithms is:

Number of Units × Savings per Unit

To determine resource savings, the per unit estimates in the algorithms will be multiplied by the number of units. Per unit savings are primarily derived from the December 2010 release of the ENERGY STAR calculator for office equipment.

ENERGY STAR Computer

 $\begin{array}{ll} \Delta kWh & = ESAV_{com} \\ \Delta kW_{peak} & = DSAV_{com} \end{array}$

ENERGY STAR Fax Machine

 $\begin{array}{ll} \Delta kWh & = ESav_{fax} \\ \Delta kW_{peak} & = DSav_{fax} \end{array}$

ENERGY STAR Copier

 $\begin{array}{ll} \Delta kWh & = ESav_{cop} \\ \Delta kW_{peak} & = DSav_{cop} \end{array}$

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ENERGY STAR Printer

 $\begin{array}{lll} \Delta kWh & = ESav_{pri} \\ \Delta kW_{peak} & = DSav_{pri} \end{array}$

ENERGY STAR Multifunction

 $\begin{array}{ll} \Delta kWh & = ESav_{mul} \\ \Delta kW_{peak} & = DSav_{mul} \end{array}$

ENERGY STAR Monitor

 $\begin{array}{ll} \Delta kWh & = ESav_{mon} \\ \Delta kW_{peak} & = DSav_{mon} \end{array}$

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Table 3-144: ENERGY STAR Office Equipment - References

	Component-Term	Type <u>Unit</u>	Values	Source
	$\mathit{ESav}_{\mathrm{com}}$, Electricity savings per purchased ENERGY STAR computer	<u>kWh</u> Fixed	See <u>Table</u> 3-146 Table	1
	$\mathit{ESav_{fax}}$, Electricity savings per purchased ENERGY STAR fax machine.		3-146	
	$\mathit{ESav}_{cop},$ Electricity savings per purchased ENERGY STAR copier.			
	ESav_{pri} , Electricity savings per purchased ENERGY STAR printer.			
	ESav_{mul} , Electricity savings per purchased ENERGY STAR multifunction machine.			
	ESav_{mon} , Electricity savings per purchased ENERGY STAR monitor.			
	${ m DS}av_{ m com},$ Summer demand savings per purchased ENERGY STAR computer.	Fixed <u>kW</u>	See <u>Table</u> 3-146Table	2
	$\mathrm{D}\mathit{Sav}_\mathit{fax},$ Summer demand savings per purchased ENERGY STAR fax machine.		3-146	
	$\mathrm{D}\mathit{Sav}_\mathit{cop},$ Summer demand savings per purchased ENERGY STAR copier.			
	$\mathrm{D}\mathit{Sav}_{pri}$, Summer demand savings per purchased ENERGY STAR printer.			
	$\mathrm{D}\mathit{Sav}_{mul},$ Summer demand savings per purchased ENERGY STAR multifunction machine.			
	$\mathrm{D}\mathit{Sav}_{mon},$ Summer demand savings per purchased ENERGY STAR monitor.			
	CF _{com} , CF _{fax} , CF _{cop} , CF _{pri} , CF _{mut} , CF _{mon} , Demand Coincidence-Factor (See Section Setion 1.5). The coincidence of average office equipment demand to summer system peak equals 1.18-for demand impacts for all office equipment reflecting-embedded coincidence in the DSav factor.	Fixed	1.18-	3

ENERGY STAR office equipment have the following measure lives:

Table 3-145: ENERGY STAR Office Equipment Measure Life⁵⁷⁴

Equipment	Commercial Life (years)
Computer	4
Monitor	4
Fax	4
Multifunction Device	6
Printer	5
Copier	6

DEEMEDFAULT SAVINGS

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⁵⁷⁴ ENERGY STAR Office Equipment Savings Calculator

Table 3-146: ENERGY STAR Office Equipment Energy and Demand Savings Values

Measure	Energy Savings (ESav)	Summer Peak Demand Savings (DSav)	Source
Computer	133 kWh	0.018 kW	1
Fax Machine (laser)	78 kWh	0.0105 kW	1
Copier (monochrome)			1
1-25 images/min	73 kWh	0.0098 kW	
26-50 images/min	151 kWh	0.0203 kW	
51+ images/min	162 kWh	0.0218 kW	
Printer (laser, monochrome)			1
1-10 images/min	26 kWh	0.0035 kW	
11-20 images/min	73 kWh	0.0098 kW	
21-30 images/min	104 kWh	0.0140 kW	
31-40 images/min	156 kWh	0.0210 kW	
41-50 images/min	133 kWh	0.0179 kW	
51+ images/min	329 kWh	0.0443 kW	
Multifunction (laser, monochrome)			1
1-10 images/min	78 kWh	0.0105 kW	
11-20 images/min	147 kWh	0.0198 kW	
21-44 images/min	253 kWh	0.0341 kW	
45-99 images/min	422 kWh	0.0569 kW	
100+ images/min	730 kWh	0.0984 kW	
Monitor	15 kWh	0.0020 kW	1

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with EDC Data Gathering.

SOURCES

- ENERGY STAR Office Equipment Savings Calculator (<u>Referenced latest version released in May 2013Calculator updated: December 2010</u>). Default values were used. Using a commercial office equipment load shape, the percentage of total savings that occur during the PJM peak demand period was calculated and multiplied by the energy savings.
- 2. Coincidence factors already embedded in summer peak demand reduction estimates.

SECTION 3: Commercial and Industrial Measures

3.9.2 OFFICE EQUIPMENT - NETWORK POWER MANAGEMENT ENABLING

Measure Name	Network Power Management Enabling
Target Sector	Commercial and Industrial Establishments
Measure Unit	One copy of licensed software installed on a PC workstation
Unit Energy Savings	<u>Fixed</u>
Unit Peak Demand Reduction	<u>Fixed</u>
Measure Life	<u>5 years⁵⁷⁵</u>
Measure Vintage	Retrofit

Over the last fewthree years, a number of strategies have evolved to save energy in desktop computers. One class of products uses software implemented at the network level for desktop computers that manipulates the internal power settings of the central processing unit (CPU) and of the monitor. These power settings are an integral part of a computer's operating system (most commonly, Microsoft Windows) including "on", "standby", "sleep", and "off" modes and can be set by users from their individual desktops.

Most individual computer users are unfamiliar with these energy—saving settings, and hence, settings are normally set by an IT administrator to minimize user complaints related to bringing the computer back from standby, sleep, or off modes. However, these strategies use a large amount of energy during times when the computer is not in active use. Studies have shown that energy consumed during non-use periods is large, and is often the majority of total energy consumed.

Qualifying software must control desktop computer and monitor power settings within a network from a central location.

ELIGIBILITY **REQUIREMENTS**

The deemed savings reported in

Table 3-147

Table 3 147 are applicable to any software that meets the following Pacific Northwest Regional Technical Forum's ("RTF") Networked Computer Power Management Control Software Specifications⁵⁷⁶:

- · Workstation is defined as the computer monitor and the PC box.
- The software shall have wake-on-LAN capability to allow networked workstations to be remotely wakened from or placed into any power-saving mode and to remotely boot or shut down ACPI-compliant workstations.
- The software shall give the IT administrator easily-accessible central control over the power management settings of networked workstations that optionally overrides settings made by users.

⁵⁷⁵ While DEER lists the EUL of electro-mechanical plug load sensors at ten years, this product is subject to the cyclical nature of the PC software and hardware industry, so a more conservative number is appropriate. This is the same value used in the SDG&E program.

<u>program.</u>
576 Network PC Power Management Presentation, Regional Technical Forum, May 4, 2010.

- The software shall be capable of applying specific power management policies to network groups, utilizing existing network grouping capabilities.
- The software shall be compatible with multiple operating systems and hardware configurations on the same network.
- The software shall monitor workstation keyboard, mouse, CPU and disk activity in determining workstation idleness.

ALGORITHMS

There are no algorithms for this measure.

DEFINITION OF TERMS

There are no definitions of terms.

DEEMED SAVINGS

The energy savings per unit found in various studies specific to the Verdiem Surveyor software varied from 33.8 kWh/year to 330 kWh/year, with an average savings of about 200 kWh/year. This includes the power savings from the PC as well as the monitor. Deemed savings are based on a research study conducted by Regional Technical Forum which involves actual field measurements of the Verdiem Surveyor product. The study reports deemed energy and demand savings for three different building types (schools, large offices and small offices) in combination with different HVAC systems types (electric heat, gas heat, and heat pumps). The deemed savings values in

Table 3-147

Table 3-147 also take into account the HVAC interactive effects. A simple average is reported for Pennsylvania.

Table 3-147: Network Power Controls, Per Unit Summary Table

Measure Name	Unit	Gross Peak kW Reduction per Unit	Gross Peak kWh Reduction per Unit	Effective Useful Life
Network PC Plug Load Power Management Software	One copy of licensed software installed on a PC workstation	0.0078 <u>0.00625</u> ⁵⁷⁷	135 ⁵⁷⁸	5

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with

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⁵⁷⁷ http://www.nwcouncil.org/energy/rtf/measures/measure.asp?id=95&decisionid=117

verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific quidelines and requirements for evaluation procedures.

EFFECTIVE USEFUL LIFE

The EUL for this technology is estimated to be five (5) years. While DEER lists the EUL of electro-mechanical plug load sensors at ten years, this product is subject to the cyclical nature of the PC software and hardware industry, so a more conservative number is appropriate. This is the same value used in the SDG&E program.

SOURCES

- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Network Computer Power Management, v3.0.
 - Office Plug Load Field Monitoring Report, Laura Moorefield et al, Ecos Consulting, Dec, 2008.
 - b. PSE PC Power Management Results, Cadmus Group, Feb, 2011.
 - c. Non-Residential Network Computer Power Management, Avista, Feb, 2011.
 - After-hours Power Status of Office Equipment and Inventory of Miscellaneous Plug-Load Equipment, LBNL, Jan 2004.
 - e. Ecos Commercial Field Research Report, 2008.
- Dimetrosky, S., Luedtke, J. S., & Seiden, K. (2005). Surveyor Network Energy Manager:
 Market Progress Evaluation Report, No. 2 (Northwest Energy Efficiency Alliance report
 #E05-136). Portland, OR: Quantec LLC.
 http://www.nwalliance.org/research/reports/136.pdf
- 3-2. Dimetrosky, S., Steiner, J., & Vellinga, N. (2006). San Diego Gas & Electric 2004-2005

 Local Energy Savers Program Evaluation Report (Study ID: SDG0212). Portland, OR:
 Quantec

 http://www.calmac.org/publications/SDGE_ESP_EMV_Report_073106_Final.pdf
- 4-3. Greenberg, D. (2004). Network Power Management Software: Saving Energy by Remote Control (E source report No. ER-04-15). Boulder, CO: Platts Research & Consulting.
- 5.4. Roth, K., Larocque, G., & Kleinman, J. (2004). Energy Consumption by Office and Telecommunications Equipment in Commercial Buildings Volume II: Energy Savings Potential (U.S. DOE contract No. DE-AM26-99FT40465). Cambridge, MA: TIAX LLC. http://www.eere.energy.gov/buildings/info/documents/pdfs/office telecom-vol2 final.pdf

3.9.3 SMART STRIP PLUG OUTLETS

Measure Name	Smart Strip Plug Outlets
Target Sector	Commercial and Industrial Establishments
Measure Unit	Smart Strip Plug Outlet
Unit Energy Savings	<u>Fixed</u>
Unit Peak Demand Reduction	<u>Fixed</u>
Measure Life	<u>5 years⁵⁷⁹</u>
Measure Vintage	Retrofit

Smart Strips are power strips that contain a number of controlled sockets with at least one uncontrolled socket. When the appliance that is plugged into the uncontrolled socket is turned off, the power strips then shuts off the items plugged into the controlled sockets. Qualified power strips must automatically turn off when equipment is unused / unoccupied.

ELIGIBILITY

This protocol documents the energy savings attributed to the installation of smart strip plugs. The most likely area of application is within commercial spaces such as isolated workstations and computer systems with standalone printers, scanners or other major peripherals that are not dependent on an uninterrupted network connection (e.g. routers and modems).

ALGORITHMS

The DSMore Michigan Database of Energy Efficiency Measures performed engineering calculations using standard standby equipment wattages for typical computer and TV systems and idle times. This commercial protocol will use the computer system assumptions except it will utilize a lower idle time for commercial office use.

The computer system usage is assumed to be 10 hours per day for 5 workdays per week. The average daily idle time including the weekend (2 days of 100% idle) is calculated as follows:

Average daily commercial computer system idle time
$$\frac{16.86 \ hours}{16.80 \ hours} = \frac{\frac{Hours \ per \ week - \ (workdays \times daily \ computer \ usage)}{days \ per \ week}}{7 \ days}$$

The energy savings and demand reduction were obtained through the following calculations:

$$\Delta kWh = \left(kW_{comp} \times Hr_{comp}\right) \times 365 = \mathbf{123.69}kWh \ (rounded \ to \ \mathbf{124}kWh)$$

$$\Delta kW_{peak} = CF \times kW_{comp} = \mathbf{0.0101}kW$$

579 DSMore Michigan Database. http://michigan.gov/mpsc/0,4639,7-159-52495_55129---,00.html

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DEFINITION OF TERMS

The parameters in the above equation are listed below.

Table 3-148: Smart Strip Calculation Assumptions

<u>Term</u> Component	<u>Unit</u> Type	Value <u>s</u>	Source
kW_{comp} , Idle kW of computer system	<u>kW</u> Fixed	0.0201	1
Hr_{comp} , Daily hours of computer idle time	<u>Hours</u> Fixed	16.86	1
CF, Coincidence Factor	None Fixed	0.50	1

DEEMEDSAVINGS

 $\begin{array}{ll} \Delta kWh & = 124 \ kWh \\ \Delta kW_{peak} & = 0.0101 \ kW \end{array}$

MEASURE LIFE

To ensure consistency with the annual savings calculation procedure used in the DSMore MI database, the measure of **5 years** is taken from DSMore.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures. The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings.

SOURCES

1. DSMore Michigan Database of Energy Efficiency Measures. http://michigan.gov/mpsc/0,4639,7-159-52495 55129---,00.html

3.10 COMPRESSED AIR

3.10.1 CYCLING REFRIGERATED THERMAL MASS DRYER

Measure Name	Cycling Refrigerated Thermal Mass Dryer
Target Sector	Commercial and Industrial Establishments
Measure Unit	Cycling Refrigerated Thermal Mass Dryer
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	10 years ⁵⁸⁰
Measure Vintage	Early Replacement

When air is compressed, water vapor in the air condenses and collects in liquid form. Some of this condensate collects in the air distribution system and can contaminate downstream components such as air tools with rust, oil, and pipe debris. Refrigerated air dryers remove the water vapor by cooling the air to its dew point and separating the condensate. Changes in production and seasonal variations in ambient air temperature lead to partial loading conditions on the dryer. Standard air dryers use a hot gas bypass system that is inefficient at partial loads. A Cycling Thermal Mass Dryer uses a thermal storage medium to store cooling capacity when the system is operated at partial loads allowing the dryer refrigerant compressor to cycle.

ELIGIBILITY

This measure is targeted to non-residential customers whose equipment is a non-cycling refrigerated air dryer with a capacity of 600 cfm or below.

Acceptable baseline conditions are a non-cycling (e.g. continuous) air dryer with a capacity of 600 cfm or below. The replacement of desiccant, deliquescent, heat-of-compression, membrane, or other types of dryers does not qualify under this measure.

Efficient conditions are a cycling thermal mass dryer with a capacity of 600 cfm or below.

ALGORITHMS

$$\Delta kWh = ((CFM \times HP_{compressor} \times \frac{CFM_{comp.}}{kW_{dryer}} \times HOURS \times (1 - APC)) \times RTD)$$

$$\Delta kW_{peak} = \frac{\Delta kWh}{HOURS} * CF$$

⁵⁸⁰Based on market activity as reported by several compressed air equipment vendors. See "Compiled Data Request Results.xls", Efficiency Vermont, Technical Reference Manual 2013-82 for details.

http://www.greenmountainpower.com/upload/photos/371TRM_User_Manual_No_2013-82-5-protected.pdf

DEFINITION OF TERMS

Table 3-149: Cycling Refrigerated Thermal Mass Dryer – Values and References

Term	Unit	Values	Source
CFM , Compressor output per hp	$\frac{CFM}{hp}$	EDC Data Gathering Default: 4	EDC Data Gathering
$HP_{compressor}$, Nominal hp rating of the air compressor motor	HP	Nameplate data	EDC Data Gathering
CFM/kW _{dryer} , Ratio of compressor CFM to dryer kW	$\frac{CFM}{kW}$	EDC Data Gathering Default: 0.0087	EDC Data Gathering 2
RTD , Chilled coil response time derate	Hours	EDC Data Gathering Default: 0.925	EDC Data Gathering 2
APC , Average compressor operating capacity	None	EDC Data Gathering Default: 65%	EDC Data Gathering 3
HOURS , Annual hours of compressor operation	hours year	EDC Data Gathering Default: See Table 3-150 Table 3-150	EDC Data Gathering
CF, Coincidence Factor	None	EDC Data Gathering Default: See <u>Table</u> 3-151 Table 3-151	EDC Data Gathering 5

Table 3-150: Annual Hours of Compressor Operation

Operation Facility Schedule (hours per day / days per week)	HOURS
Single Shift (8/5)	2080
2-Shift (16/5)	4160
3-Shift (24/5)	6240
4-Shift (24/7)	8320

Table 3-151: Coincidence Factors

Coincidence Factor	%
Single Shift (8/5)	66.7
2-Shift (16/5)	100
3-Shift (24/5)	100
4-Shift (24/7)	100

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Manufacturer's data suggests that cfm output per compressor hp ranges from 4 to 5. The lower estimate of 4 will slightly underestimate savings.
- Conversion factor based on a linear regression analysis of the relationship between air compressor full load capacity and non-cycling dryer full load kW assuming that the dryer is sized to accommodate the maximum compressor capacity. See "Compressed Air Analysis.xls" for source calculations, Efficiency Vermont, Technical Reference Manual 2013-82.
 - http://www.greenmountainpower.com/upload/photos/371TRM_User_Manual_No_2013-82-5-protected.pdf
- Based on an analysis of load profiles from 50 facilities using air compressors 40 hp and below. See "BHP Weighted Compressed Air Load Profiles.xls" for source calculations, Efficiency Vermont, Technical Reference Manual 2013-82. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- Hours account for holidays and scheduled downtime. Efficiency Vermont, Technical Reference Manual 2013-82. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- Efficiency Vermont, Technical Reference Manual 2013-82. Compressed Air Loadshape calcs (compressed_air_loadshape_calc_1-4_shifts.xls). The CF is drawn from the summer period, which is when the PA peak kW peak is calculated. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf

SECTION 3: Commercial and Industrial Measures

3.10.2 AIR-ENTRAINING AIR NOZZLE

Measure Name	Air-entraining Air Nozzle
Target Sector	Commercial and Industrial Establishments
Measure Unit	Air-entraining Air Nozzle
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	15 years ⁵⁸¹
Measure Vintage	Early Replacement

Air entraining air nozzles use compressed air to entrain and amplify atmospheric air into a stream, increasing pressure with minimal compressed air use. This decreases the compressor work necessary to provide the nozzles with compressed air. Air entraining nozzles can also reduce noise in systems with air at pressures greater than 30 psig.

ELIGIBILITY

This measure is targeted to non-residential customers whose compressed air equipment uses stationary air nozzles in a production application with an open copper tube of 1/8" or 1/4" orifice diameter.

Energy efficient conditions require replacement of an inefficient, non-air entraining air nozzle with an energy efficient air-entraining air nozzle that use less than 15 CFM at 100 psi for industrial applications.

ALGORITHMS

$$\begin{array}{ll} \Delta kWh & = (CFM_{base} - CFM_{ee}) \times COMP \times HOURS \times \% \ USE \\ \Delta kW_{peak} & = \frac{\Delta kWh}{HOURS} * CF \end{array}$$

DEFINITION OF TERMS

⁵⁸¹ PA Consulting Group (2009). *Business Programs: Measure Life Study.* Prepared for State of Wisconsin Public Service Commission. https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal_evaluationreport.pdf

Table 3-152: Air-entraining Air Nozzle – Values and References

Term	Unit	Values	Source
$\mathit{CFM}_{\mathit{base}}$, Baseline nozzle air mass flow	$CFM\left(\frac{ft^3}{min}\right)$	EDC Data Gathering Default: See <u>Table</u> 3-153 Table 3-153	1
CFM_{ee} , Energy efficient nozzle air mass flow	$CFM\left(\frac{ft^3}{min}\right)$	EDC Data Gathering Default: See <u>Table</u> 3-154Table 3-154	2
COMP , Ratio of compressor kW to CFM	$\frac{kW}{CFM}$	EDC Data Gathering Default: See <u>Table</u> 3-155 Table 3-155	3
HOURS , Annual hours of compressor operation	hours year	EDC Data Gathering Default: See <u>Table</u> 3-156Table 3-156	4
$\% \ \mathit{USE}$, Percent of hours when nozzle is in use	None	EDC Data Gathering Default: 5%	5
CF, Coincidence Factor	None	EDC Data Gathering Default: See Table 3-157	6

Table 3-153: Baseline Nozzle Mass Flow

Nozzle Diameter	Air Mass Flow (CFM)
1/8"	21
1/4"	58

Table 3-154: Air Entraining Nozzle Mass Flow

Nozzle Diameter	Air Mass Flow (CFM)
1/8"	6
1/4"	11

Table 3-155: Average Compressor kW / CFM (COMP)

Compressor Control Type	Average Compressor kW/CFM (COMP)
Modulating w/ Blowdown	0.32
Load/No Load w/ 1 gal/CFM Storage	0.32
Load/No Load w/ 3 gal/CFM Storage	0.30
Load/No Load w/ 5 gal/CFM Storage	0.28
Variable Speed w/ Unloading	0.23

Table 3-156: Annual Hours of Compressor Operation

Facility Schedule (hours per day / days per week)	HOURS
Single Shift (8/5)	2080
2-Shift (16/5)	4160
3-Shift (24/5)	6240
4-Shift (24/7)	8320

Table 3-157: Coincidence Factor

Coincidence Factor	%
Single Shift (8/5)	66.7%
2-Shift (16/5)	100%
3-Shift (24/5)	100%
4-Shift (24/7)	100%

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- 1. Machinery's Handbook 25th Edition.
- 2. Survey of Engineered Nozzle Suppliers.

SECTION 3: Commercial and Industrial Measures

- 3. Efficiency Vermont, Technical Reference Manual 2013-82. The average compressor kW/CFM values were calculated using DOE part load curves and load profile data from 50 facilities employing compressors less than or equal to 40 hp. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- 5. Assumes 50% handheld air guns and 50% stationary air nozzles. Manual air guns tend to be used less than stationary air nozzles, and a conservative estimate of 1 second of blow-off per minute of compressor run time is assumed. Stationary air nozzles are commonly more wasteful as they are often mounted on machine tools and can be manually operated resulting in the possibility of a long term open blow situation. An assumption of 5 seconds of blow-off per minute of compressor run time is used.
- Efficiency Vermont, Technical Reference Manual 2013-82. Compressed Air Loadshape calcs (compressed_air_loadshape_calc_1-4_shifts.xls). The CF is drawn from the summer period, which is when the PA peak kW peak is calculated. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf

3.10.3 No-Loss Condensate Drains

Measure Name	No-loss Condensate Drains
Target Sector	Commercial and Industrial Establishments
Measure Unit	No-loss Condensate Drains
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	5 years ⁵⁸²
Measure Vintage	Early Replacement

When air is compressed, water vapor in the air condenses and collects in the system. The water must be drained to prevent corrosion to the storage tank and piping system, and to prevent interference with other components of the compressed air system such as air dryers and filters. Many drains are controlled by a timer and are opened for a fixed amount of time on regular intervals regardless of the amount of condensate. When the drains are opened compressed air is allowed to escape without doing any purposeful work. No-loss Condensate Drains are controlled by a sensor that monitors the level of condensate and only open when there is a need to drain condensate. They close before compressed air is allowed to escape.

ELIGIBILITY

This measure is targeted to non-residential customers whose equipment is a timed drain that operates on a pre-set schedule.

Acceptable baseline conditions are compressed air systems with standard condensate drains operated by a solenoid and timer.

Energy efficient conditions are systems retrofitted with new No-loss Condensate Drains properly sized for the compressed air system.

ALGORITHMS

The following algorithms apply for No-loss Condensate Drains.

$$\begin{array}{ll} \varDelta kWh & = ALR \times COMP \times OPEN \times AF \times PNC \\ \Delta kW_{peak} & = \frac{\Delta kWh}{HOURS} * CF \end{array}$$

DEFINITION OF TERMS

⁵⁸² Based on market activity as reported by several compressed air equipment vendors. See "Compiled Air Analysis.xls," Efficiency Vermont TRM 2013 for details.

Term	Unit	Values	Source
ALR, Air Loss Rate; an hourly average rate for the timed drain dependent on drain orifice diameter and system pressure.		EDC Data Gathering Default: See <u>Table</u> 3-159Table 3-159	1
COMP, Compressor kW / CFM; the amount of electrical demand in KW required to generate one cubic foot of air at 100 PSI.		EDC Data Gathering Default: See Table 3-160 Table 3-160	2
OPEN, Hours per year drain is open		EDC Data Gathering Default: 146	3
AF, Adjustment Factor; accounts for periods when compressor is not running and the system depressurizes due to leaks and operation of time drains.	None	EDC Data Gathering Default: See <u>Table</u> 3-161Table 3-161	4
PNC, Percent Not Condensate; accounts for air loss through the drain after the condensate has been cleared and the drain remains open.	None	EDC Data Gathering Default: 0.75	4
HOURS, Annual hours of compressor operation	hours year	EDC Data Gathering Default: See <u>Table</u> 3-162 Table 3-162	5
CF, Coincidence Factor	%	EDC Data Gathering Default: See Table 3-163 Table 3-163	6

Rev Date: June 2015

Table 3-159: Average Air Loss Rates (ALR)

Pressure	Orifice Diameter (inches)					
(psig)	1/64	1/32	1/16	1/8	1/4	3/8
70	0.29	1.16	4.66	18.62	74.4	167.8
80	0.32	1.26	5.24	20.76	83.1	187.2
90	0.36	1.46	5.72	23.1	92	206.6
95	0.38	1.51	6.02	24.16	96.5	216.8
100	0.4	1.55	6.31	25.22	100.9	227
105	0.42	1.63	6.58	26.31	105.2	236.7
110	0.43	1.71	6.85	27.39	109.4	246.4
115	0.45	1.78	7.12	28.48	113.7	256.1

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Pressure (psig)	Orifice Diameter (inches)					
(psig)	1/64	1/32	1/16	1/8	1/4	3/8
120	0.46	1.86	7.39	29.56	117.9	265.8
125	0.48	1.94	7.66	30.65	122.2	275.5

For well-rounded orifices, values should be multiplied by 0.97. For sharp orifices, values should be multiplied by 0.61.
When the baseline value is unknown, use 100.9 CFM⁵⁸³.

Table 3-160: Average Compressor kW / CFM (COMP)

Compressor Control Type	Average Compressor kW/CFM (COMP)
Modulating w/ Blowdown	0.32
Load/No Load w/ 1 gal/CFM Storage	0.32
Load/No Load w/ 3 gal/CFM Storage	0.30
Load/No Load w/ 5 gal/CFM Storage	0.28
Variable Speed w/ Unloading	0.23

Table 3-161: Adjustment Factor (AF)

Compressor Operating Hours	AF
Single Shift – 2080 Hours	0.62
2-Shift – 4160 Hours	0.74
3-Shift – 6240 Hours	0.86
4-Shift – 8320 Hours	0.97

Table 3-162: Annual Hours of Compressor Operation

Facility Schedule (hours per day / days per week)	HOURS
Single Shift (8/5)	2080

⁵⁸³ Ibid.

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Compressed Air

Facility Schedule (hours per day / days per week)	HOURS
2-Shift (16/5)	4160
3-Shift (24/5)	6240
4-Shift (24/7)	8320

Table 3-163: Coincidence Factor

Coincidence Factor	%
Single Shift (8/5)	66.7%
2-Shift (16/5)	100%
3-Shift (24/5)	100%
4-Shift (24/7)	100%

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- US DOE Compressed Air Tip Sheet #3, August 2004, from Fundamentals for Compressed Air Systems Training offered by the Compressed Air Challenge. http://www1.eere.energy.gov/manufacturing/tech assistance/pdfs/compressed air3.pdf
- The average compressor kW/CFM values were calculated using DOE part load curves and load profile data from 50 facilities employing compressors less than or equal to 40 hp. Efficiency Vermont, Technical Reference Manual 2013-82. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- Assumes 10 seconds per 10 minute interval. Efficiency Vermont, Technical Reference Manual 2013-82. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf
- Based on observed data. Efficiency Vermont, Technical Reference Manual 2013-82. http://www.greenmountainpower.com/upload/photos/371TRM_User_Manual_No_2013-82-5-protected.pdf
- Accounts for holidays and scheduled downtime. Efficiency Vermont, Technical Reference Manual 2013-82.

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http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf

6. Efficiency Vermont, Technical Reference Manual 2013-82. Compressed Air Loadshape calcs (compressed_air_loadshape_calc_1-4_shifts.xls). The CF is drawn from the summer period, which is when the PA peak kW peak is calculated. http://www.greenmountainpower.com/upload/photos/371TRM User Manual No 2013-82-5-protected.pdf

Compressed Air

SECTION 3: Commercial and Industrial Measures

3.11 MISCELLANEOUS

3.11.1 ENERGY STAR SERVERS

Measure Name	ENERGY STAR Servers
Target Sector	Commercial and Industrial Establishments
Measure Unit	Variable
Unit Energy Savings	Variable
Unit Peak Demand Reduction	Variable
Measure Life	4 years ⁵⁸⁴
Measure Vintage	Replace on Burnout

ELIGIBILITY

This measure applies to the replacement of existing servers in a data center or server closet with new ENERGY STAR servers of similar computing capacity. On average, ENERGY STAR servers are 30% more efficient than standard servers. The servers operate particularly efficiently at low loads due to processor power management requirements that reduce power consumption when servers are idle.

ALGORITHMS 585

$$kW_{es} = \sum_{ES=1}^{n} kW_{es,idle} + \left[U_{es} \times \left(\frac{kW_{es,idle}}{b} - kW_{es,idle}\right)\right]$$

$$\frac{\Delta kWh}{yr} = \left[\frac{1}{(1-a)} - 1\right] \times kW_{es} \times 8,760 \frac{hours}{year}$$

$$= \left[\frac{1}{(1-a)} - 1\right] \times kW_{es}$$

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⁵⁸⁴ The three International Data Corporation (IDC) studies indicate organizations replace their servers once every three to five vears (Source 8)

years(Source 8)

585 The energy consumption and savings algorithms represented in this section were derived from the Uniform Methods Project:
Methods for Determining Energy-Efficiency Savings for Specific Measures, Draft Data Center IT Measures, 2013.

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Table 3-164: ENERGY STAR Server Measure Assumptions

Term	Unit	Values	Source
$kW_{\it es,idle}$, Power draw of ENERGY STAR server in idle mode	kW	EDC Data Gathering	1
U_{es} , utilization of ENERGY STAR server	None	EDC Data Gathering Default: See <u>Table</u> 3-165Table 3-165	EDC Data Gathering 2,3,4
a, percentage ENERGY STAR server is more efficient than "standard" or "typical" unit	None	Fixed = 30% or most current ENERGY STAR specification	5
b, ratio of idle power to full load power for an ENERGY STAR server	None	EDC Data Gathering Default: See Table 3-166 Table 3-166	EDC Data Gathering
n, number of ENERGY STAR servers	Servers	EDC Data Gathering	EDC Data Gathering
ΔkW_{peak} , peak demand savings	kW	Calculated per algorithm	7

Table 3-165: ENERGY STAR Server Utilization Default Assumptions

Server Category	Installed Processors	<i>U_{es}</i> (%)
A, B	1	15%
C, D	2	40%

Table 3-166: ENERGY STAR Server Ratio of Idle Power to Full Load Power Factors

Server Category	Installed Processors	Managed Server ⁵⁸⁶	Ratio of ES Idle/ES Full Load (b)
Α	1	No	52.1%
В	1	Yes	53.2%
С	2	No	61.3%
D	2	Yes	55.8%

⁵⁸⁶ Managed Server: A computer server that is designed for a high level of availability in a highly managed environment. For purposes of this specification, a managed server must meet all of the following criteria (from ENERGY STAR server specification 2.0):

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A. is designed to be configured with redundant power supplies; and B. contains an installed dedicated management controller (e.g., service processor).

DEFAULT SAVINGS

Default savings may be claimed using the algorithms above and the variable defaults. EDCs may also claim savings using customer specific data.

EVALUATION PROTOCOLS

When possible, perform M&V to assess the energy consumption. However, where metering of IT equipment in a data center is not allowed, follow the steps outlined.

- Invoices should be checked to confirm the number and type of ENERGY STAR servers purchased.
- If using their own estimate of active power draw, kW_{energy star}, the manager should provide a week's worth of active power draw data gathered from the uninterruptible power supply, PDUs, in-rack smart power strips, or the server itself.
- Idle power draws of servers, kW_{es,idle}, should be confirmed in the "Idle Power Typical or Single Configuration (W)" on the ENERGY STAR qualified product list⁵⁸⁷.
- If not using the default values listed in <u>Table 3-165Table 3-165Table 1-2</u>, utilization rates should be confirmed by examining the data center's server performance software.

SOURCES

- 1. An ENERGY STAR qualified server has an "Idle Power Typical or Single Configuration (W)" listed in the qualified product list for servers. The EDC should use the server make and model number to obtain the kW_{es,idle} variable used in the algorithms. The ENERGY STAR qualified server list is located at here: http://www.energystar.gov/productfinder/product/certified-enterprise-servers/results
- Utilitization of a server can be derived from a data center's server performance software.
 This data should be used, instead of the default values listed in <u>Table 3-166Table 3-166Table 1-3</u>, when possible.
- 3. The estimated utilization of the ENERGY STAR server for servers with one processor was based on the average of two sources, as follows.
 - Glanz, James. Power Pollution and The Internet, The New York Times, Spetember 22, 2012. This articile cited to sources of average utilization rates between 6 to 12%.
 - b. Stakeholders interviewed during the development of the ENERGY STAR server specification reported that the average utilization rate for servers with 1 processor is approximately 20%.
- The estimated utilization of the ENERGY STAR server for servers with two processor was based on the average of two sources, as follows.
 - a. Using Virtualization to Improve Data Center Efficiency, Green Grid White Paper, Editor: Richard Talaber, VMWare, 2009. A target of 50% server utilization is recommended when setting up a virtual host.

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⁵⁸⁷ http://www.energystar.gov/productfinder/product/certified-enterprise-servers/results

- b. Stakeholders interviewed during the development of the ENERGY STAR server specification reported that the average utilization rate for servers with two processors is approximately 30%.
- The default percentage savings on the ENERGY STAR server website was reported to be 30% on May 20th, 2014.
- 6. In December 2013, ENERGY STAR stopped including full load power data as a field in the ENERGY STAR certified product list. In order to full load power required in the Uniform Methods Project algorithm for energy efficient servers, a ratio of idle power to full load power was estimated. The idle to full load power ratios were estimated based on the ENERGY STAR qualitifed product list from November 18th, 2013. The ratios listed in Table 3-166Table 3-166_Table 1-3 are based on the average idle to full load ratios for all ENERGY STAR qualified servers in each server category.
- 7. The coincident peak demand factor was assumed to be 100% since the servers operate 24 hours per day, 365 days per year and the demand reduction associated with this measure is constant.
- 8. The three International Data Corporation (IDC) studies indicate organizations replace their servers once every three to five years
 - a. IDC (February 2012). "The Cost of Retaining Aging IT Infrastructure." Sponsored by HP. Online. http://mjf.ie/wp-content/uploads/white-papers/IDC-White-Paper the-cost-of-retaining-aging-IT-infrastructure.pdf
 - IDC (2010). "Strategies for Server Refresh." Sponsored by Dell. Online. http://i.dell.com/sites/content/business/smb/sb360/en/Documents/server-refresh-strategies.pdf
 - c. DC (August 2012). "Analyst Connection: Server Refresh Cycles: The Costs of Extending Life Cycles." Sponsored by HP/Intel. Online. http://resources.itworld.com/ccd/assets/31122/detaill

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4 AGRICULTURAL MEASURES

The following section of the TRM contains savings protocols for agricultural measures that apply to both residential and commercial & industrial sector.

4.1 AGRICULTURAL

4.1.1 AUTOMATIC MILKER TAKEOFFS

Measure Name	Automatic Milker Takeoffs
Target Sector	Agriculture (includes Small Commercial, Residential and Commercial)
Measure Unit	Per Project-Milker Takeoff
Unit Energy Savings	Varies, partially deemedVariable
Unit Peak Demand Reduction	Varries, partially deemedVariable
Measure Life	10 years ⁵⁸⁸
Measure Vintage	Retrofit

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of automatic milker takeoffs on dairy milking vacuum pump systems. Automatic milker takeoffs shut off the suction on teats once a minimum flow rate is achieved. This reduces the load on the vacuum pump.

This measure requires the installation of automatic milker takeoffs to replace pre-existing manual takeoffs on dairy milking vacuum pump systems. Equipment with existing automatic milker takeoffs is not eligible. In addition, the vacuum pump system serving the impacted milking units must be equipped with a variable speed drive (VSD) to qualify for incentives. Without a VSD, little or no savings will be realized.

ALGORITHMS

The annual energy savings are obtained through the following formulas:

⁵⁸⁸ Idaho Power Demand Side Management Potential Study – Volume II Appendices, Nexant, 2009.-

DEFINITION OF TERMS

Table 4-1: Variables for Automatic Milker Takeoffs

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
COWS, Number of cows milked per day	CowsVariableCows	Based on customer application	EDC Data Gathering
MPD, Number of milkings per day per cow	MilkingsVariable	Based on customer application	EDC Data Gathering
1		Default <u>:</u> == 2	1
ESC, Energy Savings per cow per year	<u>kWh·yr</u> cow Fixed	7.5 kWh/cow/yr	1, 2, 3, 4, 5,6
CF, Demand Coincidence factor	<u>None</u> Fixed	0.00014	6

DEFAULT SAVINGS

There are no default savings for this protocol.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for automatic milker takeoffs is 10 years. 589

SOURCES

- 1. The ESC was calculated based on the following assumptions:
 - Average herd size is 75 cows in PA (Source 2)
 - The typical dairy vacuum pump size for the average herd size is 10 horsepower
 - Based on the herd size, average pump operating hours are estimated at 8 hours per day (Source 4)
 - A 12.5% annual energy saving factor (Source 5)
- Average Pennsylvania Herd Size information came from: Hoard's Dairyman: The National Dairy Farm Magazine, March 2013. U.S. Industry Dairy Statistics, 2007-2013_a presentation_by_David W. Kammel: "Dairy Modernization: Growing Pennsylvania Family

⁵⁸⁹ Idaho Power Demand Side Management Potential Study - Volume II Appendices, Nexant, 2009.

<u>Dairy Farms"</u>, <u>Biological Systems Engineering</u>, <u>University of Wisconsin http://www.padairysummit.org/ files/live/Dairy Modernization PA Program 2014 Final. pdf</u>

- Average dairy vacuum pump size was estimated based on the Minnesota Dairy Project literature. http://www.mnproject.org/resourcecenter/Vacuum%20System%20options.pdf
- 4. Annual pump operating hours were is based on the assumption that 15-20 cows are milked per hour and two milkings occur per day.
- 5. Savings are based on the assumption that automatic milker take-offs eliminate open vacuum pump time associated with milker take-offs separating from the cow or falling off during the milking process. The following conservative assumptions were made to determine energy savings associated with the automatic milker take-offs:
 - There is 30 seconds of open vacuum pump time for every 8 cows milked.
 - The vacuum pump has the ability to turn down during these open-vacuum pump times from a 90% VFD speed to a 40% VFD speed.
 - Additionally, several case studies from the Minnesota Dairy Project include dairy pump VFD and automatic milker take-off energy savings and that are estimated at 50-70% pump savings. It is assumed that the pump VFD savings are 46%, therefore the average remaining savings can be attributed to automatic milker take-offs.

http://www.mnproject.org/resourcecenter/Vacuum%20System%20options.pdf

- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Agricultural: Variable Frequency Drives-Dairy, FY2012, V1.2.
 Accessed from RTF website http://rtf.nwcouncil.org/measures/Default.asp on February 27, 2013.
- Because this measure results in vacuum pump energy savings, the load profile and coincidence factor described in the VSD Controller on Dairy Vacuum Pump TRM protocol also apply to this measure.

4.1.2 DAIRY SCROLL COMPRESSORS

Measure Name	Dairy Scroll Compressors
Target Sector	Agriculture (includes Small Commercial, Residential and Commercial)
Measure Unit	Per cCompressor
Unit Energy Savings	Varies-Variable
Unit Peak Demand Reduction	<u>Variable</u> Varies
Measure Life	15 years ⁵⁹⁰
Measure Vintage	Replace on Burnout or New Construction

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of a scroll compressor to replace an existing reciprocating compressor or the installation of a scroll compressor in a new construction application. The compressor is used to cool milk for preservation and packaging. The energy and demand savings per cow will depend on the installed scroll compressor energy efficiency ratio (EER), operating days per year, and the presence of a precooler in the refrigeration system.

This measure requires the installation of a scroll compressor to replace an existing reciprocating compressor or to be installed in a new construction application. Existing farms replacing scroll compressors are not eligible.

ALGORITHMS

The energy and peak demand savings are dependent on the presence of a precooler in the system, and are obtained through the following formulas:

Annual Savings without a Precooler

$ \Delta kWh \frac{AkWh_{no\ precool}}{yr} \times COWS $	$= \left(\frac{CBTU_{\text{no precool}}}{EER_{base}} - \frac{CBTU_{\text{no precool}}}{EER_{ee}}\right) \times \frac{1 \cdot kW}{1000 \cdot W} \times HRS \times DAYS \times COWS$
$\Delta kW_{peak, no precool}$	$= \Delta kW h_{no\ precool} \times CF$

Annual Savings with a Precooler

$$\frac{\Delta kWh_{precool}}{yr} \times COWS = \frac{\begin{pmatrix} CBTU_{precool} & CBTU_{precool} \\ EER_{base} & EER_{ee} \end{pmatrix} \times \frac{1 \cdot kW}{1000 \cdot W} \times HRS \times DAYS}{4 \cdot kW}$$

$$\Delta kW_{peak,precool} = \Delta kWh_{precool} \times CF$$

https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal_evaluationreport.pdf

⁵⁹⁰PA Consulting Group for the State of Wisconsin Public Service Commission, Focus on Energy Evaluation. Business Programs: Measure Life Study. August 25, 2009. Appendix B

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DEFINITION OF TERMS

Table 4-2: Variables for Dairy Scroll Compressors

<u>ComponentTerm</u>	Type Unit	Value <u>s</u>	Source
$\it EER_{base}$, Baseline compressor efficiency	Variable Default_None	Baseline compressor manufacturers data based upon customer application	EDC Data Gathering
		Default: 5.85	1
$\it EER_{ee}$, Installed compressor efficiency	Variable None	From Nnameplate	EDC Data Gathering
CBTU, CBTU _{no preceet} , H_Heat load of milk per cow per day for a given refrigeration system-with no-preceeder	Btu Fixed	System without precooler: 2,864 System with precooler: 922	2, 3
CBTU _{precent} , Heat load of milk percew per day for a refrigeration-system with a precooler	Fixed-Btu Cow-day	922	2, 3
HRS, Operating hours per day	Variable hours day	Customer Customer application	EDC Data Gathering
	Default	Default: 8 hours	4
DAYS, Milking days per year	Variable	Based on customer application	EDC Data Gathering
	Default	Default: 365 days/year	3, 4
COWS, Average number of cows milked per day	Variable Cows	Based on customer application	EDC Data Gathering
CF, Demand Coincidence factor	None-	0.00014	5

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for dairy scroll compressors is 15 years 594.

⁵⁹¹PA Consulting Group for the State of Wisconsin Public Service Commission, Focus on Energy Evaluation. Business Programs: Measure Life Study. August 25, 2009. Appendix B

SOURCES

- Based on the average EER data for a variety of reciprocating compressors from Emerson Climate Technologies. http://www.emersonclimate.com/en-us/Pages/default.aspx
- 2. Based on a specific heat value of 0.93 $\frac{Btu}{lb^{\circ}F}$ and density of 8.7 lb/gallon for whole milk. American Society of Heating Refrigeration and Air-conditioning Engineers Refrigeration Handbook, 2010, Ch.19.5.
- 3. Based on delta T (temperature difference between the milk leaving the cow and the cooled milk in tank storage) of 59_°F for a system with no pre-cooler and 19_°F for a system with a pre-cooler. It was also assumed that an average cow produces 6 gallons of milk per day. Evaluation of Alliant Energy Agriculture Program, Appendix F, 2008. KEMA 2009 Evaluation of IPL Energy Efficiency Programs, Appendix F, pg. 347. http://alliantenergy.com/wcm/groups/wcm_internet/@int/documents/document/mdaw/mde y/~edisp/012895.pdf
- 4. Based on typical dairy parlor operating hours referenced for agriculture measures in California. California Public Utility Commission. Database for Energy Efficiency Resources (DEER) 2005. The DEER database assumes 20 hours of operation per day, but is based on much larger dairy farms (e.g. herd sizes > 300 cows). Therefore, the DEER default value was lowered to 8 hours per day, as the average herd size is 75 cows in Pennsylvania.
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Agricultural: Variable Frequency Drives-Dairy, FY2012, V1.2.
 Accessed from RTF website http://rtf.nwcouncil.org/measures/Default.asp on February 27, 2013.
- 5. Because the scroll compressor operates on the same operating schedule as the dairy vacuum pump equipment, the load profile and coincidence factor described in the VSD Controller on Dairy Vacuum Pump TRM protocol can be applied to this measure. However, it should be noted that the compressor and refrigeration system are thermostatically controlled and will cycle on and off during the peak demand period.

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4.1.3 HIGH EFFICIENCY VENTILATION FANS WITH AND WITHOUT THERMOSTATS

Measure Name	High Efficiency Ventilation Fans with and without Thermostats
Target Sector	Agriculture, Large Commercial, Small Commercial, Residential (includes Residential and Commercial)
Measure Unit	Per fF_an
Unit Energy Savings	Varies_ Variable
Unit Peak Demand Reduction	Varies_Variable
Measure Life	10 years ⁵⁹²
Measure Vintage	Replace on Burnout or New Construction

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of high efficiency ventilation fans to replace standard efficiency ventilation fans or the installation of a high efficiency ventilation fans in a new construction_application. The high efficiency fans move more cubic feet of air per watt than compared to standard efficiency ventilation fans. Adding a thermostat control will reduce the number of hours that the ventilation fans operate. This protocol does not apply to circulation fans.

This protocol applies to: (1) the installation of high efficiency ventilation fans in either new construction or retrofit applications where standard efficiency ventilation fans are replaced, and/or (2) the installation of a thermostat controlling either new efficient fans or existing fans. Default values are provided for dairy and swine applications. Other facility types are eligible; however, data must be collected for all default values.

ALGORITHMS

The annual energy savings are obtained through the following formulas:

$\Delta kWh_{fan} \frac{\Delta kWh_{fan}}{yr}$	$= Qty_{std} \times \left(\frac{1}{Eff_{std}}\right) \times CFM \times hours \times \frac{1}{1,000} - Qty_{high} \times \left(\frac{1}{Eff_{high}}\right) \times CFM \times hours \times \frac{1}{1,000}$
$\Delta kWh_{tstat} \frac{\Delta kWh_{tstat}}{yr}$	$= \left(\frac{1}{Eff_{installed}}\right) \times CFM \times hours_{tstat} \times \frac{1}{1,000}$
$\Delta kWh_{total} \frac{\Delta kWh_{total}}{yr}$	$= \Delta kW h_{fan} + \Delta kW h_{tstat}$
ΔkW_{peak}	$= \Delta kW h_{fan} \times CF$

DEFINITION OF TERMS

⁵⁹² New York Standard Approach for Estimating Energy Savings from Energy Efficiency Measures in Commercial and Industrial Programs, September 1, 2009, based on DEER.

http://www3.dps.ny.gov/W/PSCWeb.nsf/0/06f2fee55575bd8a852576e4006f9af7/\$FILE/TechManualNYRevised10-15-10.pdf

Table 4-3: Variables for Ventilation Fans

	Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
	Qty_{std} , Quantity of the standard efficiency fans	Variable None	Based on customer application	EDC Data Gathering
	Qty_{high} , Quantity of high efficiency fans that were installed	Variable None	Based on customer application	EDC Data Gathering
	Eff_{std} , Efficiency of the standard	Variable M	Based on customer application	EDC Data Gathering
	efficiency fan at a static pressure of 0.1 inches water	variable _W	Default values in <u>Table</u> 4-4Table 4-4	1
	Eff_{high} , Efficiency of the high efficiency fan at a static pressure of 0.1 inches water	$\frac{Variable\frac{cfm}{W}}{}$	Based on customer application. Collect the-efficiency at static pressure of 0.1 inches water	EDC Data Gathering, 2, 3
			Default values in <u>Table</u> 4-4Table 4-4	1, 2, 3
	$Eff_{installed}$, Efficiency at a static		Based on customer application. Collect the efficiency at 0.1 inches water	EDC Data Gathering, 2, 3
]] [pressure of 0.1 inches water for the installed fans controlled by the thermostat		Default values in <u>Table</u> <u>4-4Table 4-4</u> . If fans were not replaced, use the default values for Eff_{std} . If fans were replaced, use the default values for Eff_{high} .	1, 2, 3
	hours, operating hours per year	Hermal/eviable	Based on customer application	EDC Data Gathering
ļ	of the fan without thermostat	<u>Hours</u> Variable	Default use values in Table 4-5	1, 4
	CFM, cubic feet per minute of air movement	$\frac{ft^3}{min}$ Variable	Based on customer application. This can vary for pre- and post-installation if the information is known for the pre-installation case.	EDC Data Gathering
			Default values in <u>Table</u> <u>4-4</u> Table 4-4	1
	$hours_{tstat}$, reduction in operating hours of the fan due to the thermostat	<u>Hours</u> Variable	Default values in Table 4-6	4
	1,000, watts per kilowatt	$\frac{watts}{kilowatt}$	1,000	Conversion Factor
	CF, demand coincidence factor	NoneFixed	0.000197	Engineering calculations5

Table 4-4: Default values for standard and high efficiency ventilation fans for dairy and swine facilities

Fan Size (inches)	High Efficiency Fan (cfm/W at 0.1 inches water)	Standard Efficiency Fan (cfm/W at 0.1 inches water)	CFM
14-23	12.4	9.2	3,600
24-35	15.3	11.2	6,274
36-47	19.2	15.0	10,837
48 - 61	22.7	17.8	22,626

Table 4-5. Default Hours for Ventilation Fans by Facility Type by Location (No Thermostat)

Facility Type	Allentown	Erie	Harrisburg	Philadelphia	Pittsburgh	Scranton	Williamsport
Dairy - Stall Barn	5,071	4,807	5,163	5,390	5,010	4,843	5,020
Dairy – Free-Stall or Cross-Ventilated Barn	3,299	2,984	3,436	3,732	3,231	2,985	3,241
Hog Nursery or Sow House	5,864						
Hog Finishing House	4,729						

Table 4-6. Default Hours Reduced by Thermostats by Facility Type and Location

Facility Type	Allentown	Erie	Harrisburg	Philadelphia	Pittsburgh	Scranton	Williamsport
Dairy - Stall Barn	3,457	3,458	3,367	3,285	3,441	3,594	3,448
Dairy – Free- Stall or Cross- Ventilated Barn	1,685	1,635	1,640	1,627	1,662	1,736	1,669
Hog Nursery or Sow House	2,629	2,985	2,323	2,179	2,732	2,885	2,666
Hog Finishing House*	0	0	0	0	0	0	0

^{*}_Hog finishing houses base ventilation needs are based on humidity—: therefore a thermostat will not reduce the number of hours the fans operate.

DEFAULT SAVINGS

There are no default savings for this measure.

MEASURE LIFE

The measure life for high efficiency ventilation fans is 10 years. The measure life for the thermostat is 11 years.593

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

The most appropriate evaluation protocol for this measure is verification of installation coupled with assignment of stipulated energy savings. For building types which are not covered under this protocol, operating hours should also be verified.

SOURCES

- KEMA. 2009 Evaluation of IPL Energy Efficiency Programs, Appendix F, 2008. See Table H-5. http://alliantenergy.com/wcm/groups/wcm_internet/@int/documents/document/mdaw/mde-y/~edisp/012895.pdf
- Pennsylvania State University. Tunnel Ventilation for Tie Stall Dairy Barns. 2004. Downloaded from http://pubs.cas.psu.edu/freepubs/pdfs/g78.pdf. Static pressure reference point for dairy barns comes from page 3. The recommended static pressure is 0.125 to 0.1 inches water
- 3. Iowa State University. Mechanical Ventilation Design Worksheet for Swine Housing. 1999. Downloaded from http://www.extension.iastate.edu/Publications/PM1780.pdf. Static pressure reference point for swine housing comes from page 2. The recommended static pressure is 0.125 to 0.1 inches water for winter fans and 0.05 to 0.08 -inches water for summer fans. A static pressure of 0.1 inches water was assumed for dairy barns and swine houses as it is a midpoint for the recommended values.
- 4. Based on the methodology in KEMA's evaluation of the Alliant Energy Agriculture Program (reference—Source_1). Updated the hours for dairies and thermostats using TMY3 temperature data for PA, as fan run time is dependent on ambient dry-bulb temperature. For a stall barn, it was assumed 33% of fans are on 8,760 hours per year, 67% of fans are on when the temperature is above 50 degrees Fahrenheit, and 100% of the fans are on when the temperature is above 70 degrees Fahrenheit. For a cross-ventilated or free-stall barn, it was assumed 10% of fans are on 8,760 hours per year, 40% of fans are on when the temperature is above 50 degrees Fahrenheit, and 100% of the fans are on when the temperature is above 50 degrees Fahrenheit. The hours for hog facilities are based on humidity. These hours were not updated as the methodology and temperatures for determining these hours was not described in KEMA's evaluation report and could not be found elsewhere. However, Pennsylvania and lowa are in the same ASHRAE climate zone (5A) and so the lowa hours provide a good estimate for hog facilities in Pennsylvania.

⁵⁹³ New York Standard Approach for Estimating Energy Savings from Energy Efficiency-Measures in Commercial and Industrial-Programs, September 1, 2009, based on DEER.

5. The coincident peak demand factor was calculated by dividing the average peak-coincident demand kW reduction by ∆kWh savings for the fans. There are no peak-demand savings for thermostats.

4.1.4 HEAT RECLAIMERS

Measure Name	Heat Reclaimers
Target Sector	Agriculture (includes Small Commercial, Residential and Commercial)
Measure Unit	Heat Reclaimer
Unit Energy Savings	Varies-Variable
Unit Peak Demand Reduction	<u>Variable Varies</u> -
Measure Life	15 year <u>s⁵⁹⁴s⁶⁹⁵</u>
Measure Vintage	Replace on Burnout or New Construction

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of heat recovery equipment on dairy parlor milk refrigeration systems. The heat reclaimers recover heat from the refrigeration system and use it to pre-heat water used for sanitation, sterilization and cow washing.

This measure requires the installation of heat recovery equipment on dairy parlor milk refrigeration systems to heat hot water. This measure only applies to dairy parlors with electric water heating equipment.

The equipment installed must be one of the following pre-approved brands or equivalent: Century-Therm, Fre-Heater, Heat Bank, Sunset, Superheater, and or Therma-Stor.

ALGORITHMS

The energy and peak demand savings are dependent on the presence of a precooler in the refrigeration system, and are obtained through the following formulas:

Annual Savings without a precooler

$$\Delta kWh \frac{\Delta kWh_{no\ precool}}{yr} = \frac{ES_{no\ precool}ES}{\eta_{water\ heater}} \times DAYS \times COWS \times HEF$$

$$\Delta kW_{peak,no\ precool} = \frac{\Delta kWh_{no\ precool}}{\Delta kWh_{no\ precool}} \Delta kWh \times CF$$

Annual Savings with a precooler

$$\frac{\Delta kWh_{precoot}}{yr} = \frac{ES_{precoot}}{\eta_{water neater}} \times DAYS \times COWS \times HEF$$

$$\Delta kW_{peak,precoot} = \Delta kWh_{precoot} \times CF$$

DEFINITION OF TERMS

⁵⁹⁴State of Wisconsin. Focus on Energy Evaluation, Business Program: Measure Life Study Final Report: August 25, 2009. Appendix B. https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal_evaluationreport.pdf
⁶⁹⁵California Public Utility Commission. Database for Energy Efficiency Resources 2008. <u>State of Wisconsin. Focus on Energy-Evaluation, Business Program: Measure Life Study Final Report: August 25, 2009. Appendix B. https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal_evaluationreport.pdf</u> State of Pennsylvania – Technical Reference Manual – Rev Date: June 2015

Table 4-7: Variables for Heat Reclaimers

<u>Term</u> Component	Type Unit	Value <u>s</u>	Source
ES _{no precool} , Energy savings <u>for specified</u> <u>system</u> for systems with no precooler	kWh_ cow∙day	System with precooler: 0.38 System without precooler: 0.29	1,2
ES _{pecault} , Energy savings for systems with a precooler—	Ewh Fixed	0.29	1,2
DAVC Milking days per year	days year Variable days	Based on customer application	EDC Data Gathering
DAYS, Milking days per year	year Default	Default: 365	2
COWS, Average number of cows milked per day	<u>Cows</u> Variable	Based on customer application	EDC Data Gathering
HEF, Heating element factor	Variable None	Heat reclaimers with no back-up heat = 1.0 Heat reclaimers with back-up heating elements = 0.50	3
$\eta_{waterheater}$, Electric water heater efficiency	Variable_ None	Standard electric tank water heater = 0.908 High efficiency electric tank water heater = 0.93 Heat pump water heater = 2.0	4, 5
CF, Demand Coincidence factor	None Variable	0.00014	6

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

SOURCES

- Based on a specific heat value of 0.93 Btu/lb deg F and density of 8.7 lb/gallon for whole milk. American Society of Heating Refrigeration and Air-conditioning Engineers Refrigeration Handbook, 2010, Ch.19.5.
- 2. Based on a delta T (temperature difference between the milk leaving the cow and the cooled milk in tank storage) of 59°F for a system without a pre-cooler and 19°F for a system with a pre-cooler. It was also assumed that a cow produces 6 gallons of milk per day (based on two milkings per day), requires 2.2 gallons of hot water per day, and the water heater delta T (between ground water and hot water) is 70°F. Evaluation of Alliant Energy Agriculture Program, Appendix F, 2008. http://alliantenergy.com/wcm/groups/wcm_internet/@int/documents/document/mdaw/mdey/~edisp/012895.pdf
- 3. Some smaller dairy farms may not have enough space for an additional water storage tank, and will opt to install a heat reclaimer with a back-up electric resistance element. The HEF used in the savings algorithm is a conservative savings de-ration factor to account for the presence of back-up electric resistance heat. The HEF is based on the assumption that the electric resistance element in a heat reclaimer will increase the incoming ground water temperature by 40-50 °F before the water is heated by the heat reclaim coil.
- 4. Standard water heater based on minimum electric water heater efficiencies defined in Table 504.2 of the 2009 International Energy Conservation Code (IECC). High efficiency water heater based on water heater efficiencies defined in <u>Table 3-83: COP Adjustment FactorsTable 3-83: COP Adjustment FactorsTable (COP Adjustment Factors)</u> of the TRM. https://law.resource.org/pub/us/code/ibr/icc.iecc.2009.pdf
- Based on minimum heat pump water efficiencies defined by ENERGY STAR, 2009. https://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=WHH
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Agricultural: Variable Frequency Drives-Dairy, FY2012, V1.2. Accessed from RTF website http://rtf.nwcouncil.org/measures/Default.asp on February 27, 2013.
- 7. Because the water heater operates on the same operating schedule as the dairy vacuum pump equipment and refrigeration equipment, the load profile and coincidence factor described in the VSD Controller on Dairy Vacuum Pump TRM protocol can be applied to this measure.

⁵⁹⁶ California Public Utility Commission. Database for Energy Efficiency Resources 2008.

4.1.5 HIGH VOLUME LOW SPEED FANS

Measure Name	High Volume Low Speed Fans
Target Sector	Agriculture, Large Commercial, Small Commercial, Residential (includes Residential and Commercial)
Measure Unit	Per fFan
Unit Energy Savings	<u>Variable</u> Varies
Unit Peak Demand Reduction	<u>Variable</u> Varies
Measure Life	15 years ⁵⁹⁷
Measure Vintage	Replace on Burnout

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of High Volume Low Speed (HVLS) fans to replace conventional circulating fans. HVLS fans are a minimum of 16 feet long in diameter and move more cubic feet of air per watt than conventional circulating fans. Default values are provided for dairy, poultry, and swine applications. Other facility types are eligible, however, the operating hours assumptions should be reviewed and modified as appropriate.

This measure requires the installation of HVLS fans in either new construction or retrofit applications where conventional circulating fans are replaced.

ALGORITHMS

The annual energy <u>and peak demand</u> savings are obtained through the following formula<u>s</u>:

$$\begin{array}{ll} \Delta & = \times hours \\ CF & = \Delta kWh \times CF \\ & = (OR \ USE \ DEFAULT \ CF \ VALUE \ IN \ TABLE \ 1-1) \\ \Delta kW & = \frac{(W_{conventional} - W_{hvls})}{1,000} \\ \Delta kWh & = \Delta kW \times HOU \\ \Delta kW_{peak} & = \Delta kW \times CF \end{array}$$

⁵⁹⁷ State of Wisconsin. Focus on Energy Evaluation, Business Program: Measure Life Study Final Report: August 25, 2009.
Appendix B. https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal_evaluationreport.pdf

DEFINITION OF TERMS

Table 4-8: Variables for HVLS Fans

Component <u>Term</u>	Type <u>Unit</u>	Value <u>s</u>	Source
$W_{conventional}$, Watts <u>age</u> of the <u>removed</u>	<u>WWatts</u>	Based on customer application	EDC Data Gathering
conventional fans	Variable	Default values in Table 4-9	1 ,2
W_{hvls} , Wattages of the installed HVLS fan	<u>Watts</u> Variable	Based on customer application	EDC Data Gathering
		Default values in Table 4-4	<u>1</u> 2
		Based on customer application	EDC Data Gathering
hoursHOU, annual hours of operation of the fans	<u>Hours</u> Variable	Default values in Table 4-10 Table 4-10Table 4-10Table 4-10	<u>2</u> 3
1000, conversion of watts to kilowatts	watts kilowatts	1,000	Conversion Factor
CF, Demand coincidence factor	None	<u>1.0</u>	2

Table 4-9: Default Values for Conventional and HVLS Fan Wattages

Fan Size (ft)	W _{hvls}	W _{conventional}
≥ 16 and < 18	761	4,497
≥ 18 and < 20	850	5,026
≥ 20 and < 24	940	5,555
≥ 24	1,119	6,613

Table 4-10. Default Hours by Location for Dairy/Poultry/Swine Applications

Location	Hours year
Allentown	2,446
Erie	2,107
Harrisburg	2,717
Philadelphia	2,914
Pittsburgh	2,292
Scranton	2,145
Williamsport	2,371

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for HVLS fans is 15 years

SOURCES

- 1. Bioenvironmental and Structural Systems Laboratory database for circulating fans.
- 2.1. KEMA. 2009 Evaluation of IPL Energy Efficiency Programs, Appendix F Group I Programs Volume 2. See Table H-17.
 - http://alliantenergy.com/wcm/groups/wcm_internet/@int/documents/document/mdaw/mde_y/~edisp/012895.pdf
- 3-2. Number of hours above 65 degrees Fahrenheit. Based on TMY3 data. The coincidence factor has been set at 1.0 as the SWE believes all hours during the peak window will be above 65 degrees Fahrenheit.
- 4.3. Number of hours above 65 degrees Fahrenheit. Based on TMY3 data. The coincidence factor has been set at 1.0 as the SWE believes all hours during the peak window will be above 65 degrees Fahrenheit.

5.4. The coincident peak demand factor is calculated by dividing the peak coincident demand-kW reduction by Δ kWh savings. The peak coincident demand kW reduction is calculated-by $\frac{W_{conventional} - W_{harts}}{1,000}$.

4.1.6 LIVESTOCK WATERER

Measure Name	Livestock Waterer
Target Sector	Agriculture (includes Small Commercial, Residential and Commercial)
Measure Unit	Livestock WatererPer-Project
Unit Energy Savings	<u>Variable</u> Varies
Unit Peak Demand Reduction	0 kW
Measure Life	10 years ⁵⁹⁸
Measure Vintage	Replace on Burnout or New Construction

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of energy-efficient livestock waterers. In freezing climates no or low energy livestock waterers are used to prevent livestock water from freezing. These waterers are closed insulated—watering containers which_that_typically use super insulation, the-relatively warmer ground water temperatures, and the livestock's use of the waterer to keep water from freezing—and thereby capable of watering the livestock.

This measure requires the installation of an energy efficient livestock waterer that is thermostatically controlled and has a minimum of two inches of factory-installed insulation.

ALGORITHMS

The annual energy savings are obtained through the following formula:

$$\Delta kWh \frac{\Delta kWh}{yr} = QTY \times OPRHS \times ESW \times HRT$$

No demand savings are expected for this measure, as the energy savings occur during the winter months.

DEFINITION OF TERMS

⁵⁹⁸ California Public Utility Commission. Database for Energy Efficiency Resources 2005. State of Wisconsin. Focus on Energy Evaluation, Business Program: Measure Life Study Final Report: August 25, 2009. Appendix B. https://focusonenergy.com/sites/default/files/bpmeasurelifestudyfinal evaluationreport.pdf

Table 4-11: Variables for Livestock Waterers

<u>Term</u> Component	Type <u>Unit</u>	Value <u>s</u>	Source
QTY, Number of livestock waterers installed	None Variable	Based on customer application	EDC Data Gathering
OPRHS, Annual operating hours	Hours Fixed	Allentown = 1,489 Erie = 1,768 Harrisburg = 1,302 Philadelphia = 1,090 Pittsburgh = 1,360 Scranton = 1,718 Williamsport = 1,574	1
ESW, Energy Demand Savings perwatererChange in connected load (deemed)	Kilowatts waterer Fixed	0.50 kW per livestock- waterer	2, 3, 4
HRT, % heater run time	NoneFixed	80%	5

Default Savings

There are no default savings for this measure.

Evaluation Protocols

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for a livestock waterer is 10 years. 500

SOURCES

- Based on TMY3 data for various climate zones in Pennsylvania. The annual operating hours represent the annual hours when the outdoor air dry-bulb temperature is less than 32 <u>*deg-</u>F, and it is assumed that the livestock waterer electric resistance heaters are required below this temperature to prevent water freezing.
- 2. Field Study of Electrically Heated and Energy Free Automated Livestock Water Fountains Prairie Agricultural Machinery Institute, Alberta and Manitoba, 1994.

⁵⁹⁹ California Public Utility Commission. Database for Energy Efficiency Resources 2005.

- 3. Facts Automatic Livestock Waterers Fact Sheet, December 2008. http://www1.agric.gov.ab.ca/\$department/deptdocs.nsf/all/agdex5421/\$file/716c52.pdf
- Connecticut Farm Energy Program: Energy Best Management Practices Guide, 2010. http://www.ctfarmenergy.org/Pdfs/CT_Energy_BMPGuide.pdf
- 5.—The Regional Technical Forum (RTF) analyzed metered data from three baseline livestock waterers and found the average run time of electric resistance heaters in the waterers to be approximately 80% for average monthly temperatures similar to Pennsylvania climate zones. This run time factor accounts for warmer make-up water being introduced to the tank as livestock drinking occurs._
- 6-5. Downloaded on May 30th, 2013: http://rtf.nwcouncil.org/measures/measure.asp?id=87

4.1.7 VARIABLE SPEED DRIVE (VSD) CONTROLLER ON DAIRY VACUUM PUMPS

Measure Name	VSD Controller on Dairy Pumps Vacuum Pumps
Target Sector	Agriculture (includes Small Commercial, Residential and Commercial)
Measure Unit	Per dDairy <u>V</u> vacuum Ppump VSD
Unit Energy Savings	<u>Variable</u> Varies
Unit Peak Demand Reduction	Varies-Variable_
Measure Life	15 years ⁶⁰⁰
Measure Vintage	Retrofit

ELIGIBILITY

The following protocol for the calculation of energy and demand savings applies to the installation of a variable speed drive (VSD) and controls on a dairy vacuum pump. The vacuum pump operates during the milk harvest and equipment washing on a dairy farm. The vacuum pump creates negative air pressure that draws milk from the cow and assists in the milk flow from the milk receiver to either the bulk tank or the receiver bowl.

Dairy vacuum pumps are more efficient with VSDs since they tenables the motor to speed up or slow down depending on the pressure demand. The energy savings for this measure is based on pump capacity and hours of use of the pump.

This measure requires the installation of a VSD and controls on dairy vacuum pumps, or the purchase of dairy vacuum pumps with variable speed capability. Pre-existing pumps with VSD's are not eligible for this measure.

ALGORITHMS

The annual energy savings are obtained through the following formulae:

Coincidence Factor

An average of pre and post kW vacuum pump power meter data from five dairy farms in the Pacific Northwest⁶⁰¹ are used to create the vacuum pump demand load profile in Figure 16Figure 16. Because dairy vacuum pump operation does not vary based on geographical location, the average peak demand reduction obtained from these five sites can be applied to Pennsylvania. There are no seasonal variations in cow milking times, as dairy farms milk cows year round, so the load profile below applies to 365 days of operation. The average percent demand reduction

⁶⁰⁰ DEER Effective Useful Life. October 10, 2008.

⁶⁰¹ Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Agricultural: Variable Frequency Drives-Dairy, FY2012, V1.2. Accessed from RTF website http://rtf.nwcouncil.org/measures/Default.asp on February 27, 2013. Pre and post power meter data for five sites were used to establish RTF energy savings for this measure, and raw data used to generate the load profile referenced in this protocol can be found in the zip file on the "BPA Case Studies" tab.

for these five sites during the coincident peak demand period of June through September between noon and 8 pm is 46%, which is consistent with the energy savings factors and demand savings estimated for the sources cited in this protocol.

Based on this data, the demand coincidence factor is estimated by dividing the average peak coincident demand kW reduction by Δ kWh savings for a 1 horsepower motor. The result is a coincidence factor equal to 0.00014.

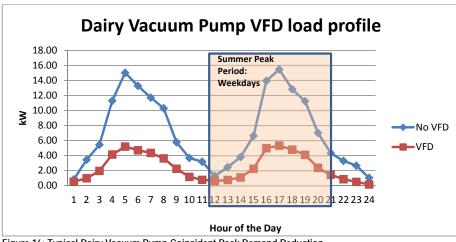


Figure 16: Typical Dairy Vacuum Pump Coincident Peak Demand Reduction

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Table 4-12: Variables for VSD Controller on Dairy Vacuum Pump

	TermComponent-	Unit Tyoe	Value <u>s</u>	Source
	Motor HP, Rated horsepower of the motor	Variable_ Horsepower	Nameplate	EDC Data Gathering
	<i>LF</i> , Load Factor. Ratio between the actual load and the rated load. The default value is 0.90	oad Factor. Ratio between the actual load and	Based on spot metering and nameplate	EDC Data Gathering
			Default <u>:</u> 90% ⁶⁰²	1
	η_{motor} , Motor efficiency at the full-rated load. For VFD installations, this can be either an energy efficient motor or standard efficiency motor.	None Variable	Nameplate	EDC Data Gathering
	ESF, Energy Savings Factor. Percent of baseline energy consumption saved by installing VFD.	None Fixed	46%	2, 3
	DHRS, Daily run hours of the motor	Variable <u>Hours</u>	Based on customer application	EDC Data Gathering
			Default <u>:</u> 8 hours day	2, 3
	ADAYS, Annual operating days	Variable <u>Days</u>	Based on customer application	EDC Data Gathering
			Default: $365 \frac{days}{year}$	2, 3
	CF, Demand Coincidence factor	None Fixed	0.00014	<u>3</u> 4

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for VSD Controllers is 15 years. 603

⁶⁰² Default Value can be used by EDC but is subject to metering and adjustment by evaluators or SWE 603 California Public Utility Commission. Database for Energy Efficiency Resources 2008.

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SOURCES

- Southern California Edison, Dairy Farm Energy Management Guide: California, p. 11, 2004.
- 2. California Public Utility Commission. Database for Energy Efficiency Resources (DEER) 2005. The DEER database assumes 20 hours of operation per day, but is based on much larger dairy farms (e.g. herd sizes > 300 cows). Therefore, the DEER default value was lowered to 8 hours per day, as the average heard size in 75 cows in Pennsylvania.
- Regional Technical Forum (RTF) as part of the Northwest Power & Conservation Council, Deemed Measures List. Agricultural: Variable Frequency Drives-Dairy, FY2012, V1.2. Accessed from RTF website http://rtf.nwcouncil.org/measures/Default.asp on February 27, 2013.
- 4. The coincident peak demand factor is calculated by dividing the average peak coincident demand kW reduction by ΔkWh savings for a 1 horsepower, 90% efficient motor. The motor efficiency was determined by averaging the efficiencies of NEMA premium efficiency motors for motor sizes between 1 and 50 horsepower.

4.1.8 Low Pressure Irrigation System

Measure Name	Low Pressure Irrigation System
Target Sector	Agriculture and Golf Courses (includes Residential and Commercial)
Measure Unit	Per Project_Irrigation System
Unit Energy Savings	Varies-Variable_
Unit Peak Demand Reduction	Varies by application_Variable
Measure Life	5 years ⁶⁰⁴
Measure Vintage	Replace on Burnout or New Construction

ELIGIBILITY

The following protocol for the measurements of energy and demand savings applies to the installation of a low-pressure irrigation system, thus reducing the amount of energy required to apply the same amount of water.

The amount of energy saved per acre will depend on the actual operating pressure decrease, the pumping plant efficiency, the amount of water applied, and the number of nozzle, sprinkler or micro irrigation system conversions made to the system.

This measure requires a minimum of 50% reduction in irrigation pumping pressure is achieved through the installation of a low-pressure irrigation system in agriculture or golf course applications. The pressure reduction can be achieved in several ways, such as nozzle or valve replacement, sprinkler head replacement, alterations ore retrofits to the pumping plant, or drip irrigation system installation, and is left up to the discretion of the owner. Pre and post retrofit pump pressure measurements are required.

ALGORITHMS

The annual energy savings are obtained through the following formulas:

Agriculture applications:

$$\frac{\Delta kWh}{-}_{-\Delta kW} = \frac{\left\{ACRES \times \left(PSI_{base} - PSI_{eff}\right) \times GPM1\right\}}{1,714 \frac{gpm \cdot psi}{HP} \times \eta_{motor}} \times 0.746 \frac{kW}{HP} \times OPRHS \frac{Irrigation Hours}{Growing Season}$$

$$\Delta kW_{peak} = \frac{\Delta kWh}{yr} \times CF$$

Golf Course applications:

$$\Delta kWh \frac{\Delta kWh}{Y^{+}} = \frac{\{ \left(PSI_{base} - PSI_{eff} \right) \times GPM2 \}}{1,714 \frac{gpm \ psi}{HP} \times \eta_{motor}} \times 0.746 \frac{kW}{HP} \times DHRS \times MONTHS \times 30 \frac{avg. \ days}{month}$$

⁶⁰⁴ DEER Effective Useful Life. October 10, 2008.

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Table 4-13: Variables for Low Pressure Irrigation Systems

Component-Term	<u>Unit</u> Type	Value <u>s</u>	Source
ACRES, Number of acres irrigated	<u>Acres</u> Variable	Based on customer application	EDC Data Gathering,1
PSI _{base} , Baseline pump pressure, must be measured and recorded prior to installing low-pressure irrigation equipment.	Pounds per square inch (psi)Variable	Based on pre retrofit pressure measurements taken by the installer	EDC Data Gathering,1
PSI_{eff} , Installed pump pressure, must be measured and recorded after the installation of low-pressure irrigation equipment by the installer.	Pounds per square inch (psi)Variable	Based on post retrofit pressure measurements taken by the installer	EDC Data Gathering,1
<i>GPM</i> 1, Pump flow rate per acre for agriculture applications.	Gallons per minute (gpm) Variable	Based on pre retrofit flow measurements taken by the installer	EDC Data Gathering,1
<i>GPM2</i> , Pump flow rate for pumping system for golf courses.	Gallons per minute (gpm)Variable	Based on pre retrofit flow measurements taken by the installer	EDC Data Gathering,1
1714. Constant used to calculate hydraulic horsepower for conversion between horsepower and pressure and flow	<u>None</u>	$= \frac{PSI \times GPM}{1714}$	Conversion Factor
OPHRS, Average irrigation hours per growing season for agriculture	HoursVariable	Based on customer application	EDC Data Gathering
DHRS, Hours of watering per day for golf courses	Hours Variable	Based on customer application	EDC Data Gathering
MONTHS, Number of months of irrigation for golf courses	Months Variab	Based on customer application	EDC Data Gathering
		Based on customer application	EDC Data Gathering
η_{motor} , Pump motor efficiency	<u>None</u> Variable	Look up pump motor efficiency based on the pump nameplate horsepower (hp) from customer application and nominal efficiencies defined in Table 3-54	2
CF, Demand coincidence factor for agriculture	None Variable	0.0026	3, 4

DEFAULT SAVINGS

There are no default savings for this measure.

EVALUATION PROTOCOLS

For most projects, the appropriate evaluation protocol is to verify installation and proper selection of default values. For projects using customer specific data for open variables, the appropriate evaluation protocol is to verify installation and proper application of TRM protocol along with verification of open variables. The Pennsylvania Phase II Evaluation Framework provides specific guidelines and requirements for evaluation procedures.

MEASURE LIFE

The measure life for low pressure irrigation systems is 5 years. 605

The measure lifeabove is based upon retrofit of existing equipment. Complete system replacement may be determined based upon the system conditions.

SOURCES

- Based on Alliant Energy program evaluation assumptions for low-flow pressure irrigation systems. Evaluation of Alliant Energy Agriculture Program, Appendix F, 2008. http://alliantenergy.com/wcm/groups/wcm_internet/@int/documents/document/mdaw/mde-y/~edisp/012895.pdf
- 2. Table 3-54_Table (Building Mechanical System Variables in Premium Efficiency Motor Calculations) contains federal motor efficiency values from NEMA EPACT efficiency motor standards by motor size and type. If existing motor nameplate data is not available, these tables will be used to establish motor efficiencies. The CF was only estimated for agricultural applications, and was determined by using the following formula CF = AkW savings per acre savings per acre.
- 3. Pennsylvania census data was used to estimate an average ΔkW savings/acre and ΔkWh/yr/savings/acre value. Pamela Kanagy. Farm and Ranch Irrigation. Pennsylvania Agricultural Statistics 2009-2010. http://www.nass.usda.gov/Statistics_by_State/Pennsylvania/Publications/Annual_Statistic al_Bulletin/2009_2010/fris.pdf =-
- 4. Irrigation Water Withdrawals, 2000 by the U.S. Geological Society. http://pubs.usgs.gov/circ/2004/circ1268/htdocs/table07.html

-Irrigation Water Withdrawals, 2000 by the U.S. Geological Society.

3. http://pubs.usgs.gov/circ/2004/circ1268/htdocs/table07.html

⁶⁰⁵-California Public Utility Commission. Database for Energy Efficiency Resources 2008.

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5 APPENDICES

5.1 APPENDIX A: MEASURE LIVES

Measure Lives Used in Cost-Effectiveness Screening August 20143

 $^*\mbox{For the purpose}$ of calculating the total Resource Cost Test for Act 129, measure cannot claim savings for more than fifteen years.

Measure	Measure Life
RESIDENTIAL SECTOR	
Lighting End-Use	
Electroluminescent Nightlight	8
LED Nightlight	8
Compact Fluorescent Light Bulb	5.2
Recessed Can Fluorescent Fixture	20*
Torchieres	10
Fixtures Other	20*
ENERGY STAR LEDs	14.7
Residential Occupancy Sensors	10
Holiday Lights	10
HVAC End-Use	
Central Air Conditioner (CAC)	14
Air Source Heat Pump	12
Central Air Conditioner proper sizing/install	14
Central Air Conditioner Quality Installation Verification	14
Central Air Conditioner Maintenance	7
Central Air Conditioner duct sealing	20
ENERGY STAR Room Air Conditioners	9
Air Source Heat Pump proper sizing/install	12
ENERGY STAR Thermostat (Central Air Conditioner)	15
ENERGY STAR Thermostat (Heat Pump)	15
Ground Source Heat Pump	30*
Room Air Conditioner Retirement	4
Furnace Whistle	14
Programmable Thermostat	11
Room AC (RAC) Retirement	4
Residential Whole House Fans	15
Ductless Mini-Split Heat Pumps	15
Fuel Switching: Electric Heat to Gas Heat	20*
Efficient Ventilation Fans with Timer	10

New Construction (NC): Single Family - gas heat with CAC	20*
NC: Single Family - oil heat with CAC	20*
NC: Single Family - all electric	20*
NC: Multiple Single Family (Townhouse) – oil heat with CAC	20*
NC: Multiple Single Family (Townhouse) - all electric	20*
NC: Multi-Family – gas heat with CAC	20*
NC: Multi-Family - oil heat with CAC	20*
NC: Multi-Family - all electric	20*
Hot Water End-Use	
Efficient Electric Water Heaters	14
Heat Pump Water Heaters	14
Low Flow Faucet Aerators	12
Low Flow Showerheads	9
Solar Water Heaters	15
Electric Water Heater Pipe Insulation	13
	40
Fuel Switching: Domestic Hot Water Electric to Gas or Propane Water Heater	13
Fuel Switching: Domestic Hot Water Electric to Oil Water Heater	8
Fuel Switching: Heat Pump Water Heater to Gas or Propane Water Heater	13
Fuel Switching: Heat Pump Water Heater to Oil Water Heater	8
Water Heater Tank Wrap	7
Appliances End-Use	
Electric ENERGY STAR Clothes Dryer with Moisture Sensor	13
Refrigerator / Freezer Recycling without replacement	8
Refrigerator / Freezer Recycling with replacement	7
ENERGY STAR Refrigerators	12
ENERGY STAR Freezers	12
ENERGY STAR Clothes Washers	11
ENERGY STAR Dishwashers	10
ENERGY STAR Dehumidifers	12
ENERGY STAR Room Air Conditioners	9
ENERGY STAR Televisions	15
ENERGY STAR Water Coolers	10
Office Equipment / Consumer Electronics End-Use	
ENERGY STAR Televisions	<u>6</u>
Smart Strip Plug Outlets	4
ENERGY STAR Computer	4
ENERGY STAR Monitor	5

ENERGY STAR Fax	4
ENERGY STAR Multifunction Device	6
ENERGY STAR Printer	5
ENERGY STAR Copier	6
Building Shell End-Use	
Ceiling / Attic and Wall Insulation	25*
Window -heat pump	20*
Window -gas heat with central air conditioning	20*
Window – electric heat without central air conditioning	20*
Window – electric heat with central air conditioning	20*
Home Audit Conservation Kits	<u>8.1</u>
Home Performance with ENERGY STAR	<u>5</u>
Agricultural End-Use	
Automatic Milker Takeoffs	10
Dairy Scroll Compressors	15
High Efficiency Ventilation Fans with or without Thermostats	10
Heat Reclaimers	15
High Volume Low Speed Fans	15
Livestock Waterer	10
Variable Speed Drive (VSD) Controller on Dairy Vacuum Pumps	15
Low Pressure Irrigation System	5
Miscellaneous	
Home Audit Conservation Kits	8.1
Home Performance with ENERGY STAR	5
Pool Pump Load Shifting	1
High Efficiency Two-Speed Pool Pump	10
Variable Speed Pool Pumps (with Load Shifting Option)	10
COMMERCIAL & INDUSTRIAL SECTOR	
Lighting End-Use	
Lighting (Non-SSL) — New/Remodel/Replacement	15
Lighting (SSL – 25,000 hours) — New/Remodel/Replacement	6
Lighting (SSL – 30,000 hours) — New/Remodel/Replacement	7
Lighting (SSL – 35,000 hours) — New/Remodel/Replacement	8
Lighting (SSL – 40,000 hours) — New/Remodel/Replacement	10
Lighting (SSL – 45,000 hours) — New/Remodel/Replacement	11
Lighting (SSL – 50,000 hours) — New/Remodel/Replacement	12
Lighting (SSL – 55,000 hours) — New/Remodel/Replacement	13

Lighting (SSL – 60,000 hours) — New/Remodel/Replacement	14
Lighting (SSL - ≥60,000 hours) - New/Remodel/Replacement	15*
Lighting Fixture Improvements	<u>13</u>
New Construction Lighting	<u>15</u>
<u>Lighting Controls</u>	<u>8</u>
<u>Traffic Lights</u>	<u>10</u>
LED Exit Signs	<u>16*</u>
LED Channel Signage_	15
LED Refrigeration Case Lighting	<u>8</u>
HVAC End-Use	
HVAC Systems — New or Replacement	15
Electric Chillers — New or Replacement	20*
Water Source and Geothermal Heat Pumps	15
Ductless Mini-Split Heat Pumps - Commercial < 5.4 tons	15
Commercial Chiller Optimization	18*
Fuel Switching: Small Commercial Electric Heat to Natural Gas/ Propane/ Oil Heat	20*
Small C/I HVAC Refrigerant Charge Correction	10
ENERGY STAR Room Air Conditioner	<u>12</u>
Controls: Guest Room Occupancy Sensor	<u>15</u>
Controls: Economizer	<u>10</u>
Motors & VFDs End-Use	
Premium Efficiency Motors — New or Replacement	20* 15
Variable Frequency Drive (VFD) Improvements — New / Retrofit	1 <u>3</u> 5
Variable Frequency Drive (VFD) Improvement for Industrial Air Compressors	20*
ECM Circulating Fan	<u>18</u>
VSD on Kitchen Exhaust Fan	<u>15</u>
Domestic Hot Water End-Use	
Electric Resistance Water Heaters_	15
Heat Pump Water Heaters_	10
Low Flow Pre-Rinse Sprayers for Retrofit Programs_	5
Low Flow Pre-Rinse Sprayers for Time of Sale / Retail Programs_ Fuel Switching: Domestic Hot Water Electric Resistance Water Heaters to	5
Gas or Propane Water Heater	13
Fuel Switching: Domestic Hot Water Electric Electric Resistance Water Heaters to Oil Water Heater_	8
Fuel Switching: Heat Pump Water Heater to Gas or Propane Water Heater_	13
Fuel Switching: Heat Pump Water Heater to Oil Water Heater_	8
Refrigeration End-Use	

	High-Efficiency Evaporator Fan Motors for Reach-In Refrigerated Cases	15
	High-Efficiency Evaporator Fan Motors for Walk-In Refrigerated Cases	15
l	Refrigeration Controls: Evaporator Fan Controllers	10
	Refrigeration — Controls: Floating Head Pressure Controls	15
	Controls: Anti-Sweat Heater Controls	12
	Controls: Evaporator Coil Defrost Control	<u>10</u>
	Refrigeration VFDVariable Speed Refrigeration Compressor	1 <u>5</u> 0
	Strip Curtains for Walk-In Freezers and Coolers	4
	Refrigeration - Night Covers for Display Cases	5
	Refrigeration Auto Closers	8
	Refrigeration Door Gaskets for Walk-in and Reach-in Coolers and Freezers	4
l	Refrigeration Special Doors with Low or No Anti-Sweat Heat for Low Temp Case	15
	Refrigeration – Suction Pipes Insulation for Walk-in Coolers and Freezers	11
	Appliances End-Use	
	ENERGY STAR Clothes Washer Multifamily	11.3
	ENERGY STAR Clothes Washer Laundromats	7.1
	ENERGY STAR Clothes Washer Laundronials	1.1
	Food Service Equipment End-Use	
	High-Efficiency Ice Machines	10
l	Controls: Beverage Machine Controls	5
	Controls: Snack Machine Controls	<u>5</u>
	ENERGY STAR Electric Steam Cooker	12
	ENERGY STAR Refrigerated Beverage Vending Machine	<u>14</u>
	Building Shell End-Use	
	Wall and Ceiling Insulation	15
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	Office Equipment / Consumer Electronics End-Use	
	ENERGY STAR Computer	4
	ENERGY STAR Monitor	4
	ENERGY STAR Fax	4
	ENERGY STAR Multifunction Device	6
	ENERGY STAR Printer	5
	ENERGY STAR Copier	6
	Office Equipment - Network Power Management Enabling	5
	Smart Strip Plug Outlets	5
ı	Compressed Air	
l	Compressed Air	

1	Cycling Refrigerated Thermal Mass Dryer	<u>10</u>
	Air-entraining Air Nozzle	<u>15</u>
1	No-Loss Condensate Drains	<u>5</u>
	Miscellaneous	
	Commercial Custom – New/Replacement	18*
	Commercial Comprehensive New Construction Design	18*
	O&M Savings	<u>3</u>
	Industrial Custom - Non Process	18*
	Industrial Custom – Process	10
	ENERGY STAR Servers	<u>4</u>
	Agricultural End-Use	
	Automatic Milker Takeoffs	10
	Dairy Scroll Compressors	15
	High Efficiency Ventilation Fans (with or without Thermostats)	10
	Heat Reclaimers	15
	High Volume Low Speed Fans	15
	Livestock Waterer	10
	Variable Speed Drive (VSD) Controller on Dairy Vacuum Pumps	15
	Low Pressure Irrigation System	5

5.2 APPENDIX B: RELATIONSHIP BETWEEN PROGRAM SAVINGS AND EVALUATION SAVINGS

There is a distinction between activities required to conduct measurement and verification of savings at the program participant level and the activities conducted by program evaluators and the SWE to validate those savings. However, the underlying standard for the measurement of the savings for both of these activities is the measurement and verification protocols approved by the PA PUC. These protocols are of two different types:

- TRM specified protocols for standard measures, originally approved in the May 2009 order adopting the TRM, and updated annually thereafter
- Interim Protocols for standard measures, reviewed and recommended by the SWE and approved for use by the Director of the CEEP, subject to modification and incorporation into succeeding TRM versions to be approved by the PA PUC

These protocols are to be uniform and used to measure and calculate savings throughout Pennsylvania. The TRM protocols are comprised of Deemed Measures and Partially Deemed Measures. Deemed Measures specify saving per energy efficiency measure and require verifying that the measure has been installed, or in cases where that is not feasible, that the measure has been purchased by a utility customer. Partially Deemed Measures require both verification of installation and the measurement or quantification of open variables in the protocol.

Stipulated and deemed numbers are valid relative to a particular classification of "standard" measures. In the determination of these values, a normal distribution of values should have been incorporated. Therefore, during the measurement and verification process, participant savings measures cannot be arbitrarily treated as "custom measures" if the category allocation is appropriate.

Custom measures are outside the scope of the TRM. The EDCs are not required to submit savings protocols for custom measures to the Commission or the SWE for each measure/technology type prior to implementing the custom measure. The Commission recommends that these protocols be established in general conformity to the IPMVP or Federal Energy Management Program M&V Guidelines. The SWE reserves the right to audit and review claimed and verified impacts of all custom measures as part of its role to perform EM&V services for the Commission.

Utility evaluators and the SWE will adjust the savings reported by program staff based on the application of the PA PUC approved protocols to a sample population and realization rates will be based on the application of these same standards. To the extent that the protocols or deemed values included in these protocols require modification, the appropriate statewide approval process will be utilized. These changes will be prospective.

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5.3 APPENDIX C: LIGHTING AUDIT AND DESIGN TOOL

The Lighting Audit and Design Tool is located on the Public Utility Commission's website at: Website Link TBD.

5.4 APPENDIX D: MOTOR & VFD AUDIT AND DESIGN TOOL

The Motor and VFD Inventory Form is located on the Public Utility Commission's website at: $\underline{\text{Website Link TBD}}$.

The Lighting Audit and Design Tool is located on the Public Utility Commission's website at: Website Link TBD.

Rev Date: June 2015

5.6 APPENDIX F: ELIGIBILITY REQUIREMENTS FOR SOLID STATE LIGHTING PRODUCTS IN COMMERCIAL AND INDUSTRIAL APPLICATIONS

The SSL market, still setting up its foundations, has been inundated with a great variety of products, including those that do not live up to manufacturers' claims. Several organizations, such as ENERGY STAR and Design Lights Consortium have responded by following standardized testing procedures and setting minimum requirements to be identified as a qualified product under those organizations.

5.6.1 SOLID STATE LIGHTING

Due to the immaturity of the SSL market, diversity of product technologies and quality, and current lack of uniform industry standards, it is impossible to point to one source as the complete list of qualifying SSL products for inclusion in Act 129 efficiency programs. A combination of industry-accepted references have been collected to generate minimum criteria for the most complete list of products while not sacrificing quality and legitimacy of savings. The following states the minimum requirements for SSL products that qualify under the TRM:

For Act 129 energy efficiency measure savings qualification, for SSL products for which there is an ENERGY STAR commercial product category⁶⁰⁶, the product shall meet the minimum ENERGY STAR requirements⁶⁰⁷ for the given product category. Products are not required to be on the ENERGY STAR Qualified Product List⁶⁰⁹, however, if a product is on the list it shall qualify for Act 129 energy efficiency programs and no additional supporting documentation shall be required. ENERGY STAR qualified commercial/non-residential product categories include:

- Omni-directional: A, BT, P, PS, S, T
- Decorative: B, BA, C, CA, DC, F, G
- Directional: BR, ER, K, MR, PAR, R
- Non-standard
- · Recessed, surface and pendant-mounted down-lights
- Under-cabinet shelf-mounted task lighting
- Portable desk task lights
- Wall wash luminaires
- Bollards

For SSL products for which there is not an ENERGY STAR commercial product category, but for which there is a DLC commercial product category⁶¹⁰, the product shall meet the minimum DLC

 $\underline{\text{http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup\&pgw_code=LTG}$

⁶⁰⁶ ENERGY STAR website for Commercial LED Lighting:

^{607 &}quot;ENERGY STAR® Program Requirements for Integral LED Lamps

Partner Commitments." *LED Lamp Specification V1.1*, modified 03/22/10. Accessed from the ENERGY STAR website on September 28, 2010. http://www.energystar.gov/ia/partners/manuf_res/downloads/IntegralLampsFINAL.pdf

⁶⁰⁸ "ENERGY STAR® Program Requirements for Solid State Lighting Luminaires" *Eligibility Criteria V1.1*, Final 12/19/08. Accessed from the ENERGY STAR website on September 28, 2010.

http://www.energystar.gov/ia/partners/product_specs/program_regs/SSL_prog_reg_V1.1.pdf

⁶⁰⁹ ENERGY STAR Qualified LED Lighting list http://www.energystar.gov/index.cfm?fuseaction=ssl.display_products_res_html

⁶¹⁰ DesignLights Consortium (DLC) Technical Requirements Table v1.4. Accessed from the DLC website on September 24, 2010. http://www.designlights.org/solidstate.manufacturer.requirements.php

requirements⁶¹¹ for the given product category. Products are not required to be on the DLC Qualified Product List⁶¹², however, if a product is on the list it shall qualify for Act 129 energy efficiency programs and no additional supporting documentation shall be required. DLC qualified commercial product categories include:

- Outdoor Pole or Arm mounted Area and Roadway Luminaires
- Outdoor Pole or arm mounted Decorative Luminaires
- **Outdoor Wall-Mounted Area Luminaires**
- Parking Garage Luminaires
- Track or Mono-point Directional Lighting Fixtures
- Refrigerated Case Lighting
- Display Case Lighting
- 2x2 Luminaires
- High-bay and Low-bay fixtures for Commercial and Industrial buildings

For SSL products that are not on either of the listed qualified products lists, they can still be considered for inclusion in Act 129 energy efficiency programs by submitting the following documentation to show compliance with the minimum product category criteria as described above:

- Manufacturer's product information sheet
- LED package/fixture specification sheet
- List the ENERGY STAR or DLC product category for which the luminaire qualifies
- Summary table listing the minimum reference criteria and the corresponding product values for the following variables:
 - Light output in lumens
 - Luminaire efficacy (Im/W)
 - Color rendering index (CRI)
 - Correlated color temperature (CCT)
 - LED lumen maintenance at 6000 hrs
 - Manufacturer's estimated lifetime for L₇₀ (70% lumen maintenance at end of useful life) (manufacturer should provide methodology for calculation and justification of product lifetime estimates)
 - Operating frequency of the lamp
- IESNA LM-79-08 test report(s) (from approved labs specified in DOE Manufacturers' Guide) containing:

612 DesignLights Consortium (DLC) Qualified Product List.

⁶¹¹ Ibid.

http://www.designlights.org/solidstate.about.QualifiedProductsList_Publicv2.php
"This Qualified Products List (QPL) of LED luminaires signifies that the proper documentation has been submitted to DesignLights (DLC) and the luminaire has met the criteria noted in the technical requirements table shown on the DesignLights website (www.designlights.org). This list is exclusively used and owned by DesignLights Members. Manufacturers, vendors and other non DesignLights members may use the QPL as displayed herein subject to the DLC Terms of Use, and are prohibited from tampering with any portion or all of its contents. For information on becoming a member please go to DesignLights.org.'

- o Photometric measurements (i.e. light output and efficacy)
- Colorimetry report (i.e. CCT and CRI)
- Electrical measurements (i.e. input voltage and current, power, power factor, etc.)
- Lumen maintenance report (select one of the two options and submit all of its corresponding required documents):
 - Option 1: Compliance through component performance (for the corresponding LED package)
 - IESNA LM-80 test report
 - In-situ temperature measurements test (ISTMT) report.
 - Schematic/photograph from LED package manufacturer that shows the specified temperature measurement point (TMP)
 - o Option 2: Compliance through luminaire performance
 - IESNA LM-79-08 report at 0 hours (same file as point c)
 - IESNA LM-79-08 report at 6000 hours after continuous operation in the appropriate ANSI/UL 1598 environment (use ANSI/UL 1574 for track lighting systems).

All supporting documentation must include a specific, relevant model or part number.

5.7 APPENDIX G: ZIP CODE MAPPING

Per Section 1.17, the following table is to be used to determine the appropriate reference city for each Pennsylvania zip code.

Zip	Reference City
15001	Pittsburgh
15003	Pittsburgh
15004	Pittsburgh
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15007	Pittsburgh
15009	Pittsburgh
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Zip	Reference City
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15784	Pittsburgh
15801	Pittsburgh
15821	Williamsport
15822	Williamsport
15823	Pittsburgh
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Zip	Reference City
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